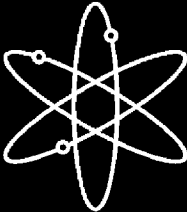


# **Common-Cause Failure Event Insights**

## **Pumps**



**Idaho National Engineering and Environmental Laboratory**



**U.S. Nuclear Regulatory Commission  
Office of Nuclear Regulatory Research  
Washington, DC 20555-0001**



# Common-Cause Failure Event Insights

## Pumps

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Prepared by  
T. E. Wierman, INEEL  
D. M. Rasmuson, NRC  
N. B. Stockton, INEEL

Idaho National Engineering and Environmental Laboratory  
Idaho Falls, ID 83415

T.R. Wolf, NRC Project Manager

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**Division of Risk Analysis and Applications**  
**Office of Nuclear Regulatory Research**  
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# **Common-Cause Failure Event Insights Volume 3 Pumps**

*T. E. Wierman  
D. M. Rasmuson, USNRC  
N. B. Stockton*





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**T. E. Wierman  
D. M. Rasmuson, USNRC  
N. B. Stockton**

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**Idaho National Engineering and Environmental Laboratory  
Risk & Reliability Assessment Department  
Lockheed Martin Idaho Technologies Company  
Idaho Falls, Idaho 83415**

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U.S. Nuclear Regulatory Commission  
Washington, D.C. 20555  
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## **ABSTRACT**

This report documents a study performed on the set of common-cause failures (CCF) of pumps from 1980 to 2000. The data studied here were derived from the NRC CCF database, which is based on US commercial nuclear power plant event data. This report is the result of an in-depth review of the pump CCF data and presents several insights about the pump CCF data. The objective of this document is to look beyond the CCF parameter estimates that can be obtained from the CCF data to gain further understanding of why CCF events occur and what measures may be taken to prevent, or at least mitigate the effect of, pump CCF events. This report presents quantitative presentation of the pump CCF data and discussion of some engineering aspects of the pump events.



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## EXECUTIVE SUMMARY

This report provides insights related to pump common-cause failure (CCF) events. These events were obtained from the U.S. Nuclear Regulatory Commission's (NRC) CCF Database. The pump CCF data contains attributes about events that are of interest in the understanding of: completeness of the failures, occurrence rate trends of the events, pump segments affected, causal factors, coupling or linking factors, event detection methods, and pump manufacturer. Distributions of these CCF characteristics and trends were analyzed and individual events were reviewed for insights.

**General Insights.** The study identified 274 events occurring at U.S. nuclear power plant (NPP) units during the period from 1980 through 2000. Thirty-three NPP units each had one CCF event during the period; 21 NPP units did not experience a CCF event. This accounts for about 50 percent of the NPP units. While only 38 NPP units experienced more than two pump CCF events, these 38 NPP units account for 76 percent of the total number of pump CCF events. Of the 274 events, 62 (23 percent) were Complete common-cause failures (failure events with all components failed due to a single cause in a short time).

**Failure Modes.** The events were classified as either fail-to-start or fail-to-run. The failure mode for the majority of the pump CCF events is fail-to-run (54 percent). The fail-to-start failure mode accounted for the other 46 percent of the events.

**Trends.** Figure ES-1 shows the trend for all pump CCF events. The decreasing trend for all pump CCF events is statistically significant with a p-value of 0.0001. There was insufficient information to determine what caused the decreasing trend in CCF events, but there were several regulatory initiatives by the NRC and industry initiatives by utilities, INPO, and EPRI involving improved operation, maintenance, testing, and inspection during the 21 years of improving performance. Both the fail-to-start and the fail-to-run failure modes for pump CCF events were similar statistically-significant decreasing trends. The trend for the Complete events from 1980-2000 is a decreasing trend and is statistically significant with a p-value = 0.0001.

**Method of Discovery.** When the method of discovery was investigated, Testing accounted for 95 events, (35 percent), 83 events (30 percent) were discovered during Demand, Inspection accounted for 69 events (25 percent), and 27 events (10 percent) were detected during Maintenance activities. Considering the extensive and frequent surveillance test requirements for pumps contained in the Technical Specifications and the standby nature of most of the pumps in this study, it is expected that a majority of the pump CCF events would be detected by Testing. The failures detected by testing tended to be Internal to Component causes attributed to wear and aging and only a small percentage of these failures resulted in Complete CCF events. It was expected that fewer failures would be detected by Demand. Analysis of events showed that over half of the events discovered by Demand were Complete or Almost Complete. The majority of events detected by Demand were attributed to design errors, human errors, and the Others category. These causes were also dominant for all Complete CCF events. This implies that testing may be effective at detecting normal wear and aging problems, but less effective at detecting failures related to design and human errors.

**Segment.** Overall, for all pumps, the highest number of events occurred in the pump segment (106 events or 39 percent). The driver and suction segments were also significant contributors (32 and 24 percent, respectively), while relatively few events involved the discharge segment. These statistics vary by system. For the emergency service water (ESW) and standby liquid control (SLC) systems, most of the failures occurred in the pump segment. However, for the auxiliary feedwater (AFW), high pressure injection (HPI), and BWR residual heat removal (RHR-B) systems, most of the failures occurred in the driver segment, and for the PWR residual heat removal (RHR-P) system, most of the failures occurred in the suction segment. Events involving the driver and suction segments were more likely to be Complete. Ninety-two percent of all Complete events occurred in these two segments.

**Piece Parts.** The most common piece parts involved in pump segment CCF events were the impellers and wear rings. The most likely piece parts involved in driver segment CCF events were circuit breakers and instrument and control circuits. The most likely piece part involved in the suction segment CCF events was piping. The most likely piece part involved in discharge segment CCF events was the valves.

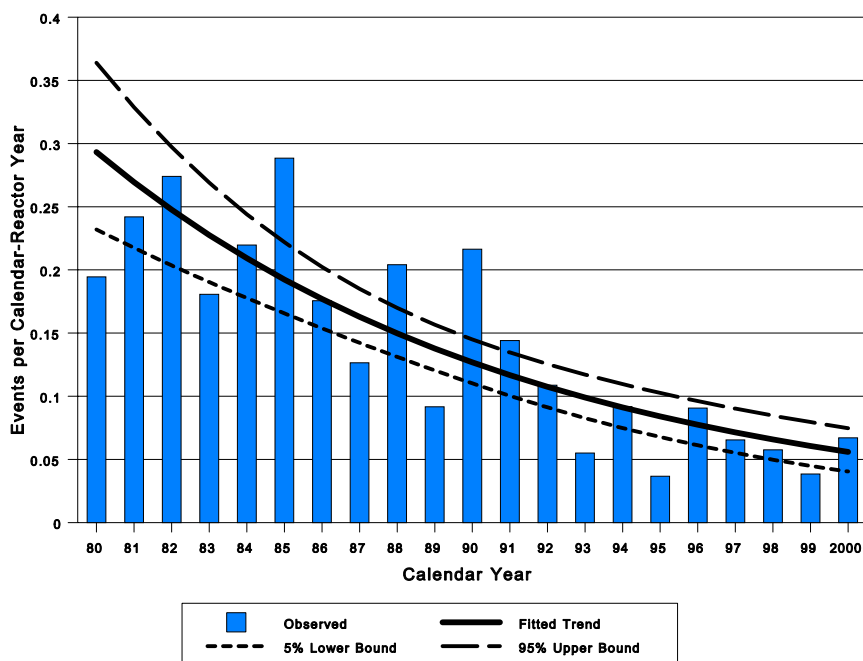


Figure ES-1. Trend for all pump CCF events. The decreasing trend is statistically significant with a p-value = 0.0001.

**Proximate Cause.** As shown in Figure ES-2, the leading proximate cause was Internal to Component, which accounted for about 39 percent of the total events; however, none of these events were Complete. Design/Construction/Installation/Manufacture Inadequacy and Human Error accounted for 24 and 20 percent of the total events, respectively. The Other and External Environment proximate causes were attributed to a small fraction of the pump CCF events.

The Internal to Component proximate cause category is the most likely for the pumps and encompasses the malfunctioning of hardware internal to the component. Internal causes result from phenomena such as normal wear or other intrinsic failure mechanisms, which are influenced by the

ambient environment of the component. Specific mechanisms include erosion, corrosion, internal contamination, fatigue, wear-out, and end of life.

The Design/Construction/Installation /Manufacture Inadequacy proximate cause group is important for pumps and encompasses events related to the design, construction, installation, and manufacture of components, both before and after the plant is operational. Included in this category are events resulting from errors in equipment and system specifications, material specifications, and calculations. Events related to maintenance activities are not included.

The Operational/Human Error proximate cause group is the next most likely for pumps and represents causes related to errors of omission or commission on the part of plant staff or contractor staff. Included in this category are accidental actions, failures to follow the correct procedures or following inadequate procedures for construction, modification, operation, maintenance, calibration, and testing. This proximate cause group also includes deficient training.

**Coupling Factors.** Maintenance was the leading coupling factor with 111 events (41 percent). The next leading coupling factor was Design with 76 events (28 percent). While not the leading coupling factor, over half (51 percent) of the Design coupled events were either Complete or Almost Complete. The Environmental and Operational coupling factors account for the majority of the remaining events (44 and 28 events, respectively). Only a small fraction of the events coupled by Environmental were Complete; however, over half (57 percent) of the events coupled by Operational were Complete. These Complete events were almost all coupled by inadequate operations procedures. Only 15 events were coupled by Quality, and three of these were Complete and affected the Driver segment.

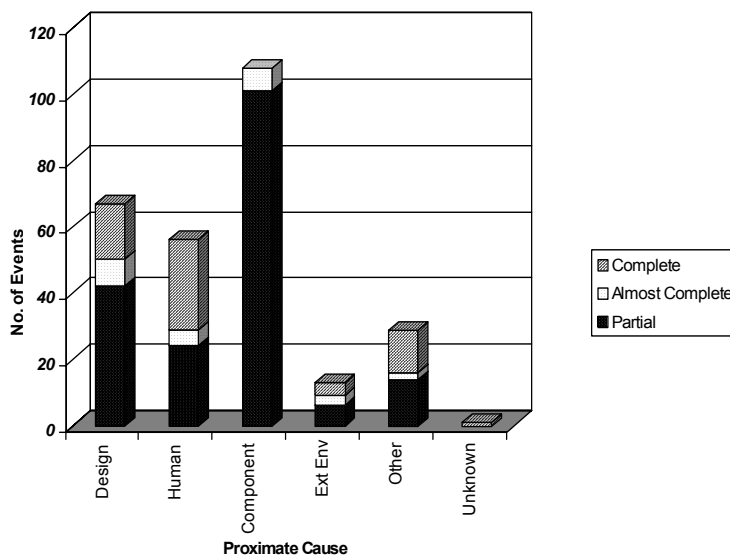


Figure ES-2. Proximate cause distribution for all pump CCF events.

**System.** Figure ES-3 shows the distribution of pump CCF events by affected system. The ESW system had the most events. Most pump CCF events in the ESW system involved problems with the pump impellers and wear rings. The RHR-P system had the largest fraction of Complete CCF events (92

percent). Most of the RHR-P system events involved loss of suction, usually during refueling outages with reduced water level in the reactor coolant system.

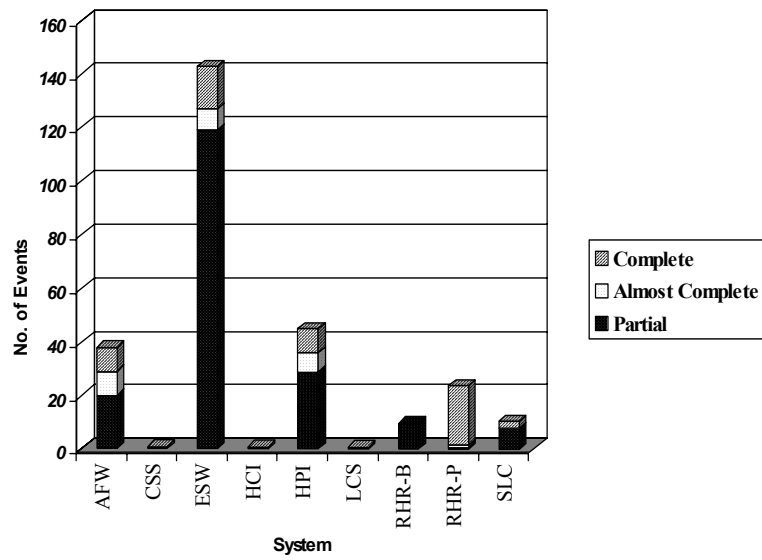


Figure ES-3. System distribution for all pump CCF events.

## FOREWORD

This report provides common-cause failure (CCF) event insights for pumps. The results, findings, conclusions, and information contained in this study, the initiating event update study, and related system reliability studies conducted by the Office of Nuclear Regulatory Research support a variety of risk-informed NRC activities. These include providing information about relevant operating experience that can be used to enhance plant inspections of risk-important systems, and information used to support staff technical reviews of proposed license amendments, including risk-informed applications. In addition, this work will be used in the development of enhanced performance indicators that will be based largely on plant-specific system and equipment performance.

Findings and conclusions from the analyses of the pump CCF data, which are based on 1980-2000 operating experience, are presented in the Executive Summary. High-level insights of the pump CCF data are presented in Section 3. Section 4 summarizes the events by sub-component. Section 5 presents pump CCF insights by the pump system. Section 6 provides information about how to obtain more detailed information for the pump CCF events. The information to support risk-informed regulatory activities related to the pump CCF data is summarized in Table F-1. This table provides a condensed index of risk-important data and results presented in discussions, tables, figures, and appendices.

Table F-1. Summary of insights from pump common-cause failure events.

| Item | Description  | Text Reference       | Page(s) | Data                      |
|------|--|----------------------|---------|---------------------------|
| 1.   | CCF trends overview                                  | Section 3.2          | 16      | Figure 3-1 – Figure 3-4   |
| 2.   | CCF segment overview                                 | Section 3.3          | 18      | Figure 3-5                |
| 3.   | CCF proximate cause overview                         | Section 3.4          | 19      | Figure 3-6                |
| 4.   | CCF coupling factor overview                         | Section 3.5          | 22      | Figure 3-7                |
| 5.   | CCF discovery method overview                        | Section 3.6          | 25      | Figure 3-8                |
| 6.   | CCF system overview                                  | Section 3.7          | 26      | Figure 3-9                |
| 7.   | Engineering Insights – Pump Segment                  | Section 4.2          | 31      | Figure 4-1 – Figure 4-3   |
| 8.   | Engineering Insights – Driver Segment                | Section 4.3          | 34      | Figure 4-4 – Figure 4-6   |
| 9.   | Engineering Insights – Suction Segment               | Section 4.4          | 38      | Figure 4-7 – Figure 4-9   |
| 10.  | Engineering Insights – Discharge Segment             | Section 4.5          | 43      | Figure 4-10 – Figure 4-12 |
| 11.  | Engineering Insights - ESW System                    | Section 5.2          | 47      | Figure 5-1 – Figure 5-4   |
| 12.  | Engineering Insights - HPI System                    | Section 5.3          | 50      | Figure 5-5 – Figure 5-8   |
| 13.  | Engineering Insights - AFW System                    | Section 5.4          | 52      | Figure 5-9 – Figure 5-12  |
| 14.  | Engineering Insights - RHR (PWR) System              | Section 5.5          | 55      | Figure 5-13 – Figure 5-16 |
| 15.  | Engineering Insights – Standby Liquid Control System | Section 5.6          | 57      | Figure 5-17 – Figure 5-20 |
| 16.  | Engineering Insights – RHR (BWR) System              | Section 5.7          | 60      | Figure 5-21 – Figure 5-24 |
| 17.  | Data Summaries                                       | Appendix A, B, and C |         |                           |

The application of results to plant-specific applications may require a more detailed review of the relevant Licensee Event Report (LER) and Nuclear Plant Reliability Data System (NPRDS) or Equipment

Performance Information and Exchange System (EPIX) data cited in this report. This review is needed to determine if generic experiences described in this report and specific aspects of the pump CCF events documented in the LER and NPRDS failure records are applicable to the design and operational features at a specific plant or site. Factors such as system design, specific pump components installed in the system, and test and maintenance practices would need to be considered in light of specific information provided in the LER and NPRDS failure records. Other documents such as logs, reports, and inspection reports that contain information about plant-specific experience (e.g., maintenance, operation, or surveillance testing) should be reviewed during plant inspections to supplement the information contained in this report.

Additional insights may be gained about plant-specific performance by examining the specific events in light of overall industry performance. In addition, a review of recent LERs and plant-specific component failure information in NPRDS or EPIX may yield indications of whether performance has undergone any significant change since the last year of this report. NPRDS archival data (through 1996) and EPIX failure data are proprietary information that can be obtained from the EPIX database through the Institute of Nuclear Power Operations (INPO). NRC staff and contractors can access that information through the EPIX database.

Common-cause failures used in this study were obtained from the common-cause failure database maintained for the NRC by the INEEL. NRC staff and contractors can access the plant-specific CCF information through the CCF database that is available on CD-ROM and has been provided to the NRC Regions and NRC Office of Nuclear Reactor Regulation (NRR). To obtain access to the NRC CCF Database, contact Dale Rasmuson [dmr@nrc.gov; (301) 415-7571] at the NRC or S. Ted Wood at the INEEL [stw@inel.gov; (208) 526-8729].

Periodic updates to the information in this report will be performed, as additional data become available. In the future, these insights will be available on the RES internal web page.

Scott F. Newberry, Director  
Division of Risk Analysis & Applications  
Office of Nuclear Regulatory Research



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## ACRONYMS

|       |   |
|-------|---|
| AFW   | auxiliary feedwater (PWR)                               |
| BWR   | boiling water reactor                                   |
| CCCG  | common-cause failure component group                    |
| CCF   | common-cause failure                                    |
| CCP   | centrifugal charging pump                               |
| CSR   | containment spray recirculation (PWR)                   |
| CVC   | chemical and volume control                             |
| EPIX  | equipment performance and information exchange          |
| ESW   | emergency service water                                 |
| FTR   | fail-to-run   |
| FTS   | fail-to-start   |
| HCI   | high pressure coolant injection (BWR)                   |
| HPI   | high pressure safety injection (PWR)                    |
| INEEL | Idaho National Engineering and Environmental Laboratory |
| INPO  | Institute of Nuclear Power Operations                   |
| IPE   | individual plant examination                            |
| LCS   | low pressure core spray (BWR)                           |
| LER   | licensee event report                                   |
| LOCA  | loss of coolant accident                                |
| MDAFP | motor-driven auxiliary feedwater pump                   |
| MDP   | motor-driven pump                                       |
| NPP   | nuclear power plant                                     |
| NPRDS | Nuclear Plant Reliability Data System                   |
| NPSH  | net positive suction head                               |
| NRC   | Nuclear Regulatory Commission                           |
| PRA   | probabilistic risk assessment                           |
| PWR   | pressurized water reactor                               |
| RCI   | reactor core isolation cooling (BWR)                    |
| RCS   | reactor coolant system                                  |
| RHR   | residual heat removal                                   |
| RHR-B | residual heat removal (BWR)                             |
| RHR-P | residual heat removal (PWR)                             |
| RHRSW | residual heat removal service water                     |
| RWST  | refueling water storage tank                            |
| SCSS  | Sequence Coding and Search System                       |
| SDC   | shutdown cooling (BWR)                                  |

|     |                              |
|-----|------------------------------|
| SG  | steam generator              |
| SI  | safety injection             |
| SLC | standby liquid control (BWR) |
| SW  | service water                |
| TDP | turbine-driven pump          |
| USI | unresolved safety issue      |

## GLOSSARY

*Application*—A particular set of CCF events selected from the common-cause failure database for use in a specific study.

*Average Impact Vector*—An average over the impact vectors for different hypotheses regarding the number of components failed in an event.

*Basic Event*—An event in a reliability logic model that represents the state in which a component or group of components is unavailable and does not require further development in terms of contributing causes.

*Common-cause Event*—A dependent failure in which two or more component fault states exist simultaneously, or within a short time interval, and are a direct result of a shared cause.

*Common-cause Basic Event*—In system modeling, a basic event that represents the unavailability of a specific set of components because of shared causes that are not explicitly represented in the system logic model as other basic events.

*Common-cause Component Group*—A group of (usually similar [in mission, manufacturer, maintenance, environment, etc.]) components that are considered to have a high potential for failure due to the same cause or causes.

*Common-cause Failure Model*—The basis for quantifying the probability of common-cause events. Examples include the beta factor, alpha factor, basic parameter, and the binomial failure rate models.

*Component*—An element of plant hardware designed to provide a particular function.

*Component Boundary*—The component boundary encompasses the set of piece parts that are considered to form the component.

*Component Degradation Value*—The assessed probability ( $0.0 \leq p \leq 1.0$ ) that a functionally- or physically-degraded component would fail to complete the mission.

*Component State*—Component state defines the component status in regard to its intended function. Two general categories of component states are defined, available, and unavailable.

*Available*—The component is available if it is capable of performing its function according to a specified success criterion. (N.B., available is not the same as availability.)

*Unavailable*—The component is unavailable if the component is unable to perform its intended function according to a stated success criterion. Two subsets of unavailable states are failure and functionally unavailable.

*Coupling Factor/Mechanism*—A set of causes and factors characterizing why and how a failure is systematically induced in several components.

*Date*—The date of the failure event, or date the failure was discovered.

*Defense*—Any operational, maintenance, and design measures taken to diminish the probability and/or consequences of common-cause failures.

*Degree of Failure*—The Degree of Failure category has three groups: Complete, Almost Complete, and Partial. The degree of failure is a categorization of a CCF event by the magnitude of three quantification parameters: component degradation value, shared cause factor, and timing factor. These parameters can be given values from zero to 1.0. The degree of failure categories are defined as follows:

*Complete*—A common-cause failure in which all redundant components are failed simultaneously as a direct result of a shared cause; i.e., the component degradation value equals 1.0 for all components, and both the timing factor and the shared cause factor are equal to 1.0.

*Almost Complete*—A common-cause failure in which one of the parameters is not equal to 1.0. Examples of events that would be termed Almost Complete are: events in which most components are completely failed and one component is degraded, or all components are completely failed but the time between failures is greater than one inspection interval.

*Partial*—All other common-cause failures (i.e., more than one of the quantification parameters is not equal to 1.0.)

*Dependent Basic Events*—Two or more basic events, A and B, are statistically dependent if, and only if,

$$P[A \cap B] = P[B | A]P[A] = P[A | B]P[B] \neq P[A]P[B],$$

where  $P[X]$  denotes the probability of event X.

*Event*—An event is the occurrence of a component state or a group of component states.

*Exposed Population*—The set of components within the plant that are potentially affected by the common-cause failure event under consideration.

*Failure*—The component is not capable of performing its specified operation according to a success criterion.

*Failure Mechanism*—The history describing the events and influences leading to a given failure.

*Failure Mode*—A description of component failure in terms of the component function that was actually or potentially unavailable.

*Failure Mode Applicability*—The analyst's probability that the specified component failure mode for a given event is appropriate to the particular application.

*Functionally Unavailable*—The component is capable of operation, but the function normally provided by the component is unavailable due to lack of proper input, lack of support function from a source outside the component (i.e., motive power, actuation signal), maintenance, testing, the improper interference of a person, etc.

*Impact Vector*—An assessment of the impact an event would have on a common-cause component group. The impact is usually measured as the number of failed components out of a set of similar components in the common-cause component group.

*Independent Basic Events*—Two basic events, A and B, are statistically independent if, and only if,

$$P[A \cap B] = P[A]P[B],$$

where  $P[X]$  denotes the probability of event X.

*Mapping*—The impact vector of an event must be “mapped up” or “mapped down” when the exposed population of the target plant is higher or lower than that of the original plant that experienced the common-cause failure. The result of mapping an impact vector is an adjusted impact vector applicable to the target plant.

*Mapping Up Factor*—A factor used to adjust the impact vector of an event when the exposed population of the target plan is higher than that of the original plant that experienced the common-cause failure.

*P-Value*—A p-value is a probability, that indicates a measure of statistical significance. The smaller the p-value, the greater the significance. A p-value of less than 0.05 is generally considered statistically significant.

*Potentially Unavailable*—The component is capable of performing its function according to a success criterion, but an incipient or degraded condition exists. (N.B., potentially unavailable is not synonymous with hypothetical.)

*Degraded*—The component is in such a state that it exhibits reduced performance but insufficient degradation to declare the component unavailable according to the specified success criterion.

*Incipient*—The component is in a condition that, if left un-remedied, could ultimately lead to a degraded or unavailable state.

*Proximate Cause*—A characterization of the condition that is readily identified as leading to failure of the component. It might alternatively be characterized as a symptom.

*Reliability Logic Model*—A logical representation of the combinations of component states that could lead to system failure. A fault tree is an example of a system logic model.

*Root Cause*—The most basic reason for a component failure, which, if corrected, could prevent recurrence. The identified root cause may vary depending on the particular defensive strategy adopted against the failure mechanism.

*Shared-Cause Factor (c)*—A number that reflects the analyst’s uncertainty ( $0.0 \leq c \leq 1.0$ ) about the existence of coupling among the failures of two or more components, i.e., whether a shared cause of failure can be clearly identified.

*Shock*—A shock is an event that occurs at a random point in time and acts on the system; i.e., all the components in the system simultaneously. There are two kinds of shocks distinguished by the potential impact of the shock event, i.e., lethal and nonlethal.

*Statistically Significant*—The term “statistically significant” means that the data are too closely correlated to be attributed to chances and consequently have a systematic relationship.

*System*—The entity that encompasses an interacting collection of components to provide a particular function or functions.

*Timing Factor ( $q$ )* —The probability ( $0.0 \leq q \leq 1.0$ ) that two or more component failures (or degraded states) separated in time represent a common-cause failure. This can be viewed as an indication of the strength-of-coupling in synchronizing failure times.



# Common-Cause Failure Event Insights for Pumps

## 1. INTRODUCTION

This report presents insights about the common-cause events that have occurred in the pump (pump) system at operating nuclear power plants.

The insights for the U.S. plants are derived from information captured in the common-cause failure (CCF) database maintained for the Nuclear Regulatory Commission (NRC) by the Idaho National Engineering and Environmental Laboratory (INEEL). The database contains CCF-related events that have occurred in U.S. commercial nuclear power plants reported in licensee event reports (LERs) and reports to the Nuclear Plant Reliability Data System (NPRDS) and the Equipment Performance Information Exchange (EPIX) system maintained by the Institute for Nuclear Power Operations (INPO)

The information presented in this report is intended to help focus NRC inspections on the more risk-important aspects of pump CCF events. Utilities can also use the information to help focus maintenance and test programs such that pump CCF events are minimized.

### 1.1 Background

The following four criteria must be met for an event to be classified as resulting from a common-cause:

- Two or more individual components must fail or be degraded, including failures during demand, inservice testing, or from deficiencies that would have resulted in a failure if a demand signal had been received;
- Two or more individual components must fail or be degraded in a select period of time such that the probabilistic risk assessment (PRA) mission would not be certain;
- The component failures or degradations must result from a single shared cause and coupling mechanism; and
- The component failures are not due to the failure of equipment outside the established component boundary.

To help resolve NRC Generic Issue 145,<sup>1</sup> *Actions to Reduce Common-Cause Failures*, and to address deficiencies related to the availability and analysis of CCF data, the NRC and the INEEL developed a CCF database that codifies information on CCF-related events that have occurred in U.S. commercial nuclear power plants from 1980 to date. The data is derived from both licensee event reports (LERs) submitted to the NRC and equipment performance reports submitted to the INPO. Accompanying the development of the CCF database was the development of CCF analysis software for investigating the CCF aspect of system reliability analyses and related risk-informed applications.

The quantitative results of this CCF data collection effort are described in the four volumes of NUREG/CR-6268, *Common-Cause Failure Database and Analysis System*.<sup>2,3,4,5</sup> Some quantitative insights about the data for use in PRA studies were also published in NUREG/CR-5497,<sup>6</sup> *Common-Cause Failure Parameter Estimations*. Copies of the CCF database together with supporting technical documentation and the analysis software are available on CD-ROM from the NRC to aid in system reliability analyses and risk-informed applications.

The CCF event data collected, classified, and compiled in the CCF database provide a unique opportunity to go beyond just estimation of CCF frequencies but to also gain more engineering insights into how and why CCF events occur. The data classification employed in the database was designed with this broader objective in mind. The data captured includes plant type, system component, piece parts, failure causes, mechanisms of propagation of failure to multiple components, their functional and physical failure modes. Other important characteristics such as defenses that could have prevented the failures are also included.

Section 1.2 of Volume 3 of NUREG/CR-6268 (Reference 4) proposes methods for classifying common-cause failures using the concepts of causes, coupling factors, and defensive mechanisms. The methods suggest a causal picture of failure with an identification of a root cause, a means by which the cause is more likely to impact a number of components simultaneously (the coupling), and the failure of the defenses against such multiple failures. Utilizing these methods, the CCF data associated with pump systems were analyzed to provide a better understanding of pump CCFs. This report presents the results of this effort.

The data analyzed are derived from the CCF database. The coding and quality assurance (QA) process for entering data into the database is as follows: Each event is coded from an LER or an NPRDS or EPIX report by analysts at the INEEL. Each analyst has access to coding guidelines (NUREG/CR-6268), which provides specific direction to the analyst about what the required information means and how to enter the information into the database. Each analyst is knowledgeable about PRA and plant systems and operations. Each event is initially coded by one analyst and reviewed by another analyst with a comparable background. Any disagreement is resolved before coding of the event is considered completed. An additional review of the events is done by another person familiar with PRA and CCF concepts. An independent outside expert in CCF and PRA then reviews the coding. Any differences are resolved and the final coding changes made in the database. The data collection, analysis, independent review, and quality assurance process are described in more detail in NUREG/CR-6268, Volumes 1 and 3 (References 2 and 4).

## **1.2 Common-Cause Failure Event Concepts**

CCFs can be thought of as resulting from the coexistence of two main factors: one that provides a susceptibility for components to fail or become unavailable due to a particular cause of failure and a coupling factor (or coupling mechanism) that creates the condition for multiple components to be affected by the same cause.

An example is a case where two relief valves fail-to-open at the required pressure due to set points being set too high. Because of personnel error (the proximate cause), each of the two valves fails due to an incorrect setpoint. What makes the two valves fail together, however, is a common calibration procedure and common maintenance personnel. These commonalities are the coupling factors of the failure event in this case.

Characterization of CCF events in terms of these key elements provides an effective means of performing engineering assessments of the CCF phenomenon including approaches to identification of plant vulnerabilities to CCFs and evaluation of the need for, and effectiveness of, defenses against them. It is equally effective in evaluation and classification of operational data and quantitative analysis of CCF frequencies.

It is evident that each component fails because of its susceptibility to the conditions created by the root cause, and the role of the coupling factor is to make those conditions common to several components. In analyzing failure events, the description of a failure in terms of the most obvious "cause" is often too

simplistic. The sequence of events that constitute a particular failure mechanism is not necessarily simple. Many different paths by which this ultimate reason for failure could be reached exist. This chain can be characterized by two useful concepts— proximate cause and root cause.

The proximate cause of a failure event is the condition that is readily identifiable as leading to the failure. The proximate cause can be regarded as a symptom of the failure cause, and it does not in itself necessarily provide a full understanding of what led to that condition. As such, it may not be the most useful characterization of failure events for the purposes of identifying appropriate corrective actions. The proximate cause classification consists of six major categories:

- Design, construction, installation, and manufacture inadequacy causes,
- Operational and human-related causes (e.g. procedural errors, maintenance errors),
- Internal to the component, including hardware-related causes and internal environmental causes,
- External environmental causes,
- State of other component, and
- Other causes.

The causal chain can be long and, without applying a criterion identifying an event in the chain as a “root cause,” is often arbitrary. Identifying root causes in relation to the implementation of defenses is a useful alternative. The root cause is therefore the most basic reason or reasons for the component failure, which if corrected, would prevent recurrence. Volume 3 of NUREG/CR-6268 (Reference 4) contains additional details on the cause categories and how CCF event causes are classified.

The coupling factor is a characteristic of a group of components or piece parts that identifies them as susceptible to the same causal mechanisms of failure – it is a characteristic that links the components. Such factors include similarity in design, location, environment, mission, and operational, maintenance, and test procedures. Coupling factors are categorized into the following five groups for analysis purposes:

- Hardware Quality,
- Hardware Design,
- Maintenance,
- Operations, and
- Environment.

Note that proximate causes of CCF events are no different from the proximate causes of single component failures.

The proximate causes and the coupling factors may appear to overlap because the same name is sometimes used as a proximate cause and as a coupling factor (e.g., design, maintenance). However, they are different. For example, maintenance, as a proximate cause, refers to errors and mistakes made during maintenance activities. As a coupling factor, maintenance refers to the similarity of maintenance among the components (e.g., same maintenance personnel, same maintenance procedures).

The defense or defensive mechanism is any operational, maintenance, or design measure taken to diminish the probability and/or consequences of a common-cause failure event. Three ways of defending against a CCF event are the following: (1) defend against the failure proximate cause, (2) defend against

the coupling factor, or (3) defend against both the proximate cause and the coupling factor. As an example, consider two redundant components in the same room as a steam line. A barrier that separates the steam line from the components is an example of defending against the proximate cause. A barrier that separates the two components is an example of defending against the coupling factor (same location). Installing barriers around each component is an example of defending against both the cause and the coupling factor.

Proximate causes of CCF events are no different from the proximate causes of single component failures. This observation suggests that defending against single component failures can have an impact on CCFs as well. Most corrective actions usually attempt to reduce the frequency of failures (single or multiple). That is, very often the approach to defending against CCFs is to defend against the cause, not the coupling. Given that a defensive strategy is established based on reducing the number of failures by addressing proximate causes, it is reasonable to postulate that if fewer component failures occur, fewer CCF events would occur.

Defenses against causes result in improving the reliability of each component but do not necessarily reduce the fraction of failures that occur due to common-cause. They typically include design control, use of qualified equipment, testing and preventive maintenance programs, procedure review, personnel training, quality control, redundancy, diversity, and barriers. It is important to remember that the susceptibility of a system of redundant components to dependent failures as opposed to independent failures is determined by the presence of coupling factors.

The above cause-defense approach does not address the way that failures are coupled. Therefore, CCF events can occur, but at a lower probability. If a defensive strategy is developed using protection against a coupling factor as a basis, the relationship among the failures is eliminated. A search for coupling factors is primarily a search for similarities among components. A search for defenses against coupling, on the other hand, is primarily a search for dissimilarities among components, including differences in the components themselves (diversity); differences in the way they are installed, operated, and maintained; and in their environment and location.

During a CCF analysis, a defense based on a coupling factor is easier to assess because the coupling mechanism among failures is more readily apparent and therefore easier to interrupt. The following defenses are oriented toward eliminating or reducing the coupling among failures: diversity, physical or functional barriers, and testing and maintenance policies. A defensive strategy based on addressing both the proximate cause and coupling factor would be the most comprehensive.

A comprehensive review should include identification of the root causes, coupling factors, and defenses in place against them. However, as discussed in NUREG/CR-5460,<sup>7</sup> *A Cause-Defense Approach to the Understanding and Analysis of Common-Cause Failures*, given the rarity of common-cause events, current weaknesses of event reporting and other practical limitations, approaching the problem from the point of view of defenses is, perhaps, the most effective and practical. A good defense can prevent a whole class of CCFs for many types of components, and in this way, the application of a procedure based on this philosophy can provide a systematic approach to screening for potential CCF mechanisms.

### **1.3 Report Structure**

This report presents an overview of the pump CCF data and insights into the characteristics of that data. This report is organized as follows: Section 2 presents a description of the pump, a short description of the associated segments, and a definition of the pump failure modes. High-level insights of all the pump CCF data are presented in Section 3. Section 4 summarizes the events by segment. Section

5 presents pump CCF insights by selected systems. Section 6 explains how to obtain more detailed information for the pump events. A glossary of terms used in this report is included in the front matter. Appendix A contains three listings of the pump CCF events sorted by proximate cause, coupling factor, and discovery method. Appendix B contains a listing of the pump CCF events sorted by the sub-component. Appendix C contains a listing of the pump CCF events sorted by the system.



## **2. PUMP COMPONENT DESCRIPTION**

### **2.1 Introduction**

Pumps are used in many safety-related systems at commercial nuclear utilities. Pumps are installed in redundant configurations to ensure the movement of water under accident conditions. Pumps provide water to makeup for the loss of inventory, loss of pressure, cooling, and the addition of chemical poisons. Many of these systems use the pumps in more than one mode of operation.

The pumps in this study are normally in standby, except for the emergency service water pumps and the chemical and volume control system pumps include in the HPI system. The systems containing pumps included in this insights study include:

- AFW      Auxiliary Feedwater System (PWR)
- CSR      Containment Spray Recirculation (PWR)
- ESW      Emergency Service Water
- HCI      High Pressure Coolant Injection (BWR)
- HPI      High Pressure Safety Injection (PWR)
- RHR-B    Residual Heat Removal (BWR)
- LCS      Low Pressure Core Spray (BWR)
- RHR-P    Residual Heat Removal (PWR)
- RCI      Reactor Core Isolation Cooling (BWR)
- SDC      Shutdown Cooling (BWR)
- SLC      Standby Liquid Control (BWR)

### **2.2 Risk Significance**

The emergency core cooling system (ECCS) is designed to supply sufficient water to the reactor vessel and reactor coolant system (RCS) to keep the core covered and to remove decay heat in the event of a loss of coolant inventory or normal core cooling. Thus, the ECCS systems play significantly in transients with a loss of secondary cooling (including loss of off-site power and station blackout), and loss of coolant accidents (LOCAs).<sup>8</sup> In general, the motor-driven and turbine-driven pumps are the most risk-important component and common-cause failures of the pumps are routinely the dominant risk contributors for the ECCS systems.

The auxiliary feedwater system (AFW) in PWRs provides a means of removing decay heat using the secondary system when the normal feedwater system is not available. The most common demands for AFW are transients with loss of secondary heat removal and loss of off-site power (including station blackout), two prominent risk contributors in PWRs. Individually, the system pumps are risk significant. Although most AFW systems employ diversity to combat common-cause failures (motor-driven and turbine-driven pumps), such failures are still significant.<sup>9</sup>

### **2.3 Component Description and Boundary**

The pumps in the systems listed above have varying characteristics such as discharge pressure, flow rate, number of stages, suction type, discharge point, and control systems. However, all pumps have a set of similar characteristics that are of interest when examining failures. Therefore, we define the pump component as the combination of the suction source, the driver, the pump, and the discharge. In

this study, we will look at the segments as well as the overall pump component. Figure 2-1 shows the component boundary as defined for this study.

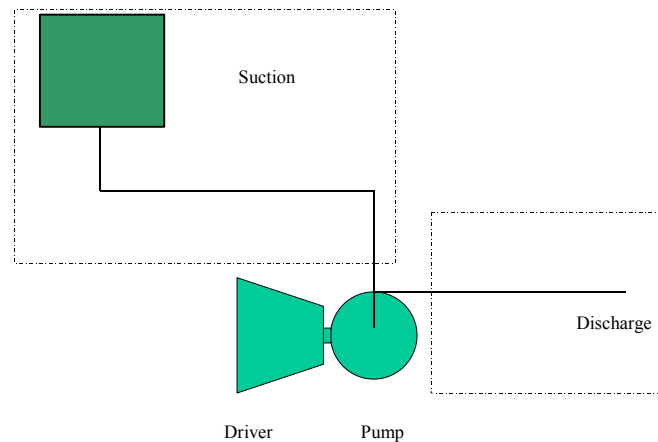


Figure 2-1. Pump component boundary drawing.

## 2.4 Segment Description

This section contains a brief description of each of the segments that comprise the pump. These descriptions are intended only to provide a general overview of the most common pumps. Failure of the pump due to external components (e.g., MOVs, check valves, and strainers) required that the components were not failed, but inhibited the pump. Otherwise, these types of components would have been classified as a failure of the specific component.

### 2.4.1 Pump

The pump segment performs the function of converting rotational energy to fluid kinetic energy to move fluid from the suction to the discharge flow paths. Most of the pumps in this study are centrifugal type pumps. The SLC system and some of the coolant charging systems employ reciprocating positive displacement pumps. The pump may include the bearings, couplings, impeller/wear rings, shaft, packing/seals, casing, lubrication, supports, and the plungers and cylinders (positive displacement).

### 2.4.2 Driver

The driver segment performs the function of providing the motive force to the pump. Most pumps are motor-driven. The driver may include the circuit breaker, bearings, lubrication system, cooling system, rotor, gearbox (positive displacement and some turbine-driven pumps), instrumentation and controls, motor or turbine, and power cables.



### 2.4.3 Suction

The suction segment performs the function of supplying the fluid to the pump. The suction includes the supply tank or other water source; manual, power-operated, or check valves; strainers; and piping. Suction segment failures are evaluated to determine the effect on pump operability. Insufficient net positive suction head (NPSH) is generally the type of event that occurs in the suction path. Low levels in water source, high temperature in the suction source, or plugged strainers are typical examples.

### 2.4.4 Discharge

The discharge segment performs the function of directing fluid to the desired flow path. The discharge includes manual valves, power-operated valves, relief valves, check valves, the recirculation flow path, and piping. Discharge segment failures are evaluated to determine the effect on pump operability. The state of the valves in the discharge path, insufficient recirculation flow, pipe leaks, etc. are typical examples.

## 2.5 System Descriptions

Figure 2-2 to Figure 2-9 are shown to provide the reader with generic representations of the system configurations discussed within this document.

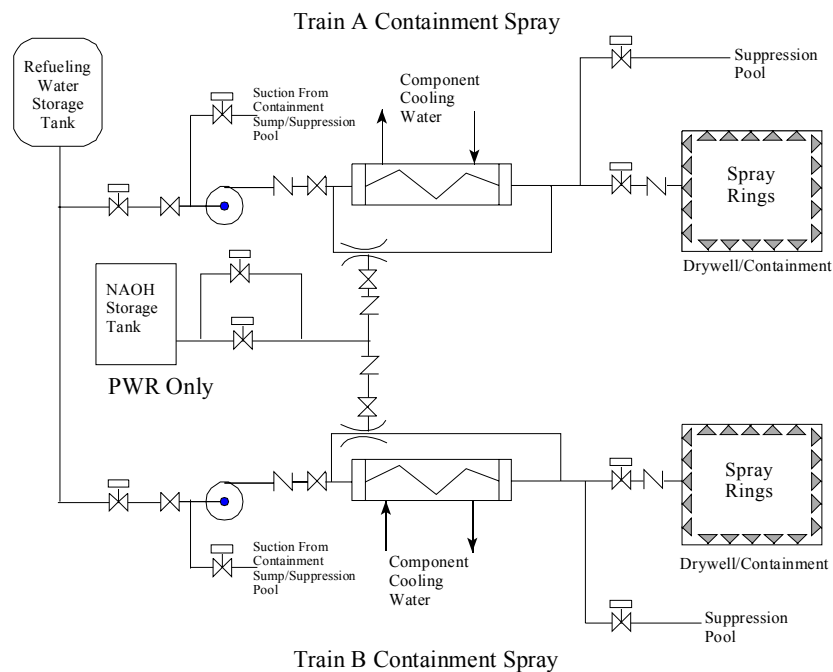


Figure 2-2. PWR Containment Spray system diagram.

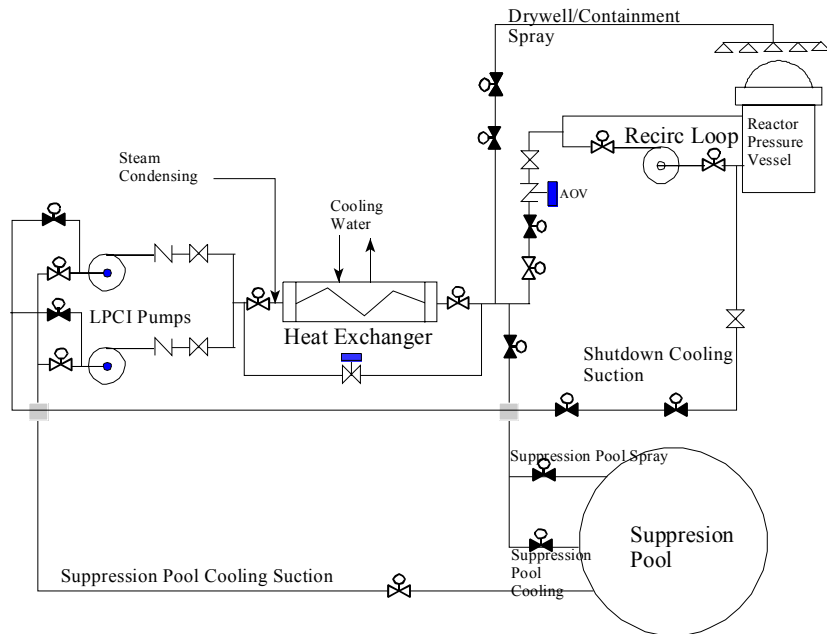


Figure 2-3. BWR RHR system diagram.

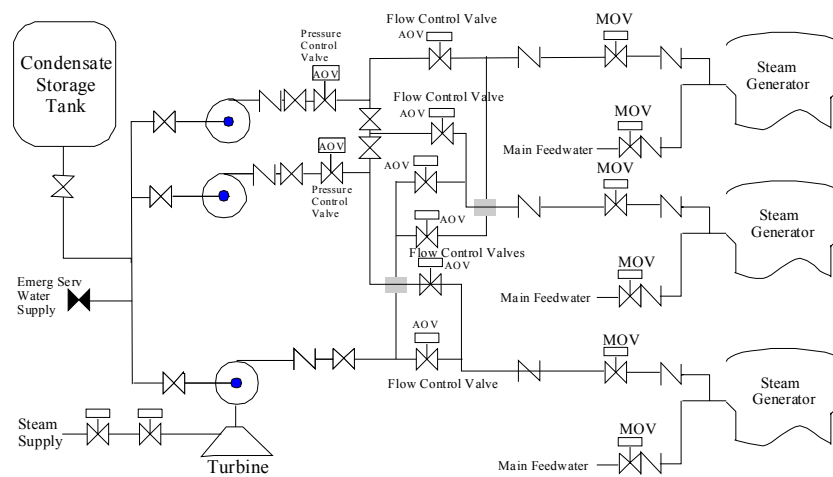


Figure 2-4. PWR Auxiliary Feedwater system diagram.

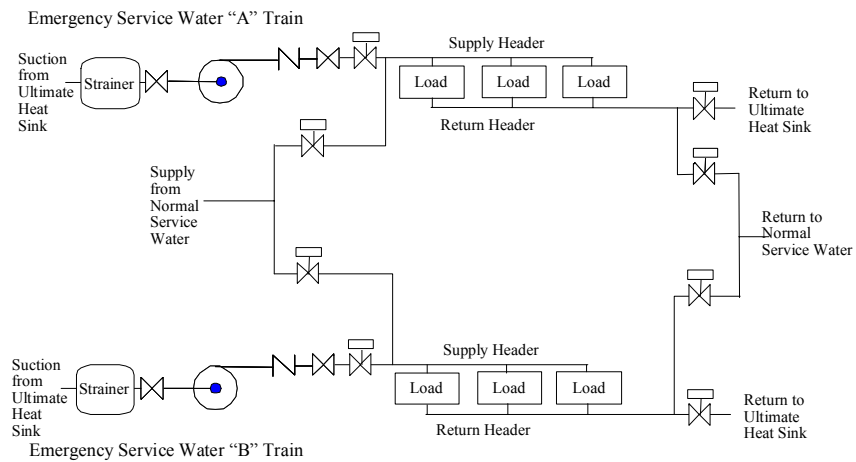


Figure 2-5. Generic Emergency Service Water system diagram.

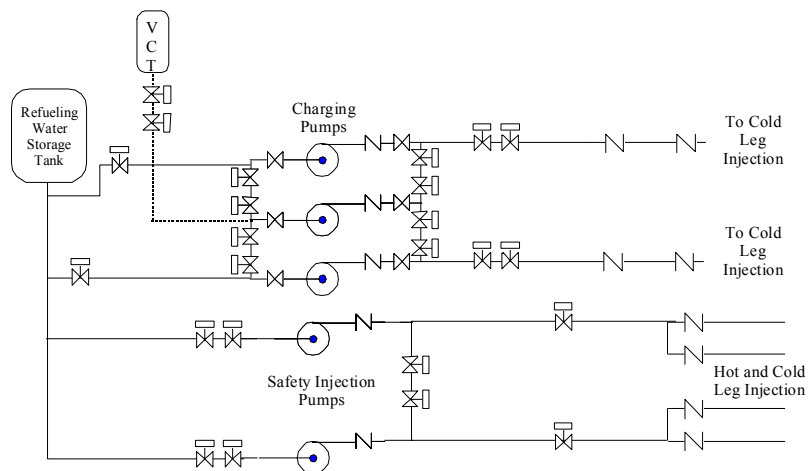


Figure 2-6. PWR High Pressure Safety Injection and Coolant Charging system diagram.

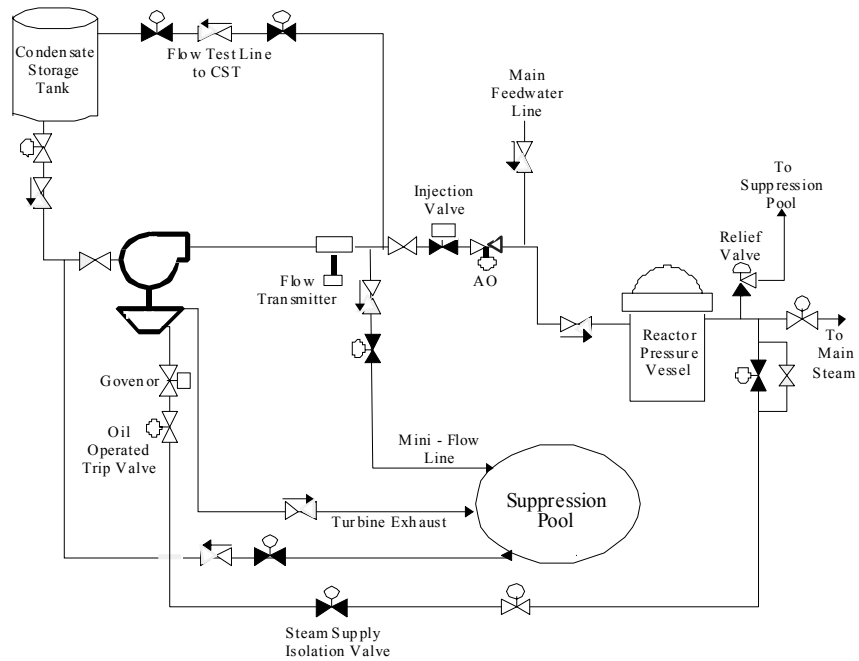


Figure 2-7. BWR High Pressure Coolant Injection and Reactor Core Isolation Cooling systems diagram.

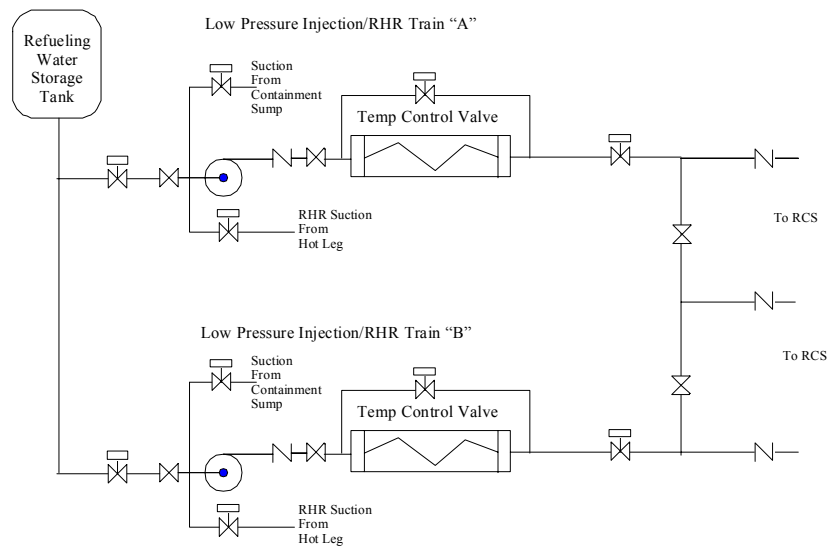


Figure 2-8. PWR RHR system diagram.

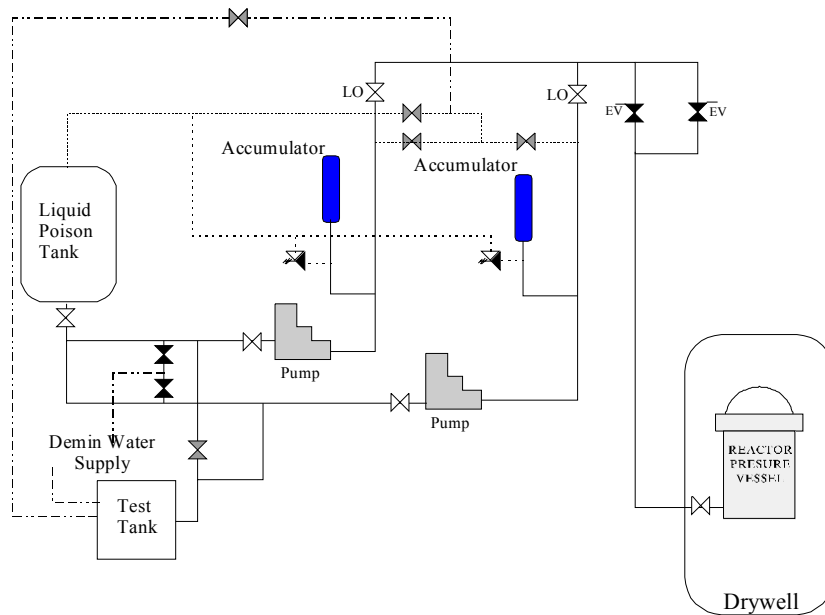


Figure 2-9. BWR Standby Liquid Control system diagram.

## 2.6 Failure Modes

Successful operation of a pump is defined for two distinct modes of operation. If the system is in the normal standby condition, it must respond to an actuation signal by starting, which consists of obtaining design discharge pressure and flow. Once running, the pump must continue to produce design flow and discharge pressure until its service is no longer needed. Failures that occurred during testing are included with the failures that occurred during plant transients requiring operation of the pumps. The respective failure modes used for evaluating the pump data are:

- |                     |  |
|---------------------|--|
| Fail-to-start (FTS) | A successful pump start is defined as the start of the pump up to the point where design flow (or minimum flow) and discharge pressure are achieved.   |
| Fail-to-run (FTR)   | A successful pump run is defined, as the continuation of full flow and discharge pressure for the time the pump is needed. In the cases where some degradation of the pump is observed, a determination is made as to the ability of the pump to perform throughout its PRA mission time (typically 24 hours). |

Pump segment failures are evaluated to determine the effect on pump operability. Pump failures include those failures that are caused by pump internals such as the impeller/wearing rings, bearings, lubrication, packing, etc.

Driver segment failures are evaluated to determine the effect on pump operability. Failures of the sensors or control circuitry to provide input in other systems (e.g., interlocks or indication) are not considered pump failures.

Suction segment failures are evaluated to determine the effect on pump operability. Insufficient net positive suction head (NPSH) is generally the type of event that occurs in the suction path. Low levels in water source, high temperature in the suction source, or plugged strainers are typical examples.

Discharge segment failures are evaluated to determine the effect on pump operability. The state of the valves in the discharge path, insufficient recirculation flow, and pipe leaks are typical examples.

Failure of the pump due to external components (e.g., MOVs, check valves, and strainers) required that the components were not failed, but inhibited the pump. Otherwise, these types of components would have been classified as a failure of the specific component.

### 3. HIGH LEVEL OVERVIEW OF PUMP INSIGHTS

#### 3.1 Introduction

This section provides an overview of CCF data for the pump component that has been collected from the NRC CCF database. The set of pump CCF events is based on industry data from 1980 to 2000. The pump CCF data contains attributes about events that are of interest in the understanding of: degree of completeness, trends, pump segment affected, causal factors, linking or coupling factors, and event detection methods.

Not all pump CCF events included in this study resulted in observed failures of multiple pumps. Many of the events included in the database, in fact, describe degraded states of the pumps where, given the conditions described, the pumps may or may not have performed as required. The CCF guidance documents (References 3 and 4) allow the use of three different quantification parameters (component degradation value, shared cause factor, and timing factor) to measure degree of failure for CCF events. Based on the values of these three parameters, a Degree of Failure was assigned to each pump CCF event.

The Degree of Failure category has three groups—Complete, Almost Complete, and Partial. Complete CCF events are CCF events in which each component within the common-cause failure component group (CCCG) fails completely due to the same cause and within a short time interval (i.e., all quantification parameters equal 1.0). Complete events are important since they show us evidence of observed CCFs of all components in a common-cause group. Complete events also dominate the parameter estimates obtained from the CCF database. All other events are termed partial CCF events (i.e., at least one quantification parameter is not equal to 1.0). A subclass of partial CCF events are those that are Almost Complete CCF events. Examples of events that would be termed Almost Complete are: events in which most components are completely failed and one component is degraded, or all components are completely failed but the time between failures is greater than one inspection interval (i.e., all but one of the quantification parameters equal 1.0).

Table 3-1 summarizes, by failure mode and degree of failure, the pump CCF events contained in this study. The majority of the pump CCF events were fail-to-run (54 percent), suggesting that often the pump must be running at rated conditions for failures to develop and/or for those failures to be detected. While most events (68 percent) were classified as Partial, a significant fraction of events (32 percent) were classified as either Complete or Almost Complete.

Table 3-1. Summary statistics of pump data.

| Failure Mode        | Degree of Failure |                 |          | Total |
|---------------------|-------------------|-----------------|----------|-------|
|                     | Partial           | Almost Complete | Complete |       |
| Fail-to-Start (FTS) | 86                | 12              | 27       | 125   |
| Fail-to-Run (FTR)   | 101               | 13              | 35       | 149   |
| Total               | 187               | 25              | 62       | 274   |

## 3.2 CCF Trends Overview

Figure 3-1 shows the yearly occurrence rate, the fitted trend, and its 90 percent uncertainty bounds for all pump CCF events over the time span of this study. The decreasing trend is statistically significant<sup>a</sup> with a p-value<sup>b</sup> of 0.0001. There was insufficient information to determine what caused the decreasing trend in CCF events, but there were several regulatory initiatives by the NRC and industry initiatives by utilities, INPO, and EPRI involving improved operation, maintenance, testing, and inspection during the 21 years of improving performance. Examples of these initiatives include improvements in testing, inspection, and maintenance associated with Generic Letter 89-13, *Problems with Service Water Systems Affecting Safety-Related Components*<sup>10</sup>, and Generic Letter 89-04, *Guidance on Developing Acceptable Inservice Testing Programs*<sup>11</sup>. Additionally, the testing and examination code for pumps has been improved significantly since 1980.

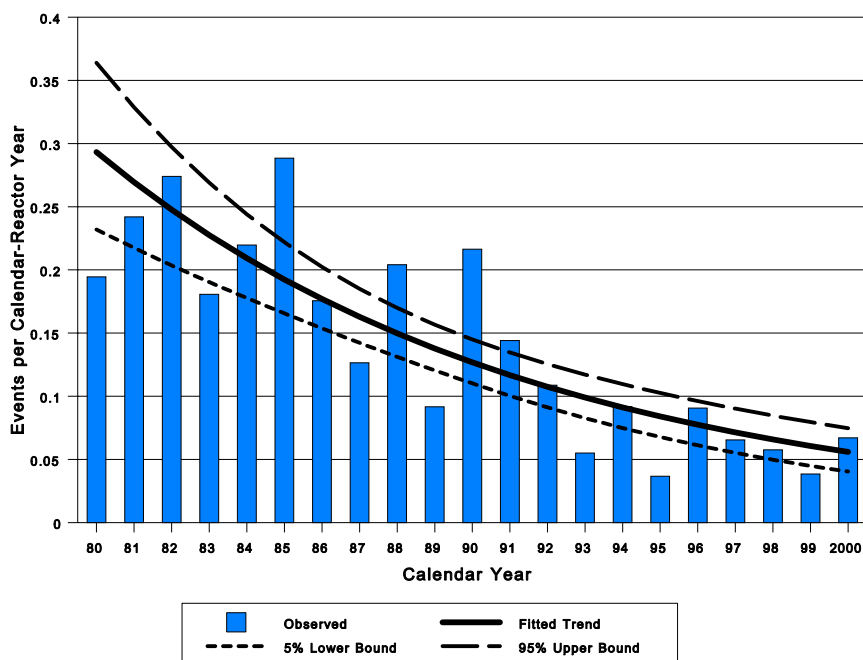


Figure 3-1. Trend for all pump CCF events. The decreasing trend is statistically significant with a p-value = 0.0001.

Figure 3-2 through Figure 3-4 show trends for subsets of the pump CCF events contained in Figure 3-1. Figure 3-2 shows the trend for Complete pump CCF events. The overall trend for Complete pump CCF events from 1980 to 2000 is also statistically significant with a p-value of 0.0001. This indicates a dramatic decrease of Complete pump CCF events, especially since the mid-1980's. Figure 3-3

a. The term “statistically significant” means that the data are too closely correlated to be attributed to chances and consequently have a systematic relationship. A p-value of less than 0.05 is generally considered to be statistically significant.

b. A p-value is a probability, with a value between zero and one, which is a measure of statistical significance. The smaller the p-value, the greater the significance. A p-value of less than 0.05 is generally considered statistically significant. A p-value of less than 0.0001 is reported as 0.0001.



and Figure 3-4 show similar statistically significant decreasing trends for both the fail-to-start and the fail-to-run failure modes for all pump CCF events, both with p-values of 0.0001.

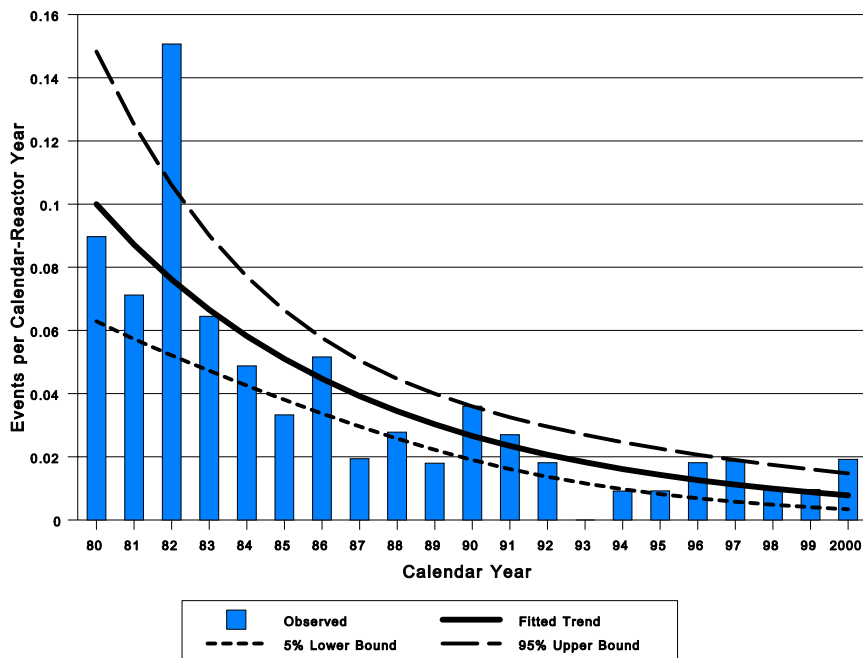


Figure 3-2. Trend for Complete pump CCF events. The decreasing trend is statistically significant with a p-value = 0.0001.

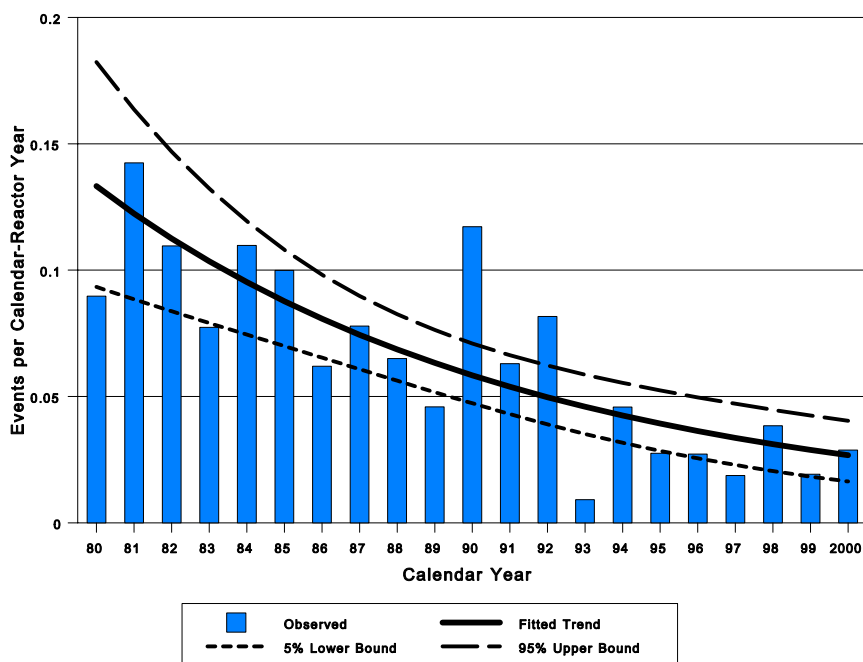


Figure 3-3. Trend for all pump CCF events for the fail-to-start failure mode. The decreasing trend is statistically significant with a p-value = 0.0001

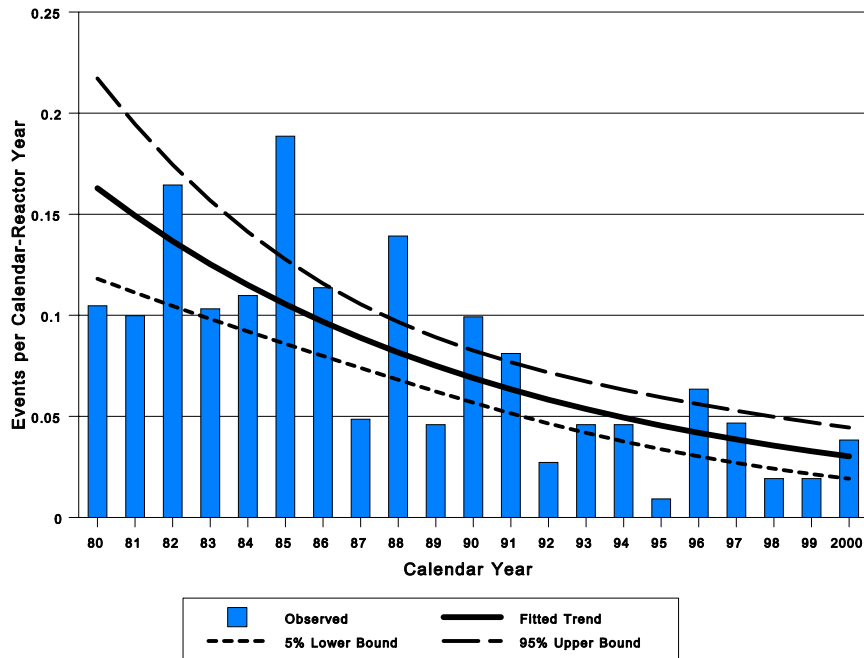


Figure 3-4. Trend for all pump CCF events for the fail-to-run failure mode. The decreasing trend is statistically significant with a p-value = 0.0001.

### 3.3 CCF Segment Overview

Pumps are complex machines and can easily be thought of as a collection of segments, each with many components. The pump CCF data were reviewed to determine the affected segment and the affected piece part in that segment. This was done to provide insights to the most vulnerable areas of the pump component to common-cause failure events. Section 2.4 describes these segments.

Figure 3-5 shows the distribution of the CCF events by pump segment. Overall, for all pumps, the highest number of events occurred in the pump segment (106 events or 39 percent). The driver and suction segments were also significant contributors (32 and 24 percent, respectively), while relatively few events involved the discharge segment. These statistics vary by system. For the ESW and SLC systems, most of the failures occurred in the pump segment. However, for the AFW, HPI, and RHR-B systems, most of the failures occurred in the driver segment, and for the RHR-P system, most of the failures occurred in the suction segment. Events involving the driver and suction segments were more likely to be Complete. Ninety-two percent of all Complete events occurred in these two segments. Section 4 of this report provides an in-depth analysis of the CCF events assigned to these segments.

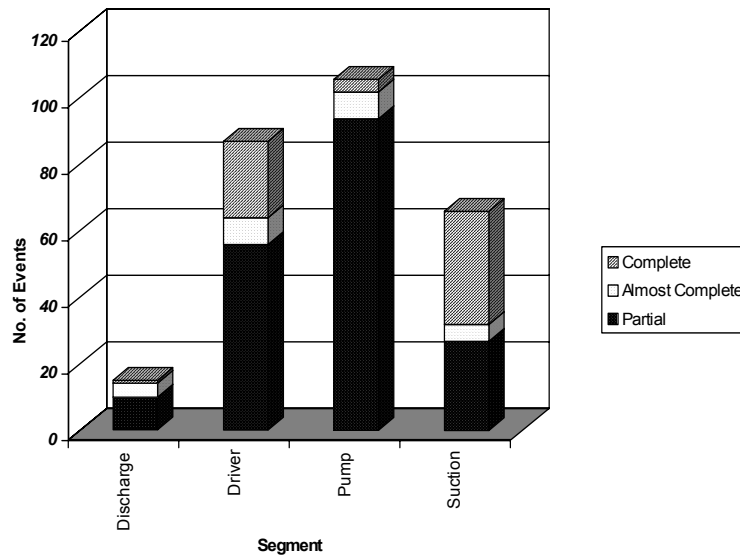


Figure 3-5. Segment distribution for all pump CCF events.

### 3.4 CCF Proximate Cause

It is evident that each component fails because of its susceptibility to the conditions created by the root cause, and the role of the coupling factor is to make those conditions common to several components. In analyzing failure events, the description of a failure in terms of the most obvious "cause" is often too simplistic. The sequence of events that constitute a particular failure mechanism is not necessarily simple. Many different paths by which this ultimate reason for failure could be reached exist. This chain can be characterized by two useful concepts— proximate cause and root cause.

A **proximate cause** of a failure event is the condition that is readily identifiable as leading to the failure. The proximate cause can be regarded as a symptom of the failure cause, and it does not in itself necessarily provide a full understanding of what led to that condition. As such, it may not be the most useful characterization of failure events for the purposes of identifying appropriate corrective actions.

The proximate cause classification consists of six major groups or classes:

- Design/Construction/Installation/Manufacture Inadequacy
- Operational/Human Error
- Internal to the component, including hardware-related causes and internal environmental causes
- External environmental causes
- Other causes
- Unknown causes.

The causal chain can be long and, without applying a criterion identifying an event in the chain as a "root cause," is often arbitrary. Identifying proximate causes in relation to the implementation of

defenses is a useful alternative. The proximate cause is therefore the most basic reason or reasons for the component failure, which if corrected, would prevent recurrence. (See Table 4-2 in Section 4.1 for a display of the major proximate cause categories and a short description.) Reference 4 contains additional details on the proximate cause categories, and how CCF event proximate causes are classified.

Figure 3-6 shows the distribution of CCF events by proximate cause. The leading proximate cause was Internal to Component, which accounted for about 39 percent of the total events; however, none of these events were Complete. Design/Construction/Installation/Manufacture Inadequacy and Human error accounted for 24 and 20 percent of the total events, respectively. The Other and External Environment proximate causes were attributed to a small fraction of the pump CCF events.

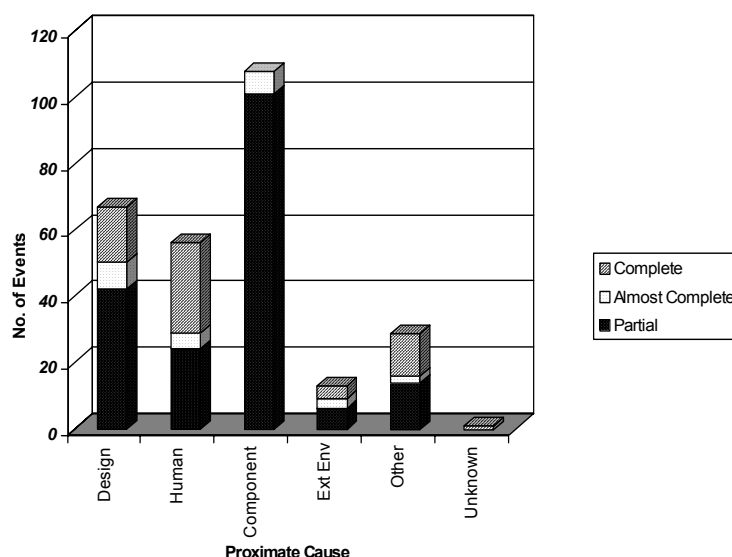


Figure 3-6. Proximate cause distribution for all pump CCF events.

Table A-1 in Appendix A presents the entire pump data set, sorted by the proximate cause. This table can be referred to when reading the following discussions to see individual events described.

The **Internal to Component** proximate cause category is dominant for pump events and involves the failure or malfunction of parts internal to the pump. Internal causes result from phenomena such as normal wear or other intrinsic failure mechanisms that are influenced by the ambient environment of the component. Specific mechanisms include erosion, corrosion, internal contamination, fatigue, wear-out, and end of life. Internal to Component failures resulted in 108 events. Of these, 61 events were classified as fail-to-run and 47 were fail-to-start. Although this is the dominant proximate cause group, there were no Complete failure events attributed to the Internal to Component proximate cause. This is because most failure mechanisms in this group are gradual in nature; infrequently causing all system components to fail at once. In addition, the lack of a large number of Complete events may be due to the method of discovery. The majority of events in this cause group were discovered by Testing. These data suggest that the testing programs are succeeding in finding and fixing gradual failures of pumps before full failure is observed.

The **Design/Construction/Installation/Manufacture Inadequacy** proximate cause category is the next most likely for pump events and encompasses events related to the design, construction,

installation, and manufacture of components, both before and after the plant is operational. Included in this category are events resulting from errors in equipment and system specifications, material specifications, and calculations. Events related to maintenance activities are not included. Design/Construction/Installation/Manufacture Inadequacy errors resulted in 67 events. The failure mode for 42 of these events was fail-to-run, and 25 events had fail-to-start as the failure mode. There were 17 Complete CCF events in this proximate cause group: 13 Complete events were fail-to-run and 4 were fail-to-start. The majority of these Complete events (11 out of 17) occurred in the Suction segment. Typically, these events were due to a lack of adequate NPSH due to design discrepancies. Instead of the loss of suction events being distributed over a large number of NPP units, two stations account for approximately 65 percent of the Suction segment CCF events with the Design, Construction, and Manufacturer proximate. The rest of the CCF events were relatively evenly distributed between the Driver segment and the Pump segment.

The **Operational/Human Error** proximate cause category is also likely for pump CCF events. This proximate cause category represents causes related to errors of omission or commission on the part of plant staff or contractor staff. Included in this category are accidental actions, failures to follow the correct procedures or following inadequate procedures for construction, modification, operation, maintenance, calibration, and testing. This proximate cause group may also include deficient training. Operational/Human Error was assigned to 56 pump CCF events. The majority of these events involved inadequate procedures and accidental action. The failure mode for 24 events was fail-to-run and 32 events had fail-to-start as the failure mode. Almost half (48 percent) of the pump CCF events in this cause category were Complete. This highlights the importance of maintenance and operations in the availability of the pump component. The majority of CCF events were discovered by either Demand or Inspection. The high number of events discovered by Demand is explained by the fact that human errors are prone to occur during operations involving system demands. In addition, maintenance personnel errors also show up when the system is called upon to function. However, for those events not discovered by system demands, Inspection discovered more events than Maintenance and Testing. Many of these events involved problems such as system misalignments, improper circuit breaker operations, Technical Specification violations (non-allowed combinations of systems/components out of service at the same time) that were discovered by plant operators. It is expected that routine Inspection would discover more of these events than Testing and Maintenance, which are conducted only periodically.

The **Other** proximate cause category is comprised of events that were caused by instrumentation and control circuit setpoint drift or failure components outside the defined pump component boundary. There were 29 events assigned to this cause category. The failure mode for 13 events was fail-to-run and 16 events had fail-to-start as the failure mode. Again, almost half (45 percent) of the pump CCF events in this cause category were Complete. The most common Complete events in this category involved an interlock dependent on either a temperature or pressure sensor that prevented pump start or an actual low level in the suction source. Therefore, this cause category is important although the total number of events was relatively small. Most of the events were discovered by Demand in lieu of Testing, Maintenance, and Inspection. This is expected due to the nature of CCF events in this proximate cause group. The dependencies outside the pump component that initiate these CCF events may not be the specific target of system component testing; therefore, it is reasonable that more events would be discovered during system operation than by less-frequent test surveillance. In addition, because CCF events that occur due to the state of other components typically are indirectly initiated by failure of other components, they may not be readily apparent during routine inspections and maintenance. Fourteen events (48 percent) affected the Driver segment. This is reasonable to expect because the pump Drivers are dependent on a large number of other components, such as circuit breakers, instruments, interlocks and controls. The other important segment is Suction, with 11 events. This is a reflection of the number of events in the RHR-P system related to loss of suction due to system configuration.

The **External Environment** proximate cause category represents causes related to a harsh environment that are not within the component design specifications. Specific mechanisms include chemical reactions, electromagnetic interference, fire or smoke, impact loads, moisture (sprays, floods, etc.), radiation, abnormally high or low temperature, vibration load, and acts of nature (high wind, snow, etc.). There were 13 pump CCF events in this cause category. The failure mode for eight events was fail-to-run, and five events had fail-to-start as the failure mode. There were four Complete CCF events in attributed to External Environment.

The **Unknown** proximate cause category is used when the cause of the component state cannot be identified. There was one Complete, fail-to-run event in this cause category that occurred in the Suction segment.

### 3.5 CCF Coupling Factor

Closely connected to the proximate cause is the concept of **coupling factor**. A coupling factor is a characteristic of a component group or piece parts that links them together so that they are more susceptible to the same causal mechanisms of failure. Such factors include similarity in design, location, environment, mission, and operational, maintenance, design, manufacturer, and test procedures. These factors have also been referred to as examples of coupling mechanisms, but because they really identify a potential for common susceptibility, it is preferable to think of these factors as characteristics of a common-cause component group. Reference 4 contains additional detail about the coupling factors.

The coupling factor classification consists of five major classes:

- Hardware Quality based coupling factors,
- Design-based coupling factors,
- Maintenance coupling factors,
- Operational coupling factors, and
- Environmental coupling factors.

Figure 3-7 shows the coupling factor distribution for the pump CCF events. Maintenance was the leading coupling factor with 111 events (40 percent). The next leading coupling factor was Design with 76 events (28 percent). While not the leading coupling factor, over half (51 percent) of the Design, coupled events were either Complete or Almost Complete. The Environmental and Operational coupling factors account for the majority of the remaining events (44 and 28 events, respectively). Only a small fraction of the events coupled by Environmental were Complete; however, over half (57 percent) of the events coupled by Operational were Complete. These Complete events were almost all coupled by inadequate operations procedures. Only 15 events were coupled by Quality, and three of these were Complete and affected the Driver segment.

Table A-2 in Appendix A presents the entire pump data set, sorted by the coupling factor. This table can be referred to when reading the following discussions to see individual events described.

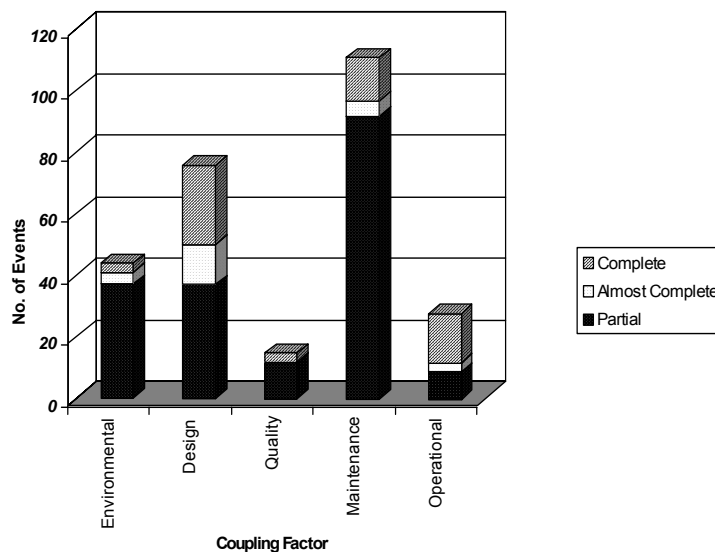


Figure 3-7. Coupling factor distribution for all pump CCF events.

The **Maintenance** coupling factor indicates that the maintenance frequency, procedures, or personnel provided the linkage among the events. Most of the pump CCF events with this coupling factor were coupled by maintenance/test schedules (74 out of 111) and maintenance/test procedures (23 out of 111). Internal to Component was the most prevalent proximate cause to be linked by maintenance (75 events). The maintenance linkage to the component failure proximate cause usually indicated that maintenance that is more frequent could have prevented the CCF mechanism. Very few of these events actually resulted in Complete CCF events, and most were detected as incipient failures. Examples of these are:

- The circuit breakers associated with the Auxiliary Feedwater Pumps failed to close as required. The cause of the failure was the binding in the operating mechanism due to accumulated dirt and lack of lubrication.
- The AFW pumps failed to start due to steam binding. The cause of the steam binding was determined to be leakage past the downstream AFW system check valves.
- Two of three ESW pumps failed to start on demand. The cause was determined to be bad couplings between the pumps and drivers. The cause was determined to be lack of periodic maintenance and inspection.
- The two gland seal retaining bolts inside the centrifugal charging pump speed increaser lube oil pump were found to be backed out allowing the gland seal to loosen. This resulted in reduced oil flow to the speed increaser causing significant damage. Other centrifugal charging pumps (CCPs) were inspected, and the same gland seal bolts as on the first pump were found loosened. The cause of the bolts backing out was determined to be lack of a periodic adjustment of the gland seal bolts.

The **Design** coupling factor indicates that the failures were linked by the components having the same design and component parts or by the system configuration. Design/Construction/Installation/Manufacture was the most prevalent proximate cause to be linked by Design (45 events). This means that design errors and inadequacies were both the cause and the link between the events. Examples of these events are:

- A modification design error removed a start permissive interlock contact. This flaw de-energized the auxiliary lube oil pump; consequently, when one AFW pump was started it ran for 2.5 seconds and tripped on low oil pressure. Further investigation showed that both units AFW pumps would be affected in the same way.
- Both RHR-P pumps failed to run due to high bearing temperatures caused by inadequate bearing clearances and using the wrong lubricating oil, which had too high a viscosity. Inadequate vendor design information resulted in the higher viscosity oil being used.
- During the performance of a special test to determine the available net positive suction head of the SLC Pumps, the pumps began to cavitate unexpectedly. The causes of this event were determined to be inadequate modification testing and errors in the original design calculations.
- During a unit load shed test, the service water pumps lost suction and tripped. The loss of suction pressure was caused by a loss of prime in the condenser circulating water siphon flow system. The event was attributed to poor system design.

The **Environmental** coupling factor propagates a failure mechanism via identical external or internal environmental characteristics. Internal to Component was the most prevalent proximate cause to be linked by Environmental (29 events). Examples of these events are:

- Failure of the HPI Pumps due to clam and sludge fouling of the pump lube oil coolers.
- A CCP seized during surveillance testing. Subsequent inspection revealed resin particles and metal shavings in the pump casings and suction lines for all the charging pumps.

The **Operational** based coupling factor links the CCF events via inadequate operations procedures and operations staff errors. Human Error was the dominant proximate cause for events linked by Operational factors (25 events). Examples of these events are:

- HPI pumps not restored to service before a mode change as required by Technical Specifications due to a procedural inadequacy.
- The CCPs were erroneously placed in pull-to-lock when required to operable.
- During a routine Control Board walk-down it was discovered that the AFW pump discharge MOVs were closed. Subsequent investigation revealed the AFW system had not been previously placed in standby readiness per the operating procedure after the system was secured.

The **Quality** based coupling factor propagates a failure mechanism among several components by manufacturing and installation errors. Design was the dominant proximate cause for events linked by Quality based coupling factors (12 events). Examples of these events are:

- During surveillance testing, neither motor-driven AFW pump would start. The pump control circuit was found with auto-start defeat switches labeled backwards, causing all auto-starts except



the low-low steam generator level to be defeated. This was an original installation error resulting from an inadequate design change process.

- Both motor-driven AFW pumps failed to start when the operator tried to start them manually. While preparing a design change, the designer failed to review all the unit specific documentation associated with the motor-driven AFW pump wiring and made the erroneous assumption that both units switchgear compartment internal wiring was identical. In fact, the wiring for each unit was different. Consequently, when the design change was installed, it was installed in accordance with the erroneous design.

### 3.6 CCF Discovery Method Overview

An important facet of these CCF events is the way in which the failures were discovered. Each CCF event was reviewed and categorized into one of four discovery categories: Test, Maintenance, Demand, or Inspection. These categories are defined as:

|             |   |
|-------------|---|
| Test        | The equipment failure was discovered either during the performance of a scheduled test or because of such a test. These tests are typically periodic surveillance tests, but may be any of the other tests performed at nuclear power plants, e.g., post-maintenance tests and special systems tests. |
| Maintenance | The equipment failure was discovered during maintenance activities. This typically occurs during preventative maintenance activities.   |
| Demand      | The equipment failure was discovered during a demand for the equipment. The demand can be in response to an automatic actuation of a safety system or during normal system operation.   |
| Inspection  | The equipment failure was discovered by personnel, typically during system tours or by operator observations.   |

Figure 3-8 shows the distribution of how the events were discovered or detected. Testing accounted for 95 events, (35 percent), 83 events (30 percent) were discovered during Demand, Inspection accounted for 69 events (25 percent), and 27 events (10 percent) were detected during Maintenance activities. Considering the extensive and frequent surveillance test requirements for pumps contained in Technical Specifications, it is expected that a majority of the pump CCF events would be detected by Testing. The intent of testing programs is to detect degradation and initiate corrective actions before total failure. The failures detected by testing tended to be Internal to Component causes attributed to wear and aging and only a small percentage of these failures resulted in Complete CCF events. It was expected that fewer failures would be detected by Demand. Analysis of events showed that over half of the events discovered by Demand were Complete or Almost Complete. The majority of events detected by Demand were attributed to design errors, human errors, and the Others. These causes were also dominant for all Complete CCF events. This implies that testing may be effective at detecting normal wear and aging problems, but less effective at detecting failures related to design and human errors.

Table A-3 in Appendix A presents the entire pump data set, sorted by the discovery method. This table can be referred to when reading the following discussions to see individual events described.

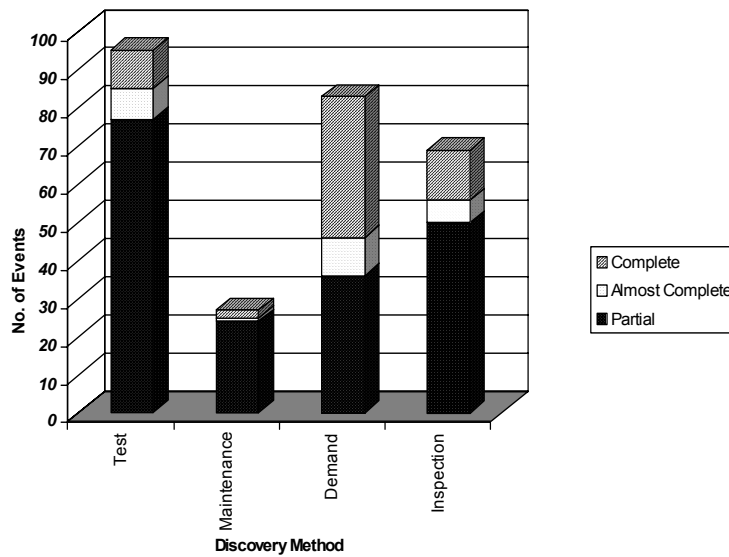


Figure 3-8. Discovery method distribution for all pump CCF events.

### 3.7 Pump CCF System Observations

Figure 3-9 shows the distribution of pump CCF events by system and the degree of failure. The ESW system had the most events. Most pump CCF events in the ESW system involved problems with the pump impellers and wear rings. The RHR-P system had the largest fraction of Complete CCF events (92 percent). Most of the RHR-P system events involved loss of suction, usually during refueling outages with reduced water level in the RCS. Section 5 of this report provides an in-depth analysis of the pump CCF events in these systems.

### 3.8 Other Pump CCF Observations

Figure 3-10 shows the distribution of pump CCF events among the NPP units. The data are based on 109 NPP units represented in the insights CCF studies. Eighty-eight of the NPP units included in this study (81 percent) experienced at least one pump CCF event, and 55 NPP units had more than one pump CCF event. While only 38 NPP units experienced more than two pump CCF events, these 38 NPP units account for 76 percent of the total number of pump CCF events.

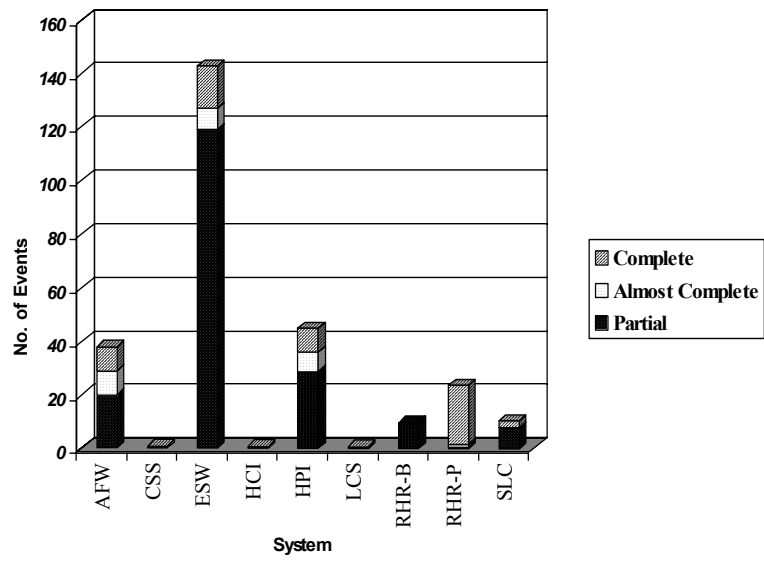


Figure 3-9. Distribution of pump CCF events by system.

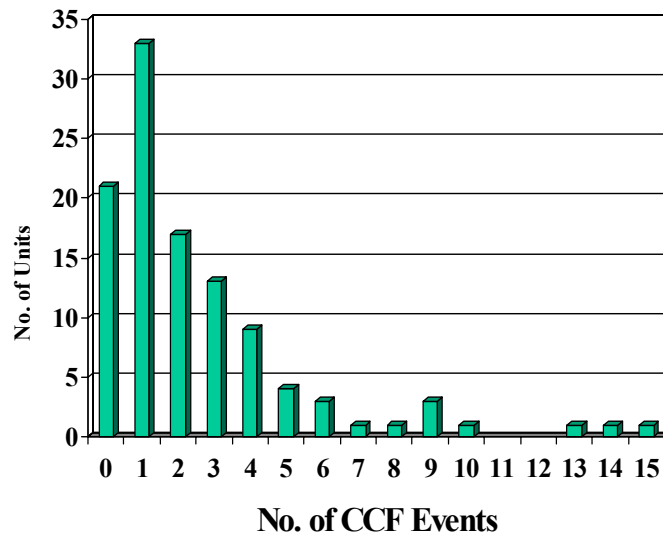


Figure 3-10. Distribution of NPP units experiencing a multiplicity of CCFs for all pump CCF events.



## 4. ENGINEERING INSIGHTS BY PUMP SEGMENT

### 4.1 Introduction

This section presents an overview of the pump CCF data that have been collected from the NRC CCF database, grouped by the affected segment. Pumps are relatively complex machines and can easily be thought of as a collection of segments, each with many components. The pump CCF data were reviewed to determine the affected segment and the affected piece part in that segment. This was done to determine which pump segments and piece-parts are most vulnerable to common-cause failure events. For the descriptions of the pump and its segments, see Section 2.4.

Table 4-1 summarizes the CCF events by segment. The rest of this section provides discussions of pump segment, summarizing selected attributes of that segment. At the end of each discussion is a list of the Complete pump CCF events, with identification of the proximate cause, the failure mode, and a short description of the event. For a listing of all pump CCF events by segment, see Appendix B.

Table 4-1. Summary of segments.

| Segment   | Sub-Section | Partial | Almost Complete | Complete | Total | Percent |
|-----------|-------------|---------|-----------------|----------|-------|---------|
| Pump      | 4.2         | 94      | 8               | 4        | 106   | 38.7%   |
| Driver    | 4.3         | 56      | 8               | 23       | 87    | 31.8%   |
| Suction   | 4.4         | 27      | 5               | 34       | 66    | 24.1%   |
| Discharge | 4.5         | 10      | 4               | 1        | 15    | 5.5%    |
| Total     |             | 187     | 25              | 62       | 274   | 100.0%  |

The majority of the pump CCF events originated in the pump segment followed by the driver and suction segments. The majority of Complete CCF events occurred in the driver and suction segments. There were relatively few events involving the discharge segment. The Complete events in the driver segment were dominated by instrument and control failures and circuit breaker failures. The Complete events in the suction segment were dominated by lack or loss of suction head. The failure mode for the majority of CCF events in the pump and suction segments was fail-to run. However, the failure mode for the majority of events in the driver segment was fail-to-start.

In this study, the proximate causes of the pump CCF events in the NRC CCF database have been grouped into higher-order proximate cause categories to facilitate the graphical depiction of proximate causes. Table 4-2 contains a hierarchical mapping of the proximate causes of pump CCF events into the higher-order groups. Since the graph x-axis labels are restricted in length, the proximate cause category names have been shortened and are shown in parenthesis in Table 4-2. Table 4-2 also describes each of these groups.

Table 4-2. Proximate cause hierarchy.

|   |
|---|
| <div data-bbox="250 331 769 487"> <div>PROXIMATE CAUSE</div> </div> <div data-bbox="282 508 786 1675"> <ul style="list-style-type: none"> <li>Design/Const./Install./Manufacture (Design) <ul style="list-style-type: none"> <li>Design Error</li> <li>Manufacturing Error</li> <li>Installation/Construction Error</li> <li>Design Modification Error</li> </ul> </li> <li>Operational/Human Error (Human) <ul style="list-style-type: none"> <li>Accidental Action</li> <li>Inadequate/Incorrect Procedure</li> <li>Failure to Follow Procedure</li> <li>Inadquate Training</li> <li>Inadequate Maintenance</li> </ul> </li> <li>External Environment (Ext Env) <ul style="list-style-type: none"> <li>Fire/Smoke</li> <li>Humidity/Moisture</li> <li>High/Low Temperature</li> <li>Electromagnetic Field</li> <li>Radiation</li> <li>Bio-organisms</li> <li>Contamination/Dust/Dirt</li> <li>Acts of Nature <ul style="list-style-type: none"> <li>- Wind</li> <li>- Flood</li> <li>- Lightning</li> <li>- Snow/Ice</li> </ul> </li> </ul> </li> <li>Internal to Component (Component)</li> <li>Other <ul style="list-style-type: none"> <li>State of Other Component</li> <li>Setpoint Drift</li> </ul> </li> <li>Unknown</li> </ul> </div> |
|---|

## 4.2 Pump

There were 106 pump CCF events affecting the pump segment (see Table B-1 in Appendix B, items 103 – 208). Of these 106 events, 37 were fail-to-start and 69 were fail-to-run. Only four of the pump segment events were Complete CCF events. Table 4-3 contains a summary of these events by proximate cause group and degree of failure. Figure 4-1 displays the events by proximate cause and failure mode.

Table 4-3. CCF events in pump segment by cause group and degree of failure.

| Proximate Cause Group                                    | Complete | Almost Complete | Partial | Total | Percent |
|--|----------|-----------------|---------|-------|---------|
| Design/Construction/Installation/ Manufacture Inadequacy | 1        | 3               | 13      | 17    | 16.0%   |
| Internal to Component                                    |          | 3               | 75      | 78    | 73.6%   |
| Operational/Human  | 2        | 1               | 4       | 7     | 6.6%    |
| External Environment                                     | 1        | 1               | 1       | 3     | 2.8%    |
| Other  |          |                 | 1       | 1     | 0.9%    |
| Total  | 4        | 8               | 94      | 106   | 100.0%  |

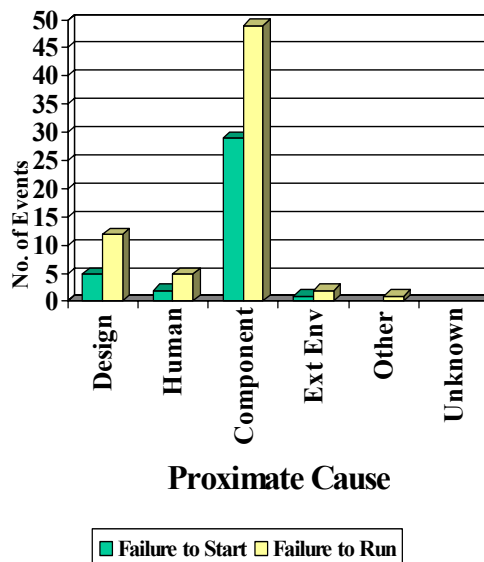


Figure 4-1. Distribution of proximate causes for the pump segment.

The Design/Construction/Installation/Manufacture Inadequacy proximate cause group had 17 events, of which one was Complete and three were Almost Complete (see Table B-1 in Appendix B, items 103 – 119). The causes were primarily due to installation of improper materials that led to corrosion and or failure of piece parts. Other causes included installation errors and failures due to improper design specifications.

The Internal to Component proximate cause group had 78 events, of which none were Complete and three were Almost Complete (see Table B-1 in Appendix B, items 123 – 200). The causes included failed bearings, failed and leaking seals/packing, worn impellers and wear rings due to aging and normal wear and erosion damage of pump internals.

The Operational/Human Error proximate cause group had seven events of which two were Complete and one was Almost Complete (see Table B-1 in Appendix B, items 201 – 207). The causes of these events included pump failures due to misalignment, failures due to maintenance personnel errors such as improper pump assembly or failure to add sufficient lubricant, and gas binding of pumps due to failure to follow procedures.

There were three events with External Environment as the proximate cause group. One of these events was Complete and one was Almost Complete (see Table B-1 in Appendix B, items 120 – 122). The causes for these events included damage from water spray, foreign material in the process fluid and damage due to air entrainment. The Other proximate cause group (see Table B-1 in Appendix B, item 208) contains one event, which was partial. The event involved loss of pump cooling water.

Testing was the most likely method of discovery for CCF events involving the pump segment (59 out of 106 events) as shown in Figure 4-2. The pumps are frequently tested and typically in standby during power operations. Inspection and Demand are the next most likely discovery methods (27 and 13 events, respectively). The most common piece parts involved in pump segment CCF events were the impellers and wear rings, as shown in Figure 4-3.

Table 4-4 lists the short descriptions by proximate cause for the Complete events. The descriptions of all pump CCF events can be found in Appendix A.

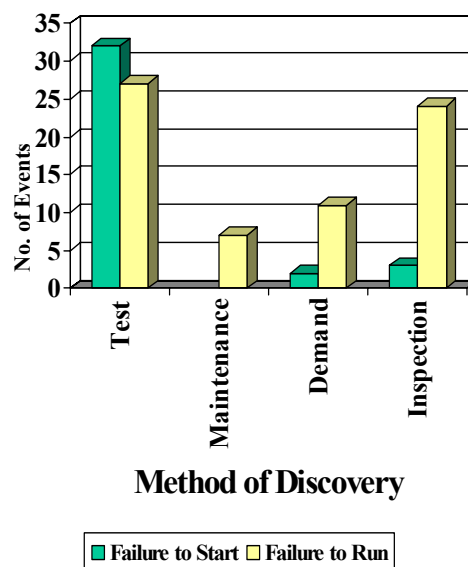


Figure 4-2. Distribution of the method of discovery for the pump segment.



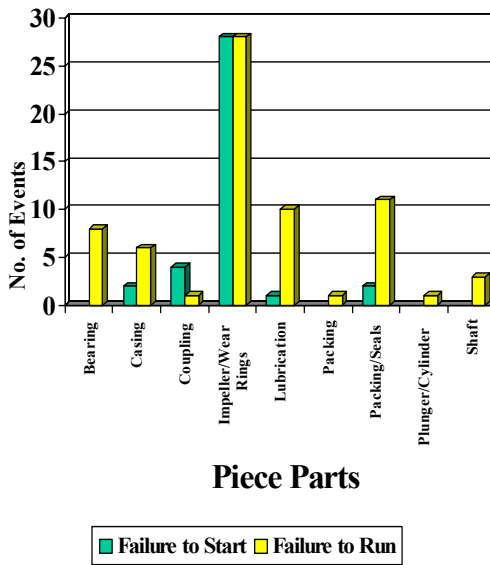


Figure 4-3. Distribution of the affected piece parts for the pump segment.

Table 4-4. Pump segment event short descriptions for Complete events.

| System | Proximate Cause Group  | Failure Mode     | Description  |
|--------|--|------------------|--|
| ESW    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Failure to Run   | Both charging pump service water pumps failed. A carbon cap screw failed allowing the impeller of one pump to bind on the casing. The ensuing leakage shorted the motor windings of the other pump.  |
| HPI    | Operational/<br>Human Error  | Failure to Start | A routine preventive maintenance (oil change) was mistakenly performed on the north charging pump instead of the south as scheduled. Since the south pump was previously cleared for this oil change, and the test pump was valved out, none of these three pumps were in service as required by tech specs for the approximately 20 minutes it took to change the oil in the north pump.  |
| RHR-P  | External<br>Environment  | Failure to Start | Following a trip, water was found spraying from both low head safety injection pump wedge control rod seals. Both pumps were declared inoperable. Postulated failure on the seals was from a minor flow induced pressure transient.  |
| RHR-P  | Operational/<br>Human Error  | Failure to Start | Both loops of the residual heat removal system were declared inoperable due to gas binding of both RHR pumps. The gas binding was caused by entry of nitrogen gas into the reactor coolant system from accumulator. The root cause of this event has been attributed to personnel error. Personnel did not comply with the specific requirements in the accumulator discharge check valve full flow test procedure due to inattention to detail. |

### 4.3 Driver

There were 87 pump CCF events affecting the driver segment, of which 24 were Complete events and 6 were Almost Complete (see Table B-1 in Appendix B, items 16 – 102). The failure mode for the majority of the pump CCF events involving the driver was fail-to-start (68 events). Only 19 events involved fail-to-run. The most likely proximate cause was Operational/Human, followed by Internal to Component and Design/Construction/ Installation/Manufacture Inadequacy. Table 4-5 contains a summary of these events by proximate cause group and degree of failure. Figure 4-4 shows the distribution of events for the driver segment by proximate cause and failure mode.

Table 4-5. CCF events in the driver segment by cause group and degree of failure.

| Proximate Cause Group                                    | Complete | Almost Complete | Partial | Total | Percent |
|--|----------|-----------------|---------|-------|---------|
| Design/Construction/Installation/ Manufacture Inadequacy | 5        | 3               | 10      | 18    | 20.7%   |
| Internal to Component                                    |          | 3               | 16      | 19    | 21.8%   |
| Operational/Human  | 12       | 2               | 15      | 29    | 33.3%   |
| External Environment                                     | 2        |                 | 4       | 6     | 6.9%    |
| Other  | 4        |                 | 11      | 15    | 17.2%   |
| Total  | 23       | 8               | 56      | 87    | 100.0%  |

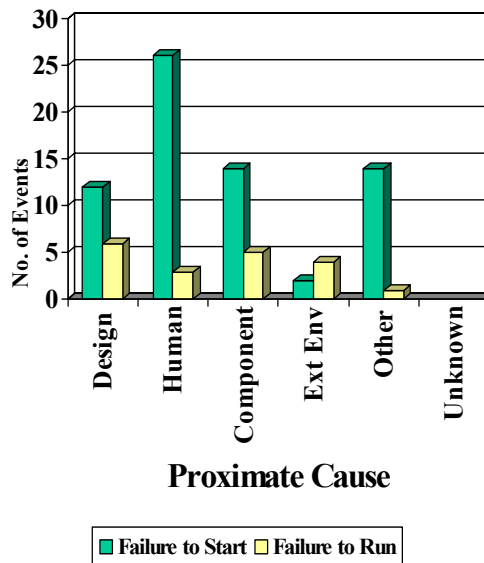


Figure 4-4. Distribution of proximate causes for the driver segment.

There were 18 events involving the driver segment in the Design/Construction/Installation/Manufacture Inadequacy proximate cause group, of which five were Complete and three were Almost Complete (see Table B-1 in Appendix B, items 16 – 33). Most of these events were caused by design related errors with instruments and control circuits.

There were 19 pump CCF events involving the driver segment with Internal to Component as the proximate cause, of which none were Complete and three were Almost Complete (see Table B-1 in Appendix B, items 40 – 58). Most of these events involved circuit breaker failures due to worn internal parts and binding.

A third of the CCF events attributed to the driver segment were assigned to the Operational/ Human Error proximate cause group (see Table B-1 in Appendix B, items 59 – 87). There were 29 driver failures with this proximate cause, of which 12 were Complete and two were Almost Complete. The causes of these events included operations and maintenance personnel errors such as improper lineups, poor maintenance, work on the wrong components, and inadequate procedures.

External Environment was the proximate cause for six driver segment events (see Table B-1 in Appendix B, items 34 – 39). Two of these events were Complete and none were Almost Complete. Causes for these events included foreign material contamination, flooding, low ambient temperatures.

Other was determined to be the proximate cause for 15 driver segment events (see Table B-1 in Appendix B, items 88 – 102). Four of these were Complete and none were Almost Complete. Most of these events were caused by instrument problems or failures of valves and piping in other systems.

Inspection was the most likely method of discovery for driver events (29 events) as shown in Figure 4-5, followed closely by Demands and testing. The most likely piece parts involving driver segment events were circuit breakers, instruments, and control circuits as shown in Figure 4-6.

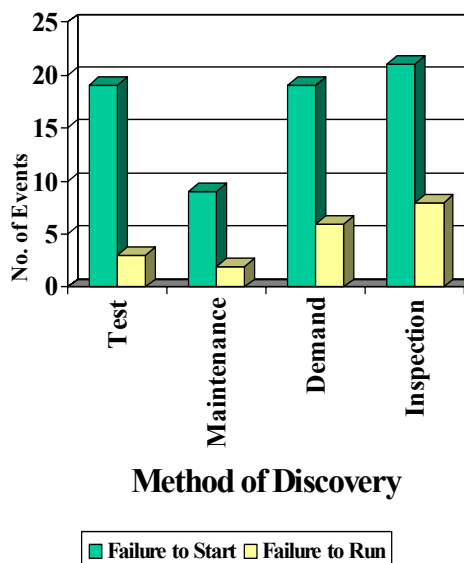


Figure 4-5. Distribution of the method of discovery for the driver segment.

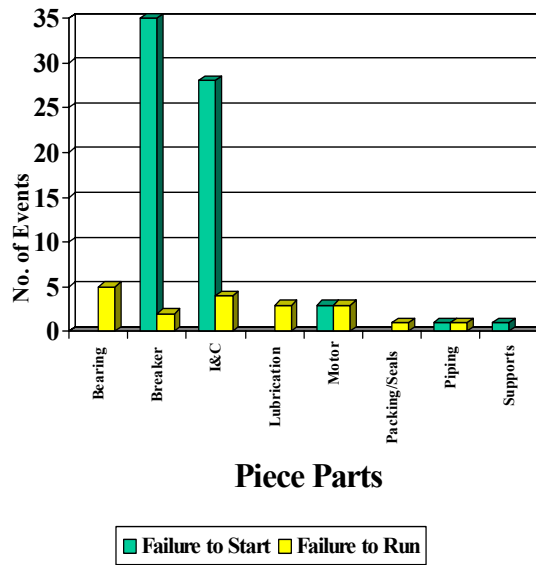


Figure 4-6. Distribution of the affected piece part for the driver segment.

Table 4-6 lists the short descriptions by proximate cause for the Complete events, the events that failed all the pumps. The descriptions of all pump CCF events can be found in Appendix A.

Table 4-6. Driver segment event short descriptions for Complete events.

| System | Proximate Cause Group  | Failure Mode     | Description  |
|--------|--|------------------|--|
| AFW    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Failure to Start | During surveillance testing, neither motor-driven AFW pump would start. The pump control circuit was found with autostart defeat switches labeled backwards, causing all autostarts except the low-low steam generator level to be defeated. The labels were corrected and the links were closed. The original installation error was the result of an inadequate design change process that did not require sufficient verification and testing of the modification.  |
| LCS    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Failure to Start | Relay extra contacts left connected during construction, prevented Core Spray pump start with emergency diesel generator breakers racked out.  |
| RHR-P  | Operational/ Human Error   | Failure to Start | All RHR pumps de-energized to replace RHR Relief valve. T.S. allows this condition for 1 hour. Operated in the mode in excess of 5 hours.  |
| ESW    | Other  | Failure to Start | Following a reactor scram, an attempt to initiate suppression pool cooling revealed that both RHRSW loops were inoperable as neither loop's pumps could be started. Low suction header pressure lockout signals in each loop prevented starting each loop's pumps. Plugging of the sensing line to each loop's suction header pressure switch prevented both switches from sensing actual pressure, although a lack of operating fluid in one switch and an open power supply breaker to the other switch also would have prevented pumps from starting. |
| HPI    | Operational/ Human Error   | Failure to Start | During the draining of the reactor coolant system, both centrifugal charging pumps were rendered inoperable. The initial conditions in the draining procedure contained a confusing statement, which led to an erroneous assumption that both CCP breakers had to be racked out and tagged.  |

| System | Proximate Cause Group                                      | Failure Mode     | Description  |
|--------|--|------------------|--|
| AFW    | Operational/ Human Error                                   | Failure to Start | An operator incorrectly secured the diesel and steam-driven AFW pumps, which prevented their restart on low SG level.  |
| AFW    | External Environment                                       | Failure to Start | Both AFW pumps failed to start. The problem was traced to two relays (1 per pump). Examination of the relays revealed open circuiting and severe degradation of the insulation.  |
| AFW    | Operational/ Human Error                                   | Failure to Start | Both AFW pumps failed to start when tested, due to the circuit breakers not being racked in properly.  |
| SLC    | Other  | Failure to Start | During a test, both Squib Valve Detonators shorted after firing and the Control Power Transformer fuse blew causing the pump motor trip. This was caused by improper fuse coordination between the Control Power Transformer fuse and the Squib Valve Detonator fuses. The redundant system's Squib Valve was also fired during this test, without running the associated pump, and one of the Squib Valve Detonators shorted after firing. The same fuse coordination problem existed for both systems.   |
| RHR-P  | Other  | Failure to Start | Two LPI pumps, when given a start signal, would not start. An ongoing investigation revealed the probable root cause of the event to be poor electrical contact of the breaker auxiliary stabs for the pumps.  |
| HPI    | Operational/ Human Error                                   | Failure to Start | With alternate CCP pump out-of-service, the remaining operable pump was erroneously placed in pull-to-lock.  |
| HPI    | Operational/ Human Error                                   | Failure to Start | HPI pumps not restored before mode change due to procedural inadequacy.  |
| AFW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Start | Both motor-driven auxiliary feedwater pumps failed to start when the operator tried to start them manually. While preparing a design change, the designer failed to review all the unit specific documentation associated with the motor-driven AFW pump wiring and made the erroneous assumption that both units switchgear compartment internal wiring was identical. In fact, the wiring for each unit was different. Consequently, when the design change was installed, it was installed in accordance with the erroneous design. The wiring discrepancy was corrected and the motor-driven AFW pumps were tested and returned to service.                            |
| AFW    | Operational/ Human Error                                   | Failure to Start | During testing one AFW pump was tested and other was tested without returning first to auto. Both pumps were unavailable at the same time. The procedure was the cause.  |
| ESW    | Operational/ Human Error                                   | Failure to Start | An emergency service water pump failed to start and was declared inoperable. Further investigation determined that the failure of the pump to start was due to a tripped emergency engine shutdown device. Operations personnel performing the testing did not recognize the need to reset it prior to starting the pump. Examination of the other two ESW pumps revealed that their emergency shutdown devices were also in the tripped condition.  |
| HPI    | Operational/ Human Error                                   | Failure to Start | By opening incorrect breaker, HPI pump tripped while others were unavailable.  |
| HPI    | External Environment                                       | Failure to Run   | It was determined that the common minimum flow path return line for the safety injection pumps to the refueling water storage tank was frozen. Previous actions to investigate problems with the freeze protection system were unsuccessful in preventing development of this condition. The two HPI pumps were declared inoperable with this return line frozen. A faulty ambient temperature switch for the RWST heat trace system prevented the heat trace from activating and was subsequently replaced. In addition, administrative controls did not sufficiently recognize the safety significance of flow through this line and the need to ensure flow capability. |
| CSR    | Operational/ Human Error                                   | Failure to Start | CSR control power de-energized prior to mode change. Technical Specification violation. Inadequate procedure review.   |
| AFW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Start | A modification design error (in 1983-1984) removed a start permissive interlock contact. At cold shutdown this de-energized the auxiliary lube oil pump, consequently, when one AFW pump was started it ran for 2.5 seconds and tripped on low oil pressure. Further investigation showed that both units AFW pumps would be affected in the same way. The design error combined with insufficient post modification testing led to this CCF event.  |
| RHR-P  | Operational/ Human Error                                   | Failure to Start | Both trains of RHR were rendered inoperable for two minutes, while performing an operational readiness test surveillance procedure. The surveillance procedure required that the one RHR train pump be placed in pull to lock and the other train heat exchanger flow control valve throttled to 30-40% open. The procedure directed the   |

| System | Proximate Cause Group                          | Failure Mode     | Description  |
|--------|--|------------------|--|
|        | Error  | to Start         | operators to perform operations that resulted in both trains of RHR being inoperable   |
| RHR-P  | Operational/ Human Error                       | Failure to Start | The switches for the containment spray and recirculation pumps were in a trip pullout when the Technical Specifications and plant procedures required the pumps to be operable.  |
| HCI    | Other  | Failure to Start | Water entered the HCI and RCI steam supply lines, rendering both pumps inoperable. Failed reactor vessel instrumentation allowed water to overflow and fill the HCI/RCI steam lines. Pumps were unavailable.   |
| RHR-P  | Design/ Construction/ Installation/ Inadequacy | Failure to Run   | Both RHR/LPI pumps fail-to-run due to improper oil in system. High bearing temperatures occurred when the pumps were operated. This was due to the wrong lube oil being used, which had too high a viscosity. Inadequate vender design information resulted in the higher viscosity oil being used and additional exacerbating problems such as insufficient bearing clearances. |

## 4.4 Suction

Sixty-six events affected the suction segment of the pumps (see Table B-1 in Appendix B, items 209 – 274). Thirty-four were Complete events. The most likely proximate cause was Design/Construction/Installation/Manufacture. The failure mode was fail-to-run for 54 events and fail-to-start for 12 events. Table 4-7 contains a summary of these events by proximate cause group and degree of failure. Figure 4-7 shows the distribution of events by proximate cause and failure mode.

Table 4-7. CCF events in the suction segment by cause group and degree of failure.

| Proximate Cause Group                                    | Complete | Almost Complete | Partial | Total | Percent |
|--|----------|-----------------|---------|-------|---------|
| Design/Construction/Installation/ Manufacture Inadequacy | 11       | 2               | 16      | 29    | 43.9%   |
| Internal to Component                                    |          |                 | 6       | 6     | 9.1%    |
| Operational/Human  | 12       | 1               | 4       | 17    | 25.8%   |
| External Environment                                     | 1        |                 |         | 1     | 1.5%    |
| Other  | 9        | 2               | 1       | 12    | 18.2%   |
| Unknown  | 1        |                 |         | 1     | 1.5%    |
| Total  | 34       | 5               | 27      | 66    | 100.0%  |

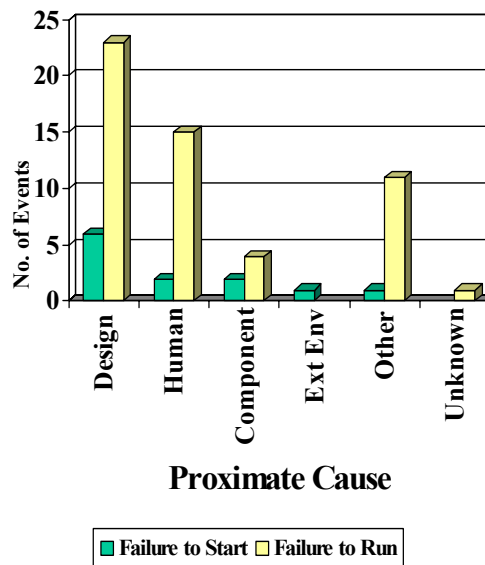


Figure 4-7. Distribution of proximate causes for the suction segment.

The Design/Construction/Installation/Manufacture Inadequacy proximate cause group had 29 events, of which 11 were Complete and two were Almost Complete (see Table B-1 in Appendix B, items 209 – 237). Most of these events involved inadequate net positive suction head due to poor system design.

The Internal to Component proximate cause group had six events, of which none were Complete or Almost Complete (see Table B-1 in Appendix B, items 239 – 244). All of these events involved blocked suctions due to foreign material intrusion.

There were 17 events assigned to the Operational/Human Error proximate cause group. Twelve of these events were Complete and one was Almost Complete (see Table B-1 in Appendix B, items 245 – 261). These events mostly involve inadequate procedures and personnel errors related loss of pump suction due to improper venting or system lineups. This has the largest (35 percent) contribution to the Complete suction events.

Other was identified as the proximate cause of 12 events, of which nine were Complete and two were Almost Complete (see Table B-1 in Appendix B, items 262 – 273). These events involved failures of other components impacting pump suction, such as leaking or blocked valves, failed vent valves, and erroneous level instruments.

There was one event assigned to each of the External Environment and Unknown proximate cause groups, and both these events were Complete (see Table B-1 in Appendix B, items 238 and 274). One event involved loss of RHR pump suction, the cause of which could not be determined or repeated. The other event was caused by boron solidification in the suction piping.

Demand was the most likely method of discovery for the suction segment events (39 of 68 events) as shown in Figure 4-8. Since most events were attributed to design problems and human error, this implies that testing has not been effective in detecting failures with these causes. The most likely

piece part involved in the suction segment CCF events was piping as shown in Figure 4-9. The piping piece part indicates that something caused a loss of NPSH to the pumps that is not a valve, strainer, etc.

Table 4-8 lists the short descriptions by proximate cause for the Complete events. The descriptions of all pump CCF events can be found in Appendix A.

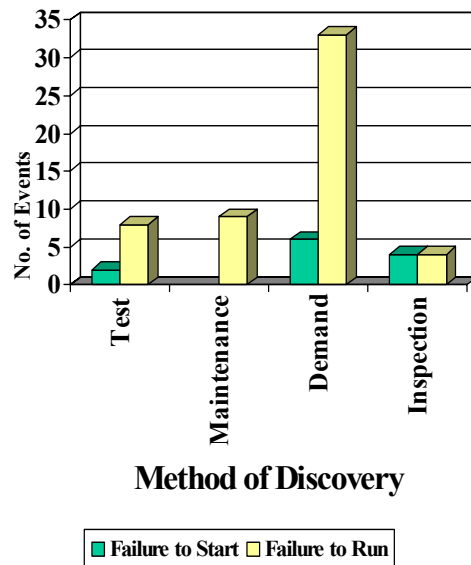


Figure 4-8. Distribution of the method of discovery for the suction segment.

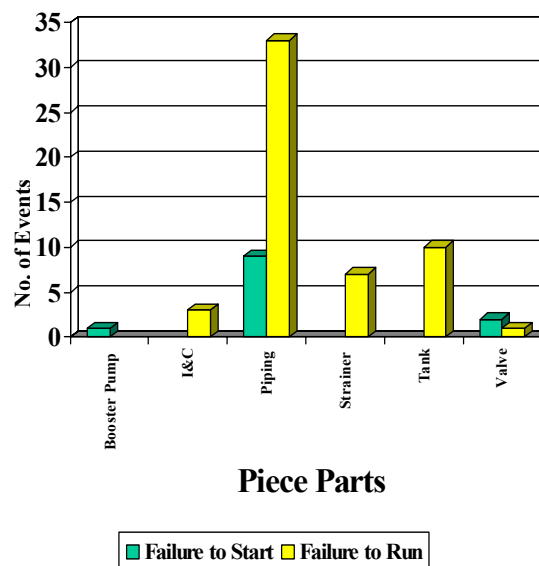


Figure 4-9. Distribution of the affected piece parts for the suction segment.



Table 4-8. Suction segment event short descriptions for Complete events.

| System | Proximate Cause Group                                      | Failure Mode     | Description   |
|--------|--|------------------|---|
| AFW    | Operational/ Human Error                                   | Failure to Run   | Both emergency feedwater pumps lost feed pump suction. The emergency feedwater pump suction flashed to steam due to the feedwater train flashing and forcing hot water back through the startup and blowdown tanks and into the feedwater pump suction. To prevent this recurrence, the operating procedures have been changed to require isolating the startup and blowdown effluent as a source of emergency feedwater suction prior to increasing power. |
| RHR-P  | Other  | Failure to Run   | A complete loss of RHR flow occurred while plant operators were increasing RHR heat exchanger flow by closing down on the heat exchanger bypass valve.  |
| RHR-P  | Operational/ Human Error                                   | Failure to Run   | While attempting to increase RHR flow, the plant experienced a total loss of flow due to the pumps being air-bound. The pump was not vented when starting to increase flow. Operating procedures have been changed to have an operator present while changing flow in the RHR system. There have been losses of RHR flow in the past because the pumps were air-bound and methods are being investigated to improve the system design.                      |
| RHR-P  | Operational/ Human Error                                   | Failure to Run   | The reactor vessel vent eductor was in service in preparation for refueling with RHR operating. A low flow alarm was received and low flow and low motor current were indicated. A second pump was started and became air-bound. Putting the vessel vent eductor system into service was the root cause of the incident.  |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | Increasing flow to chillers robs NPSH from charging service water pumps.  |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| RHR-P  | Other  | Failure to Run   | Temporary coolant loop level indicator showed level slowly increasing over a period of days. The system was periodically drained to maintain 65 percent indicated level. A RHR pump lost suction on reduction of actual level. The second pump was started, and lost suction. Indication drift was due to evaporation of reference leg.   |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | The use of service water by the chillers can cause a loss of suction pressure to the Charging Water Service Water pumps.  |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |
| HPI    | Other  | Failure to Start | Hydrogen from the suction dampener got into suction piping and failed both CCPs.  |

| System | Proximate Cause Group                                      | Failure Mode     | Description   |
|--------|--|------------------|---|
| RHR-P  | Other  | Failure to Run   | RHR Suction lost due to erroneous RCS level while draining the RCS.   |
| RHR-P  | Operational/ Human Error                                   | Failure to Run   | Shutdown cooling was lost due to nitrogen intrusion because of backflushing a filter in the purification system.  |
| RHR-P  | Operational/ Human Error                                   | Failure to Run   | Suction was lost to both RHR pumps. RHR flow was less than 3000 gpm and pump amps were fluctuating prior to taking corrective action. Each of these events appear to have been caused by a slow decrease in RCS level in conjunction with the vortex action at the pump suction.  |
| RHR-P  | Other  | Failure to Run   | With unit drained to centerline of the nozzles, suction to both RHR pumps was lost for 36 minutes. Suction to the RHR pumps was lost because of ambiguous reactor coolant system level indication while drained to centerline of the nozzles. The actual RCS level was lower than observed.   |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| RHR-P  | Other  | Failure to Run   | The RHR pumps began to cavitate and eventually both pumps were stopped. The reactor vessel level gauge being used to provide an indication that the level was approaching the vessel flange level had been isolated (reactor coolant drain tank isolation valve had been closed during an attempt to reduce leakage). Additionally, procedures did not require visual monitoring of cavity level.   |
| RHR-P  | Unknown  | Failure to Run   | RHR pumps cavitated. Unable to repeat. Unknown cause.   |
| HPI    | External Environment                                       | Failure to Start | Boron solidification in the suction and gas binding of pumps led to the failure of all three safety injection pumps. Flushing procedures inadequate.  |
| RHR-P  | Operational/ Human Error                                   | Failure to Run   | The control room operators started a second residual heat removal pump in preparation for removing the operating RHR pump from service. With both pumps running, flow became excessive for the half-loop condition causing cavitation and air binding of both pumps. To prevent recurrence the procedure which controls the operation of the RHR pumps has been changed to include specific instructions to stop the operating pump prior to starting the second pump while at half-loop. |
| RHR-P  | Other  | Failure to Run   | Both RHR pumps were unable to operate due to the introduction of air into the RHR system. The incident occurred during the drain down of the RCS, when the level of the RCS was being monitored via a standpipe off the centerline of one of the RCS loops. The isolation valve to which the standpipe was attached became clogged sometime during the drain down and falsely indicated above centerline when in fact the level was below the RHR suction line (below centerline).        |
| RHR-P  | Operational/ Human Error                                   | Failure to Run   | Swap over of RHR pumps resulted in both trains becoming inoperable due to air injection into the suction of the pumps. This required both pumps to be vented and required RCS level to be raised to prevent a possible recurrence of the vortex problem.  |
| ESW    | Operational/ Human Error                                   | Failure to Run   | A service water strainer was placed in service without being vented resulting in air binding system and loss of charging pump service water pumps.  |
| ESW    | Operational/ Human Error                                   | Failure to Run   | Failure to properly vent and fill a newly installed pipe introduced air into the charging pump service water system.  |
| ESW    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | Loss of prime in the condenser circulating water siphon flow system caused loss of low pressure service water pumps. Pumps lost suction during a test due to poor design.   |
| RHR-P  | Other  | Failure to Run   | SDC pumps cavitated due to lowering RCS level. Level indication was in error.   |

| System | Proximate Cause Group                                      | Failure Mode     | Description   |
|--------|--|------------------|---|
| RHR-P  | Other  | Failure to Run   | RHR flow was interrupted when both RHR trains became inoperable due to air bound RHR pumps. The loss of RCS inventory to the reactor coolant drain tank due to a leaking valve caused a decrease in RCS water level, vortexing in the pumps' suction line, and air entrainment in the RHR pumps.  |
| ESW    | Operational/ Human Error                                   | Failure to Run   | The procedure failed to adequately caution the operator to slowly fill a drained line. Rapid filling resulted in a loss of NPSH to the charging service water pumps.  |
| SLC    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | During the performance of a special test on the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.   |
| SLC    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | During the performance of a special test on Unit 1 to determine the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.                           |
| ESW    | Operational/ Human Error                                   | Failure to Run   | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test. |
| ESW    | Operational/ Human Error                                   | Failure to Run   | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test. |
| HPI    | Design/ Construction/ Manufacture/ Installation Inadequacy | Failure to Run   | HPI pumps fail due to operation with inadequate suction head. Two pumps damaged due to operation with inadequate suction, but all three system pumps were unavailable due to the loss of the suction source. Suction source level instrumentation was the cause.  |
| SDC    | Other  | Failure to Start | SDC pump suction high temperature interlock failed, causing all three SDC pumps to be inoperable.   |

## 4.5 Discharge

Fifteen events affected the discharge segment of the pumps, of which one event was Complete and four were Almost Complete (see Table B-1 in Appendix B, items 1 – 15). No one proximate cause was dominant. The failure mode for eight events was fail-to-start and the failure mode for seven events was fail-to-run. Table 4-9 contains a summary of these events by proximate cause group and degree of failure. Figure 4-10 shows the distribution of events by proximate cause and failure mode.

Table 4-9. CCF events in the discharge segment by cause group and degree of failure.

| Proximate Cause Group                                    | Complete | Almost Complete | Partial | Total | Percent |
|--|----------|-----------------|---------|-------|---------|
| Design/Construction/Installation/ Manufacture Inadequacy |          |                 | 3       | 3     | 20.0%   |
| Internal to Component                                    |          | 1               | 4       | 5     | 33.3%   |
| Operational/Human  | 1        | 1               | 1       | 3     | 20.0%   |
| External Environment                                     |          | 2               | 1       | 3     | 20.0%   |
| Other  |          |                 | 1       | 1     | 6.7%    |
| Total  | 1        | 4               | 10      | 15    | 100.0%  |

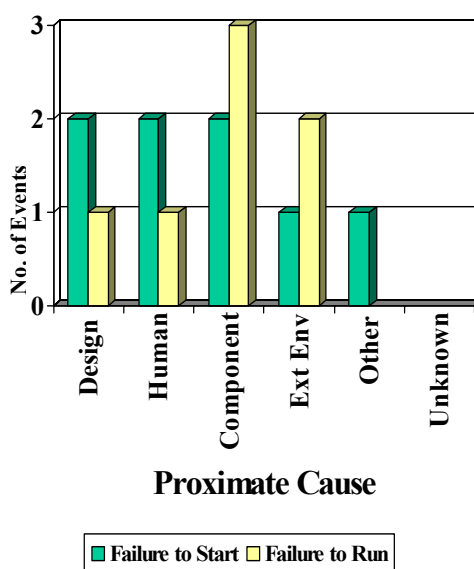


Figure 4-10. Distribution of proximate causes for the discharge segment.

The Design/Construction/Installation/Manufacture Inadequacy proximate cause group had three events, of which none were Complete or Almost Complete (see Table B-1 in Appendix B, items 1 – 3). These events involved failure of the discharge flow controller, pumps dead-headed by other operating pumps, and discharge valve failure.

The Internal to Component proximate cause group had five events, of which none were Complete and one was Almost Complete (see Table B-1 in Appendix B, items 7 – 11). These events involved degradation of discharge valves and line blockage.

The Operational/Human Error proximate cause group contains three events, with one Complete event and one Almost Complete event (see Table B-1 in Appendix B, items 12 – 14). Two of these events were due to inadvertent valve closures in the discharge flow path. The third event was due to

procedural problems that allowed pumps to be run with no flow or beyond the maximum allowable flow rate.

External Environment was the proximate cause for three events, two of which were Almost Complete (see Table B-1 in Appendix B, items 4 – 6). These events were caused by voiding in the discharge lines due to high temperatures, voiding due to air entrainment, and blockage due to foreign material intrusion.

The Other proximate cause group was identified for one Partial event, which was caused by failure of an automatic vent valve on the pump discharge lines (see Table B-1 in Appendix B, item 15).

The method of detection was rather evenly split among demand, inspection, and testing for the discharge segment events as shown in Figure 4-11. Most discharge segment events involved the state of the valves in the discharge of the pumps as shown in Figure 4-12. Table 4-10 lists the short description for the Complete discharge segment event. The descriptions of all pump CCF events can be found in Appendix A.

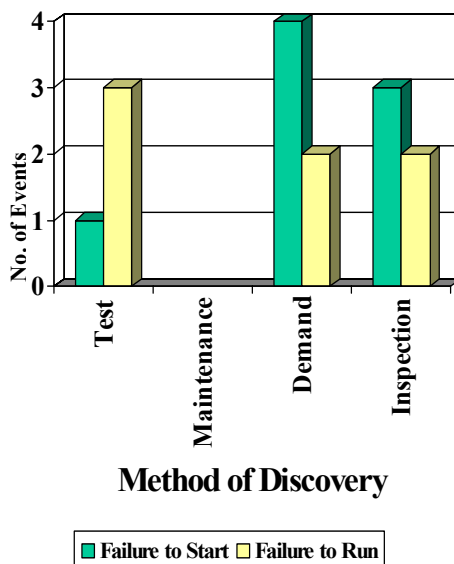


Figure 4-11. Distribution of the method of discovery for the discharge segment.

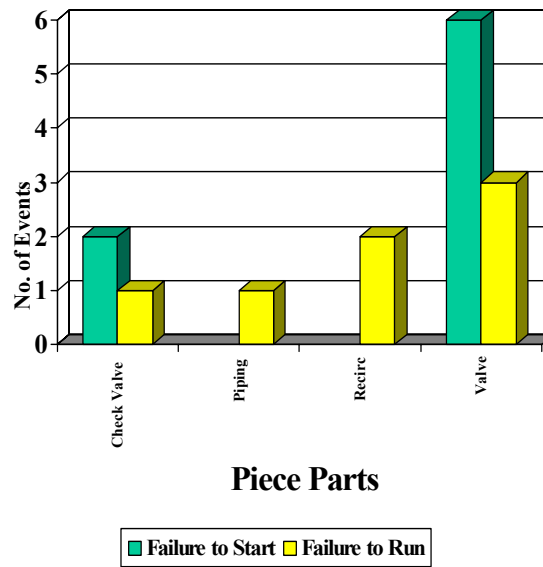


Figure 4-12. Distribution of the affected piece parts for the discharge segment.

Table 4-10. Discharge segment event short description for the Complete event.

| System | Proximate Cause Group    | Failure Mode     | Description   |
|--------|--------------------------|------------------|---|
| AFW    | Operational/ Human Error | Failure to Start | Following a trip, the AFW Pumps were secured and the discharge flow control valves for the Motor-driven Pumps were closed. Later, an operator discovered during a routine Control Board walkdown that the valves were closed. Subsequent investigation revealed the AFW system had not been placed in standby readiness per the operating procedure after the system was secured. |

## 5. ENGINEERING INSIGHTS BY PUMP SYSTEM

### 5.1 Introduction

This section presents an overview of the CCF data for the pump component that have been collected from the NRC CCF database, grouped by the system. Each discussion of a system summarizes selected attributes of that system. Table 5-1 shows the summary of the event counts by system and the degree of failure. For a listing of all pump CCF events, by system, see Appendix C.

Table 5-1. Summary of systems.

| <b>System</b> | <b>Sub-Section</b> | <b>Partial</b> | <b>Almost Complete</b> | <b>Complete</b> | <b>Total</b> | <b>Percent</b> |
|---------------|--------------------|----------------|------------------------|-----------------|--------------|----------------|
| ESW           | 5.2                | 119            | 8                      | 16              | 143          | 52.2%          |
| HPI           | 5.3                | 29             | 7                      | 9               | 45           | 16.4%          |
| AFW           | 5.4                | 20             | 9                      | 9               | 38           | 13.9%          |
| RHR-P         | 5.5                | 1              | 1                      | 22              | 24           | 8.8%           |
| SLC           | 5.6                | 8              |                        | 3               | 11           | 4.0%           |
| RHR-B         | 5.8                | 10             |                        |                 | 10           | 3.6%           |
| CSS           | 5.8                |                |                        | 1               | 1            | 0.4%           |
| HCI           | 5.8                |                |                        | 1               | 1            | 0.4%           |
| LCS           | 5.8                |                |                        | 1               | 1            | 0.4%           |
| <b>Total</b>  |                    | 187            | 25                     | 62              | 274          | 100.0%         |

### 5.2 Emergency Service Water

One hundred and forty three pump CCF events affected pumps in the ESW system (see Table C-1 in Appendix C, items 40 – 182). Figure 5-1 through Figure 5-4 show selected distributions graphically. The Internal to Component was the dominant proximate cause (51 percent of the events for this system) affecting both the fail-to-start and fail-to-run. The most likely discovery method was testing. Most pump CCF events in the ESW system involved problems with the pump impellers and wear rings. Consistent with this, most of the failures involved the pump segment (50 percent).

Sixteen of the ESW pump CCF events were Complete. The set of Complete CCF events is dominated by two units at a single facility, accounting of 14 of the 16 events. Most these events occurred in the early 1980s and involved a design configuration issue, which caused the ESW pumps to fail when suction water was diverted for the chillers. Most of the other events involved air introduction into the ESW suction path. Very few of the Complete and Almost Complete events are attributed to the impeller or wearing rings. However, the ESW pumps CCFs are dominated by this piece part.

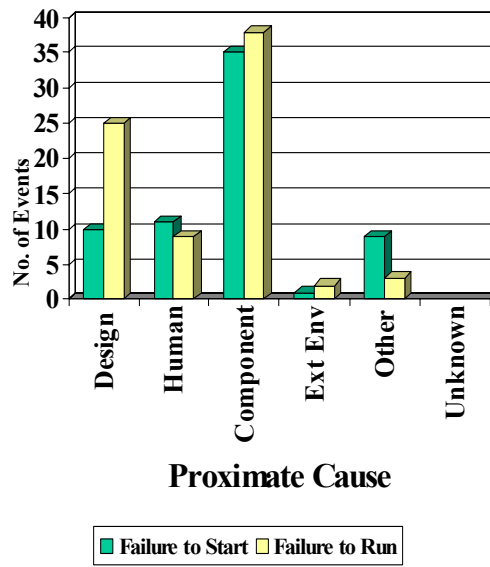


Figure 5-1. Proximate cause distribution for the ESW system.

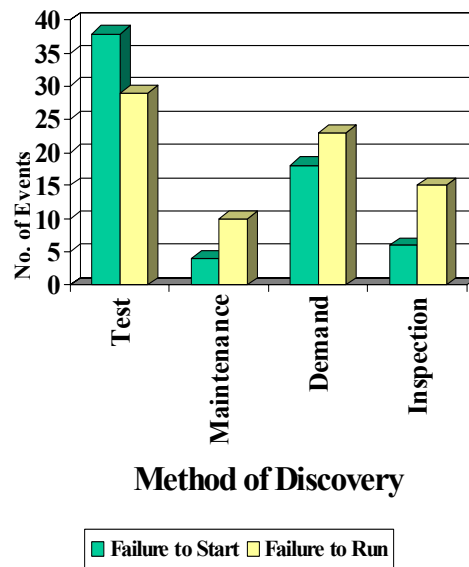


Figure 5-2. Method of discovery distribution for the ESW system.



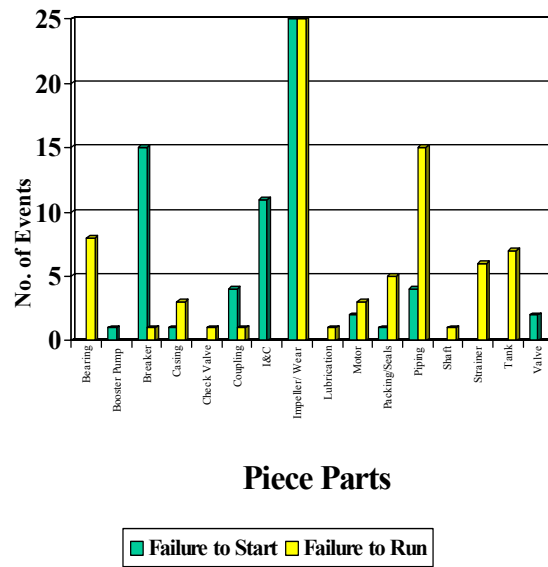


Figure 5-3. Piece part distribution for the ESW system.

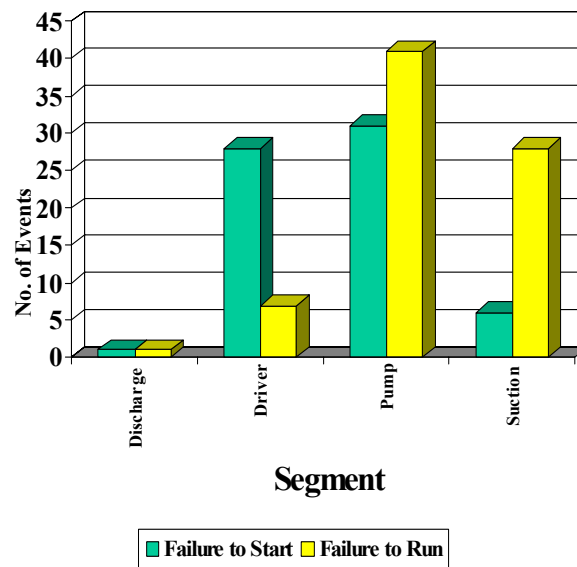


Figure 5-4. Segment distribution for the ESW system.

### 5.3 High Pressure Injection

Forty-five pump CCF events affected pumps in the HPI system (see Table C-1 in Appendix C, items 184 – 228). Figure 5-5 through Figure 5-8 show selected distributions graphically. The most likely proximate causes were the Internal to Component, Design/Construction/Installation/Manufacture Inadequacy, and Operational/Human Error. The failure mode for 26 events was fail-to-run and the failure mode for 19 events was fail-to-start. The most likely discovery method was Inspection.

Nine of the HPI pump CCF events were Complete and seven events were Almost Complete. Most of these events involve line blockage (foreign material, bio-fouling, boron solidification, frozen lines) or system misalignment. For all HPI events, the dominant failed piece parts were lubrication, piping, instruments, and control circuits and circuit breakers. Sixteen events involved failure of the driver segment while 13 events involved the pump segment.

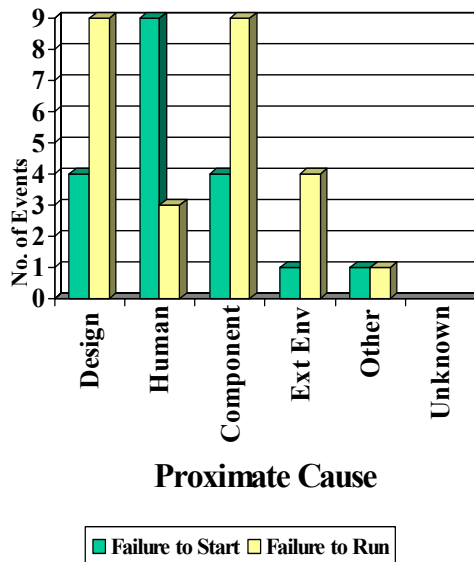


Figure 5-5. Proximate cause distribution for the HPI system.

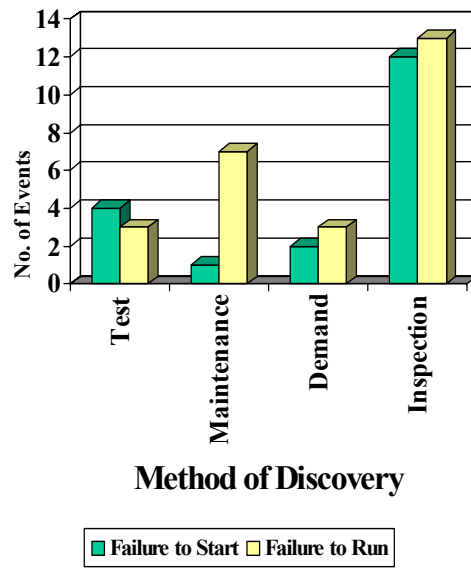


Figure 5-6. Method of discovery distribution for the HPI system.

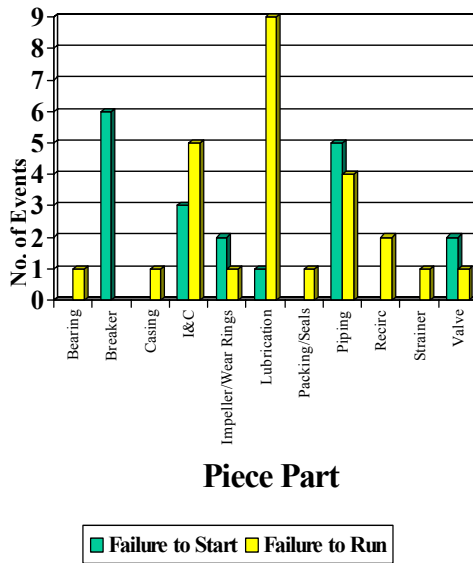


Figure 5-7. Piece part distribution for the HPI system.

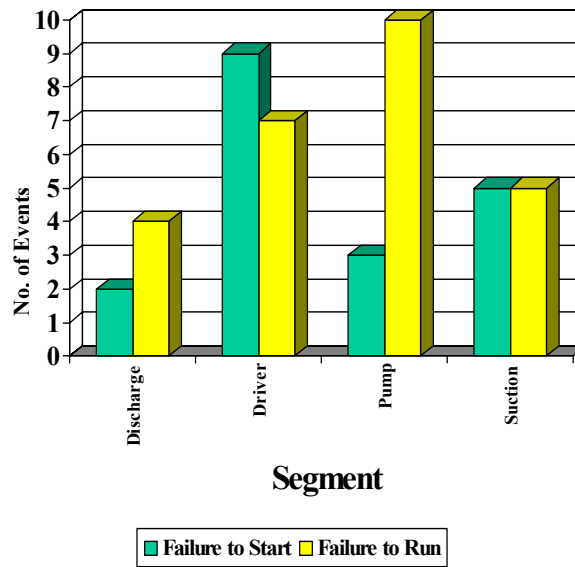


Figure 5-8. Segment distribution for the HPI system.

## 5.4 Auxiliary Feedwater

Thirty-eight pump CCF events affected pumps in the AFW system (see Table C-1 in Appendix C, items 1 – 38). Figure 5-9 through Figure 5-12 show selected distributions graphically. The most likely proximate cause was Design/Construction/ Installation/Manufacture Inadequacy (37 percent), followed by Internal to Component (26 percent) and Operational/Human Error (21 percent). The failure mode for 18 events was fail-to-run and the failure mode for 20 events was fail-to-start. The most likely discovery method was Demands. There were nine Complete and nine Almost Complete AFW pump CCF events. Almost half the AFW pump CCF events were observed safety-significant events. The last Complete AFW pump CCF event occurred in 1994.

The dominant piece parts involved in the AFW pump Complete and Almost Complete CCF events were instrument and control circuits. Examples follow: Degraded relays, permissive interlock, interlock improperly engaged, pumps not returned to automatic, autostart defeat switches labeled backwards, incorrect modification of pump circuitry. These events involved human error, failed equipment, improper operation, and bad design. Consistent with this, most of the events involved the driver segment with a dominant failure mode of fail-to-start. Another important contribution was the leaking of check valves that caused the AFW pumps to become steam bound.

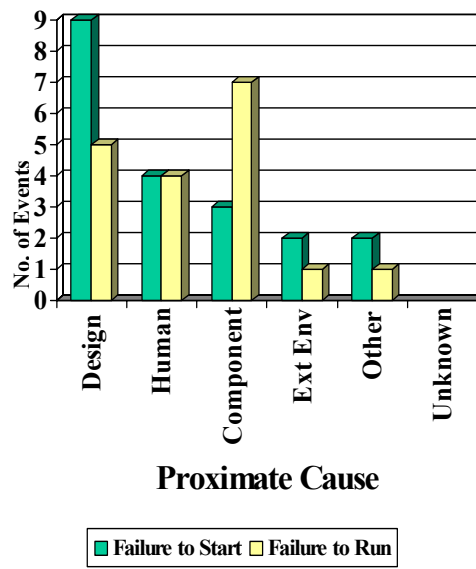


Figure 5-9. Proximate cause distribution for the AFW system.

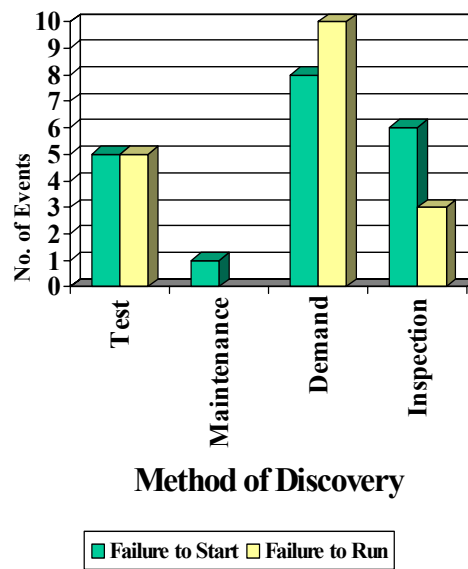


Figure 5-10. Method of discovery distribution for the AFW system.

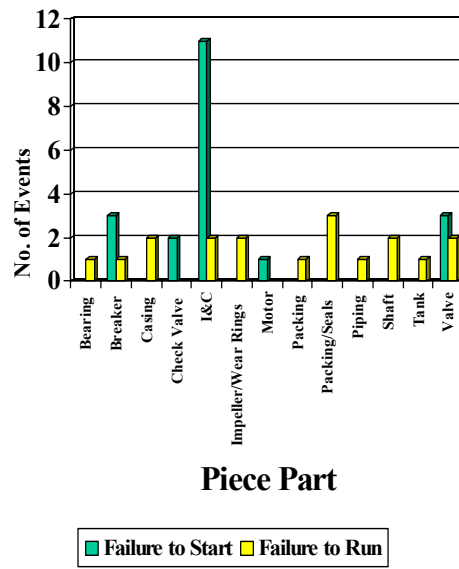


Figure 5-11. Piece part distribution for the AFW system.

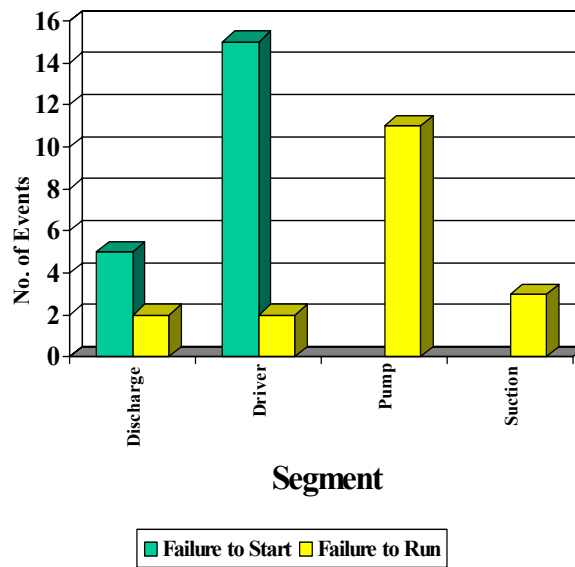


Figure 5-12. Segment distribution for the AFW system.

## 5.5 Residual Heat Removal (PWR)

Twenty-four pump CCF events affected pumps in the RHR-P system (see Table C-1 in Appendix C, items 240 – 263). Figure 5-13 through Figure 5-16 show selected distributions graphically. The RHR-P system had the largest fraction of Complete CCF events (92 percent). One event was Almost Complete. Consistent with this, the dominant proximate causes were Operational/Human Error and Other, and the dominant method of discovery was Demands. The pump CCF data indicates that events caused by human error or component failures outside the pump boundary are more likely to be Complete events and are more likely to be detected by demand than by testing, maintenance or inspection. The failure mode for most RHR-P system CCF events was fail-to-run (18 events). The last Complete RHR-P pump CCF event was in 2000, indicating that the overall problems with RHR-P pumps have not been completely addressed. However, the last loss of suction CCF event was in 1987, which indicates that this failure mode has been addressed.

The Suction segment and the piping piece part (piping was used as the piece part for the loss of suction events) dominate the events in this system. Most of the RHR-P system events involved loss of suction, usually during refueling outages with reduced water level in the RCS. These events occurred repeatedly, but were caused by different mechanisms including suction vortexing, air entrainment, operator error, and malfunctioning level instruments. All 16 of the suction segment events were either Complete or Almost Complete. Four of the remaining Complete events were due to improper system lineups caused by human error.

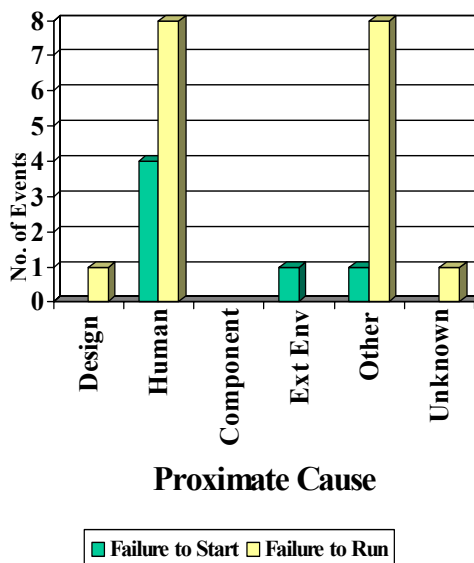


Figure 5-13. Proximate cause distribution for the RHR-P system.

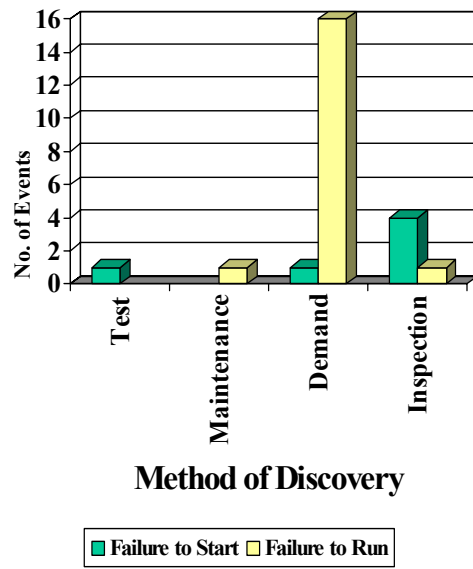


Figure 5-14. Method of discovery distribution for the RHR-P system.

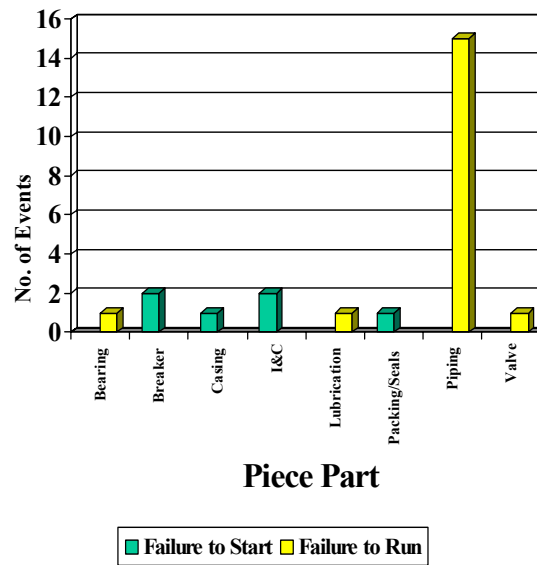


Figure 5-15. Piece part distribution for the RHR-P system.



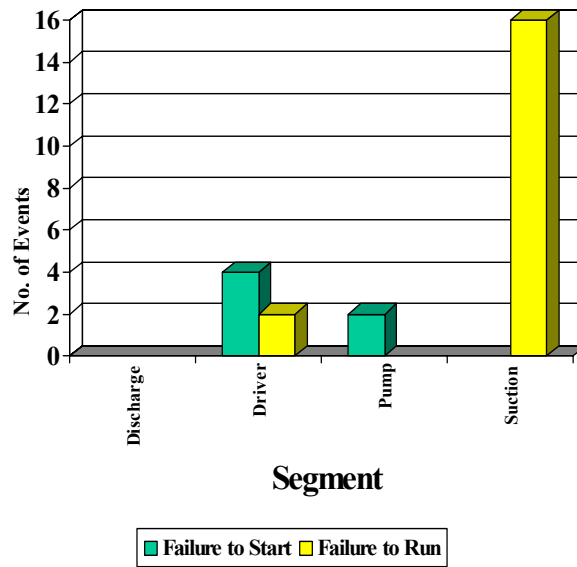


Figure 5-16. Segment distribution for the RHR-P system.

## 5.6 Standby Liquid Control

Eleven pump CCF events affected pumps in the SLC system (see Table C-1 in Appendix C, items 264 – 274). Figure 5-17 through Figure 5-20 show selected distributions for the SLC system. The dominant proximate cause was Internal to Component (64 percent) and the dominant failure mode was fail-to-run (73 percent). The most likely discovery methods were inspection and testing. A variety of piece parts failed, affecting mostly the pump segment. Three of the SLC system CCF events were Complete and none were Almost Complete. One of the Complete events involved a short circuit in the pump control circuit and two events involved inadequate pump suction head. The Partial SLC pump CCF events were associated with worn internals and leaks.

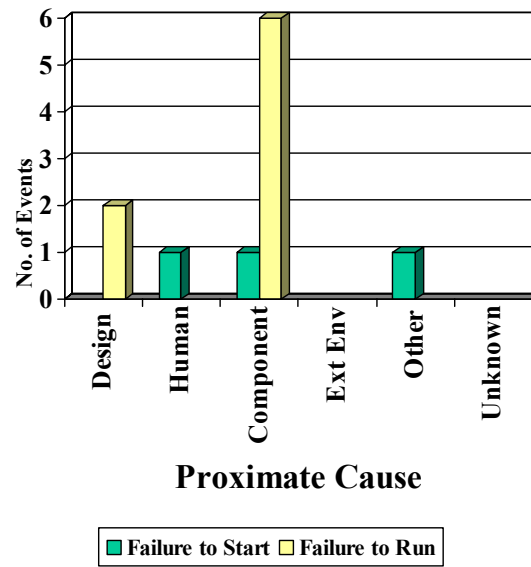


Figure 5-17. Proximate cause distribution for the SLC system.

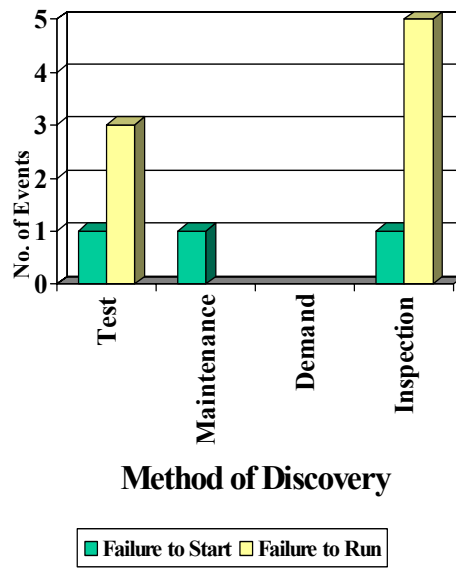


Figure 5-18. Method of discovery distribution for the SLC system.

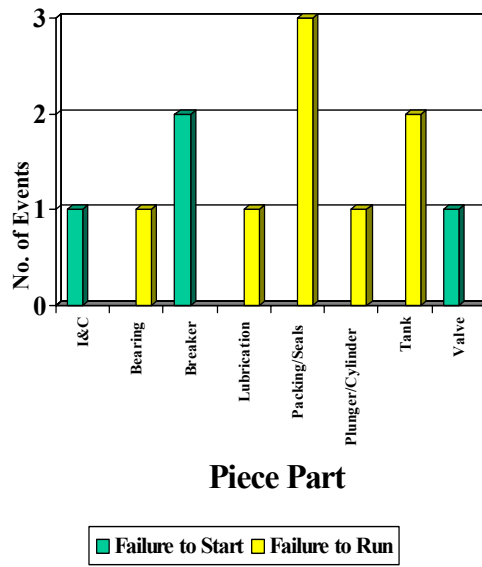


Figure 5-19. Piece part distribution for the SLC system.

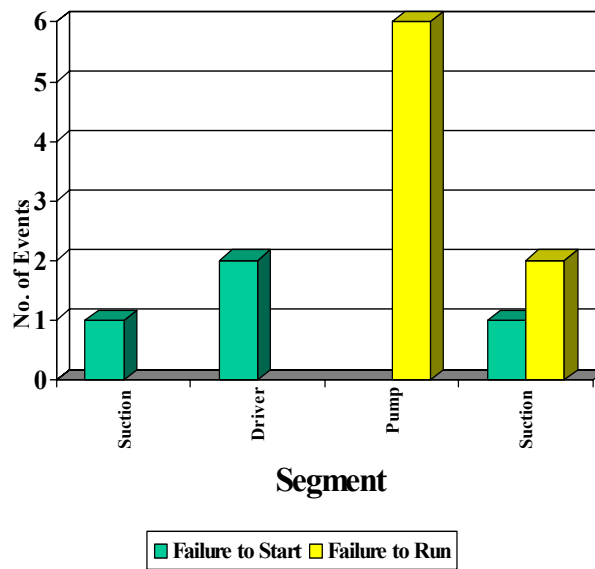


Figure 5-20. Segment distribution for the SLC system.

## 5.7 Residual Heat Removal (BWR)

Ten pump CCF events affected pumps in the RHR-B system (see Table C-1 in Appendix C, items 230 – 239). Figure 5-21 through Figure 5-24 show selected distributions for the RHR-B system. The most likely proximate cause was Internal to Component (50 percent) and the dominant failure mode was fail-to-start (80 percent). The most likely discovery method was Testing and half of the events involved circuit breaker failures. None of the RHR-B system CCF events were classified as either Complete or Almost Complete.

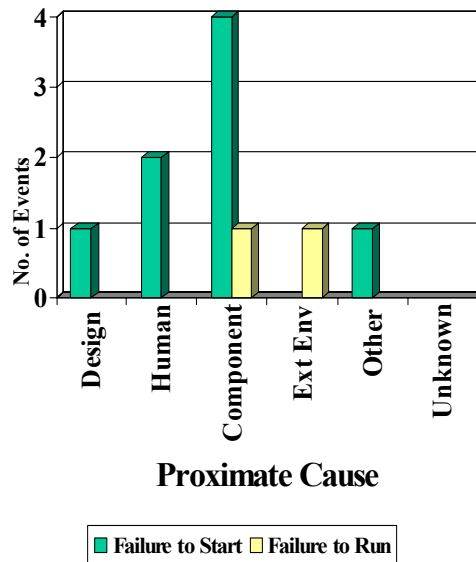


Figure 5-21. Proximate cause distribution for the RHR-B system.

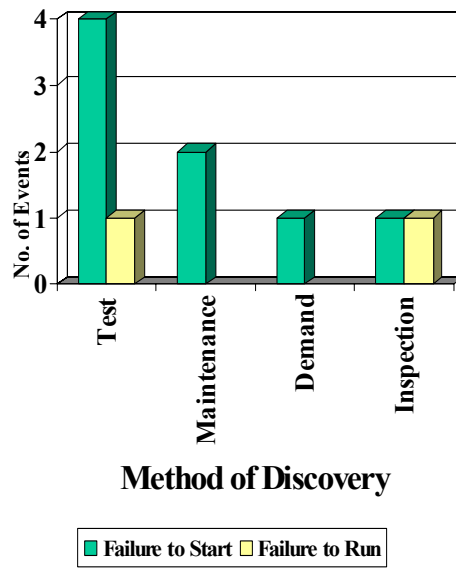


Figure 5-22. Method of discovery distribution for the RHR-B system.

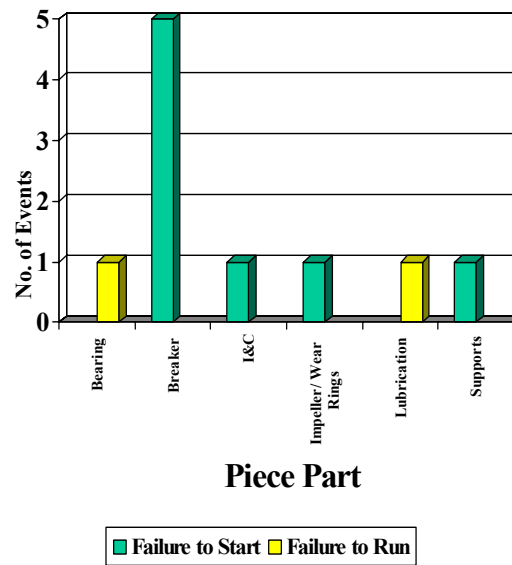


Figure 5-23. Piece part distribution for the RHR-B system.

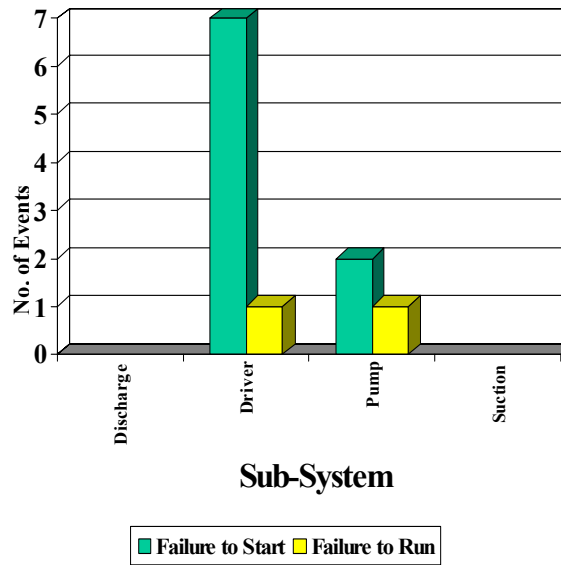


Figure 5-24. Segment distribution for the RHR-B system.

## 5.8 Other Systems

Three pump CCF events affected pumps in the CSS, HCI/RCI, and LCS systems. The small number of events in these systems precludes the presentation of CCF parameter charts. These events are included in this study since they are of interest. All of the events for these systems were Complete. The CSS event (Appendix C, Table C-1, item 39) involved the removal of control power prior to mode change. The HCI/RCI event (Appendix C, Table C-1, item 183) involved failure of both systems due to overfilling the reactor vessel, which filled the steam supply lines with water. The HCI count is low because it requires coincident failure of RCI. Most HCI failures were independent or RCI. In the LCS system (Appendix C, Table C-1, item 229), the CCF event involved improperly wired relays, which prevented auto start of the pumps under certain conditions.

## **6. HOW TO OBTAIN MORE DETAILED INFORMATION**

The pump CCF insights for the U.S. plants are derived from information contained in the CCF Database maintained for the NRC by the INEEL. The database contains CCF-related events that have occurred in U.S. commercial nuclear power plants reported in LERs, NPRDS failure records, and EPIX failure records. The NPRDS and EPIX information is proprietary. Thus, the information presented in the report has been presented in such a way to keep the information proprietary.

The subset of the CCF database presented in this volume is based on the pump component data from 1980 through 2000. The information contained in the CCF Database consists of coded fields and a descriptive narrative taken verbatim from LERs or NPRDS/EPIX failure records. The database was searched on component type (MDP and TDP) and failure mode. The failure modes selected were fail-to-start and fail-to-run. The additional fields, (e.g., proximate cause, coupling factor, shared cause factor, and component degradation values), along with the information contained in the narrative, were used to glean the insights presented in this report. The detailed records and narratives can be obtained from the CCF Database and from respective LERs and NPRDS/EPIX failure records.

The CCF Database was designed so that information can be easily obtained by defining searches. Searches can be made on any coded fields. That is, plant, date, component type, system, proximate cause, coupling factor, shared cause factor, reactor type, reactor vendor, CCG size, defensive mechanism, degree of failure, or any combination of these coded fields. The results for most of the figures in the report can be obtained or a subset of the information can be obtained by selecting specific values for the fields of interest. The identified records can then be reviewed and reports generated if desired. To obtain access to the NRC CCF Database, contact Dale Rasmuson at the NRC or Ted Wood at the INEEL.





## 7. REFERENCES

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## **Appendix A**

### **Data Summary**



# **Appendix A**

## **Data Summary**

This appendix is a summary of the data evaluated in the common-cause failure (CCF) data collection effort for pumps. The tables in this appendix support the charts in Chapter 3. Each table is sorted alphabetically, by the first four columns.

## Appendix A

|   |    |
|---|----|
| Table A-1. Pump CCF event summary, sorted by proximate cause. ....  | 3  |
| Table A-2. Pump CCF event summary, sorted by coupling factor. ....  | 25 |
| Table A-3. Pump CCF events, sorted by the method of discovery. .... | 48 |

Table A-1. Pump CCF event summary, sorted by proximate cause.

| Item | Proximate Cause  | Coupling Factor | Segment   | Discovery Method | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|--|-----------------|-----------|------------------|-------------|--------|------|------------------|-------------------|--|
| 1    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Discharge | Demand           | Valve       | AFW    | 1986 | Failure to Start | Partial           | Both the turbine driven and motor driven AFW pumps could not produce full flow because the cages in their discharge valve trapped debris and plugged.  |
| 2    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Discharge | Demand           | Valve       | AFW    | 1985 | Failure to Start | Partial           | Controller problems in the steam and diesel driven AFW pumps caused the pumps to trip on low suction pressure. The pump discharge flow controller valves were also not set properly after last maintenance. Low suction trips were due to design error.  |
| 3    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver    | Demand           | Lubrication | RHR-P  | 2000 | Failure to Run   | Complete          | Both RHR/LPI pumps fail to run due to improper oil in system. High bearing temperatures occurred when the pumps were operated. This was due to the wrong lube oil being used, which had too high a viscosity. Inadequate vender design information resulted in the higher viscosity oil being used and additional exacerbating problems such as insufficient bearing clearances. |
| 4    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver    | Demand           | I&C         | AFW    | 1981 | Failure to Start | Almost Complete   | Two AFW pumps failed to automatically start due to low suction pressure trips. A modification was installed to prevent this. This effect was discovered previously, but apparently had not been corrected prior to an attempt to start the pumps three weeks later.  |
| 5    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver    | Demand           | I&C         | AFW    | 1997 | Failure to Run   | Partial           | One actual AFW pump failure due to spurious electronic overspeed trip. Determined that all three pumps were susceptible to spurious overspeed trips.   |
| 6    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver    | Demand           | I&C         | AFW    | 1981 | Failure to Start | Almost Complete   | A modification to the control instrumentation for two AFW pumps resulted in a backfeed situation such that when called upon to start, both pumps would not start.  |
| 7    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver    | Inspection       | I&C         | AFW    | 1994 | Failure to Start | Partial           | Single failure would prevent auto initiation of AFW. Circuit design did not provide separation required by standards and code. The single failure identified was a short circuit across two conductors of the actuation relays associated with the initiation logic matrix.  |
| 8    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver    | Inspection       | Lubrication | HPI    | 2000 | Failure to Run   | Partial           | CVC makeup oil pump motor too small for certain accidents.   |

| Item | Proximate Cause  | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 9    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver  | Inspection       | Supports            | RHR-B  | 1986 | Failure to Start | Partial           | RHR motor internal supports were cracked due to stress and vibration. Design improvements were made.  |
| 10   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver  | Maintenance      | I&C                 | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |
| 11   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver  | Maintenance      | I&C                 | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |
| 12   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver  | Test             | I&C                 | AFW    | 1981 | Failure to Start | Almost Complete   | Two low suction pressure trips for the AFW pumps were mis-calibrated, which prevented the pumps from starting.  |
| 13   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver  | Test             | I&C                 | AFW    | 1992 | Failure to Start | Complete          | A modification design error (in 1983-1984) removed a start permissive interlock contact. At cold shutdown this de-energized the auxiliary lube oil pump, consequently, when one AFW pump was started it ran for 2.5 seconds and tripped on low oil pressure. Further investigation showed that both units AFW pumps would be affected in the same way. The design error combined with insufficient post modification testing led to this CCF event.   |
| 14   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Driver  | Test             | Breaker             | HPI    | 1980 | Failure to Start | Partial           | Upon testing the safety injection pumps it was found that the 6900-v breakers would lock-out preventing pump start if they were given a close signal for >0.32 seconds when a trip condition existed. There is no indication to operations when this locked-out condition exists. The breaker appears to be available for service when it actually is not. The only means of clearing the condition is to remove and reinstall the fuses at the breaker or manually change the state of the relays.                             |
| 15   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | All four emergency service water pumps showed cavitation damage. Two of the pumps had minor damage and were placed back in service. Recirculation cavitation occurs at flows significantly less than design.  |
| 16   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1981 | Failure to Run   | Complete          | Both charging pump service water pumps failed. A carbon cap screw failed allowing the impeller of one pump to bind on the casing. The ensuing leakage shorted the motor windings of the other pump.   |

| Item | Proximate Cause  | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 17   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1996 | Failure to Run   | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 18   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Pump    | Test             | Shaft               | AFW    | 1988 | Failure to Run   | Partial           | The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 19   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Pump    | Test             | Coupling            | ESW    | 1994 | Failure to Start | Partial           | Pump produced no flow when started. A shaft coupling failed. Material was determined to be brittle and have low impact properties. The coupling was replaced on all pumps with a type of material more suitable for this application.   |
| 20   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Pump    | Test             | Shaft               | AFW    | 1988 | Failure to Run   | Almost Complete   | An auxiliary feedwater pump failed its performance test. Subsequent inspection of the pump internals revealed significant damage, including a split in the center shaft sleeve. The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 21   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping              | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |
| 22   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping              | ESW    | 1983 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Pump Service Water pumps.   |
| 23   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping              | ESW    | 1982 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 24   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping              | ESW    | 1983 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |



| Item | Proximate Cause  | Coupling Factor | Segment | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--|-----------------|---------|------------------|------------|--------|------|------------------|-------------------|---|
| 25   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping     | ESW    | 1982 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 26   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping     | ESW    | 1981 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 27   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping     | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |
| 28   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping     | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the Charging Water Service Water pumps.  |
| 29   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping     | ESW    | 1996 | Failure to Start | Partial           | Freezing of diesel generator service water piping in intake bay. Inadequate initial design.   |
| 30   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Demand           | Piping     | ESW    | 1981 | Failure to Run   | Complete          | Increasing flow to chillers robs NPSH from charging service water pumps.  |
| 31   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Inspection       | Piping     | HPI    | 1988 | Failure to Run   | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the suction piping.   |
| 32   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Inspection       | Piping     | HPI    | 1991 | Failure to Start | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the alternate boration line and the gravity feed line from the boric acid storage tank.   |
| 33   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Inspection       | Piping     | HPI    | 1988 | Failure to Start | Partial           | It was determined that various pipes of the safety injection system and chemical volume and control system collected or trapped gas which might affect the functions of these systems. There was a concern that the gas pockets may adversely effect pump operation. Voids were detected in some of the high head SI pump piping. |

| Item | Proximate Cause  | Coupling Factor | Segment | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--|-----------------|---------|------------------|------------|--------|------|------------------|-------------------|---|
| 34   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Inspection       | Piping     | HPI    | 1990 | Failure to Start | Partial           | A quantity of gas was found in the centrifugal charging pump suction header that exceeded the maximum allowed gas volume. It was subsequently determined that hydrogen gas had been coming out of solution on both units and accumulating in the suction piping as a probable result of gas stripping by the CCP miniflow orifices. In addition, entrainment of hydrogen bubbles from the volume control tank to the CCP suction pipe may be a contributor as well. |
| 35   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Maintenance      | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 36   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Maintenance      | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 37   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Maintenance      | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 38   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Maintenance      | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 39   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Maintenance      | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 40   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Maintenance      | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 41   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Test             | Tank       | SLC    | 1991 | Failure to Run   | Complete          | During the performance of a special test on the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.   |

| Item | Proximate Cause  | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 42   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Test             | Tank                | SLC    | 1991 | Failure to Run   | Complete          | During the performance of a special test on Unit 1 to determine the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.   |
| 43   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Test             | Piping              | AFW    | 1999 | Failure to Run   | Partial           | All AFW trains declared inoperable due to inadequate suction flow capability from the nuclear service water alternate source. Inadequate flow caused by corroded piping. Piping is undersized so there is little margin for piping degradation. Since this is 1 of 4 suction sources, the safety significance is limited.   |
| 44   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Test             | Valve               | ESW    | 1983 | Failure to Start | Partial           | Low discharge pressure was caused by insufficient suction pressure. Service water flow to parallel components was adjusted.   |
| 45   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Design          | Suction | Test             | Tank                | ESW    | 1986 | Failure to Run   | Complete          | Loss of prime in the condenser circulating water siphon flow system caused loss of low pressure service water pumps. Pumps lost suction during a test due to poor design.   |
| 46   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Environmental   | Driver  | Inspection       | Piping              | HPI    | 2000 | Failure to Run   | Partial           | Microbiologically induced corrosion leak on service water lines to two charging/HPI pump lube oil coolers.  |
| 47   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Environmental   | Pump    | Demand           | Impeller/Wear Rings | ESW    | 2000 | Failure to Start | Almost Complete   | Two of the River Water pumps tripped on overcurrent when they were attempted to be started. The trips were a result of physical contact between the impeller and the lower casing liner of the pumps. This condition was due to differential thermal expansion between the pump shaft and the pump casing as a result of an elevated seal injection water temperature. The elevated temperature was due to an abnormal configuration of the Filtered Water System (the backup seal water supply). |
| 48   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Environmental   | Pump    | Inspection       | Lubrication         | HPI    | 1995 | Failure to Run   | Partial           | High lube oil temperatures were observed during HPI pump operation. Zinc particles from anode were discovered plugging the lube oil coolers. Accelerated corrosion was attributed to a corrosion inhibitor that was added to the system, which chemically interacted with the zinc.   |
| 49   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Environmental   | Pump    | Test             | Coupling            | ESW    | 1987 | Failure to Start | Partial           | Test showed two ESW pumps failed. Pump shafts were corroded and found to be made of incorrect material.   |

| Item | Proximate Cause  | Coupling Factor | Segment   | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--|-----------------|-----------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 50   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Environmental   | Suction   | Inspection       | Strainer            | ESW    | 2000 | Failure to Run   | Partial           | RHRSW Pumps Failed to Develop flow/pressure. Debris in intake structure. Requires modifications to the traveling Water Screen.  |
| 51   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance     | Pump      | Inspection       | Packing/Seals       | ESW    | 1997 | Failure to Run   | Partial           | Both ESW pumps leaking greater than 4 gpm because of inappropriate material for packing and sleeve (nitronic 60).   |
| 52   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance     | Pump      | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.  |
| 53   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance     | Pump      | Test             | Casing              | ESW    | 1997 | Failure to Run   | Almost Complete   | Both ESW pumps failed due to installation of wrong material for pump casing flanges by vendor during pump overhaul. The vendor overhauled the pumps without changing material. The plant returned the pumps to the warehouse also without verifying material.   |
| 54   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance     | Suction   | Demand           | I&C                 | HPI    | 1997 | Failure to Run   | Complete          | HPI pumps fail due to operation with inadequate suction head. Two pumps damaged due to operation with inadequate suction, but all three system pumps were unavailable due to the loss of the suction source. Suction source level instrumentation was the cause.  |
| 55   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Operational     | Discharge | Test             | Check Valve         | ESW    | 1999 | Failure to Run   | Partial           | Two ESW pumps had low flow due to interaction with the two other pumps when all four pumps were running.  |
| 56   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Driver    | Demand           | I&C                 | AFW    | 1989 | Failure to Start | Complete          | Both motor driven auxiliary feedwater pumps failed to start when the operator tried to start them manually. While preparing a design change, the designer failed to review all the unit specific documentation associated with the motor-driven AFW pump wiring and made the erroneous assumption that both units switchgear compartment internal wiring was identical. In fact, the wiring for each unit was different. Consequently, when the design change was installed, it was installed in accordance with the erroneous design. The wiring discrepancy was corrected and the motor-driven AFW pumps were tested and returned to service. |
| 57   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Driver    | Demand           | Breaker             | ESW    | 1996 | Failure to Start | Partial           | Two RHRSW pumps fail to start due to breaker failures. Wrong contacts were installed. Design called for contacts to have a minimum current interrupt rating of 6 amps; contacts installed (that subsequently failed) had current interrupt rating of only 2.2 amps.   |

| Item | Proximate Cause  | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 58   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Driver  | Demand           | Motor               | ESW    | 1987 | Failure to Start | Partial           | ESW pump motors tripped on overcurrent. The overcurrent trip was due to a ground and a short on the pump motor.   |
| 59   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Driver  | Test             | Breaker             | LCS    | 1980 | Failure to Start | Complete          | Relay extra contacts left connected during construction, prevented Core Spray pump start with emergency diesel generator breakers racked out.   |
| 60   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Driver  | Test             | I&C                 | AFW    | 1980 | Failure to Start | Complete          | During surveillance testing, neither motor-driven AFW pump would start. The pump control circuit was found with autostart defeat switches labeled backwards, causing all autostarts except the low-low steam generator level to be defeated. The labels were corrected and the links were closed. The original installation error was the result of an inadequate design change process that did not require sufficient verification and testing of the modification.   |
| 61   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps drawing excessive current. Carbon steel snap rings corroded allowing impeller to come in contact with casing. The third pump, although not exhibiting abnormal current, had similar corrosion   |
| 62   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1996 | Failure to Run   | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 63   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Pump    | Inspection       | Casing              | AFW    | 1983 | Failure to Run   | Partial           | Two AFW pumps thrust tolerance was out of specification. These events were caused by improperly installed balancing drum parts. One turbine driven and one motor driven pump was involved.  |
| 64   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Pump    | Inspection       | Casing              | HPI    | 1987 | Failure to Run   | Partial           | During inspection of a centrifugal charging pump, a portion of the stainless steel cladding on the inside surface of the pump casing exhibited corrosion. Corrosion of the pump casing was through the stainless steel cladding into the carbon steel base material. Inspection of the other CCP revealed similar corrosion. The cause of this event was a manufacturing deficiency. Corrosion observed at the pump casing discharge nozzle was attributed to a cladding breakthrough during final machining. Corrosion observed at the pump casing inlet end was attributed to either over-machining of the cladding or inadequate overlay of two adjacent weld beads.   |

| Item | Proximate Cause  | Coupling Factor | Segment   | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|--|-----------------|-----------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 65   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Pump      | Test             | Impeller/Wear Rings | ESW    | 1986 | Failure to Start | Partial           | Testing of the service water system disclosed that the performance of the three service water pumps was below requirements. The condition is the result of both an inadequate system design and the installation of replacement impellers, which were not modified by the vendor to improve performance, as were the original impellers.   |
| 66   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Suction   | Demand           | Piping              | ESW    | 1984 | Failure to Start | Partial           | Both RHR service water pumps tripped as a result of inadequate venting of suction header resulting from poor orientation of the vent line.   |
| 67   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Quality         | Suction   | Inspection       | Piping              | HPI    | 1988 | Failure to Run   | Partial           | Vortex breakers had not been installed in the containment emergency sumps. Vortex breakers are required to be installed in the containment emergency sumps to prevent the formation of vortices which could adversely affect performance of safety injection pumps during the safety injection and containment spray systems were declared inoperable.   |
| 68   | External Environment   | Design          | Discharge | Demand           | Check Valve         | AFW    | 1983 | Failure to Start | Almost Complete   | Hot water in the AFW pump casings caused the pumps to become vapor bound. The hot water was from leaking check valves upstream of the pumps. This event occurred once on the turbine driven pump and 5 times on the motor driven pump.   |
| 69   | External Environment   | Design          | Discharge | Inspection       | Piping              | HPI    | 1994 | Failure to Run   | Partial           | Due to a leaking socket weld in the common recirculation line, all three SI pumps were declared inoperable. The underlying cause of the leak was a crack in the socket weld in the common recirculation line, caused by pipe displacement from air entrainment and pump misalignment.  |
| 70   | External Environment   | Design          | Pump      | Inspection       | Bearing             | HPI    | 1991 | Failure to Run   | Almost Complete   | Charging/safety pumps beyond operational limits. Damage was found to the thrust bearings. Air was introduced into this train of chilled water during modifications and testing being performed on the system. This air became trapped in high points of either, or both of, the supply and return chilled water lines to the charging pump. At the reduced flow rate, sufficient cooling was not available and oil temperature increased to the point where bearing damage occurred. |
| 71   | External Environment   | Environmental   | Discharge | Test             | Recirc              | HPI    | 1992 | Failure to Run   | Almost Complete   | Safety Injection pumps were declared inoperable due to an observed declining trend in the pump's recirculation flow. The cause of the Safety Injection pump reduced recirculation flow is attributed to foreign material blockage within the associated minimum flow recirculation line flow orifice.  |
| 72   | External Environment   | Environmental   | Driver    | Demand           | Motor               | ESW    | 1985 | Failure to Run   | Partial           | Two service water motors failed on demand as a result of cement dust contamination.  |
| 73   | External Environment   | Environmental   | Driver    | Demand           | I&C                 | AFW    | 1984 | Failure to Start | Complete          | Both AFW pumps failed to start. The problem was traced to two relays (1 per pump). Examination of the relays revealed open circuiting and severe degradation of the insulation.  |
| 74   | External Environment   | Environmental   | Driver    | Maintenance      | Motor               | ESW    | 1987 | Failure to Start | Partial           | During an extended service water bay flooding incident, one ESW pump was found grounded by testing, later two more pumps were found to be failed also.   |
| 75   | External Environment   | Environmental   | Driver    | Test             | Bearing             | RHR-B  | 1991 | Failure to Run   | Partial           | Two LCI pumps were declared inoperable due to high motor vibration.  |
| 76   | External Environment   | Environmental   | Pump      | Inspection       | Coupling            | ESW    | 1993 | Failure to Run   | Partial           | Entrained debris caused ESW pump shaft coupling to fail. Plant equipment did not prevent this debris from entering pump.   |

| Item | Proximate Cause       | Coupling Factor | Segment   | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------------|-----------------|-----------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 77   | External Environment  | Environmental   | Pump      | Inspection       | Packing/Seals       | RHR-P  | 1985 | Failure to Start | Complete          | Following a trip, water was found spraying from both low head safety injection pump wedge control rod seals. Both pumps were declared inoperable. Postulated failure on the seals was from a minor flow induced pressure transient.  |
| 78   | External Environment  | Environmental   | Suction   | Demand           | Piping              | HPI    | 1984 | Failure to Start | Complete          | Boron solidification in the suction and gas binding of pumps led to the failure of all three safety injection pumps. Flushing procedures inadequate.   |
| 79   | External Environment  | Maintenance     | Driver    | Demand           | Breaker             | AFW    | 1990 | Failure to Run   | Partial           | AFW pumps circuit breakers degraded.   |
| 80   | External Environment  | Operational     | Driver    | Inspection       | I&C                 | HPI    | 1990 | Failure to Run   | Complete          | It was determined that the common minimum flow path return line for the safety injection pumps to the refueling water storage tank was frozen. Previous actions to investigate problems with the freeze protection system were unsuccessful in preventing development of this condition. The two HPI pumps were declared inoperable with this return line frozen. A faulty ambient temperature switch for the RWST heat trace system prevented the heat trace from activating and was subsequently replaced. In addition, administrative controls did not sufficiently recognize the safety significance of flow through this line and the need to ensure flow capability. |
| 81   | Internal to Component | Design          | Driver    | Demand           | Breaker             | ESW    | 2000 | Failure to Start | Almost Complete   | Two ESW pumps failed to start due to their breakers failing to close. The breakers' prop spring bracket has slipped thus preventing proper interfacing between the prop and the prop pin.  |
| 82   | Internal to Component | Design          | Driver    | Inspection       | I&C                 | ESW    | 1982 | Failure to Start | Partial           | Open circuit breaker resulted in loss of two RHR service water pumps.  |
| 83   | Internal to Component | Design          | Pump      | Inspection       | Lubrication         | HPI    | 1981 | Failure to Run   | Partial           | Corrosion of HPI pump cooler heads. Improper material led to corrosion   |
| 84   | Internal to Component | Environmental   | Discharge | Demand           | Valve               | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavtrol cages for these valves were clogged with shredded Asiatic clam shells.   |
| 85   | Internal to Component | Environmental   | Discharge | Demand           | Valve               | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavtrol cages for these valves were clogged with shredded Asiatic clam shells.   |
| 86   | Internal to Component | Environmental   | Discharge | Test             | Recirc              | HPI    | 1991 | Failure to Run   | Partial           | Something in HPI pump recirculation line was restricting flow. The piece later dislodged and no identification was made. Both SI pumps had inadequate recirculation flow.  |
| 87   | Internal to Component | Environmental   | Pump      | Demand           | Impeller/Wear Rings | ESW    | 1994 | Failure to Run   | Partial           | Raw water pump currents stayed high after starting. The primary cause of these events was determined to be elevated sand content in the river, resulting in excessive sand accumulation around the suction area of the pumps.  |
| 88   | Internal to Component | Environmental   | Pump      | Inspection       | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Marine growth in suction.   |
| 89   | Internal to Component | Environmental   | Pump      | Inspection       | Lubrication         | HPI    | 1983 | Failure to Run   | Partial           | Oysters and miscellaneous mollusks plugged HPI oil coolers. Two pumps were required to be shutdown due to rising lubricating oil temperatures.   |

| Item | Proximate Cause       | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------------|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 90   | Internal to Component | Environmental   | Pump    | Inspection       | Packing/Seals       | ESW    | 1994 | Failure to Run   | Partial           | Backup seal water regulators did not provide required flow during testing on two pumps. The third pump lost seal flow while operating. The cause was attributed to plugged lines.  |
| 91   | Internal to Component | Environmental   | Pump    | Maintenance      | Packing/Seals       | ESW    | 1985 | Failure to Run   | Partial           | First pump developed seal leak due to sand. Second pump had high bearing temperatures due to trash clogging cooling water lines.   |
| 92   | Internal to Component | Environmental   | Pump    | Maintenance      | Lubrication         | HPI    | 1980 | Failure to Run   | Partial           | HPI pump lube oil cooler with tube leak allowed water into oil reservoir.  |
| 93   | Internal to Component | Environmental   | Pump    | Maintenance      | Lubrication         | HPI    | 1986 | Failure to Run   | Almost Complete   | Clams/sludge fouling of lube oil cooler caused high temperature alarms on two HPI pumps.   |
| 94   | Internal to Component | Environmental   | Pump    | Maintenance      | Lubrication         | HPI    | 1991 | Failure to Run   | Partial           | HPI pump lube oil cooler leaks. Degraded tubes.  |
| 95   | Internal to Component | Environmental   | Pump    | Test             | Bearing             | ESW    | 1992 | Failure to Run   | Partial           | Abrasive particles present in ocean water produced accelerated wear of shaft bearing journals.   |
| 96   | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | HPI    | 1984 | Failure to Run   | Almost Complete   | One HPI pump seized, the second would have seized if operated.   |
| 97   | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1995 | Failure to Start | Partial           | Marine growth caused low flow and speed condition for two service water pumps  |
| 98   | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. A rag was found in one impeller and a plastic bottle in the other.   |
| 99   | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1982 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.                     |
| 100  | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1982 | Failure to Run   | Partial           | Low ESW pump head values were caused excessive wear of pump impeller due to foreign material in the service water.   |
| 101  | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1993 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head values. The low pump heads were caused by excessive wear of pump impeller due to sand in the service water.                                 |
| 102  | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1991 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted. |
| 103  | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1995 | Failure to Start | Partial           | Pumps failed performance test. Sand in water eroded pump internals. Pump lift was adjusted.  |
| 104  | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.  |
| 105  | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1994 | Failure to Start | Partial           | Degraded performance identified during testing. Sand in water was causing accelerated wear of the pump internals. Lift was adjusted for three pumps and one pump internals were replaced.                                |
| 106  | Internal to Component | Environmental   | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Run   | Partial           | ESW pump impeller lift out of adjustment.  |
| 107  | Internal to Component | Environmental   | Suction | Demand           | Piping              | ESW    | 1986 | Failure to Start | Partial           | RHR service water pumps failed flow testing due to blocked suctions and abnormal wear of impellers.  |
| 108  | Internal to Component | Environmental   | Suction | Demand           | Strainer            | ESW    | 1980 | Failure to Run   | Partial           | Foreign material was allowed to enter the suction of the charging pump service water pumps resulting in low flow conditions.   |



| Item | Proximate Cause       | Coupling Factor | Segment   | Discovery Method | Piece Part    | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------------|-----------------|-----------|------------------|---------------|--------|------|------------------|-------------------|---|
| 109  | Internal to Component | Environmental   | Suction   | Inspection       | Strainer      | ESW    | 1984 | Failure to Run   | Partial           | Two RHR service water pumps had blown seals and sparks and smoke between the bearing housing and shaft. A piece of hard rubber valve liner was found in the pumps.  |
| 110  | Internal to Component | Environmental   | Suction   | Test             | Strainer      | ESW    | 1990 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by suction blockage due to foreign material in the service water.   |
| 111  | Internal to Component | Environmental   | Suction   | Test             | Piping        | ESW    | 1990 | Failure to Start | Partial           | ESW pumps failed flow testing. Foreign material blocked the suction.  |
| 112  | Internal to Component | Environmental   | Suction   | Test             | Strainer      | ESW    | 1982 | Failure to Run   | Partial           | Failures occurred on residual heat removal service water pumps. The pumps failed to meet flow and pressure requirements. Failure was due to debris lodging in pump impellers. Source of debris was maintenance activities, broken traveling water screens, and the inadvertent opening of a RHR minimum flow line which washed materials into suction pit.  |
| 113  | Internal to Component | Maintenance     | Discharge | Inspection       | Check Valve   | AFW    | 1990 | Failure to Start | Almost Complete   | Leakage past AFW check valves caused AFW pumps to become steam bound. Closed motor operated valve in line. Scheduled check valves for replacement next outage.  |
| 114  | Internal to Component | Maintenance     | Discharge | Test             | Valve         | HPI    | 1984 | Failure to Start | Partial           | CCP pump low flow rates due to inaccuracies in positioning the throttle valves.   |
| 115  | Internal to Component | Maintenance     | Driver    | Demand           | I&C           | ESW    | 1991 | Failure to Start | Partial           | Two ESW pumps failed to start due to failed breakers. Inadequate maintenance.   |
| 116  | Internal to Component | Maintenance     | Driver    | Demand           | Breaker       | RHR-B  | 1987 | Failure to Start | Partial           | RHR pump breakers failed to close when operated remotely from the control room. It was found that the latch roller bearings and the cam follower bearing (internal piece parts of the breaker) were not operating correctly. This prevented the trip latch assembly from resetting and allowing the breaker to close.   |
| 117  | Internal to Component | Maintenance     | Driver    | Demand           | Lubrication   | HPI    | 1984 | Failure to Run   | Partial           | Charging pump lube oil cooler fan motor trips on thermal overload. Probable cause: normal wear on motor resulting in increased friction replaced worn motor with spare. During routine inservice testing found that another charging pump lube oil cooler fan motor had a current imbalance. Probable cause: normal aging of motor insulation has resulted in a current imbalance.  |
| 118  | Internal to Component | Maintenance     | Driver    | Inspection       | Bearing       | ESW    | 1981 | Failure to Run   | Partial           | ESW motor to pump alignment problems. Bearings worn out.  |
| 119  | Internal to Component | Maintenance     | Driver    | Inspection       | Bearing       | ESW    | 1985 | Failure to Run   | Partial           | One service water pump motor upper bearing oil reservoir leaking from cover plate. Another service water pump motor upper oil cooler oil reservoir leaking.   |
| 120  | Internal to Component | Maintenance     | Driver    | Inspection       | Breaker       | ESW    | 1996 | Failure to Start | Partial           | ESW pump breakers fail due to misalignment of the breaker mechanism and internals developed over the years of operation.  |
| 121  | Internal to Component | Maintenance     | Driver    | Inspection       | Packing/Seals | HPI    | 1988 | Failure to Run   | Almost Complete   | Smoke was discovered coming from the speed increaser unit for a centrifugal charging pump. Investigation found the two gland seal retaining bolts inside the speed increaser lube oil pump backed out allowing the gland seal to loosen. The gland seal being loosened, caused reduced oil flow to the speed increaser internals and ultimate damage. Other CCPs were inspected, and the same gland seal bolts as on the first pump were found loosened. The cause of the bolts backing out was determined to be lack of a periodic adjustment of the gland seal bolts. |
| 122  | Internal to Component | Maintenance     | Driver    | Maintenance      | Breaker       | ESW    | 1985 | Failure to Start | Partial           | Two raw water pump breaker main wipes were out of adjustment.   |
| 123  | Internal to Component | Maintenance     | Driver    | Maintenance      | Breaker       | HPI    | 1991 | Failure to Start | Partial           | HPI pump breakers failed due to a broken pawl, and a broken closing coil.   |

| Item | Proximate Cause       | Coupling Factor | Segment | Discovery Method | Piece Part       | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------------|-----------------|---------|------------------|------------------|--------|------|------------------|-------------------|---|
| 124  | Internal to Component | Maintenance     | Driver  | Maintenance      | Breaker          | AFW    | 1992 | Failure to Start | Partial           | With the unit in a refueling outage, following repairs to a motor driven auxiliary feedwater pump local/remote switch of the circuit breaker, personnel found that the switch contacts would not close. This failure rendered one of three auxiliary feedwater pumps inoperable. The cause of the failure appears to be due to dirty/corroded contacts on the switch. |
| 125  | Internal to Component | Maintenance     | Driver  | Maintenance      | Breaker          | SLC    | 1999 | Failure to Start | Partial           | SLC Pump Breakers Fail to pickup on degraded voltage test   |
| 126  | Internal to Component | Maintenance     | Driver  | Test             | Bearing          | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps exhibited vibration. Attributed to normal wear.   |
| 127  | Internal to Component | Maintenance     | Driver  | Test             | Breaker          | RHR-B  | 1997 | Failure to Start | Partial           | Breaker latch check switch failed on both pumps. Lack of lubrication.   |
| 128  | Internal to Component | Maintenance     | Driver  | Test             | Breaker          | ESW    | 1998 | Failure to Start | Partial           | Two RHR service water pump breakers would not close due to dirty contacts in breakers.  |
| 129  | Internal to Component | Maintenance     | Driver  | Test             | Breaker          | ESW    | 1998 | Failure to Start | Partial           | Service water pumps fail to start due to circuit breaker failures. Pump breakers failed to close due to failures of the charging spring/motor and closing spring motor.   |
| 130  | Internal to Component | Maintenance     | Driver  | Test             | Breaker          | AFW    | 1997 | Failure to Start | Almost Complete   | The circuit breakers associated with the AFW Pumps failed to close as required. The root cause of the failure was the binding in the operating mechanism. The plunger apparently did not always complete its upward movement to close and latch the breaker, due to accumulated dirt and lubricants.  |
| 131  | Internal to Component | Maintenance     | Driver  | Test             | Breaker          | RHR-B  | 1986 | Failure to Start | Partial           | RHR pump circuit breakers failed during a start for testing. Bend switch and binding mechanism. Attributed to inadequate maintenance.   |
| 132  | Internal to Component | Maintenance     | Pump    | Demand           | Casing           | ESW    | 1998 | Failure to Start | Partial           | Two ESW pump started and ran, but would not develop sufficient pressure or flow rate. Exact cause not known for either failure, however, one pump was noted to have microbiological induced corrosion fouling on internal surfaces.   |
| 133  | Internal to Component | Maintenance     | Pump    | Demand           | Bearing          | AFW    | 1984 | Failure to Run   | Partial           | One ESW bearing failed and pump seized; second motor bearing failed.  |
| 134  | Internal to Component | Maintenance     | Pump    | Demand           | Packing/Seals    | AFW    | 1998 | Failure to Run   | Partial           | AFW MDP and TDPs failed due to incorrect packing installed.   |
| 135  | Internal to Component | Maintenance     | Pump    | Inspection       | Packing/Seals    | ESW    | 1989 | Failure to Run   | Partial           | ESW pump excessive packing leakage.   |
| 136  | Internal to Component | Maintenance     | Pump    | Inspection       | Casing           | ESW    | 1986 | Failure to Run   | Partial           | Cracked seal water and vent lines.  |
| 137  | Internal to Component | Maintenance     | Pump    | Inspection       | Bearing          | ESW    | 1987 | Failure to Run   | Partial           | Service water pumps had high shaft vibration. The excessive vibrations caused by worn bearings and shaft sleeves.   |
| 138  | Internal to Component | Maintenance     | Pump    | Inspection       | Packing/Seals    | AFW    | 1990 | Failure to Run   | Partial           | Both motor-driven aux. feedwater pumps had excessive packing leaks, due to worn packing.  |
| 139  | Internal to Component | Maintenance     | Pump    | Inspection       | Lubrication      | RHR-B  | 1990 | Failure to Run   | Partial           | Both pump motor oil coolers were leaking due to aging of components. The first case involved through wall corrosion and the pump was immediately removed from service. The second case was a packing leak.  |
| 140  | Internal to Component | Maintenance     | Pump    | Inspection       | Plunger/Cylinder | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pump seal was leaking excessively. The cause of this failure was normal wear of the plungers, packing, and head gaskets for the plungers (piece parts of the pump).  |
| 141  | Internal to Component | Maintenance     | Pump    | Inspection       | Packing/Seals    | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking profusely at the packing. The failure of the packing was attributed to normal wear.  |

| Item | Proximate Cause       | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------------|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 142  | Internal to Component | Maintenance     | Pump    | Inspection       | Packing/Seals       | SLC    | 1987 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing adjusted.   |
| 143  | Internal to Component | Maintenance     | Pump    | Inspection       | Packing/Seals       | ESW    | 1986 | Failure to Run   | Partial           | Excessive packing leakage. Both events occurred after previous maintenance had been performed for the same problems.   |
| 144  | Internal to Component | Maintenance     | Pump    | Inspection       | Packing             | AFW    | 1986 | Failure to Run   | Partial           | The packing was worn on both the motor-driven and one turbine-driven aux. feedwater pump, causing high temperature on one packing gland, and excessive leaking on the other pump.  |
| 145  | Internal to Component | Maintenance     | Pump    | Inspection       | Bearing             | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. Loose fittings and lack of thread sealant.  |
| 146  | Internal to Component | Maintenance     | Pump    | Inspection       | Packing/Seals       | SLC    | 1988 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing replaced.   |
| 147  | Internal to Component | Maintenance     | Pump    | Inspection       | Casing              | ESW    | 1988 | Failure to Run   | Partial           | RHR service water pumps. Pump diffuser eroded on first pump and a through wall casing leak developed on the second.  |
| 148  | Internal to Component | Maintenance     | Pump    | Inspection       | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. The cause of the failure is suspected to be binding.  |
| 149  | Internal to Component | Maintenance     | Pump    | Maintenance      | Bearing             | ESW    | 1985 | Failure to Run   | Partial           | High ESW pump vibration was caused by wearing of the upper bearings.   |
| 150  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | ESW pump performance decreased 15% and 8% respectively since last test. Pumps were replaced.   |
| 151  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1994 | Failure to Run   | Partial           | Two ESW pumps had internal deterioration, one of which was indicated by high vibration readings.   |
| 152  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW impeller gaps too wide. Gaps adjusted.   |
| 153  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | HPI    | 1985 | Failure to Start | Partial           | The CCPs were tested and had low flow rates. The most probable cause is attributed to observed degradation of the pumps. The CCPs are subject to normal wear associated with their secondary duty of providing normal charging flow. |
| 154  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1984 | Failure to Run   | Partial           | Containment spray raw water pumps failed flow tests. Aging and normal wear.  |
| 155  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to brackish water corrosion.   |
| 156  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1984 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 157  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1987 | Failure to Run   | Partial           | ESW pump low flow. Worn impellers.   |
| 158  | Internal to Component | Maintenance     | Pump    | Test             | Coupling            | ESW    | 1987 | Failure to Start | Almost Complete   | Two ESW pumps had failed couplings. Cause attributed to abnormal stress.   |
| 159  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1989 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 160  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | The charging pump service water pumps degraded. Caused by expected wear of pump due to erosion and corrosion properties of the process fluid involved  |
| 161  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | ESW pumps had worn impellers and one had a plugged strainer.   |

| Item | Proximate Cause       | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------------|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 162  | Internal to Component | Maintenance     | Pump    | Test             | Packing/Seals       | ESW    | 1981 | Failure to Start | Partial           | RHR service water pumps failed to meet flow requirements due to seal water leakage and pump wearout.   |
| 163  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1991 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 164  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 165  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1982 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 166  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. One pump also exhibited high vibration.   |
| 167  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1994 | Failure to Start | Partial           | Two ESW pumps had low discharge pressure during testing. Each pump had worn internals and both pump internals were replaced.   |
| 168  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed due to worn internals.  |
| 169  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.  |
| 170  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1984 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.  |
| 171  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Run   | Partial           | ESW pumps had worn and cracked impellers. Aging and normal wear.   |
| 172  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1998 | Failure to Start | Partial           | Two ESW pumps failed to develop adequate flow/pressure - pumps degraded.   |
| 173  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 174  | Internal to Component | Maintenance     | Pump    | Test             | Bearing             | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 175  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 176  | Internal to Component | Maintenance     | Pump    | Test             | Shaft               | ESW    | 1993 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.   |
| 177  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | Emergency service water pumps discharge pressure below allowable limits. Causes were loose impellers, dropped impeller, and worn internals.  |
| 178  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted. |
| 179  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 180  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.   |
| 181  | Internal to Component | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1983 | Failure to Run   | Partial           | RHR Service Water pumps failed flow tests due to wearout and had to be rebuilt.  |

| Item | Proximate Cause          | Coupling Factor | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------------------------|-----------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 182  | Internal to Component    | Maintenance     | Pump    | Test             | Impeller/Wear Rings | RHR-B  | 1985 | Failure to Start | Partial           | The first pump failed to meet required flow rate. The second was drawing excessive amperage. Both conditions were attributed to worn internals.   |
| 183  | Internal to Component    | Maintenance     | Pump    | Test             | Impeller/Wear Rings | HPI    | 1983 | Failure to Start | Partial           | SI pump and both CCPs failed to meet the minimum head curve requirements. The cause of pump head capacity degradation has been attributed to normal pump operation. The inability to balance flows has been attributed to the lower head capacity of the pumps.   |
| 184  | Internal to Component    | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | Wear caused high ESW pump bearing temperatures, vibration, and low amperage/flow.   |
| 185  | Internal to Component    | Maintenance     | Pump    | Test             | Impeller/Wear Rings | ESW    | 1981 | Failure to Start | Partial           | Loss of Service Water pump due to wearout at end of life.   |
| 186  | Internal to Component    | Maintenance     | Pump    | Test             | Lubrication         | SLC    | 1992 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. The gasket between the crankcase frame cap and the gear housing cover was worn.  |
| 187  | Internal to Component    | Maintenance     | Pump    | Test             | Coupling            | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.   |
| 188  | Internal to Component    | Quality         | Pump    | Demand           | Impeller/Wear Rings | AFW    | 1988 | Failure to Run   | Partial           | Following a plant trip, it was discovered that the auxiliary feedwater pumps had internal damage. Some channel ring vanes had chips missing, and several parts were found in the SG auxiliary feedwater piping.   |
| 189  | Operational/ Human Error | Design          | Driver  | Demand           | I&C                 | ESW    | 1980 | Failure to Start | Partial           | Instrument isolation valve closed causing a low suction trip signal to two RHRSW pumps.   |
| 190  | Operational/ Human Error | Design          | Driver  | Inspection       | Breaker             | ESW    | 1984 | Failure to Start | Partial           | During an attempt to perform preventive maintenance for unit one's RHR service water pumps, plant personnel mistakenly disconnected the motor leads for unit two's RHR service water pump.  |
| 191  | Operational/ Human Error | Design          | Driver  | Test             | Breaker             | AFW    | 1985 | Failure to Start | Complete          | Both AFW pumps failed to start when tested, due to the circuit breakers not being racked in properly.   |
| 192  | Operational/ Human Error | Design          | Pump    | Demand           | Impeller/Wear Rings | AFW    | 1990 | Failure to Run   | Almost Complete   | Due to a combination of management error and procedural deficiency, the turbine driven auxiliary feedwater pump was run deadheaded. The operation damaged the pump. When the pump was manually tripped, steam vented back into the suction line, caused another AFW pump to also trip, on a low suction pressure signal.  |
| 193  | Operational/ Human Error | Design          | Suction | Demand           | Piping              | RHR-P  | 1980 | Failure to Run   | Complete          | The reactor vessel vent eductor was in service in preparation for refueling with RHR operating. A low flow alarm was received and low flow and low motor current were indicated. A second pump was started and became air-bound. Putting the vessel vent eductor system into service was the root cause of the incident.  |
| 194  | Operational/ Human Error | Design          | Suction | Demand           | Piping              | RHR-P  | 1985 | Failure to Run   | Complete          | Swap over of RHR pumps resulted in both trains becoming inoperable due to air injection into the suction of the pumps. This required both pumps to be vented and required RCS level to be raised to prevent a possible recurrence of the vortex problem.  |
| 195  | Operational/ Human Error | Design          | Suction | Demand           | Tank                | AFW    | 1980 | Failure to Run   | Complete          | Both emergency feedwater pumps lost feed pump suction. The emergency feedwater pump suction flashed to steam due to the feedwater train flashing and forcing hot water back through the startup and blowdown tanks and into the feedwater pump suction. To prevent this recurrence, the operating procedures have been changed to require isolating the startup and blowdown effluent as a source of emergency feedwater suction prior to increasing power. |

| Item | Proximate Cause          | Coupling Factor | Segment | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------------------------|-----------------|---------|------------------|------------|--------|------|------------------|-------------------|---|
| 196  | Operational/ Human Error | Design          | Suction | Demand           | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | Suction was lost to both RHR pumps. RHR flow was less than 3000 gpm and pump amps were fluctuating prior to taking corrective action. Each of these events appear to have been caused by a slow decrease in RCS level in conjunction with the vortex action at the pump suction.  |
| 197  | Operational/ Human Error | Design          | Suction | Demand           | Piping     | RHR-P  | 1984 | Failure to Run   | Almost Complete   | On two occasions, RHR pumps cavitated due to low RCS level while draining the RCS.  |
| 198  | Operational/ Human Error | Maintenance     | Driver  | Demand           | Breaker    | ESW    | 1988 | Failure to Run   | Partial           | Service water pump high dropout over current protection devices were less than running current conditions and trip setpoints did not account for changing load conditions due to modified impellers. Three pump trips had occurred.   |
| 199  | Operational/ Human Error | Maintenance     | Driver  | Demand           | Breaker    | ESW    | 1987 | Failure to Start | Partial           | One breaker failed to linkage alignment and second from loose relay connections. Inadequate maintenance.  |
| 200  | Operational/ Human Error | Maintenance     | Driver  | Demand           | Breaker    | ESW    | 1993 | Failure to Start | Partial           | Operations personnel were attempting to swap the running service water pump with the idle service water pump. Personnel placed the control switch to start and the service water pump did not start. Breaker malfunction. Later, another service water pump failed to start because of the breaker.   |
| 201  | Operational/ Human Error | Maintenance     | Driver  | Inspection       | Bearing    | RHR-P  | 1988 | Failure to Run   | Partial           | Residual heat removal pump motor upper bearing housings were observed to be leaking oil. The cause of the failure was attributed to a lack of sealant being applied and gasket installed after the last maintenance was performed on the motor bearing housing.   |
| 202  | Operational/ Human Error | Maintenance     | Driver  | Inspection       | I&C        | RHR-P  | 1992 | Failure to Start | Complete          | Both trains of RHR were rendered inoperable for two minutes, while performing an operational readiness test surveillance procedure. The surveillance procedure required that the one RHR train pump be placed in pull to lock and the other train heat exchanger flow control valve throttled to 30-40% open. The procedure directed the operators to perform operations that resulted in both trains of RHR being inoperable                             |
| 203  | Operational/ Human Error | Maintenance     | Driver  | Inspection       | I&C        | AFW    | 1990 | Failure to Start | Complete          | During testing one AFW pump was tested and other was tested without returning first to auto. Both pumps were unavailable at the same time. The procedure was the cause.   |
| 204  | Operational/ Human Error | Maintenance     | Driver  | Inspection       | Breaker    | RHR-P  | 1981 | Failure to Start | Complete          | All RHR pumps de-energized to replace RHR Relief valve. T.S. allows this condition for 1 hour. Operated in the mode in excess of 5 hours.   |
| 205  | Operational/ Human Error | Maintenance     | Driver  | Maintenance      | Breaker    | RHR-B  | 1991 | Failure to Start | Partial           | While performing preventive maintenance calibration check on the protective relays for a residual heat removal pump motor 4kv breaker, it was found that all overcurrent relays for two pumps were out of calibration   |
| 206  | Operational/ Human Error | Maintenance     | Driver  | Maintenance      | Breaker    | RHR-B  | 1990 | Failure to Start | Partial           | RHR pump breaker overcurrent trips out of calibration.  |
| 207  | Operational/ Human Error | Maintenance     | Driver  | Test             | Motor      | ESW    | 1994 | Failure to Run   | Partial           | Leak test of the containment cooling service water pump vault watertight door revealed excessive leakage. Flooding and leakage past this door would make inoperable two of four containment cooling service water pumps. Procedural inadequacy was cited as the cause for the degraded door seals.  |
| 208  | Operational/ Human Error | Maintenance     | Driver  | Test             | I&C        | ESW    | 1989 | Failure to Start | Partial           | Emergency equipment service water pump relays were not reset following a load shedding test 30 hours before.  |
| 209  | Operational/ Human Error | Maintenance     | Pump    | Demand           | Casing     | AFW    | 1983 | Failure to Run   | Partial           | During testing, the outboard bearing temperature was high on the turbine-driven AFW pump, due to improper balance drum clearances, caused by improper maintenance. The procedure will be modified and the balance drum clearance reset. While the unit was starting up, the motor-driven AFW pump outboard bearing temperature was high. Excessive thrust bearing clearance caused the balance drum to unbalance, causing the thrust bearing to overheat. |

| Item | Proximate Cause          | Coupling Factor | Segment   | Discovery Method | Piece Part    | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------------------------|-----------------|-----------|------------------|---------------|--------|------|------------------|-------------------|---|
| 210  | Operational/ Human Error | Maintenance     | Pump      | Maintenance      | Lubrication   | HPI    | 1991 | Failure to Run   | Partial           | Following an overhaul of the HPI pumps. Too much oil flow led to excessive oil leakage, which would have failed HPI pumps before end of mission.  |
| 211  | Operational/ Human Error | Maintenance     | Pump      | Test             | Packing/Seals | AFW    | 1996 | Failure to Run   | Partial           | During the performance of Steam-Driven Emergency Feedwater Pump testing, sparks were observed emanating from the outboard mechanical seal area. The sparks appeared to be due to a mechanical interference within the mechanical seal assembly. The pump mechanical seal was disassembled and determined to have been improperly installed during the last refueling outage. The evaluation identified a mechanical seal design deficiency and inadequate corrective action for a previously identified event as the primary causes for this event. A contributing cause for this event was found to be inadequate predictive maintenance techniques. The electric AFW pump exhibited the same problem. |
| 212  | Operational/ Human Error | Maintenance     | Pump      | Test             | Casing        | RHR-P  | 1989 | Failure to Start | Complete          | Both loops of the residual heat removal system were declared inoperable due to gas binding of both RHR pumps. The gas binding was caused by entry of nitrogen gas into the reactor coolant system from accumulator. The root cause of this event has been attributed to personnel error. Personnel did not comply with the specific requirements in the accumulator discharge check valve full flow test procedure due to inattention to detail.  |
| 213  | Operational/ Human Error | Maintenance     | Suction   | Demand           | Piping        | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.   |
| 214  | Operational/ Human Error | Maintenance     | Suction   | Demand           | Piping        | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.   |
| 215  | Operational/ Human Error | Maintenance     | Suction   | Inspection       | Valve         | SLC    | 1991 | Failure to Start | Partial           | SLC pumps were potentially inoperable during part of test due to valve lineup.  |
| 216  | Operational/ Human Error | Maintenance     | Suction   | Maintenance      | Piping        | RHR-P  | 1982 | Failure to Run   | Complete          | Shutdown cooling was lost due to nitrogen intrusion because of backflushing a filter in the purification system.  |
| 217  | Operational/ Human Error | Maintenance     | Suction   | Maintenance      | Strainer      | HPI    | 1985 | Failure to Run   | Partial           | Strainers found still installed in the suction piping of the high-pressure injection pumps was a condition not considered in the operating design. The strainers were found during maintenance to repair a slight flange leak. The strainers had been placed in the suction piping during construction and were to be in place during system flushing to prevent any debris from reaching the pumps. However, the strainers should have been removed after system flushing prior to functional testing  |
| 218  | Operational/ Human Error | Operational     | Discharge | Inspection       | Valve         | AFW    | 1994 | Failure to Start | Complete          | Following a trip, the AFW Pumps were secured and the discharge flow control valves for the Motor Driven Pumps were closed. Later, an operator discovered during a routine Control Board walkdown that the valves were closed. Subsequent investigation revealed the AFW system had not been placed in standby readiness per the operating procedure after the system was secured.   |
| 219  | Operational/ Human Error | Operational     | Discharge | Inspection       | Valve         | HPI    | 1987 | Failure to Start | Almost Complete   | While attempting to fill the safety injection accumulators, it was discovered that two of three SI pumps had been isolated from the high head injection flowpath.   |
| 220  | Operational/ Human Error | Operational     | Discharge | Inspection       | Valve         | HPI    | 1993 | Failure to Run   | Partial           | One AFW pump failed due to incorrect procedure which allowed pump to be run without flow, other AFW pump was allowed to run past max flow rate. It is unclear whether these mistakes were due to inadequate procedures or staff errors, but it was assumed to be a failure to follow procedure.   |

| Item | Proximate Cause          | Coupling Factor | Segment | Discovery Method | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------------------------|-----------------|---------|------------------|-------------|--------|------|------------------|-------------------|---|
| 221  | Operational/ Human Error | Operational     | Driver  | Demand           | I&C         | ESW    | 1981 | Failure to Start | Partial           | Alarm circuit breaker was de-energized resulting in a loss of two RHR service water pumps.  |
| 222  | Operational/ Human Error | Operational     | Driver  | Demand           | I&C         | AFW    | 1983 | Failure to Start | Complete          | An operator incorrectly secured the diesel and steam driven AFW pumps, which prevented their restart on low SG level.   |
| 223  | Operational/ Human Error | Operational     | Driver  | Inspection       | I&C         | HPI    | 1990 | Failure to Start | Partial           | Both safety injection pumps were in the pull-to-lock position. With the switches in pull-to-lock, the pumps would not have automatically started upon receipt of an initiating signal. This event was caused by cognitive personnel error by a utility licensed operator in failure to follow an approved procedure.  |
| 224  | Operational/ Human Error | Operational     | Driver  | Inspection       | Breaker     | HPI    | 1989 | Failure to Start | Partial           | HPI Pump B not retested, then HPI Pump A removed from service.  |
| 225  | Operational/ Human Error | Operational     | Driver  | Inspection       | Breaker     | HPI    | 1990 | Failure to Start | Complete          | By opening incorrect breaker, HPI pump tripped while others were unavailable.   |
| 226  | Operational/ Human Error | Operational     | Driver  | Inspection       | I&C         | HPI    | 1992 | Failure to Start | Almost Complete   | Two charging pumps and one charging pump service water pump were removed from service simultaneously which is a condition not allowed by technical specifications.  |
| 227  | Operational/ Human Error | Operational     | Driver  | Inspection       | Breaker     | HPI    | 1988 | Failure to Start | Complete          | HPI pumps not restored before mode change due to procedural inadequacy.   |
| 228  | Operational/ Human Error | Operational     | Driver  | Inspection       | Breaker     | ESW    | 1981 | Failure to Start | Almost Complete   | Control breakers for two ESW pumps were open due to inadvertent operator action.  |
| 229  | Operational/ Human Error | Operational     | Driver  | Inspection       | I&C         | HPI    | 1988 | Failure to Start | Complete          | With alternate CCP pump out-of-service, the remaining operable pump was erroneously placed in pull-to-lock.   |
| 230  | Operational/ Human Error | Operational     | Driver  | Inspection       | Breaker     | CSS    | 1991 | Failure to Start | Complete          | CSR control power de-energized prior to mode change. Technical Specification violation. Inadequate procedure review.  |
| 231  | Operational/ Human Error | Operational     | Driver  | Inspection       | Breaker     | HPI    | 1982 | Failure to Start | Complete          | During the draining of the reactor coolant system, both centrifugal charging pumps were rendered inoperable. The initial conditions in the draining procedure contained a confusing statement, which led to an erroneous assumption that both CCP breakers had to be racked out and tagged.   |
| 232  | Operational/ Human Error | Operational     | Driver  | Inspection       | I&C         | RHR-P  | 1995 | Failure to Start | Complete          | The switches for the containment spray and recirculation pumps were in a trip pullout when the Technical Specifications and plant procedures required the pumps to be operable.   |
| 233  | Operational/ Human Error | Operational     | Driver  | Test             | I&C         | ESW    | 1990 | Failure to Start | Complete          | An emergency service water pump failed to start and was declared inoperable. Further investigation determined that the failure of the pump to start was due to a tripped emergency engine shutdown device. Operations personnel performing the testing did not recognize the need to reset it prior to starting the pump. Examination of the other two ESW pumps revealed that their emergency shutdown devices were also in the tripped condition. |
| 234  | Operational/ Human Error | Operational     | Pump    | Inspection       | Lubrication | HPI    | 1983 | Failure to Start | Complete          | A routine preventive maintenance (oil change) was mistakenly performed on the north charging pump instead of the south as scheduled. Since the south pump was previously cleared for this oil change, and the test pump was valved out, none of these three pumps were in service as required by tech specs for the approximately 20 minutes it took to change the oil in the north pump.   |
| 235  | Operational/ Human Error | Operational     | Pump    | Maintenance      | Lubrication | ESW    | 1993 | Failure to Run   | Partial           | Low pressure RHR bearing oil level not maintained high enough when new smaller sightglass installed. Second event the sightglass was broken when adding oil.  |
| 236  | Operational/ Human Error | Operational     | Suction | Demand           | Piping      | ESW    | 1986 | Failure to Run   | Complete          | Failure to properly vent and fill a newly installed pipe introduced air into the charging pump service water system.  |



| Item | Proximate Cause          | Coupling Factor | Segment | Discovery Method | Piece Part   | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------------------------|-----------------|---------|------------------|--------------|--------|------|------------------|-------------------|---|
| 237  | Operational/ Human Error | Operational     | Suction | Demand           | Piping       | RHR-P  | 1984 | Failure to Run   | Complete          | The control room operators started a second residual heat removal pump in preparation for removing the operating RHR pump from service. With both pumps running, flow became excessive for the half-loop condition causing cavitation and air binding of both pumps. To prevent recurrence the procedure which controls the operation of the RHR pumps has been changed to include specific instructions to stop the operating pump prior to starting the second pump while at half-loop. |
| 238  | Operational/ Human Error | Operational     | Suction | Demand           | Booster Pump | ESW    | 1980 | Failure to Start | Partial           | The service water RHR booster pump was de-energized during maintenance. The attempt to start service water pumps failed due to low suction pressure.  |
| 239  | Operational/ Human Error | Operational     | Suction | Demand           | Piping       | RHR-P  | 1980 | Failure to Run   | Complete          | While attempting to increase RHR flow, the plant experienced a total loss of flow due to the pumps being air-bound. The pump was not vented when starting to increase flow. Operating procedures have been changed to have an operator present while changing flow in the RHR system. There have been losses of RHR flow in the past because the pumps were air-bound and methods are being investigated to improve the system design.  |
| 240  | Operational/ Human Error | Operational     | Suction | Demand           | Piping       | ESW    | 1988 | Failure to Run   | Complete          | The procedure failed to adequately caution the operator to slowly fill a drained line. Rapid filling resulted in a loss of NPSH to the charging service water pumps.  |
| 241  | Operational/ Human Error | Operational     | Suction | Maintenance      | Strainer     | ESW    | 1986 | Failure to Run   | Complete          | A service water strainer was placed in service without being vented resulting in air binding system and loss of charging pump service water pumps.  |
| 242  | Operational/ Human Error | Operational     | Suction | Test             | Piping       | ESW    | 1989 | Failure to Run   | Partial           | Inadequate procedure led to air binding of operating ESW pumps.   |
| 243  | Operational/ Human Error | Quality         | Driver  | Inspection       | Breaker      | ESW    | 1992 | Failure to Start | Partial           | The fit between an ESW pump breaker primary disconnects and the associated breaker cubicle stabs was inadequate. The poor fit between the disconnects and the stabs led to arcing in the breaker cubicle when the pump was started, resulting in a fire. Shortly after identifying the cause of the fire, the remaining ESW breakers, which had recently been replaced along with the failed breaker, as part of a design modification package, were found to be inadequate also.         |
| 244  | Operational/ Human Error | Quality         | Driver  | Test             | I&C          | ESW    | 1982 | Failure to Start | Partial           | Two ESW pumps failed to start. One ESW pump failed to function as a result of loose wires on relay terminals in both pump logic schemes, a loose states link and an instantaneous contact found out of adjustment on the other pump logic scheme.   |
| 245  | Other                    | Design          | Driver  | Demand           | I&C          | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure.  |
| 246  | Other                    | Design          | Driver  | Demand           | Piping       | HCI    | 1999 | Failure to Start | Complete          | Water entered the HCI and RCI steam supply lines, rendering both pumps inoperable. Failed reactor vessel instrumentation allowed water to overflow and fill the HCI/RCI steam lines. Pumps were unavailable.  |
| 247  | Other                    | Design          | Driver  | Demand           | I&C          | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure. This is a second event two months later.   |
| 248  | Other                    | Design          | Driver  | Inspection       | I&C          | AFW    | 1983 | Failure to Start | Partial           | Both AFW pumps had to be rendered inoperable to allow repairs to actuation circuitry.   |

| Item | Proximate Cause | Coupling Factor | Segment | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|-----------------|---------|------------------|------------|--------|------|------------------|-------------------|--|
| 249  | Other           | Design          | Driver  | Test             | I&C        | RHR-B  | 1982 | Failure to Start | Partial           | A functional test revealed a sliding link in control room panel open. Further investigation revealed a total of four links open. These links, left open, negated all autostart capability of 2 of 4 RHR pumps. It could not be determined why these four links were open.  |
| 250  | Other           | Design          | Driver  | Test             | I&C        | ESW    | 1992 | Failure to Start | Partial           | Valve position contacts prevented ESW pump circuit breakers from closing. Poor design resulted in water intrusion in the valve limit switch box.   |
| 251  | Other           | Design          | Driver  | Test             | Breaker    | SLC    | 1986 | Failure to Start | Complete          | During a test, both Squib Valve Detonators shorted after firing and the Control Power Transformer fuse blew causing the pump motor trip. This was caused by improper fuse coordination between the Control Power Transformer fuse and the Squib Valve Detonator fuses. The redundant system's Squib Valve was also fired during this test, without running the associated pump, and one of the Squib Valve Detonators shorted after firing. The same fuse coordination problem existed for both systems.   |
| 252  | Other           | Design          | Suction | Demand           | Valve      | RHR-P  | 1984 | Failure to Run   | Complete          | Both RHR pumps were unable to operate due to the introduction of air into the RHR system. The incident occurred during the drain down of the RCS, when the level of the RCS was being monitored via a standpipe off the centerline of one of the RCS loops. The isolation valve to which the standpipe was attached became clogged sometime during the drain down and falsely indicated above centerline when in fact the level was below the RHR suction line (below centerline).   |
| 253  | Other           | Design          | Suction | Demand           | Piping     | ESW    | 1980 | Failure to Run   | Almost Complete   | Air ingress exceeded the air removal capability of the constant vent valves. A design change was implemented to remove the air compressor cooling from the service water system.   |
| 254  | Other           | Design          | Suction | Demand           | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | With unit drained to centerline of the nozzles, suction to both RHR pumps was lost for 36 minutes. Suction to the RHR pumps was lost because of ambiguous reactor coolant system level indication while drained to centerline of the nozzles. The actual RCS level was lower than observed.  |
| 255  | Other           | Design          | Suction | Demand           | Piping     | RHR-P  | 1987 | Failure to Run   | Complete          | RHR flow was interrupted when both RHR trains became inoperable due to air bound RHR pumps. The loss of RCS inventory to the reactor coolant drain tank due to a leaking valve caused a decrease in RCS water level, vortexing in the pumps' suction line, and air entrainment in the RHR pumps.   |
| 256  | Other           | Design          | Suction | Demand           | Piping     | HPI    | 1982 | Failure to Start | Complete          | Hydrogen from the suction dampener got into suction piping and failed both CCPs.   |
| 257  | Other           | Design          | Suction | Demand           | I&C        | HPI    | 1997 | Failure to Run   | Partial           | Letdown storage tank reference leg not full, which gave erroneous indication of sufficient tank level. One HPI pump severely damaged, other pump not as damaged, and could have run. The root cause was a combination of a design weakness of a common reference leg for the Letdown storage tank level instruments and a leaking instrument fitting due to an inadequate work practice.   |
| 258  | Other           | Design          | Suction | Demand           | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | RHR Suction lost due to erroneous RCS level while draining the RCS.  |
| 259  | Other           | Design          | Suction | Test             | I&C        | AFW    | 1985 | Failure to Run   | Almost Complete   | Testing of the turbine driven AFW pump resulted in a low suction trip of the motor driven pump. The turbine driven pump had a faulty governor. It was during the post maintenance test of turbine driven pump that speed oscillations occurred causing pressure oscillations in the suction of the motor driven pump that was in service. Foreign material in the suction gauge protectors resulted in the pressure sensors sensing only the low pressures and not the high pressures of the oscillations, so the motor driven pump tripped on low pressure. |

| Item | Proximate Cause | Coupling Factor | Segment   | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|-----------------|-----------|------------------|------------|--------|------|------------------|-------------------|--|
| 260  | Other           | Environmental   | Driver    | Inspection       | Motor      | ESW    | 1981 | Failure to Run   | Partial           | The float guide failed in a RHRSW pump air valve and caused the valve to fail open and flood pump room.  |
| 261  | Other           | Environmental   | Driver    | Inspection       | Motor      | AFW    | 1990 | Failure to Start | Partial           | Both motor driven AFW pumps were sprayed when a service water pipe developed a through wall leak.  |
| 262  | Other           | Maintenance     | Discharge | Demand           | Valve      | ESW    | 1980 | Failure to Start | Partial           | RHR service water pumps were started to put torus cooling in service. When these pumps would not deliver required discharge pressure, they were declared inoperable. The seal in an air release valve was bad, allowing a vent on the discharge line.  |
| 263  | Other           | Maintenance     | Driver    | Demand           | Breaker    | RHR-P  | 1987 | Failure to Start | Complete          | Two LPI pumps, when given a start signal, would not start. An ongoing investigation revealed the probable root cause of the event to be poor electrical contact of the breaker auxiliary stabs for the pumps.  |
| 264  | Other           | Maintenance     | Driver    | Demand           | I&C        | ESW    | 1982 | Failure to Start | Complete          | Following a reactor scram, an attempt to initiate suppression pool cooling revealed that both RHRSW loops were inoperable as neither loop's pumps could be started. Low suction header pressure lockout signals in each loop prevented starting each loop's pumps. Plugging of the sensing line to each loop's suction header pressure switch prevented both switches from sensing actual pressure, although a lack of operating fluid in one switch and an open power supply breaker to the other switch also would have prevented pumps from starting. |
| 265  | Other           | Maintenance     | Driver    | Maintenance      | Breaker    | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker failures, broken screw, no lubrication, and a bent track  |
| 266  | Other           | Maintenance     | Driver    | Maintenance      | Breaker    | ESW    | 1982 | Failure to Start | Partial           | ESW pump circuit breakers found damaged. Defective arc chute and cracked secondary coupler.  |
| 267  | Other           | Maintenance     | Driver    | Test             | Breaker    | ESW    | 1984 | Failure to Start | Partial           | ESW pump breakers tripped due to failed voltage control devices.   |
| 268  | Other           | Maintenance     | Driver    | Test             | Breaker    | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker overcurrent trip devices tripping too low.  |
| 269  | Other           | Maintenance     | Suction   | Demand           | Piping     | RHR-P  | 1986 | Failure to Run   | Complete          | SDC pumps cavitated due to lowering RCS level. Level indication was in error.  |
| 270  | Other           | Maintenance     | Suction   | Demand           | Piping     | RHR-P  | 1981 | Failure to Run   | Complete          | Temporary coolant loop level indicator showed level slowly increasing over a period of days. The system was periodically drained to maintain 65 percent indicated level. A RHR pump lost suction on reduction of actual level. The second pump was started, and lost suction. Indication drift was due to evaporation of reference leg.  |
| 271  | Other           | Maintenance     | Suction   | Demand           | Piping     | RHR-P  | 1980 | Failure to Run   | Complete          | A complete loss of RHR flow occurred while plant operators were increasing RHR heat exchanger flow by closing down on the heat exchanger bypass valve.   |
| 272  | Other           | Maintenance     | Suction   | Demand           | Piping     | RHR-P  | 1983 | Failure to Run   | Complete          | The RHR pumps began to cavitate and eventually both pumps were stopped. The reactor vessel level gauge being used to provide an indication that the level was approaching the vessel flange level had been isolated (reactor coolant drain tank isolation valve had been closed during an attempt to reduce leakage). Additionally, procedures did not require visual monitoring of cavity level.  |
| 273  | Other           | Operational     | Pump      | Inspection       | Bearing    | ESW    | 1991 | Failure to Run   | Partial           | Lube oil cooling water isolated during a test. Pumps continued to run with no cooling.   |
| 274  | Unknown         | Design          | Suction   | Demand           | Piping     | RHR-P  | 1983 | Failure to Run   | Complete          | RHR pumps cavitated. Unable to repeat. Unknown cause.  |

Table A-2. Pump CCF event summary, sorted by coupling factor.

| Item | Coupling Factor | Proximate Cause  | Segment   | Discovery Method | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|--|-----------|------------------|-------------|--------|------|------------------|-------------------|--|
| 1    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Discharge | Demand           | Valve       | AFW    | 1985 | Failure to Start | Partial           | Controller problems in the steam and diesel driven AFW pumps caused the pumps to trip on low suction pressure. The pump discharge flow controller valves were also not set properly after last maintenance. Low suction trips were due to design error.  |
| 2    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Discharge | Demand           | Valve       | AFW    | 1986 | Failure to Start | Partial           | Both the turbine driven and motor driven AFW pumps could not produce full flow because the cages in their discharge valve trapped debris and plugged.  |
| 3    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Demand           | I&C         | AFW    | 1981 | Failure to Start | Almost Complete   | Two AFW pumps failed to automatically start due to low suction pressure trips. A modification was installed to prevent this. This effect was discovered previously, but apparently had not been corrected prior to an attempt to start the pumps three weeks later.  |
| 4    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Demand           | Lubrication | RHR-P  | 2000 | Failure to Run   | Complete          | Both RHR/LPI pumps fail to run due to improper oil in system. High bearing temperatures occurred when the pumps were operated. This was due to the wrong lube oil being used, which had too high a viscosity. Inadequate vender design information resulted in the higher viscosity oil being used and additional exacerbating problems such as insufficient bearing clearances. |
| 5    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Demand           | I&C         | AFW    | 1997 | Failure to Run   | Partial           | One actual AFW pump failure due to spurious electronic overspeed trip. Determined that all three pumps were susceptible to spurious overspeed trips.   |
| 6    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Demand           | I&C         | AFW    | 1981 | Failure to Start | Almost Complete   | A modification to the control instrumentation for two AFW pumps resulted in a backfeed situation such that when called upon to start, both pumps would not start.  |
| 7    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Inspection       | Lubrication | HPI    | 2000 | Failure to Run   | Partial           | CVC makeup oil pump motor too small for certain accidents.   |
| 8    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Inspection       | I&C         | AFW    | 1994 | Failure to Start | Partial           | Single failure would prevent auto initiation of AFW. Circuit design did not provide separation required by standards and code. The single failure identified was a short circuit across two conductors of the actuation relays associated with the initiation logic matrix.  |

| Item | Coupling Factor | Proximate Cause  | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 9    | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Inspection       | Supports            | RHR-B  | 1986 | Failure to Start | Partial           | RHR motor internal supports were cracked due to stress and vibration. Design improvements were made.  |
| 10   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Maintenance      | I&C                 | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |
| 11   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Maintenance      | I&C                 | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |
| 12   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Test             | I&C                 | AFW    | 1981 | Failure to Start | Almost Complete   | Two low suction pressure trips for the AFW pumps were mis-calibrated, which prevented the pumps from starting.  |
| 13   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Test             | I&C                 | AFW    | 1992 | Failure to Start | Complete          | A modification design error (in 1983-1984) removed a start permissive interlock contact. At cold shutdown this de-energized the auxiliary lube oil pump, consequently, when one AFW pump was started it ran for 2.5 seconds and tripped on low oil pressure. Further investigation showed that both units AFW pumps would be affected in the same way. The design error combined with insufficient post modification testing led to this CCF event.   |
| 14   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Test             | Breaker             | HPI    | 1980 | Failure to Start | Partial           | Upon testing the safety injection pumps it was found that the 6900-v breakers would lock-out preventing pump start if they were given a close signal for >0.32 seconds when a trip condition existed. There is no indication to operations when this locked-out condition exists. The breaker appears to be available for service when it actually is not. The only means of clearing the condition is to remove and reinstall the fuses at the breaker or manually change the state of the relays.                             |
| 15   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1981 | Failure to Run   | Complete          | Both charging pump service water pumps failed. A carbon cap screw failed allowing the impeller of one pump to bind on the casing. The ensuing leakage shorted the motor windings of the other pump.   |

| Item | Coupling Factor | Proximate Cause  | Segment | Discovery Method | Piece Part             | System | Year | Failure Mode        | Degree of Failure  | Description   |
|------|-----------------|--|---------|------------------|------------------------|--------|------|---------------------|--------------------|---|
| 16   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Demand           | Impeller/Wear<br>Rings | ESW    | 1996 | Failure<br>to Run   | Partial            | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 17   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Demand           | Impeller/Wear<br>Rings | ESW    | 1986 | Failure<br>to Run   | Partial            | All four emergency service water pumps showed cavitation damage. Two of the pumps had minor damage and were placed back in service. Recirculation cavitation occurs at flows significantly less than design.  |
| 18   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Test             | Shaft                  | AFW    | 1988 | Failure<br>to Run   | Almost<br>Complete | An auxiliary feedwater pump failed its performance test. Subsequent inspection of the pump internals revealed significant damage, including a split in the center shaft sleeve. The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 19   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Test             | Coupling               | ESW    | 1994 | Failure<br>to Start | Partial            | Pump produced no flow when started. A shaft coupling failed. Material was determined to be brittle and have low impact properties. The coupling was replaced on all pumps with a type of material more suitable for this application.   |
| 20   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Test             | Shaft                  | AFW    | 1988 | Failure<br>to Run   | Partial            | The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 21   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping                 | ESW    | 1981 | Failure<br>to Run   | Complete           | Increasing flow to chillers robs NPSH from charging service water pumps.  |
| 22   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping                 | ESW    | 1983 | Failure<br>to Run   | Complete           | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 23   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping                 | ESW    | 1982 | Failure<br>to Run   | Complete           | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |

| Item | Coupling Factor | Proximate Cause  | Segment | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--|---------|------------------|------------|--------|------|------------------|-------------------|---|
| 24   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping     | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |
| 25   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping     | ESW    | 1981 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 26   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping     | ESW    | 1982 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 27   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping     | ESW    | 1983 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Pump Service Water pumps.   |
| 28   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping     | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the Charging Water Service Water pumps.  |
| 29   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping     | ESW    | 1996 | Failure to Start | Partial           | Freezing of diesel generator service water piping in intake bay. Inadequate initial design.   |
| 30   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping     | ESW    | 1982 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 31   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Inspection       | Piping     | HPI    | 1991 | Failure to Start | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the alternate boration line and the gravity feed line from the boric acid storage tank. |
| 32   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Inspection       | Piping     | HPI    | 1988 | Failure to Run   | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the suction piping.   |

| Item | Coupling Factor | Proximate Cause  | Segment | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--|---------|------------------|------------|--------|------|------------------|-------------------|---|
| 33   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Inspection       | Piping     | HPI    | 1990 | Failure to Start | Partial           | A quantity of gas was found in the centrifugal charging pump suction header that exceeded the maximum allowed gas volume. It was subsequently determined that hydrogen gas had been coming out of solution on both units and accumulating in the suction piping as a probable result of gas stripping by the CCP miniflow orifices. In addition, entrainment of hydrogen bubbles from the volume control tank to the CCP suction pipe may be a contributor as well. |
| 34   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Inspection       | Piping     | HPI    | 1988 | Failure to Start | Partial           | It was determined that various pipes of the safety injection system and chemical volume and control system collected or trapped gas which might affect the functions of these systems. There was a concern that the gas pockets may adversely effect pump operation. Voids were detected in some of the high head SI pump piping.   |
| 35   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Maintenance      | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 36   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Maintenance      | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 37   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Maintenance      | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 38   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Maintenance      | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 39   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Maintenance      | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 40   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Maintenance      | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |



| Item | Coupling Factor | Proximate Cause  | Segment   | Discovery Method | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|--|-----------|------------------|-------------|--------|------|------------------|-------------------|--|
| 41   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Test             | Tank        | SLC    | 1991 | Failure to Run   | Complete          | During the performance of a special test on the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.  |
| 42   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Test             | Tank        | SLC    | 1991 | Failure to Run   | Complete          | During the performance of a special test on Unit 1 to determine the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.  |
| 43   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Test             | Piping      | AFW    | 1999 | Failure to Run   | Partial           | All AFW trains declared inoperable due to inadequate suction flow capability from the nuclear service water alternate source. Inadequate flow caused by corroded piping. Piping is undersized so there is little margin for piping degradation. Since this is 1 of 4 suction sources, the safety significance is limited.  |
| 44   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Test             | Tank        | ESW    | 1986 | Failure to Run   | Complete          | Loss of prime in the condenser circulating water siphon flow system caused loss of low pressure service water pumps. Pumps lost suction during a test due to poor design.  |
| 45   | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Test             | Valve       | ESW    | 1983 | Failure to Start | Partial           | Low discharge pressure was caused by insufficient suction pressure. Service water flow to parallel components was adjusted.  |
| 46   | Design          | External Environment   | Discharge | Demand           | Check Valve | AFW    | 1983 | Failure to Start | Almost Complete   | Hot water in the AFW pump casings caused the pumps to become vapor bound. The hot water was from leaking check valves upstream of the pumps. This event occurred once on the turbine driven pump and 5 times on the motor driven pump.   |
| 47   | Design          | External Environment   | Discharge | Inspection       | Piping      | HPI    | 1994 | Failure to Run   | Partial           | Due to a leaking socket weld in the common recirculation line, all three SI pumps were declared inoperable. The underlying cause of the leak was a crack in the socket weld in the common recirculation line, caused by pipe displacement from air entrainment and pump misalignment.  |
| 48   | Design          | External Environment   | Pump      | Inspection       | Bearing     | HPI    | 1991 | Failure to Run   | Almost Complete   | Charging/safety pumps beyond operational limits. Damage was found to the thrust bearings. Air was introduced into this train of chilled water during modifications and testing being performed on the system. This air became trapped in high points of either, or both of, the supply and return chilled water lines to the charging pump. At the reduced flow rate, sufficient cooling was not available and oil temperature increased to the point where bearing damage occurred. |
| 49   | Design          | Internal to Component  | Driver    | Demand           | Breaker     | ESW    | 2000 | Failure to Start | Almost Complete   | Two ESW pumps failed to start due to their breakers failing to close. The breakers' prop spring bracket has slipped thus preventing proper interfacing between the prop and the prop pin.  |
| 50   | Design          | Internal to Component  | Driver    | Inspection       | I&C         | ESW    | 1982 | Failure to Start | Partial           | Open circuit breaker resulted in loss of two RHR service water pumps.  |

| Item | Coupling Factor | Proximate Cause          | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--------------------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 51   | Design          | Internal to Component    | Pump    | Inspection       | Lubrication         | HPI    | 1981 | Failure to Run   | Partial           | Corrosion of HPI pump cooler heads. Improper material led to corrosion  |
| 52   | Design          | Operational/ Human Error | Driver  | Demand           | I&C                 | ESW    | 1980 | Failure to Start | Partial           | Instrument isolation valve closed causing a low suction trip signal to two RHRSW pumps.   |
| 53   | Design          | Operational/ Human Error | Driver  | Inspection       | Breaker             | ESW    | 1984 | Failure to Start | Partial           | During an attempt to perform preventive maintenance for unit one's RHR service water pumps, plant personnel mistakenly disconnected the motor leads for unit two's RHR service water pump.  |
| 54   | Design          | Operational/ Human Error | Driver  | Test             | Breaker             | AFW    | 1985 | Failure to Start | Complete          | Both AFW pumps failed to start when tested, due to the circuit breakers not being racked in properly.   |
| 55   | Design          | Operational/ Human Error | Pump    | Demand           | Impeller/Wear Rings | AFW    | 1990 | Failure to Run   | Almost Complete   | Due to a combination of management error and procedural deficiency, the turbine driven auxiliary feedwater pump was run deadheaded. The operation damaged the pump. When the pump was manually tripped, steam vented back into the suction line, caused another AFW pump to also trip, on a low suction pressure signal.  |
| 56   | Design          | Operational/ Human Error | Suction | Demand           | Piping              | RHR-P  | 1982 | Failure to Run   | Complete          | Suction was lost to both RHR pumps. RHR flow was less than 3000 gpm and pump amps were fluctuating prior to taking corrective action. Each of these events appear to have been caused by a slow decrease in RCS level in conjunction with the vortex action at the pump suction.  |
| 57   | Design          | Operational/ Human Error | Suction | Demand           | Piping              | RHR-P  | 1984 | Failure to Run   | Almost Complete   | On two occasions, RHR pumps cavitated due to low RCS level while draining the RCS.  |
| 58   | Design          | Operational/ Human Error | Suction | Demand           | Piping              | RHR-P  | 1980 | Failure to Run   | Complete          | The reactor vessel vent eductor was in service in preparation for refueling with RHR operating. A low flow alarm was received and low flow and low motor current were indicated. A second pump was started and became air-bound. Putting the vessel vent eductor system into service was the root cause of the incident.  |
| 59   | Design          | Operational/ Human Error | Suction | Demand           | Tank                | AFW    | 1980 | Failure to Run   | Complete          | Both emergency feedwater pumps lost feed pump suction. The emergency feedwater pump suction flashed to steam due to the feedwater train flashing and forcing hot water back through the startup and blowdown tanks and into the feedwater pump suction. To prevent this recurrence, the operating procedures have been changed to require isolating the startup and blowdown effluent as a source of emergency feedwater suction prior to increasing power. |
| 60   | Design          | Operational/ Human Error | Suction | Demand           | Piping              | RHR-P  | 1985 | Failure to Run   | Complete          | Swap over of RHR pumps resulted in both trains becoming inoperable due to air injection into the suction of the pumps. This required both pumps to be vented and required RCS level to be raised to prevent a possible recurrence of the vortex problem.  |
| 61   | Design          | Other                    | Driver  | Demand           | Piping              | HCI    | 1999 | Failure to Start | Complete          | Water entered the HCI and RCI steam supply lines, rendering both pumps inoperable. Failed reactor vessel instrumentation allowed water to overflow and fill the HCI/RCI steam lines. Pumps were unavailable.  |
| 62   | Design          | Other                    | Driver  | Demand           | I&C                 | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure. This is a second event two months later.                   |

| Item | Coupling Factor | Proximate Cause | Segment | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|-----------------|---------|------------------|------------|--------|------|------------------|-------------------|--|
| 63   | Design          | Other           | Driver  | Demand           | I&C        | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure.   |
| 64   | Design          | Other           | Driver  | Inspection       | I&C        | AFW    | 1983 | Failure to Start | Partial           | Both AFW pumps had to be rendered inoperable to allow repairs to actuation circuitry.  |
| 65   | Design          | Other           | Driver  | Test             | Breaker    | SLC    | 1986 | Failure to Start | Complete          | During a test, both Squib Valve Detonators shorted after firing and the Control Power Transformer fuse blew causing the pump motor trip. This was caused by improper fuse coordination between the Control Power Transformer fuse and the Squib Valve Detonator fuses. The redundant system's Squib Valve was also fired during this test, without running the associated pump, and one of the Squib Valve Detonators shorted after firing. The same fuse coordination problem existed for both systems. |
| 66   | Design          | Other           | Driver  | Test             | I&C        | ESW    | 1992 | Failure to Start | Partial           | Valve position contacts prevented ESW pump circuit breakers from closing. Poor design resulted in water intrusion in the valve limit switch box.   |
| 67   | Design          | Other           | Driver  | Test             | I&C        | RHR-B  | 1982 | Failure to Start | Partial           | A functional test revealed a sliding link in control room panel open. Further investigation revealed a total of four links open. These links, left open, negated all autostart capability of 2 of 4 RHR pumps. It could not be determined why these four links were open.  |
| 68   | Design          | Other           | Suction | Demand           | Piping     | HPI    | 1982 | Failure to Start | Complete          | Hydrogen from the suction dampener got into suction piping and failed both CCPs.   |
| 69   | Design          | Other           | Suction | Demand           | I&C        | HPI    | 1997 | Failure to Run   | Partial           | Letdown storage tank reference leg not full, which gave erroneous indication of sufficient tank level. One HPI pump severely damaged, other pump not as damaged, and could have run. The root cause was a combination of a design weakness of a common reference leg for the Letdown storage tank level instruments and a leaking instrument fitting due to an inadequate work practice.   |
| 70   | Design          | Other           | Suction | Demand           | Piping     | ESW    | 1980 | Failure to Run   | Almost Complete   | Air ingress exceeded the air removal capability of the constant vent valves. A design change was implemented to remove the air compressor cooling from the service water system.   |
| 71   | Design          | Other           | Suction | Demand           | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | With unit drained to centerline of the nozzles, suction to both RHR pumps was lost for 36 minutes. Suction to the RHR pumps was lost because of ambiguous reactor coolant system level indication while drained to centerline of the nozzles. The actual RCS level was lower than observed.  |
| 72   | Design          | Other           | Suction | Demand           | Valve      | RHR-P  | 1984 | Failure to Run   | Complete          | Both RHR pumps were unable to operate due to the introduction of air into the RHR system. The incident occurred during the drain down of the RCS, when the level of the RCS was being monitored via a standpipe off the centerline of one of the RCS loops. The isolation valve to which the standpipe was attached became clogged sometime during the drain down and falsely indicated above centerline when in fact the level was below the RHR suction line (below centerline).                       |
| 73   | Design          | Other           | Suction | Demand           | Piping     | RHR-P  | 1987 | Failure to Run   | Complete          | RHR flow was interrupted when both RHR trains became inoperable due to air bound RHR pumps. The loss of RCS inventory to the reactor coolant drain tank due to a leaking valve caused a decrease in RCS water level, vortexing in the pumps' suction line, and air entrainment in the RHR pumps.   |
| 74   | Design          | Other           | Suction | Demand           | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | RHR Suction lost due to erroneous RCS level while draining the RCS.  |

| Item | Coupling Factor | Proximate Cause  | Segment   | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|--|-----------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 75   | Design          | Other  | Suction   | Test             | I&C                 | AFW    | 1985 | Failure to Run   | Almost Complete   | Testing of the turbine driven AFW pump resulted in a low suction trip of the motor driven pump. The turbine driven pump had a faulty governor. It was during the post maintenance test of turbine driven pump that speed oscillations occurred causing pressure oscillations in the suction of the motor driven pump that was in service. Foreign material in the suction gauge protectors resulted in the pressure sensors sensing only the low pressures and not the high pressures of the oscillations, so the motor driven pump tripped on low pressure. |
| 76   | Design          | Unknown  | Suction   | Demand           | Piping              | RHR-P  | 1983 | Failure to Run   | Complete          | RHR pumps cavitated. Unable to repeat. Unknown cause.  |
| 77   | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Inspection       | Piping              | HPI    | 2000 | Failure to Run   | Partial           | Microbiologically induced corrosion leak on service water lines to two charging/HPI pump lube oil coolers.   |
| 78   | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Demand           | Impeller/Wear Rings | ESW    | 2000 | Failure to Start | Almost Complete   | Two of the River Water pumps tripped on overcurrent when they were attempted to be started. The trips were a result of physical contact between the impeller and the lower casing liner of the pumps. This condition was due to differential thermal expansion between the pump shaft and the pump casing as a result of an elevated seal injection water temperature. The elevated temperature was due to an abnormal configuration of the Filtered Water System (the backup seal water supply).  |
| 79   | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Inspection       | Lubrication         | HPI    | 1995 | Failure to Run   | Partial           | High lube oil temperatures were observed during HPI pump operation. Zinc particles from anode were discovered plugging the lube oil coolers. Accelerated corrosion was attributed to a corrosion inhibitor that was added to the system, which chemically interacted with the zinc.  |
| 80   | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Test             | Coupling            | ESW    | 1987 | Failure to Start | Partial           | Test showed two ESW pumps failed. Pump shafts were corroded and found to be made of incorrect material.  |
| 81   | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Inspection       | Strainer            | ESW    | 2000 | Failure to Run   | Partial           | RHRWS Pumps Failed to Develop flow/pressure. Debris in intake structure. Requires modifications to the traveling Water Screen.   |
| 82   | Environmental   | External Environment   | Discharge | Test             | Recirc              | HPI    | 1992 | Failure to Run   | Almost Complete   | Safety Injection pumps were declared inoperable due to an observed declining trend in the pump's recirculation flow. The cause of the Safety Injection pump reduced recirculation flow is attributed to foreign material blockage within the associated minimum flow recirculation line flow orifice.  |
| 83   | Environmental   | External Environment   | Driver    | Demand           | Motor               | ESW    | 1985 | Failure to Run   | Partial           | Two service water motors failed on demand as a result of cement dust contamination.  |
| 84   | Environmental   | External Environment   | Driver    | Demand           | I&C                 | AFW    | 1984 | Failure to Start | Complete          | Both AFW pumps failed to start. The problem was traced to two relays (1 per pump). Examination of the relays revealed open circuiting and severe degradation of the insulation.  |

| Item | Coupling Factor | Proximate Cause       | Segment   | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|-----------------------|-----------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 85   | Environmental   | External Environment  | Driver    | Maintenance      | Motor               | ESW    | 1987 | Failure to Start | Partial           | During an extended service water bay flooding incident, one ESW pump was found grounded by testing, later two more pumps were found to be failed also.  |
| 86   | Environmental   | External Environment  | Driver    | Test             | Bearing             | RHR-B  | 1991 | Failure to Run   | Partial           | Two LCI pumps were declared inoperable due to high motor vibration.   |
| 87   | Environmental   | External Environment  | Pump      | Inspection       | Coupling            | ESW    | 1993 | Failure to Run   | Partial           | Entrained debris caused ESW pump shaft coupling to fail. Plant equipment did not prevent this debris from entering pump.  |
| 88   | Environmental   | External Environment  | Pump      | Inspection       | Packing/Seals       | RHR-P  | 1985 | Failure to Start | Complete          | Following a trip, water was found spraying from both low head safety injection pump wedge control rod seals. Both pumps were declared inoperable. Postulated failure on the seals was from a minor flow induced pressure transient.   |
| 89   | Environmental   | External Environment  | Suction   | Demand           | Piping              | HPI    | 1984 | Failure to Start | Complete          | Boron solidification in the suction and gas binding of pumps led to the failure of all three safety injection pumps. Flushing procedures inadequate.  |
| 90   | Environmental   | Internal to Component | Discharge | Demand           | Valve               | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitrol cages for these valves were clogged with shredded Asiatic clam shells. |
| 91   | Environmental   | Internal to Component | Discharge | Demand           | Valve               | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitrol cages for these valves were clogged with shredded Asiatic clam shells. |
| 92   | Environmental   | Internal to Component | Discharge | Test             | Recirc              | HPI    | 1991 | Failure to Run   | Partial           | Something in HPI pump recirculation line was restricting flow. The piece later dislodged and no identification was made. Both SI pumps had inadequate recirculation flow.   |
| 93   | Environmental   | Internal to Component | Pump      | Demand           | Impeller/Wear Rings | ESW    | 1994 | Failure to Run   | Partial           | Raw water pump currents stayed high after starting. The primary cause of these events was determined to be elevated sand content in the river, resulting in excessive sand accumulation around the suction area of the pumps.   |
| 94   | Environmental   | Internal to Component | Pump      | Inspection       | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Marine growth in suction.  |
| 95   | Environmental   | Internal to Component | Pump      | Inspection       | Packing/Seals       | ESW    | 1994 | Failure to Run   | Partial           | Backup seal water regulators did not provide required flow during testing on two pumps. The third pump lost seal flow while operating. The cause was attributed to plugged lines.   |
| 96   | Environmental   | Internal to Component | Pump      | Inspection       | Lubrication         | HPI    | 1983 | Failure to Run   | Partial           | Oysters and miscellaneous mollusks plugged HPI oil coolers. Two pumps were required to be shutdown due to rising lubricating oil temperatures.  |
| 97   | Environmental   | Internal to Component | Pump      | Maintenance      | Lubrication         | HPI    | 1980 | Failure to Run   | Partial           | HPI pump lube oil cooler with tube leak allowed water into oil reservoir.   |
| 98   | Environmental   | Internal to Component | Pump      | Maintenance      | Lubrication         | HPI    | 1986 | Failure to Run   | Almost Complete   | Clams/sludge fouling of lube oil cooler caused high temperature alarms on two HPI pumps.  |
| 99   | Environmental   | Internal to Component | Pump      | Maintenance      | Lubrication         | HPI    | 1991 | Failure to Run   | Partial           | HPI pump lube oil cooler leaks. Degraded tubes.   |
| 100  | Environmental   | Internal to Component | Pump      | Maintenance      | Packing/Seals       | ESW    | 1985 | Failure to Run   | Partial           | First pump developed seal leak due to sand. Second pump had high bearing temperatures due to trash clogging cooling water lines.  |

| Item | Coupling Factor | Proximate Cause       | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|-----------------------|---------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 101  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1995 | Failure to Start | Partial           | Pumps failed performance test. Sand in water eroded pump internals. Pump lift was adjusted.  |
| 102  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1982 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.   |
| 103  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | HPI    | 1984 | Failure to Run   | Almost Complete   | One HPI pump seized, the second would have seized if operated.   |
| 104  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1982 | Failure to Run   | Partial           | Low ESW pump head values were caused excessive wear of pump impeller due to foreign material in the service water.   |
| 105  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1993 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to sand in the service water.   |
| 106  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.  |
| 107  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Run   | Partial           | ESW pump impeller lift out of adjustment.  |
| 108  | Environmental   | Internal to Component | Pump    | Test             | Bearing             | ESW    | 1992 | Failure to Run   | Partial           | Abrasive particles present in ocean water produced accelerated wear of shaft bearing journals.   |
| 109  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1995 | Failure to Start | Partial           | Marine growth caused low flow and speed condition for two service water pumps  |
| 110  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1994 | Failure to Start | Partial           | Degraded performance identified during testing. Sand in water was causing accelerated wear of the pump internals. Lift was adjusted for three pumps and one pump internals were replaced.  |
| 111  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. A rag was found in one impeller and a plastic bottle in the other.   |
| 112  | Environmental   | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1991 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted.   |
| 113  | Environmental   | Internal to Component | Suction | Demand           | Strainer            | ESW    | 1980 | Failure to Run   | Partial           | Foreign material was allowed to enter the suction of the charging pump service water pumps resulting in low flow conditions.   |
| 114  | Environmental   | Internal to Component | Suction | Demand           | Piping              | ESW    | 1986 | Failure to Start | Partial           | RHR service water pumps failed flow testing due to blocked suctions and abnormal wear of impellers.  |
| 115  | Environmental   | Internal to Component | Suction | Inspection       | Strainer            | ESW    | 1984 | Failure to Run   | Partial           | Two RHR service water pumps had blown seals and sparks and smoke between the bearing housing and shaft. A piece of hard rubber valve liner was found in the pumps.   |
| 116  | Environmental   | Internal to Component | Suction | Test             | Piping              | ESW    | 1990 | Failure to Start | Partial           | ESW pumps failed flow testing. Foreign material blocked the suction.   |
| 117  | Environmental   | Internal to Component | Suction | Test             | Strainer            | ESW    | 1990 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by suction blockage due to foreign material in the service water.  |
| 118  | Environmental   | Internal to Component | Suction | Test             | Strainer            | ESW    | 1982 | Failure to Run   | Partial           | Failures occurred on residual heat removal service water pumps. The pumps failed to meet flow and pressure requirements. Failure was due to debris lodging in pump impellers. Source of debris was maintenance activities, broken traveling water screens, and the inadvertent opening of a RHR minimum flow line which washed materials into suction pit. |

| Item | Coupling Factor | Proximate Cause  | Segment   | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|--|-----------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 119  | Environmental   | Other  | Driver    | Inspection       | Motor               | AFW    | 1990 | Failure to Start | Partial           | Both motor driven AFW pumps were sprayed when a service water pipe developed a through wall leak.  |
| 120  | Environmental   | Other  | Driver    | Inspection       | Motor               | ESW    | 1981 | Failure to Run   | Partial           | The float guide failed in a RHRSW pump air valve and caused the valve to fail open and flood pump room.  |
| 121  | Maintenance     | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Inspection       | Packing/Seals       | ESW    | 1997 | Failure to Run   | Partial           | Both ESW pumps leaking greater than 4 gpm because of inappropriate material for packing and sleeve (nitronic 60).  |
| 122  | Maintenance     | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.   |
| 123  | Maintenance     | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Test             | Casing              | ESW    | 1997 | Failure to Run   | Almost Complete   | Both ESW pumps failed due to installation of wrong material for pump casing flanges by vendor during pump overhaul. The vendor overhauled the pumps without changing material. The plant returned the pumps to the warehouse also without verifying material.  |
| 124  | Maintenance     | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Demand           | I&C                 | HPI    | 1997 | Failure to Run   | Complete          | HPI pumps fail due to operation with inadequate suction head. Two pumps damaged due to operation with inadequate suction, but all three system pumps were unavailable due to the loss of the suction source. Suction source level instrumentation was the cause.   |
| 125  | Maintenance     | External Environment   | Driver    | Demand           | Breaker             | AFW    | 1990 | Failure to Run   | Partial           | AFW pumps circuit breakers degraded.   |
| 126  | Maintenance     | Internal to Component  | Discharge | Inspection       | Check Valve         | AFW    | 1990 | Failure to Start | Almost Complete   | Leakage past AFW check valves caused AFW pumps to become steam bound. Closed motor operated valve in line. Scheduled check valves for replacement next outage.   |
| 127  | Maintenance     | Internal to Component  | Discharge | Test             | Valve               | HPI    | 1984 | Failure to Start | Partial           | CCP pump low flow rates due to inaccuracies in positioning the throttle valves.  |
| 128  | Maintenance     | Internal to Component  | Driver    | Demand           | I&C                 | ESW    | 1991 | Failure to Start | Partial           | Two ESW pumps failed to start due to failed breakers. Inadequate maintenance.  |
| 129  | Maintenance     | Internal to Component  | Driver    | Demand           | Lubrication         | HPI    | 1984 | Failure to Run   | Partial           | Charging pump lube oil cooler fan motor trips on thermal overload. Probable cause: normal wear on motor resulting in increased friction replaced worn motor with spare. During routine inservice testing found that another charging pump lube oil cooler fan motor had a current imbalance. Probable cause: normal aging of motor insulation has resulted in a current imbalance. |
| 130  | Maintenance     | Internal to Component  | Driver    | Demand           | Breaker             | RHR-B  | 1987 | Failure to Start | Partial           | RHR pump breakers failed to close when operated remotely from the control room. It was found that the latch roller bearings and the cam follower bearing (internal piece parts of the breaker) were not operating correctly. This prevented the trip latch assembly from resetting and allowing the breaker to close.  |

| Item | Coupling Factor | Proximate Cause       | Segment | Discovery Method | Piece Part    | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|-----------------------|---------|------------------|---------------|--------|------|------------------|-------------------|---|
| 131  | Maintenance     | Internal to Component | Driver  | Inspection       | Packing/Seals | HPI    | 1988 | Failure to Run   | Almost Complete   | Smoke was discovered coming from the speed increaser unit for a centrifugal charging pump. Investigation found the two gland seal retaining bolts inside the speed increaser lube oil pump backed out allowing the gland seal to loosen. The gland seal being loosened, caused reduced oil flow to the speed increaser internals and ultimate damage. Other CCPs were inspected, and the same gland seal bolts as on the first pump were found loosened. The cause of the bolts backing out was determined to be lack of a periodic adjustment of the gland seal bolts. |
| 132  | Maintenance     | Internal to Component | Driver  | Inspection       | Bearing       | ESW    | 1981 | Failure to Run   | Partial           | ESW motor to pump alignment problems. Bearings worn out.  |
| 133  | Maintenance     | Internal to Component | Driver  | Inspection       | Breaker       | ESW    | 1996 | Failure to Start | Partial           | ESW pump breakers fail due to misalignment of the breaker mechanism and internals developed over the years of operation.  |
| 134  | Maintenance     | Internal to Component | Driver  | Inspection       | Bearing       | ESW    | 1985 | Failure to Run   | Partial           | One service water pump motor upper bearing oil reservoir leaking from cover plate. Another service water pump motor upper oil cooler oil reservoir leaking.   |
| 135  | Maintenance     | Internal to Component | Driver  | Maintenance      | Breaker       | ESW    | 1985 | Failure to Start | Partial           | Two raw water pump breaker main wipes were out of adjustment.   |
| 136  | Maintenance     | Internal to Component | Driver  | Maintenance      | Breaker       | SLC    | 1999 | Failure to Start | Partial           | SLC Pump Breakers Fail to pickup on degraded voltage test   |
| 137  | Maintenance     | Internal to Component | Driver  | Maintenance      | Breaker       | HPI    | 1991 | Failure to Start | Partial           | HPI pump breakers failed due to a broken pawl, and a broken closing coil.   |
| 138  | Maintenance     | Internal to Component | Driver  | Maintenance      | Breaker       | AFW    | 1992 | Failure to Start | Partial           | With the unit in a refueling outage, following repairs to a motor driven auxiliary feedwater pump local/remote switch of the circuit breaker, personnel found that the switch contacts would not close. This failure rendered one of three auxiliary feedwater pumps inoperable. The cause of the failure appears to be due to dirty/corroded contacts on the switch.   |
| 139  | Maintenance     | Internal to Component | Driver  | Test             | Breaker       | ESW    | 1998 | Failure to Start | Partial           | Two RHR service water pump breakers would not close due to dirty contacts in breakers.  |
| 140  | Maintenance     | Internal to Component | Driver  | Test             | Breaker       | RHR-B  | 1997 | Failure to Start | Partial           | Breaker latch check switch failed on both pumps. Lack of lubrication.   |
| 141  | Maintenance     | Internal to Component | Driver  | Test             | Bearing       | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps exhibited vibration. Attributed to normal wear.   |
| 142  | Maintenance     | Internal to Component | Driver  | Test             | Breaker       | ESW    | 1998 | Failure to Start | Partial           | Service water pumps fail to start due to circuit breaker failures. Pump breakers failed to close due to failures of the charging spring/motor and closing spring motor.   |
| 143  | Maintenance     | Internal to Component | Driver  | Test             | Breaker       | AFW    | 1997 | Failure to Start | Almost Complete   | The circuit breakers associated with the AFW Pumps failed to close as required. The root cause of the failure was the binding in the operating mechanism. The plunger apparently did not always complete its upward movement to close and latch the breaker, due to accumulated dirt and lubricants.  |
| 144  | Maintenance     | Internal to Component | Driver  | Test             | Breaker       | RHR-B  | 1986 | Failure to Start | Partial           | RHR pump circuit breakers failed during a start for testing. Bend switch and binding mechanism. Attributed to inadequate maintenance.   |
| 145  | Maintenance     | Internal to Component | Pump    | Demand           | Bearing       | AFW    | 1984 | Failure to Run   | Partial           | One ESW bearing failed and pump seized; second motor bearing failed.  |
| 146  | Maintenance     | Internal to Component | Pump    | Demand           | Casing        | ESW    | 1998 | Failure to Start | Partial           | Two ESW pump started and ran, but would not develop sufficient pressure or flow rate. Exact cause not known for either failure, however, one pump was noted to have microbiological induced corrosion fouling on internal surfaces.   |



| Item | Coupling Factor | Proximate Cause       | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|-----------------------|---------|------------------|---------------------|--------|------|------------------|-------------------|--|
| 147  | Maintenance     | Internal to Component | Pump    | Demand           | Packing/Seals       | AFW    | 1998 | Failure to Run   | Partial           | AFW MDP and TDPs failed due to incorrect packing installed.  |
| 148  | Maintenance     | Internal to Component | Pump    | Inspection       | Bearing             | ESW    | 1987 | Failure to Run   | Partial           | Service water pumps had high shaft vibration. The excessive vibrations caused by worn bearings and shaft sleeves.  |
| 149  | Maintenance     | Internal to Component | Pump    | Inspection       | Packing/Seals       | AFW    | 1990 | Failure to Run   | Partial           | Both motor-driven aux. feedwater pumps had excessive packing leaks, due to worn packing.   |
| 150  | Maintenance     | Internal to Component | Pump    | Inspection       | Bearing             | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. Loose fittings and lack of thread sealant.  |
| 151  | Maintenance     | Internal to Component | Pump    | Inspection       | Casing              | ESW    | 1986 | Failure to Run   | Partial           | Cracked seal water and vent lines.   |
| 152  | Maintenance     | Internal to Component | Pump    | Inspection       | Packing/Seals       | ESW    | 1989 | Failure to Run   | Partial           | ESW pump excessive packing leakage.  |
| 153  | Maintenance     | Internal to Component | Pump    | Inspection       | Packing/Seals       | ESW    | 1986 | Failure to Run   | Partial           | Excessive packing leakage. Both events occurred after previous maintenance had been performed for the same problems.   |
| 154  | Maintenance     | Internal to Component | Pump    | Inspection       | Casing              | ESW    | 1988 | Failure to Run   | Partial           | RHR service water pumps. Pump diffuser eroded on first pump and a through wall casing leak developed on the second.  |
| 155  | Maintenance     | Internal to Component | Pump    | Inspection       | Lubrication         | RHR-B  | 1990 | Failure to Run   | Partial           | Both pump motor oil coolers were leaking due to aging of components. The first case involved through wall corrosion and the pump was immediately removed from service. The second case was a packing leak. |
| 156  | Maintenance     | Internal to Component | Pump    | Inspection       | Packing             | AFW    | 1986 | Failure to Run   | Partial           | The packing was worn on both the motor-driven and one turbine-driven aux. feedwater pump, causing high temperature on one packing gland, and excessive leaking on the other pump.                          |
| 157  | Maintenance     | Internal to Component | Pump    | Inspection       | Packing/Seals       | SLC    | 1987 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing adjusted.   |
| 158  | Maintenance     | Internal to Component | Pump    | Inspection       | Plunger/Cylinder    | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pump seal was leaking excessively. The cause of this failure was normal wear of the plungers, packing, and head gaskets for the plungers (piece parts of the pump).                 |
| 159  | Maintenance     | Internal to Component | Pump    | Inspection       | Packing/Seals       | SLC    | 1988 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing replaced.   |
| 160  | Maintenance     | Internal to Component | Pump    | Inspection       | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. The cause of the failure is suspected to be binding.  |
| 161  | Maintenance     | Internal to Component | Pump    | Inspection       | Packing/Seals       | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking profusely at the packing. The failure of the packing was attributed to normal wear.   |
| 162  | Maintenance     | Internal to Component | Pump    | Maintenance      | Bearing             | ESW    | 1985 | Failure to Run   | Partial           | High ESW pump vibration was caused by wearing of the upper bearings.   |
| 163  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed due to worn internals.  |
| 164  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1984 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head values. The low pump heads were caused by wear and aging of internals.  |
| 165  | Maintenance     | Internal to Component | Pump    | Test             | Coupling            | ESW    | 1987 | Failure to Start | Almost Complete   | Two ESW pumps had failed couplings. Cause attributed to abnormal stress.   |

| Item | Coupling Factor | Proximate Cause       | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|-----------------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 166  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.   |
| 167  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1982 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.   |
| 168  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1989 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 169  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | HPI    | 1985 | Failure to Start | Partial           | The CCPs were tested and had low flow rates. The most probable cause is attributed to observed degradation of the pumps. The CCPs are subject to normal wear associated with their secondary duty of providing normal charging flow.                            |
| 170  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1994 | Failure to Start | Partial           | Two ESW pumps had low discharge pressure during testing. Each pump had worn internals and both pump internals were replaced.  |
| 171  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | RHR-B  | 1985 | Failure to Start | Partial           | The first pump failed to meet required flow rate. The second was drawing excessive amperage. Both conditions were attributed to worn internals.   |
| 172  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1984 | Failure to Run   | Partial           | Containment spray raw water pumps failed flow tests. Aging and normal wear.   |
| 173  | Maintenance     | Internal to Component | Pump    | Test             | Shaft               | ESW    | 1993 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.  |
| 174  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.  |
| 175  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | HPI    | 1983 | Failure to Start | Partial           | SI pump and both CCPs failed to meet the minimum head curve requirements. The cause of pump head capacity degradation has been attributed to normal pump operation. The inability to balance flows has been attributed to the lower head capacity of the pumps. |
| 176  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW impeller gaps too wide. Gaps adjusted.  |
| 177  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1998 | Failure to Start | Partial           | Two ESW pumps failed to develop adequate flow/pressure - pumps degraded.  |
| 178  | Maintenance     | Internal to Component | Pump    | Test             | Lubrication         | SLC    | 1992 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. The gasket between the crankcase frame cap and the gear housing cover was worn.  |
| 179  | Maintenance     | Internal to Component | Pump    | Test             | Coupling            | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.   |
| 180  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1981 | Failure to Start | Partial           | Loss of Service Water pump due to wearout at end of life.   |
| 181  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | Wear caused high ESW pump bearing temperatures, vibration, and low amperage/flow.   |
| 182  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | ESW pump performance decreased 15% and 8% respectively since last test. Pumps were replaced.  |
| 183  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | ESW pumps had worn impellers and one had a plugged strainer.  |
| 184  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1994 | Failure to Run   | Partial           | Two ESW pumps had internal deterioration, one of which was indicated by high vibration readings.  |
| 185  | Maintenance     | Internal to Component | Pump    | Test             | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. One pump also exhibited high vibration.  |

| Item | Coupling Factor | Proximate Cause          | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--------------------------|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 186  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to brackish water corrosion.  |
| 187  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1987 | Failure to Run   | Partial           | ESW pump low flow. Worn impellers.  |
| 188  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Run   | Partial           | ESW pumps had worn and cracked impellers. Aging and normal wear.  |
| 189  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | The charging pump service water pumps degraded. Caused by expected wear of pump due to erosion and corrosion properties of the process fluid involved   |
| 190  | Maintenance     | Internal to Component    | Pump    | Test             | Packing/Seals       | ESW    | 1981 | Failure to Start | Partial           | RHR service water pumps failed to meet flow requirements due to seal water leakage and pump wearout.  |
| 191  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1991 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 192  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 193  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1984 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.   |
| 194  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 195  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | Emergency service water pumps discharge pressure below allowable limits. Causes were loose impellers, dropped impeller, and worn internals.   |
| 196  | Maintenance     | Internal to Component    | Pump    | Test             | Bearing             | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 197  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1983 | Failure to Run   | Partial           | RHR Service Water pumps failed flow tests due to wearout and had to be rebuilt.   |
| 198  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 199  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 200  | Maintenance     | Internal to Component    | Pump    | Test             | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted.  |
| 201  | Maintenance     | Operational/ Human Error | Driver  | Demand           | Breaker             | ESW    | 1993 | Failure to Start | Partial           | Operations personnel were attempting to swap the running service water pump with the idle service water pump. Personnel placed the control switch to start and the service water pump did not start. Breaker malfunction. Later, another service water pump failed to start because of the breaker. |
| 202  | Maintenance     | Operational/ Human Error | Driver  | Demand           | Breaker             | ESW    | 1988 | Failure to Run   | Partial           | Service water pump high dropout over current protection devices were less than running current conditions and trip setpoints did not account for changing load conditions due to modified impellers. Three pump trips had occurred.   |
| 203  | Maintenance     | Operational/ Human Error | Driver  | Demand           | Breaker             | ESW    | 1987 | Failure to Start | Partial           | One breaker failed to linkage alignment and second from loose relay connections. Inadequate maintenance.  |
| 204  | Maintenance     | Operational/ Human Error | Driver  | Inspection       | Breaker             | RHR-P  | 1981 | Failure to Start | Complete          | All RHR pumps de-energized to replace RHR Relief valve. T.S. allows this condition for 1 hour. Operated in the mode in excess of 5 hours.   |

| Item | Coupling Factor | Proximate Cause          | Segment | Discovery Method | Piece Part    | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--------------------------|---------|------------------|---------------|--------|------|------------------|-------------------|---|
| 205  | Maintenance     | Operational/ Human Error | Driver  | Inspection       | I&C           | RHR-P  | 1992 | Failure to Start | Complete          | Both trains of RHR were rendered inoperable for two minutes, while performing an operational readiness test surveillance procedure. The surveillance procedure required that the one RHR train pump be placed in pull to lock and the other train heat exchanger flow control valve throttled to 30-40% open. The procedure directed the operators to perform operations that resulted in both trains of RHR being inoperable   |
| 206  | Maintenance     | Operational/ Human Error | Driver  | Inspection       | Bearing       | RHR-P  | 1988 | Failure to Run   | Partial           | Residual heat removal pump motor upper bearing housings were observed to be leaking oil. The cause of the failure was attributed to a lack of sealant being applied and gasket installed after the last maintenance was performed on the motor bearing housing.   |
| 207  | Maintenance     | Operational/ Human Error | Driver  | Inspection       | I&C           | AFW    | 1990 | Failure to Start | Complete          | During testing one AFW pump was tested and other was tested without returning first to auto. Both pumps were unavailable at the same time. The procedure was the cause.   |
| 208  | Maintenance     | Operational/ Human Error | Driver  | Maintenance      | Breaker       | RHR-B  | 1990 | Failure to Start | Partial           | RHR pump breaker overcurrent trips out of calibration.  |
| 209  | Maintenance     | Operational/ Human Error | Driver  | Maintenance      | Breaker       | RHR-B  | 1991 | Failure to Start | Partial           | While performing preventive maintenance calibration check on the protective relays for a residual heat removal pump motor 4kv breaker, it was found that all overcurrent relays for two pumps were out of calibration   |
| 210  | Maintenance     | Operational/ Human Error | Driver  | Test             | I&C           | ESW    | 1989 | Failure to Start | Partial           | Emergency equipment service water pump relays were not reset following a load shedding test 30 hours before.  |
| 211  | Maintenance     | Operational/ Human Error | Driver  | Test             | Motor         | ESW    | 1994 | Failure to Run   | Partial           | Leak test of the containment cooling service water pump vault watertight door revealed excessive leakage. Flooding and leakage past this door would make inoperable two of four containment cooling service water pumps. Procedural inadequacy was cited as the cause for the degraded door seals.  |
| 212  | Maintenance     | Operational/ Human Error | Pump    | Demand           | Casing        | AFW    | 1983 | Failure to Run   | Partial           | During testing, the outboard bearing temperature was high on the turbine-driven AFW pump, due to improper balance drum clearances, caused by improper maintenance. The procedure will be modified and the balance drum clearance reset. While the unit was starting up, the motor-driven AFW pump outboard bearing temperature was high. Excessive thrust bearing clearance caused the balance drum to unbalance, causing the thrust bearing to overheat.   |
| 213  | Maintenance     | Operational/ Human Error | Pump    | Maintenance      | Lubrication   | HPI    | 1991 | Failure to Run   | Partial           | Following an overhaul of the HPI pumps. Too much oil flow led to excessive oil leakage, which would have failed HPI pumps before end of mission.  |
| 214  | Maintenance     | Operational/ Human Error | Pump    | Test             | Casing        | RHR-P  | 1989 | Failure to Start | Complete          | Both loops of the residual heat removal system were declared inoperable due to gas binding of both RHR pumps. The gas binding was caused by entry of nitrogen gas into the reactor coolant system from accumulator. The root cause of this event has been attributed to personnel error. Personnel did not comply with the specific requirements in the accumulator discharge check valve full flow test procedure due to inattention to detail.  |
| 215  | Maintenance     | Operational/ Human Error | Pump    | Test             | Packing/Seals | AFW    | 1996 | Failure to Run   | Partial           | During the performance of Steam-Driven Emergency Feedwater Pump testing, sparks were observed emanating from the outboard mechanical seal area. The sparks appeared to be due to a mechanical interference within the mechanical seal assembly. The pump mechanical seal was disassembled and determined to have been improperly installed during the last refueling outage. The evaluation identified a mechanical seal design deficiency and inadequate corrective action for a previously identified event as the primary causes for this event. A contributing cause for this event was found to be inadequate predictive maintenance techniques. The electric AFW pump exhibited the same problem. |

| Item | Coupling Factor | Proximate Cause          | Segment   | Discovery Method | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|--------------------------|-----------|------------------|------------|--------|------|------------------|-------------------|--|
| 216  | Maintenance     | Operational/ Human Error | Suction   | Demand           | Piping     | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.  |
| 217  | Maintenance     | Operational/ Human Error | Suction   | Demand           | Piping     | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.  |
| 218  | Maintenance     | Operational/ Human Error | Suction   | Inspection       | Valve      | SLC    | 1991 | Failure to Start | Partial           | SLC pumps were potentially inoperable during part of test due to valve lineup.   |
| 219  | Maintenance     | Operational/ Human Error | Suction   | Maintenance      | Strainer   | HPI    | 1985 | Failure to Run   | Partial           | Strainers found still installed in the suction piping of the high-pressure injection pumps was a condition not considered in the operating design. The strainers were found during maintenance to repair a slight flange leak. The strainers had been placed in the suction piping during construction and were to be in place during system flushing to prevent any debris from reaching the pumps. However, the strainers should have been removed after system flushing prior to functional testing   |
| 220  | Maintenance     | Operational/ Human Error | Suction   | Maintenance      | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | Shutdown cooling was lost due to nitrogen intrusion because of backflushing a filter in the purification system.   |
| 221  | Maintenance     | Other                    | Discharge | Demand           | Valve      | ESW    | 1980 | Failure to Start | Partial           | RHR service water pumps were started to put torus cooling in service. When these pumps would not deliver required discharge pressure, they were declared inoperable. The seal in an air release valve was bad, allowing a vent on the discharge line.  |
| 222  | Maintenance     | Other                    | Driver    | Demand           | I&C        | ESW    | 1982 | Failure to Start | Complete          | Following a reactor scram, an attempt to initiate suppression pool cooling revealed that both RHRSW loops were inoperable as neither loop's pumps could be started. Low suction header pressure lockout signals in each loop prevented starting each loop's pumps. Plugging of the sensing line to each loop's suction header pressure switch prevented both switches from sensing actual pressure, although a lack of operating fluid in one switch and an open power supply breaker to the other switch also would have prevented pumps from starting. |
| 223  | Maintenance     | Other                    | Driver    | Demand           | Breaker    | RHR-P  | 1987 | Failure to Start | Complete          | Two LPI pumps, when given a start signal, would not start. An ongoing investigation revealed the probable root cause of the event to be poor electrical contact of the breaker auxiliary stabs for the pumps.  |
| 224  | Maintenance     | Other                    | Driver    | Maintenance      | Breaker    | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker failures, broken screw, no lubrication, and a bent track  |
| 225  | Maintenance     | Other                    | Driver    | Maintenance      | Breaker    | ESW    | 1982 | Failure to Start | Partial           | ESW pump circuit breakers found damaged. Defective arc chute and cracked secondary coupler.  |
| 226  | Maintenance     | Other                    | Driver    | Test             | Breaker    | ESW    | 1984 | Failure to Start | Partial           | ESW pump breakers tripped due to failed voltage control devices.   |
| 227  | Maintenance     | Other                    | Driver    | Test             | Breaker    | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker overcurrent trip devices tripping too low.  |
| 228  | Maintenance     | Other                    | Suction   | Demand           | Piping     | RHR-P  | 1980 | Failure to Run   | Complete          | A complete loss of RHR flow occurred while plant operators were increasing RHR heat exchanger flow by closing down on the heat exchanger bypass valve.   |

| Item | Coupling Factor | Proximate Cause  | Segment   | Discovery Method | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|-----------------|--|-----------|------------------|-------------|--------|------|------------------|-------------------|--|
| 229  | Maintenance     | Other  | Suction   | Demand           | Piping      | RHR-P  | 1981 | Failure to Run   | Complete          | Temporary coolant loop level indicator showed level slowly increasing over a period of days. The system was periodically drained to maintain 65 percent indicated level. A RHR pump lost suction on reduction of actual level. The second pump was started, and lost suction. Indication drift was due to evaporation of reference leg.  |
| 230  | Maintenance     | Other  | Suction   | Demand           | Piping      | RHR-P  | 1986 | Failure to Run   | Complete          | SDC pumps cavitated due to lowering RCS level. Level indication was in error.  |
| 231  | Maintenance     | Other  | Suction   | Demand           | Piping      | RHR-P  | 1983 | Failure to Run   | Complete          | The RHR pumps began to cavitate and eventually both pumps were stopped. The reactor vessel level gauge being used to provide an indication that the level was approaching the vessel flange level had been isolated (reactor coolant drain tank isolation valve had been closed during an attempt to reduce leakage). Additionally, procedures did not require visual monitoring of cavity level.  |
| 232  | Operational     | Design/ Construction/ Manufacture/ Installation Inadequacy | Discharge | Test             | Check Valve | ESW    | 1999 | Failure to Run   | Partial           | Two ESW pumps had low flow due to interaction with the two other pumps when all four pumps were running.   |
| 233  | Operational     | External Environment                                       | Driver    | Inspection       | I&C         | HPI    | 1990 | Failure to Run   | Complete          | It was determined that the common minimum flow path return line for the safety injection pumps to the refueling water storage tank was frozen. Previous actions to investigate problems with the freeze protection system were unsuccessful in preventing development of this condition. The two HPI pumps were declared inoperable with this return line frozen. A faulty ambient temperature switch for the RWST heat trace system prevented the heat trace from activating and was subsequently replaced. In addition, administrative controls did not sufficiently recognize the safety significance of flow through this line and the need to ensure flow capability. |
| 234  | Operational     | Operational/ Human Error                                   | Discharge | Inspection       | Valve       | HPI    | 1987 | Failure to Start | Almost Complete   | While attempting to fill the safety injection accumulators, it was discovered that two of three SI pumps had been isolated from the high head injection flowpath.  |
| 235  | Operational     | Operational/ Human Error                                   | Discharge | Inspection       | Valve       | HPI    | 1993 | Failure to Run   | Partial           | One AFW pump failed due to incorrect procedure which allowed pump to be run without flow, other AFW pump was allowed to run past max flow rate. It is unclear whether these mistakes were due to inadequate procedures or staff errors, but it was assumed to be a failure to follow procedure.  |
| 236  | Operational     | Operational/ Human Error                                   | Discharge | Inspection       | Valve       | AFW    | 1994 | Failure to Start | Complete          | Following a trip, the AFW Pumps were secured and the discharge flow control valves for the Motor Driven Pumps were closed. Later, an operator discovered during a routine Control Board walkdown that the valves were closed. Subsequent investigation revealed the AFW system had not been placed in standby readiness per the operating procedure after the system was secured.  |
| 237  | Operational     | Operational/ Human Error                                   | Driver    | Demand           | I&C         | ESW    | 1981 | Failure to Start | Partial           | Alarm circuit breaker was de-energized resulting in a loss of two RHR service water pumps.   |
| 238  | Operational     | Operational/ Human Error                                   | Driver    | Demand           | I&C         | AFW    | 1983 | Failure to Start | Complete          | An operator incorrectly secured the diesel and steam driven AFW pumps, which prevented their restart on low SG level.  |
| 239  | Operational     | Operational/ Human Error                                   | Driver    | Inspection       | I&C         | HPI    | 1988 | Failure to Start | Complete          | With alternate CCP pump out-of-service, the remaining operable pump was erroneously placed in pull-to-lock.  |
| 240  | Operational     | Operational/ Human Error                                   | Driver    | Inspection       | I&C         | RHR-P  | 1995 | Failure to Start | Complete          | The switches for the containment spray and recirculation pumps were in a trip pullout when the Technical Specifications and plant procedures required the pumps to be operable.  |

| Item | Coupling Factor | Proximate Cause          | Segment | Discovery Method | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--------------------------|---------|------------------|-------------|--------|------|------------------|-------------------|---|
| 241  | Operational     | Operational/ Human Error | Driver  | Inspection       | Breaker     | ESW    | 1981 | Failure to Start | Almost Complete   | Control breakers for two ESW pumps were open due to inadvertent operator action.  |
| 242  | Operational     | Operational/ Human Error | Driver  | Inspection       | Breaker     | HPI    | 1988 | Failure to Start | Complete          | HPI pumps not restored before mode change due to procedural inadequacy.   |
| 243  | Operational     | Operational/ Human Error | Driver  | Inspection       | I&C         | HPI    | 1992 | Failure to Start | Almost Complete   | Two charging pumps and one charging pump service water pump were removed from service simultaneously which is a condition not allowed by technical specifications.  |
| 244  | Operational     | Operational/ Human Error | Driver  | Inspection       | I&C         | HPI    | 1990 | Failure to Start | Partial           | Both safety injection pumps were in the pull-to-lock position. With the switches in pull-to-lock, the pumps would not have automatically started upon receipt of an initiating signal. This event was caused by cognitive personnel error by a utility licensed operator in failure to follow an approved procedure.  |
| 245  | Operational     | Operational/ Human Error | Driver  | Inspection       | Breaker     | HPI    | 1982 | Failure to Start | Complete          | During the draining of the reactor coolant system, both centrifugal charging pumps were rendered inoperable. The initial conditions in the draining procedure contained a confusing statement, which led to an erroneous assumption that both CCP breakers had to be racked out and tagged.   |
| 246  | Operational     | Operational/ Human Error | Driver  | Inspection       | Breaker     | HPI    | 1989 | Failure to Start | Partial           | HPI Pump B not retested, then HPI Pump A removed from service.  |
| 247  | Operational     | Operational/ Human Error | Driver  | Inspection       | Breaker     | HPI    | 1990 | Failure to Start | Complete          | By opening incorrect breaker, HPI pump tripped while others were unavailable.   |
| 248  | Operational     | Operational/ Human Error | Driver  | Inspection       | Breaker     | CSS    | 1991 | Failure to Start | Complete          | CSR control power de-energized prior to mode change. Technical Specification violation. Inadequate procedure review.  |
| 249  | Operational     | Operational/ Human Error | Driver  | Test             | I&C         | ESW    | 1990 | Failure to Start | Complete          | An emergency service water pump failed to start and was declared inoperable. Further investigation determined that the failure of the pump to start was due to a tripped emergency engine shutdown device. Operations personnel performing the testing did not recognize the need to reset it prior to starting the pump. Examination of the other two ESW pumps revealed that their emergency shutdown devices were also in the tripped condition. |
| 250  | Operational     | Operational/ Human Error | Pump    | Inspection       | Lubrication | HPI    | 1983 | Failure to Start | Complete          | A routine preventive maintenance (oil change) was mistakenly performed on the north charging pump instead of the south as scheduled. Since the south pump was previously cleared for this oil change, and the test pump was valved out, none of these three pumps were in service as required by tech specs for the approximately 20 minutes it took to change the oil in the north pump.   |
| 251  | Operational     | Operational/ Human Error | Pump    | Maintenance      | Lubrication | ESW    | 1993 | Failure to Run   | Partial           | Low pressure RHR bearing oil level not maintained high enough when new smaller sightglass installed. Second event the sightglass was broken when adding oil.  |
| 252  | Operational     | Operational/ Human Error | Suction | Demand           | Piping      | RHR-P  | 1980 | Failure to Run   | Complete          | While attempting to increase RHR flow, the plant experienced a total loss of flow due to the pumps being air-bound. The pump was not vented when starting to increase flow. Operating procedures have been changed to have an operator present while changing flow in the RHR system. There have been losses of RHR flow in the past because the pumps were air-bound and methods are being investigated to improve the system design.              |
| 253  | Operational     | Operational/ Human Error | Suction | Demand           | Piping      | ESW    | 1986 | Failure to Run   | Complete          | Failure to properly vent and fill a newly installed pipe introduced air into the charging pump service water system.  |

| Item | Coupling Factor | Proximate Cause  | Segment | Discovery Method | Piece Part   | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--|---------|------------------|--------------|--------|------|------------------|-------------------|---|
| 254  | Operational     | Operational/ Human Error                                   | Suction | Demand           | Piping       | RHR-P  | 1984 | Failure to Run   | Complete          | The control room operators started a second residual heat removal pump in preparation for removing the operating RHR pump from service. With both pumps running, flow became excessive for the half-loop condition causing cavitation and air binding of both pumps. To prevent recurrence the procedure which controls the operation of the RHR pumps has been changed to include specific instructions to stop the operating pump prior to starting the second pump while at half-loop.   |
| 255  | Operational     | Operational/ Human Error                                   | Suction | Demand           | Booster Pump | ESW    | 1980 | Failure to Start | Partial           | The service water RHR booster pump was de-energized during maintenance. The attempt to start service water pumps failed due to low suction pressure.  |
| 256  | Operational     | Operational/ Human Error                                   | Suction | Demand           | Piping       | ESW    | 1988 | Failure to Run   | Complete          | The procedure failed to adequately caution the operator to slowly fill a drained line. Rapid filling resulted in a loss of NPSH to the charging service water pumps.  |
| 257  | Operational     | Operational/ Human Error                                   | Suction | Maintenance      | Strainer     | ESW    | 1986 | Failure to Run   | Complete          | A service water strainer was placed in service without being vented resulting in air binding system and loss of charging pump service water pumps.  |
| 258  | Operational     | Operational/ Human Error                                   | Suction | Test             | Piping       | ESW    | 1989 | Failure to Run   | Partial           | Inadequate procedure led to air binding of operating ESW pumps.   |
| 259  | Operational     | Other  | Pump    | Inspection       | Bearing      | ESW    | 1991 | Failure to Run   | Partial           | Lube oil cooling water isolated during a test. Pumps continued to run with no cooling.  |
| 260  | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | Demand           | Motor        | ESW    | 1987 | Failure to Start | Partial           | ESW pump motors tripped on overcurrent. The overcurrent trip was due to a ground and a short on the pump motor.   |
| 261  | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | Demand           | Breaker      | ESW    | 1996 | Failure to Start | Partial           | Two RHRSW pumps fail to start due to breaker failures. Wrong contacts were installed. Design called for contacts to have a minimum current interrupt rating of 6 amps; contacts installed (that subsequently failed) had current interrupt rating of only 2.2 amps.   |
| 262  | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | Demand           | I&C          | AFW    | 1989 | Failure to Start | Complete          | Both motor driven auxiliary feedwater pumps failed to start when the operator tried to start them manually. While preparing a design change, the designer failed to review all the unit specific documentation associated with the motor-driven AFW pump wiring and made the erroneous assumption that both units switchgear compartment internal wiring was identical. In fact, the wiring for each unit was different. Consequently, when the design change was installed, it was installed in accordance with the erroneous design. The wiring discrepancy was corrected and the motor-driven AFW pumps were tested and returned to service. |
| 263  | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | Test             | Breaker      | LCS    | 1980 | Failure to Start | Complete          | Relay extra contacts left connected during construction, prevented Core Spray pump start with emergency diesel generator breakers racked out.   |



| Item | Coupling Factor | Proximate Cause  | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 264  | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Test             | I&C                 | AFW    | 1980 | Failure to Start | Complete          | During surveillance testing, neither motor-driven AFW pump would start. The pump control circuit was found with autostart defeat switches labeled backwards, causing all autostarts except the low-low steam generator level to be defeated. The labels were corrected and the links were closed. The original installation error was the result of an inadequate design change process that did not require sufficient verification and testing of the modification.   |
| 265  | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps drawing excessive current. Carbon steel snap rings corroded allowing impeller to come in contact with casing. The third pump, although not exhibiting abnormal current, had similar corrosion   |
| 266  | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Demand           | Impeller/Wear Rings | ESW    | 1996 | Failure to Run   | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 267  | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Inspection       | Casing              | AFW    | 1983 | Failure to Run   | Partial           | Two AFW pumps thrust tolerance was out of specification. These events were caused by improperly installed balancing drum parts. One turbine driven and one motor driven pump was involved.  |
| 268  | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Inspection       | Casing              | HPI    | 1987 | Failure to Run   | Partial           | During inspection of a centrifugal charging pump, a portion of the stainless steel cladding on the inside surface of the pump casing exhibited corrosion. Corrosion of the pump casing was through the stainless steel cladding into the carbon steel base material. Inspection of the other CCP revealed similar corrosion. The cause of this event was a manufacturing deficiency. Corrosion observed at the pump casing discharge nozzle was attributed to a cladding breakthrough during final machining. Corrosion observed at the pump casing inlet end was attributed to either over-machining of the cladding or inadequate overlay of two adjacent weld beads.   |
| 269  | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Test             | Impeller/Wear Rings | ESW    | 1986 | Failure to Start | Partial           | Testing of the service water system disclosed that the performance of the three service water pumps was below requirements. The condition is the result of both an inadequate system design and the installation of replacement impellers, which were not modified by the vendor to improve performance, as were the original impellers.  |
| 270  | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Demand           | Piping              | ESW    | 1984 | Failure to Start | Partial           | Both RHR service water pumps tripped as a result of inadequate venting of suction header resulting from poor orientation of the vent line.  |

| Item | Coupling Factor | Proximate Cause  | Segment | Discovery Method | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------------|--|---------|------------------|---------------------|--------|------|------------------|-------------------|---|
| 271  | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Inspection       | Piping              | HPI    | 1988 | Failure to Run   | Partial           | Vortex breakers had not been installed in the containment emergency sumps. Vortex breakers are required to be installed in the containment emergency sumps to prevent the formation of vortices which could adversely affect performance of safety injection pumps during the safety injection and containment spray systems were declared inoperable.  |
| 272  | Quality         | Internal to Component                                      | Pump    | Demand           | Impeller/Wear Rings | AFW    | 1988 | Failure to Run   | Partial           | Following a plant trip, it was discovered that the auxiliary feedwater pumps had internal damage. Some channel ring vanes had chips missing, and several parts were found in the SG auxiliary feedwater piping.   |
| 273  | Quality         | Operational/ Human Error                                   | Driver  | Inspection       | Breaker             | ESW    | 1992 | Failure to Start | Partial           | The fit between an ESW pump breaker primary disconnects and the associated breaker cubicle stabs was inadequate. The poor fit between the disconnects and the stabs led to arcing in the breaker cubicle when the pump was started, resulting in a fire. Shortly after identifying the cause of the fire, the remaining ESW breakers, which had recently been replaced along with the failed breaker, as part of a design modification package, were found to be inadequate also. |
| 274  | Quality         | Operational/ Human Error                                   | Driver  | Test             | I&C                 | ESW    | 1982 | Failure to Start | Partial           | Two ESW pumps failed to start. One ESW pump failed to function as a result of loose wires on relay terminals in both pump logic schemes, a loose states link and an instantaneous contact found out of adjustment on the other pump logic scheme.   |

Table A-3. Pump CCF events, sorted by the method of discovery.

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment   | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--|-----------|---------------------|--------|------|------------------|-------------------|--|
| 1    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Discharge | Valve               | AFW    | 1985 | Failure to Start | Partial           | Controller problems in the steam and diesel driven AFW pumps caused the pumps to trip on low suction pressure. The pump discharge flow controller valves were also not set properly after last maintenance. Low suction trips were due to design error.  |
| 2    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Discharge | Valve               | AFW    | 1986 | Failure to Start | Partial           | Both the turbine driven and motor driven AFW pumps could not produce full flow because the cages in their discharge valve trapped debris and plugged.  |
| 3    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | I&C                 | AFW    | 1981 | Failure to Start | Almost Complete   | A modification to the control instrumentation for two AFW pumps resulted in a backfeed situation such that when called upon to start, both pumps would not start.  |
| 4    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | I&C                 | AFW    | 1981 | Failure to Start | Almost Complete   | Two AFW pumps failed to automatically start due to low suction pressure trips. A modification was installed to prevent this. This effect was discovered previously, but apparently had not been corrected prior to an attempt to start the pumps three weeks later.  |
| 5    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | I&C                 | AFW    | 1997 | Failure to Run   | Partial           | One actual AFW pump failure due to spurious electronic overspeed trip. Determined that all three pumps were susceptible to spurious overspeed trips.   |
| 6    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Lubrication         | RHR-P  | 2000 | Failure to Run   | Complete          | Both RHR/LPI pumps fail to run due to improper oil in system. High bearing temperatures occurred when the pumps were operated. This was due to the wrong lube oil being used, which had too high a viscosity. Inadequate vendor design information resulted in the higher viscosity oil being used and additional exacerbating problems such as insufficient bearing clearances. |
| 7    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Impeller/Wear Rings | ESW    | 1981 | Failure to Run   | Complete          | Both charging pump service water pumps failed. A carbon cap screw failed allowing the impeller of one pump to bind on the casing. The ensuing leakage shorted the motor windings of the other pump.  |
| 8    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | All four emergency service water pumps showed cavitation damage. Two of the pumps had minor damage and were placed back in service. Recirculation cavitation occurs at flows significantly less than design.   |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part          | System | Year | Failure Mode   | Degree of Failure | Description   |
|------|------------------|-----------------|--|---------|---------------------|--------|------|----------------|-------------------|---|
| 9    | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Impeller/Wear Rings | ESW    | 1996 | Failure to Run | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 10   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1983 | Failure to Run | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Pump Service Water pumps.   |
| 11   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1982 | Failure to Run | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 12   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1983 | Failure to Run | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 13   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1982 | Failure to Run | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |
| 14   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1982 | Failure to Run | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.   |
| 15   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1981 | Failure to Run | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 16   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1981 | Failure to Run | Complete          | Increasing flow to chillers robs NPSH from charging service water pumps.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment   | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|-----------|---------------------|--------|------|------------------|-------------------|---|
| 17   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Piping              | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the Charging Water Service Water pumps.  |
| 18   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Piping              | ESW    | 1982 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 19   | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Piping              | ESW    | 1996 | Failure to Start | Partial           | Freezing of diesel generator service water piping in intake bay. Inadequate initial design.   |
| 20   | Demand           | Design          | External Environment   | Discharge | Check Valve         | AFW    | 1983 | Failure to Start | Almost Complete   | Hot water in the AFW pump casings caused the pumps to become vapor bound. The hot water was from leaking check valves upstream of the pumps. This event occurred once on the turbine driven pump and 5 times on the motor driven pump.  |
| 21   | Demand           | Design          | Internal to Component  | Driver    | Breaker             | ESW    | 2000 | Failure to Start | Almost Complete   | Two ESW pumps failed to start due to their breakers failing to close. The breakers' prop spring bracket has slipped thus preventing proper interfacing between the prop and the prop pin.   |
| 22   | Demand           | Design          | Operational/ Human Error   | Driver    | I&C                 | ESW    | 1980 | Failure to Start | Partial           | Instrument isolation valve closed causing a low suction trip signal to two RHRSW pumps.   |
| 23   | Demand           | Design          | Operational/ Human Error   | Pump      | Impeller/Wear Rings | AFW    | 1990 | Failure to Run   | Almost Complete   | Due to a combination of management error and procedural deficiency, the turbine driven auxiliary feedwater pump was run deadheaded. The operation damaged the pump. When the pump was manually tripped, steam vented back into the suction line, caused another AFW pump to also trip, on a low suction pressure signal.  |
| 24   | Demand           | Design          | Operational/ Human Error   | Suction   | Tank                | AFW    | 1980 | Failure to Run   | Complete          | Both emergency feedwater pumps lost feed pump suction. The emergency feedwater pump suction flashed to steam due to the feedwater train flashing and forcing hot water back through the startup and blowdown tanks and into the feedwater pump suction. To prevent this recurrence, the operating procedures have been changed to require isolating the startup and blowdown effluent as a source of emergency feedwater suction prior to increasing power. |
| 25   | Demand           | Design          | Operational/ Human Error   | Suction   | Piping              | RHR-P  | 1982 | Failure to Run   | Complete          | Suction was lost to both RHR pumps. RHR flow was less than 3000 gpm and pump amps were fluctuating prior to taking corrective action. Each of these events appear to have been caused by a slow decrease in RCS level in conjunction with the vortex action at the pump suction.  |
| 26   | Demand           | Design          | Operational/ Human Error   | Suction   | Piping              | RHR-P  | 1984 | Failure to Run   | Almost Complete   | On two occasions, RHR pumps cavitated due to low RCS level while draining the RCS.  |
| 27   | Demand           | Design          | Operational/ Human Error   | Suction   | Piping              | RHR-P  | 1980 | Failure to Run   | Complete          | The reactor vessel vent eductor was in service in preparation for refueling with RHR operating. A low flow alarm was received and low flow and low motor current were indicated. A second pump was started and became air-bound. Putting the vessel vent eductor system into service was the root cause of the incident.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause          | Segment | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--------------------------|---------|------------|--------|------|------------------|-------------------|--|
| 28   | Demand           | Design          | Operational/ Human Error | Suction | Piping     | RHR-P  | 1985 | Failure to Run   | Complete          | Swap over of RHR pumps resulted in both trains becoming inoperable due to air injection into the suction of the pumps. This required both pumps to be vented and required RCS level to be raised to prevent a possible recurrence of the vortex problem.   |
| 29   | Demand           | Design          | Other                    | Driver  | I&C        | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure.   |
| 30   | Demand           | Design          | Other                    | Driver  | Piping     | HCI    | 1999 | Failure to Start | Complete          | Water entered the HCI and RCI steam supply lines, rendering both pumps inoperable. Failed reactor vessel instrumentation allowed water to overflow and fill the HCI/RCI steam lines. Pumps were unavailable.   |
| 31   | Demand           | Design          | Other                    | Driver  | I&C        | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure. This is a second event two months later.  |
| 32   | Demand           | Design          | Other                    | Suction | Valve      | RHR-P  | 1984 | Failure to Run   | Complete          | Both RHR pumps were unable to operate due to the introduction of air into the RHR system. The incident occurred during the drain down of the RCS, when the level of the RCS was being monitored via a standpipe off the centerline of one of the RCS loops. The isolation valve to which the standpipe was attached became clogged sometime during the drain down and falsely indicated above centerline when in fact the level was below the RHR suction line (below centerline). |
| 33   | Demand           | Design          | Other                    | Suction | Piping     | RHR-P  | 1987 | Failure to Run   | Complete          | RHR flow was interrupted when both RHR trains became inoperable due to air bound RHR pumps. The loss of RCS inventory to the reactor coolant drain tank due to a leaking valve caused a decrease in RCS water level, vortexing in the pumps' suction line, and air entrainment in the RHR pumps.   |
| 34   | Demand           | Design          | Other                    | Suction | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | With unit drained to centerline of the nozzles, suction to both RHR pumps was lost for 36 minutes. Suction to the RHR pumps was lost because of ambiguous reactor coolant system level indication while drained to centerline of the nozzles. The actual RCS level was lower than observed.  |
| 35   | Demand           | Design          | Other                    | Suction | Piping     | RHR-P  | 1982 | Failure to Run   | Complete          | RHR Suction lost due to erroneous RCS level while draining the RCS.  |
| 36   | Demand           | Design          | Other                    | Suction | Piping     | ESW    | 1980 | Failure to Run   | Almost Complete   | Air ingress exceeded the air removal capability of the constant vent valves. A design change was implemented to remove the air compressor cooling from the service water system.   |
| 37   | Demand           | Design          | Other                    | Suction | I&C        | HPI    | 1997 | Failure to Run   | Partial           | Letdown storage tank reference leg not full, which gave erroneous indication of sufficient tank level. One HPI pump severely damaged, other pump not as damaged, and could have run. The root cause was a combination of a design weakness of a common reference leg for the Letdown storage tank level instruments and a leaking instrument fitting due to an inadequate work practice.   |
| 38   | Demand           | Design          | Other                    | Suction | Piping     | HPI    | 1982 | Failure to Start | Complete          | Hydrogen from the suction dampener got into suction piping and failed both CCPs.   |
| 39   | Demand           | Design          | Unknown                  | Suction | Piping     | RHR-P  | 1983 | Failure to Run   | Complete          | RHR pumps cavitated. Unable to repeat. Unknown cause.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment   | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|-----------|---------------------|--------|------|------------------|-------------------|---|
| 40   | Demand           | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump      | Impeller/Wear Rings | ESW    | 2000 | Failure to Start | Almost Complete   | Two of the River Water pumps tripped on overcurrent when they were attempted to be started. The trips were a result of physical contact between the impeller and the lower casing liner of the pumps. This condition was due to differential thermal expansion between the pump shaft and the pump casing as a result of an elevated seal injection water temperature. The elevated temperature was due to an abnormal configuration of the Filtered Water System (the backup seal water supply). |
| 41   | Demand           | Environmental   | External Environment   | Driver    | Motor               | ESW    | 1985 | Failure to Run   | Partial           | Two service water motors failed on demand as a result of cement dust contamination.   |
| 42   | Demand           | Environmental   | External Environment   | Driver    | I&C                 | AFW    | 1984 | Failure to Start | Complete          | Both AFW pumps failed to start. The problem was traced to two relays (1 per pump). Examination of the relays revealed open circuiting and severe degradation of the insulation.   |
| 43   | Demand           | Environmental   | External Environment   | Suction   | Piping              | HPI    | 1984 | Failure to Start | Complete          | Boron solidification in the suction and gas binding of pumps led to the failure of all three safety injection pumps. Flushing procedures inadequate.  |
| 44   | Demand           | Environmental   | Internal to Component  | Discharge | Valve               | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitol cages for these valves were clogged with shredded Asiatic clam shells.                                      |
| 45   | Demand           | Environmental   | Internal to Component  | Discharge | Valve               | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitol cages for these valves were clogged with shredded Asiatic clam shells.                                      |
| 46   | Demand           | Environmental   | Internal to Component  | Pump      | Impeller/Wear Rings | ESW    | 1994 | Failure to Run   | Partial           | Raw water pump currents stayed high after starting. The primary cause of these events was determined to be elevated sand content in the river, resulting in excessive sand accumulation around the suction area of the pumps.   |
| 47   | Demand           | Environmental   | Internal to Component  | Suction   | Strainer            | ESW    | 1980 | Failure to Run   | Partial           | Foreign material was allowed to enter the suction of the charging pump service water pumps resulting in low flow conditions.  |
| 48   | Demand           | Environmental   | Internal to Component  | Suction   | Piping              | ESW    | 1986 | Failure to Start | Partial           | RHR service water pumps failed flow testing due to blocked suctions and abnormal wear of impellers.   |
| 49   | Demand           | Maintenance     | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | I&C                 | HPI    | 1997 | Failure to Run   | Complete          | HPI pumps fail due to operation with inadequate suction head. Two pumps damaged due to operation with inadequate suction, but all three system pumps were unavailable due to the loss of the suction source. Suction source level instrumentation was the cause.  |
| 50   | Demand           | Maintenance     | External Environment   | Driver    | Breaker             | AFW    | 1990 | Failure to Run   | Partial           | AFW pumps circuit breakers degraded.  |
| 51   | Demand           | Maintenance     | Internal to Component  | Driver    | Breaker             | RHR-B  | 1987 | Failure to Start | Partial           | RHR pump breakers failed to close when operated remotely from the control room. It was found that the latch roller bearings and the cam follower bearing (internal piece parts of the breaker) were not operating correctly. This prevented the trip latch assembly from resetting and allowing the breaker to close.   |

| Item | Discovery Method | Coupling Factor | Proximate Cause          | Segment   | Piece Part    | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--------------------------|-----------|---------------|--------|------|------------------|-------------------|---|
| 52   | Demand           | Maintenance     | Internal to Component    | Driver    | I&C           | ESW    | 1991 | Failure to Start | Partial           | Two ESW pumps failed to start due to failed breakers. Inadequate maintenance.   |
| 53   | Demand           | Maintenance     | Internal to Component    | Driver    | Lubrication   | HPI    | 1984 | Failure to Run   | Partial           | Charging pump lube oil cooler fan motor trips on thermal overload. Probable cause: normal wear on motor resulting in increased friction replaced worn motor with spare. During routine inservice testing found that another charging pump lube oil cooler fan motor had a current imbalance. Probable cause: normal aging of motor insulation has resulted in a current imbalance.  |
| 54   | Demand           | Maintenance     | Internal to Component    | Pump      | Casing        | ESW    | 1998 | Failure to Start | Partial           | Two ESW pump started and ran, but would not develop sufficient pressure or flow rate. Exact cause not known for either failure, however, one pump was noted to have microbiological induced corrosion fouling on internal surfaces.   |
| 55   | Demand           | Maintenance     | Internal to Component    | Pump      | Bearing       | AFW    | 1984 | Failure to Run   | Partial           | One ESW bearing failed and pump seized; second motor bearing failed.  |
| 56   | Demand           | Maintenance     | Internal to Component    | Pump      | Packing/Seals | AFW    | 1998 | Failure to Run   | Partial           | AFW MDP and TDPs failed due to incorrect packing installed.   |
| 57   | Demand           | Maintenance     | Operational/ Human Error | Driver    | Breaker       | ESW    | 1993 | Failure to Start | Partial           | Operations personnel were attempting to swap the running service water pump with the idle service water pump. Personnel placed the control switch to start and the service water pump did not start. Breaker malfunction. Later, another service water pump failed to start because of the breaker.   |
| 58   | Demand           | Maintenance     | Operational/ Human Error | Driver    | Breaker       | ESW    | 1987 | Failure to Start | Partial           | One breaker failed to linkage alignment and second from loose relay connections. Inadequate maintenance.  |
| 59   | Demand           | Maintenance     | Operational/ Human Error | Driver    | Breaker       | ESW    | 1988 | Failure to Run   | Partial           | Service water pump high dropout over current protection devices were less than running current conditions and trip setpoints did not account for changing load conditions due to modified impellers. Three pump trips had occurred.   |
| 60   | Demand           | Maintenance     | Operational/ Human Error | Pump      | Casing        | AFW    | 1983 | Failure to Run   | Partial           | During testing, the outboard bearing temperature was high on the turbine-driven AFW pump, due to improper balance drum clearances, caused by improper maintenance. The procedure will be modified and the balance drum clearance reset. While the unit was starting up, the motor-driven AFW pump outboard bearing temperature was high. Excessive thrust bearing clearance caused the balance drum to unbalance, causing the thrust bearing to overheat. |
| 61   | Demand           | Maintenance     | Operational/ Human Error | Suction   | Piping        | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.   |
| 62   | Demand           | Maintenance     | Operational/ Human Error | Suction   | Piping        | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.   |
| 63   | Demand           | Maintenance     | Other                    | Discharge | Valve         | ESW    | 1980 | Failure to Start | Partial           | RHR service water pumps were started to put torus cooling in service. When these pumps would not deliver required discharge pressure, they were declared inoperable. The seal in an air release valve was bad, allowing a vent on the discharge line.   |
| 64   | Demand           | Maintenance     | Other                    | Driver    | Breaker       | RHR-P  | 1987 | Failure to Start | Complete          | Two LPI pumps, when given a start signal, would not start. An ongoing investigation revealed the probable root cause of the event to be poor electrical contact of the breaker auxiliary stabs for the pumps.   |



| Item | Discovery Method | Coupling Factor | Proximate Cause          | Segment | Piece Part   | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--------------------------|---------|--------------|--------|------|------------------|-------------------|--|
| 65   | Demand           | Maintenance     | Other                    | Driver  | I&C          | ESW    | 1982 | Failure to Start | Complete          | Following a reactor scram, an attempt to initiate suppression pool cooling revealed that both RHRSW loops were inoperable as neither loop's pumps could be started. Low suction header pressure lockout signals in each loop prevented starting each loop's pumps. Plugging of the sensing line to each loop's suction header pressure switch prevented both switches from sensing actual pressure, although a lack of operating fluid in one switch and an open power supply breaker to the other switch also would have prevented pumps from starting. |
| 66   | Demand           | Maintenance     | Other                    | Suction | Piping       | RHR-P  | 1981 | Failure to Run   | Complete          | Temporary coolant loop level indicator showed level slowly increasing over a period of days. The system was periodically drained to maintain 65 percent indicated level. A RHR pump lost suction on reduction of actual level. The second pump was started, and lost suction. Indication drift was due to evaporation of reference leg.  |
| 67   | Demand           | Maintenance     | Other                    | Suction | Piping       | RHR-P  | 1980 | Failure to Run   | Complete          | A complete loss of RHR flow occurred while plant operators were increasing RHR heat exchanger flow by closing down on the heat exchanger bypass valve.   |
| 68   | Demand           | Maintenance     | Other                    | Suction | Piping       | RHR-P  | 1983 | Failure to Run   | Complete          | The RHR pumps began to cavitate and eventually both pumps were stopped. The reactor vessel level gauge being used to provide an indication that the level was approaching the vessel flange level had been isolated (reactor coolant drain tank isolation valve had been closed during an attempt to reduce leakage). Additionally, procedures did not require visual monitoring of cavity level.  |
| 69   | Demand           | Maintenance     | Other                    | Suction | Piping       | RHR-P  | 1986 | Failure to Run   | Complete          | SDC pumps cavitated due to lowering RCS level. Level indication was in error.  |
| 70   | Demand           | Operational     | Operational/ Human Error | Driver  | I&C          | AFW    | 1983 | Failure to Start | Complete          | An operator incorrectly secured the diesel and steam driven AFW pumps, which prevented their restart on low SG level.  |
| 71   | Demand           | Operational     | Operational/ Human Error | Driver  | I&C          | ESW    | 1981 | Failure to Start | Partial           | Alarm circuit breaker was de-energized resulting in a loss of two RHR service water pumps.   |
| 72   | Demand           | Operational     | Operational/ Human Error | Suction | Piping       | ESW    | 1988 | Failure to Run   | Complete          | The procedure failed to adequately caution the operator to slowly fill a drained line. Rapid filling resulted in a loss of NPSH to the charging service water pumps.   |
| 73   | Demand           | Operational     | Operational/ Human Error | Suction | Piping       | RHR-P  | 1980 | Failure to Run   | Complete          | While attempting to increase RHR flow, the plant experienced a total loss of flow due to the pumps being air-bound. The pump was not vented when starting to increase flow. Operating procedures have been changed to have an operator present while changing flow in the RHR system. There have been losses of RHR flow in the past because the pumps were air-bound and methods are being investigated to improve the system design.   |
| 74   | Demand           | Operational     | Operational/ Human Error | Suction | Piping       | ESW    | 1986 | Failure to Run   | Complete          | Failure to properly vent and fill a newly installed pipe introduced air into the charging pump service water system.   |
| 75   | Demand           | Operational     | Operational/ Human Error | Suction | Booster Pump | ESW    | 1980 | Failure to Start | Partial           | The service water RHR booster pump was de-energized during maintenance. The attempt to start service water pumps failed due to low suction pressure.   |
| 76   | Demand           | Operational     | Operational/ Human Error | Suction | Piping       | RHR-P  | 1984 | Failure to Run   | Complete          | The control room operators started a second residual heat removal pump in preparation for removing the operating RHR pump from service. With both pumps running, flow became excessive for the half-loop condition causing cavitation and air binding of both pumps. To prevent recurrence the procedure which controls the operation of the RHR pumps has been changed to include specific instructions to stop the operating pump prior to starting the second pump while at half-loop.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|---------|---------------------|--------|------|------------------|-------------------|---|
| 77   | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | I&C                 | AFW    | 1989 | Failure to Start | Complete          | Both motor driven auxiliary feedwater pumps failed to start when the operator tried to start them manually. While preparing a design change, the designer failed to review all the unit specific documentation associated with the motor-driven AFW pump wiring and made the erroneous assumption that both units switchgear compartment internal wiring was identical. In fact, the wiring for each unit was different. Consequently, when the design change was installed, it was installed in accordance with the erroneous design. The wiring discrepancy was corrected and the motor-driven AFW pumps were tested and returned to service.   |
| 78   | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Breaker             | ESW    | 1996 | Failure to Start | Partial           | Two RHRSW pumps fail to start due to breaker failures. Wrong contacts were installed. Design called for contacts to have a minimum current interrupt rating of 6 amps; contacts installed (that subsequently failed) had current interrupt rating of only 2.2 amps.   |
| 79   | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Motor               | ESW    | 1987 | Failure to Start | Partial           | ESW pump motors tripped on overcurrent. The overcurrent trip was due to a ground and a short on the pump motor.   |
| 80   | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Impeller/Wear Rings | ESW    | 1996 | Failure to Run   | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 81   | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Impeller/Wear Rings | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps drawing excessive current. Carbon steel snap rings corroded allowing impeller to come in contact with casing. The third pump, although not exhibiting abnormal current, had similar corrosion   |
| 82   | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Piping              | ESW    | 1984 | Failure to Start | Partial           | Both RHR service water pumps tripped as a result of inadequate venting of suction header resulting from poor orientation of the vent line.  |
| 83   | Demand           | Quality         | Internal to Component  | Pump    | Impeller/Wear Rings | AFW    | 1988 | Failure to Run   | Partial           | Following a plant trip, it was discovered that the auxiliary feedwater pumps had internal damage. Some channel ring vanes had chips missing, and several parts were found in the SG auxiliary feedwater piping.   |
| 84   | Inspection       | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver  | Lubrication         | HPI    | 2000 | Failure to Run   | Partial           | CVC makeup oil pump motor too small for certain accidents.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment   | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--|-----------|-------------|--------|------|------------------|-------------------|--|
| 85   | Inspection       | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | I&C         | AFW    | 1994 | Failure to Start | Partial           | Single failure would prevent auto initiation of AFW. Circuit design did not provide separation required by standards and code. The single failure identified was a short circuit across two conductors of the actuation relays associated with the initiation logic matrix.  |
| 86   | Inspection       | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Driver    | Supports    | RHR-B  | 1986 | Failure to Start | Partial           | RHR motor internal supports were cracked due to stress and vibration. Design improvements were made.   |
| 87   | Inspection       | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Piping      | HPI    | 1990 | Failure to Start | Partial           | A quantity of gas was found in the centrifugal charging pump suction header that exceeded the maximum allowed gas volume. It was subsequently determined that hydrogen gas had been coming out of solution on both units and accumulating in the suction piping as a probable result of gas stripping by the CCP miniflow orifices. In addition, entrainment of hydrogen bubbles from the volume control tank to the CCP suction pipe may be a contributor as well.                  |
| 88   | Inspection       | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Piping      | HPI    | 1988 | Failure to Run   | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the suction piping.  |
| 89   | Inspection       | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Piping      | HPI    | 1988 | Failure to Start | Partial           | It was determined that various pipes of the safety injection system and chemical volume and control system collected or trapped gas which might affect the functions of these systems. There was a concern that the gas pockets may adversely effect pump operation. Voids were detected in some of the high head SI pump piping.  |
| 90   | Inspection       | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction   | Piping      | HPI    | 1991 | Failure to Start | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the alternate boration line and the gravity feed line from the boric acid storage tank.  |
| 91   | Inspection       | Design          | External Environment   | Discharge | Piping      | HPI    | 1994 | Failure to Run   | Partial           | Due to a leaking socket weld in the common recirculation line, all three SI pumps were declared inoperable. The underlying cause of the leak was a crack in the socket weld in the common recirculation line, caused by pipe displacement from air entrainment and pump misalignment.  |
| 92   | Inspection       | Design          | External Environment   | Pump      | Bearing     | HPI    | 1991 | Failure to Run   | Almost Complete   | Charging/safety pumps beyond operational limits. Damage was found to the thrust bearings. Air was introduced into this train of chilled water during modifications and testing being performed on the system. This air became trapped in high points of either, or both of, the supply and return chilled water lines to the charging pump. At the reduced flow rate, sufficient cooling was not available and oil temperature increased to the point where bearing damage occurred. |
| 93   | Inspection       | Design          | Internal to Component  | Driver    | I&C         | ESW    | 1982 | Failure to Start | Partial           | Open circuit breaker resulted in loss of two RHR service water pumps.  |
| 94   | Inspection       | Design          | Internal to Component  | Pump      | Lubrication | HPI    | 1981 | Failure to Run   | Partial           | Corrosion of HPI pump cooler heads. Improper material led to corrosion   |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|---------|---------------------|--------|------|------------------|-------------------|---|
| 95   | Inspection       | Design          | Operational/ Human Error                                   | Driver  | Breaker             | ESW    | 1984 | Failure to Start | Partial           | During an attempt to perform preventive maintenance for unit one's RHR service water pumps, plant personnel mistakenly disconnected the motor leads for unit two's RHR service water pump.  |
| 96   | Inspection       | Design          | Other  | Driver  | I&C                 | AFW    | 1983 | Failure to Start | Partial           | Both AFW pumps had to be rendered inoperable to allow repairs to actuation circuitry.   |
| 97   | Inspection       | Environmental   | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | Piping              | HPI    | 2000 | Failure to Run   | Partial           | Microbiologically induced corrosion leak on service water lines to two charging/HPI pump lube oil coolers.  |
| 98   | Inspection       | Environmental   | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump    | Lubrication         | HPI    | 1995 | Failure to Run   | Partial           | High lube oil temperatures were observed during HPI pump operation. Zinc particles from anode were discovered plugging the lube oil coolers. Accelerated corrosion was attributed to a corrosion inhibitor that was added to the system, which chemically interacted with the zinc. |
| 99   | Inspection       | Environmental   | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Strainer            | ESW    | 2000 | Failure to Run   | Partial           | RHRSW Pumps Failed to Develop flow/pressure. Debris in intake structure. Requires modifications to the traveling Water Screen.  |
| 100  | Inspection       | Environmental   | External Environment                                       | Pump    | Coupling            | ESW    | 1993 | Failure to Run   | Partial           | Entrained debris caused ESW pump shaft coupling to fail. Plant equipment did not prevent this debris from entering pump.  |
| 101  | Inspection       | Environmental   | External Environment                                       | Pump    | Packing/Seals       | RHR-P  | 1985 | Failure to Start | Complete          | Following a trip, water was found spraying from both low head safety injection pump wedge control rod seals. Both pumps were declared inoperable. Postulated failure on the seals was from a minor flow induced pressure transient.   |
| 102  | Inspection       | Environmental   | Internal to Component                                      | Pump    | Packing/Seals       | ESW    | 1994 | Failure to Run   | Partial           | Backup seal water regulators did not provide required flow during testing on two pumps. The third pump lost seal flow while operating. The cause was attributed to plugged lines.   |
| 103  | Inspection       | Environmental   | Internal to Component                                      | Pump    | Lubrication         | HPI    | 1983 | Failure to Run   | Partial           | Oysters and miscellaneous mollusks plugged HPI oil coolers. Two pumps were required to be shutdown due to rising lubricating oil temperatures.  |
| 104  | Inspection       | Environmental   | Internal to Component                                      | Pump    | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Marine growth in suction.  |
| 105  | Inspection       | Environmental   | Internal to Component                                      | Suction | Strainer            | ESW    | 1984 | Failure to Run   | Partial           | Two RHR service water pumps had blown seals and sparks and smoke between the bearing housing and shaft. A piece of hard rubber valve liner was found in the pumps.  |
| 106  | Inspection       | Environmental   | Other  | Driver  | Motor               | ESW    | 1981 | Failure to Run   | Partial           | The float guide failed in a RHRSW pump air valve and caused the valve to fail open and flood pump room.   |
| 107  | Inspection       | Environmental   | Other  | Driver  | Motor               | AFW    | 1990 | Failure to Start | Partial           | Both motor driven AFW pumps were sprayed when a service water pipe developed a through wall leak.   |
| 108  | Inspection       | Maintenance     | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump    | Packing/Seals       | ESW    | 1997 | Failure to Run   | Partial           | Both ESW pumps leaking greater than 4 gpm because of inappropriate material for packing and sleeve (nitronic 60).   |

| Item | Discovery Method | Coupling Factor | Proximate Cause       | Segment   | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|-----------------------|-----------|---------------------|--------|------|------------------|-------------------|---|
| 109  | Inspection       | Maintenance     | Internal to Component | Discharge | Check Valve         | AFW    | 1990 | Failure to Start | Almost Complete   | Leakage past AFW check valves caused AFW pumps to become steam bound. Closed motor operated valve in line. Scheduled check valves for replacement next outage.  |
| 110  | Inspection       | Maintenance     | Internal to Component | Driver    | Breaker             | ESW    | 1996 | Failure to Start | Partial           | ESW pump breakers fail due to misalignment of the breaker mechanism and internals developed over the years of operation.  |
| 111  | Inspection       | Maintenance     | Internal to Component | Driver    | Bearing             | ESW    | 1985 | Failure to Run   | Partial           | One service water pump motor upper bearing oil reservoir leaking from cover plate. Another service water pump motor upper oil cooler oil reservoir leaking.   |
| 112  | Inspection       | Maintenance     | Internal to Component | Driver    | Packing/Seals       | HPI    | 1988 | Failure to Run   | Almost Complete   | Smoke was discovered coming from the speed increaser unit for a centrifugal charging pump. Investigation found the two gland seal retaining bolts inside the speed increaser lube oil pump backed out allowing the gland seal to loosen. The gland seal being loosened, caused reduced oil flow to the speed increaser internals and ultimate damage. Other CCPs were inspected, and the same gland seal bolts as on the first pump were found loosened. The cause of the bolts backing out was determined to be lack of a periodic adjustment of the gland seal bolts. |
| 113  | Inspection       | Maintenance     | Internal to Component | Driver    | Bearing             | ESW    | 1981 | Failure to Run   | Partial           | ESW motor to pump alignment problems. Bearings worn out.  |
| 114  | Inspection       | Maintenance     | Internal to Component | Pump      | Bearing             | ESW    | 1987 | Failure to Run   | Partial           | Service water pumps had high shaft vibration. The excessive vibrations caused by worn bearings and shaft sleeves.   |
| 115  | Inspection       | Maintenance     | Internal to Component | Pump      | Casing              | ESW    | 1986 | Failure to Run   | Partial           | Cracked seal water and vent lines.  |
| 116  | Inspection       | Maintenance     | Internal to Component | Pump      | Packing/Seals       | ESW    | 1989 | Failure to Run   | Partial           | ESW pump excessive packing leakage.   |
| 117  | Inspection       | Maintenance     | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. The cause of the failure is suspected to be binding.   |
| 118  | Inspection       | Maintenance     | Internal to Component | Pump      | Packing/Seals       | SLC    | 1988 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing replaced.  |
| 119  | Inspection       | Maintenance     | Internal to Component | Pump      | Bearing             | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. Loose fittings and lack of thread sealant.   |
| 120  | Inspection       | Maintenance     | Internal to Component | Pump      | Lubrication         | RHR-B  | 1990 | Failure to Run   | Partial           | Both pump motor oil coolers were leaking due to aging of components. The first case involved through wall corrosion and the pump was immediately removed from service. The second case was a packing leak.  |
| 121  | Inspection       | Maintenance     | Internal to Component | Pump      | Casing              | ESW    | 1988 | Failure to Run   | Partial           | RHR service water pumps. Pump diffuser eroded on first pump and a through wall casing leak developed on the second.   |
| 122  | Inspection       | Maintenance     | Internal to Component | Pump      | Packing/Seals       | SLC    | 1987 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing adjusted.  |
| 123  | Inspection       | Maintenance     | Internal to Component | Pump      | Plunger/Cylinder    | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pump seal was leaking excessively. The cause of this failure was normal wear of the plungers, packing, and head gaskets for the plungers (piece parts of the pump).  |
| 124  | Inspection       | Maintenance     | Internal to Component | Pump      | Packing/Seals       | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking profusely at the packing. The failure of the packing was attributed to normal wear.  |
| 125  | Inspection       | Maintenance     | Internal to Component | Pump      | Packing             | AFW    | 1986 | Failure to Run   | Partial           | The packing was worn on both the motor-driven and one turbine-driven aux. feedwater pump, causing high temperature on one packing gland, and excessive leaking on the other pump.   |

| Item | Discovery Method | Coupling Factor | Proximate Cause          | Segment   | Piece Part    | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--------------------------|-----------|---------------|--------|------|------------------|-------------------|--|
| 126  | Inspection       | Maintenance     | Internal to Component    | Pump      | Packing/Seals | ESW    | 1986 | Failure to Run   | Partial           | Excessive packing leakage. Both events occurred after previous maintenance had been performed for the same problems.   |
| 127  | Inspection       | Maintenance     | Internal to Component    | Pump      | Packing/Seals | AFW    | 1990 | Failure to Run   | Partial           | Both motor-driven aux. feedwater pumps had excessive packing leaks, due to worn packing.   |
| 128  | Inspection       | Maintenance     | Operational/ Human Error | Driver    | I&C           | AFW    | 1990 | Failure to Start | Complete          | During testing one AFW pump was tested and other was tested without returning first to auto. Both pumps were unavailable at the same time. The procedure was the cause.  |
| 129  | Inspection       | Maintenance     | Operational/ Human Error | Driver    | Breaker       | RHR-P  | 1981 | Failure to Start | Complete          | All RHR pumps de-energized to replace RHR Relief valve. T.S. allows this condition for 1 hour. Operated in the mode in excess of 5 hours.  |
| 130  | Inspection       | Maintenance     | Operational/ Human Error | Driver    | I&C           | RHR-P  | 1992 | Failure to Start | Complete          | Both trains of RHR were rendered inoperable for two minutes, while performing an operational readiness test surveillance procedure. The surveillance procedure required that the one RHR train pump be placed in pull to lock and the other train heat exchanger flow control valve throttled to 30-40% open. The procedure directed the operators to perform operations that resulted in both trains of RHR being inoperable  |
| 131  | Inspection       | Maintenance     | Operational/ Human Error | Driver    | Bearing       | RHR-P  | 1988 | Failure to Run   | Partial           | Residual heat removal pump motor upper bearing housings were observed to be leaking oil. The cause of the failure was attributed to a lack of sealant being applied and gasket installed after the last maintenance was performed on the motor bearing housing.  |
| 132  | Inspection       | Maintenance     | Operational/ Human Error | Suction   | Valve         | SLC    | 1991 | Failure to Start | Partial           | SLC pumps were potentially inoperable during part of test due to valve lineup.   |
| 133  | Inspection       | Operational     | External Environment     | Driver    | I&C           | HPI    | 1990 | Failure to Run   | Complete          | It was determined that the common minimum flow path return line for the safety injection pumps to the refueling water storage tank was frozen. Previous actions to investigate problems with the freeze protection system were unsuccessful in preventing development of this condition. The two HPI pumps were declared inoperable with this return line frozen. A faulty ambient temperature switch for the RWST heat trace system prevented the heat trace from activating and was subsequently replaced. In addition, administrative controls did not sufficiently recognize the safety significance of flow through this line and the need to ensure flow capability. |
| 134  | Inspection       | Operational     | Operational/ Human Error | Discharge | Valve         | HPI    | 1987 | Failure to Start | Almost Complete   | While attempting to fill the safety injection accumulators, it was discovered that two of three SI pumps had been isolated from the high head injection flowpath.  |
| 135  | Inspection       | Operational     | Operational/ Human Error | Discharge | Valve         | HPI    | 1993 | Failure to Run   | Partial           | One AFW pump failed due to incorrect procedure which allowed pump to be run without flow, other AFW pump was allowed to run past max flow rate. It is unclear whether these mistakes were due to inadequate procedures or staff errors, but it was assumed to be a failure to follow procedure.  |
| 136  | Inspection       | Operational     | Operational/ Human Error | Discharge | Valve         | AFW    | 1994 | Failure to Start | Complete          | Following a trip, the AFW Pumps were secured and the discharge flow control valves for the Motor Driven Pumps were closed. Later, an operator discovered during a routine Control Board walkdown that the valves were closed. Subsequent investigation revealed the AFW system had not been placed in standby readiness per the operating procedure after the system was secured.  |
| 137  | Inspection       | Operational     | Operational/ Human Error | Driver    | I&C           | HPI    | 1990 | Failure to Start | Partial           | Both safety injection pumps were in the pull-to-lock position. With the switches in pull-to-lock, the pumps would not have automatically started upon receipt of an initiating signal. This event was caused by cognitive personnel error by a utility licensed operator in failure to follow an approved procedure.   |
| 138  | Inspection       | Operational     | Operational/ Human Error | Driver    | Breaker       | CSS    | 1991 | Failure to Start | Complete          | CSR control power de-energized prior to mode change. Technical Specification violation. Inadequate procedure review.   |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|---------|-------------|--------|------|------------------|-------------------|---|
| 139  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | Breaker     | HPI    | 1988 | Failure to Start | Complete          | HPI pumps not restored before mode change due to procedural inadequacy.   |
| 140  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | Breaker     | HPI    | 1989 | Failure to Start | Partial           | HPI Pump B not retested, then HPI Pump A removed from service.  |
| 141  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | I&C         | RHR-P  | 1995 | Failure to Start | Complete          | The switches for the containment spray and recirculation pumps were in a trip pullout when the Technical Specifications and plant procedures required the pumps to be operable.   |
| 142  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | Breaker     | ESW    | 1981 | Failure to Start | Almost Complete   | Control breakers for two ESW pumps were open due to inadvertent operator action.  |
| 143  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | I&C         | HPI    | 1992 | Failure to Start | Almost Complete   | Two charging pumps and one charging pump service water pump were removed from service simultaneously which is a condition not allowed by technical specifications.  |
| 144  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | Breaker     | HPI    | 1990 | Failure to Start | Complete          | By opening incorrect breaker, HPI pump tripped while others were unavailable.   |
| 145  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | I&C         | HPI    | 1988 | Failure to Start | Complete          | With alternate CCP pump out-of-service, the remaining operable pump was erroneously placed in pull-to-lock.   |
| 146  | Inspection       | Operational     | Operational/ Human Error                                   | Driver  | Breaker     | HPI    | 1982 | Failure to Start | Complete          | During the draining of the reactor coolant system, both centrifugal charging pumps were rendered inoperable. The initial conditions in the draining procedure contained a confusing statement, which led to an erroneous assumption that both CCP breakers had to be racked out and tagged.   |
| 147  | Inspection       | Operational     | Operational/ Human Error                                   | Pump    | Lubrication | HPI    | 1983 | Failure to Start | Complete          | A routine preventive maintenance (oil change) was mistakenly performed on the north charging pump instead of the south as scheduled. Since the south pump was previously cleared for this oil change, and the test pump was valved out, none of these three pumps were in service as required by tech specs for the approximately 20 minutes it took to change the oil in the north pump.   |
| 148  | Inspection       | Operational     | Other  | Pump    | Bearing     | ESW    | 1991 | Failure to Run   | Partial           | Lube oil cooling water isolated during a test. Pumps continued to run with no cooling.  |
| 149  | Inspection       | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump    | Casing      | AFW    | 1983 | Failure to Run   | Partial           | Two AFW pumps thrust tolerance was out of specification. These events were caused by improperly installed balancing drum parts. One turbine driven and one motor driven pump was involved.  |
| 150  | Inspection       | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump    | Casing      | HPI    | 1987 | Failure to Run   | Partial           | During inspection of a centrifugal charging pump, a portion of the stainless steel cladding on the inside surface of the pump casing exhibited corrosion. Corrosion of the pump casing was through the stainless steel cladding into the carbon steel base material. Inspection of the other CCP revealed similar corrosion. The cause of this event was a manufacturing deficiency. Corrosion observed at the pump casing discharge nozzle was attributed to a cladding breakthrough during final machining. Corrosion observed at the pump casing inlet end was attributed to either over-machining of the cladding or inadequate overlay of two adjacent weld beads. |
| 151  | Inspection       | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Piping      | HPI    | 1988 | Failure to Run   | Partial           | Vortex breakers had not been installed in the containment emergency sumps. Vortex breakers are required to be installed in the containment emergency sumps to prevent the formation of vortices which could adversely affect performance of safety injection pumps during the safety injection and containment spray systems were declared inoperable.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|---------|------------|--------|------|------------------|-------------------|---|
| 152  | Inspection       | Quality         | Operational/ Human Error                                   | Driver  | Breaker    | ESW    | 1992 | Failure to Start | Partial           | The fit between an ESW pump breaker primary disconnects and the associated breaker cubicle stabs was inadequate. The poor fit between the disconnects and the stabs led to arcing in the breaker cubicle when the pump was started, resulting in a fire. Shortly after identifying the cause of the fire, the remaining ESW breakers, which had recently been replaced along with the failed breaker, as part of a design modification package, were found to be inadequate also.   |
| 153  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | I&C        | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |
| 154  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | I&C        | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |
| 155  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 156  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 157  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Tank       | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 158  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 159  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Tank       | ESW    | 1985 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |



| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part    | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--|---------|---------------|--------|------|------------------|-------------------|--|
| 160  | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Tank          | ESW    | 1990 | Failure to Run   | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.  |
| 161  | Maintenance      | Environmental   | External Environment                                       | Driver  | Motor         | ESW    | 1987 | Failure to Start | Partial           | During an extended service water bay flooding incident, one ESW pump was found grounded by testing, later two more pumps were found to be failed also.   |
| 162  | Maintenance      | Environmental   | Internal to Component                                      | Pump    | Packing/Seals | ESW    | 1985 | Failure to Run   | Partial           | First pump developed seal leak due to sand. Second pump had high bearing temperatures due to trash clogging cooling water lines.   |
| 163  | Maintenance      | Environmental   | Internal to Component                                      | Pump    | Lubrication   | HPI    | 1986 | Failure to Run   | Almost Complete   | Clams/sludge fouling of lube oil cooler caused high temperature alarms on two HPI pumps.   |
| 164  | Maintenance      | Environmental   | Internal to Component                                      | Pump    | Lubrication   | HPI    | 1991 | Failure to Run   | Partial           | HPI pump lube oil cooler leaks. Degraded tubes.  |
| 165  | Maintenance      | Environmental   | Internal to Component                                      | Pump    | Lubrication   | HPI    | 1980 | Failure to Run   | Partial           | HPI pump lube oil cooler with tube leak allowed water into oil reservoir.  |
| 166  | Maintenance      | Maintenance     | Internal to Component                                      | Driver  | Breaker       | ESW    | 1985 | Failure to Start | Partial           | Two raw water pump breaker main wipes were out of adjustment.  |
| 167  | Maintenance      | Maintenance     | Internal to Component                                      | Driver  | Breaker       | HPI    | 1991 | Failure to Start | Partial           | HPI pump breakers failed due to a broken pawl, and a broken closing coil.  |
| 168  | Maintenance      | Maintenance     | Internal to Component                                      | Driver  | Breaker       | SLC    | 1999 | Failure to Start | Partial           | SLC Pump Breakers Fail to pickup on degraded voltage test  |
| 169  | Maintenance      | Maintenance     | Internal to Component                                      | Driver  | Breaker       | AFW    | 1992 | Failure to Start | Partial           | With the unit in a refueling outage, following repairs to a motor driven auxiliary feedwater pump local/remote switch of the circuit breaker, personnel found that the switch contacts would not close. This failure rendered one of three auxiliary feedwater pumps inoperable. The cause of the failure appears to be due to dirty/corroded contacts on the switch.  |
| 170  | Maintenance      | Maintenance     | Internal to Component                                      | Pump    | Bearing       | ESW    | 1985 | Failure to Run   | Partial           | High ESW pump vibration was caused by wearing of the upper bearings.   |
| 171  | Maintenance      | Maintenance     | Operational/ Human Error                                   | Driver  | Breaker       | RHR-B  | 1990 | Failure to Start | Partial           | RHR pump breaker overcurrent trips out of calibration.   |
| 172  | Maintenance      | Maintenance     | Operational/ Human Error                                   | Driver  | Breaker       | RHR-B  | 1991 | Failure to Start | Partial           | While performing preventive maintenance calibration check on the protective relays for a residual heat removal pump motor 4kv breaker, it was found that all overcurrent relays for two pumps were out of calibration  |
| 173  | Maintenance      | Maintenance     | Operational/ Human Error                                   | Pump    | Lubrication   | HPI    | 1991 | Failure to Run   | Partial           | Following an overhaul of the HPI pumps. Too much oil flow led to excessive oil leakage, which would have failed HPI pumps before end of mission.   |
| 174  | Maintenance      | Maintenance     | Operational/ Human Error                                   | Suction | Piping        | RHR-P  | 1982 | Failure to Run   | Complete          | Shutdown cooling was lost due to nitrogen intrusion because of backflushing a filter in the purification system.   |
| 175  | Maintenance      | Maintenance     | Operational/ Human Error                                   | Suction | Strainer      | HPI    | 1985 | Failure to Run   | Partial           | Strainers found still installed in the suction piping of the high-pressure injection pumps was a condition not considered in the operating design. The strainers were found during maintenance to repair a slight flange leak. The strainers had been placed in the suction piping during construction and were to be in place during system flushing to prevent any debris from reaching the pumps. However, the strainers should have been removed after system flushing prior to functional testing |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part  | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|---------|-------------|--------|------|------------------|-------------------|---|
| 176  | Maintenance      | Maintenance     | Other  | Driver  | Breaker     | ESW    | 1982 | Failure to Start | Partial           | ESW pump circuit breakers found damaged. Defective arc chute and cracked secondary coupler.   |
| 177  | Maintenance      | Maintenance     | Other  | Driver  | Breaker     | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker failures, broken screw, no lubrication, and a bent track   |
| 178  | Maintenance      | Operational     | Operational/ Human Error                                   | Pump    | Lubrication | ESW    | 1993 | Failure to Run   | Partial           | Low pressure RHR bearing oil level not maintained high enough when new smaller sightglass installed. Second event the sightglass was broken when adding oil.  |
| 179  | Maintenance      | Operational     | Operational/ Human Error                                   | Suction | Strainer    | ESW    | 1986 | Failure to Run   | Complete          | A service water strainer was placed in service without being vented resulting in air binding system and loss of charging pump service water pumps.  |
| 180  | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | Breaker     | HPI    | 1980 | Failure to Start | Partial           | Upon testing the safety injection pumps it was found that the 6900-v breakers would lock-out preventing pump start if they were given a close signal for >0.32 seconds when a trip condition existed. There is no indication to operations when this locked-out condition exists. The breaker appears to be available for service when it actually is not. The only means of clearing the condition is to remove and reinstall the fuses at the breaker or manually change the state of the relays. |
| 181  | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | I&C         | AFW    | 1981 | Failure to Start | Almost Complete   | Two low suction pressure trips for the AFW pumps were mis-calibrated, which prevented the pumps from starting.  |
| 182  | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver  | I&C         | AFW    | 1992 | Failure to Start | Complete          | A modification design error (in 1983-1984) removed a start permissive interlock contact. At cold shutdown this de-energized the auxiliary lube oil pump, consequently, when one AFW pump was started it ran for 2.5 seconds and tripped on low oil pressure. Further investigation showed that both units AFW pumps would be affected in the same way. The design error combined with insufficient post modification testing led to this CCF event.   |
| 183  | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump    | Shaft       | AFW    | 1988 | Failure to Run   | Partial           | The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 184  | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump    | Coupling    | ESW    | 1994 | Failure to Start | Partial           | Pump produced no flow when started. A shaft coupling failed. Material was determined to be brittle and have low impact properties. The coupling was replaced on all pumps with a type of material more suitable for this application.   |
| 185  | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump    | Shaft       | AFW    | 1988 | Failure to Run   | Almost Complete   | An auxiliary feedwater pump failed its performance test. Subsequent inspection of the pump internals revealed significant damage, including a split in the center shaft sleeve. The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 186  | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | Suction | Piping      | AFW    | 1999 | Failure to Run   | Partial           | All AFW trains declared inoperable due to inadequate suction flow capability from the nuclear service water alternate source. Inadequate flow caused by corroded piping. Piping is undersized so there is little margin for piping degradation. Since this is 1 of 4 suction sources, the safety significance is limited.   |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment | Piece Part | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--|---------|------------|--------|------|------------------|-------------------|--|
| 187  | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Valve      | ESW    | 1983 | Failure to Start | Partial           | Low discharge pressure was caused by insufficient suction pressure. Service water flow to parallel components was adjusted.  |
| 188  | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Tank       | SLC    | 1991 | Failure to Run   | Complete          | During the performance of a special test on the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.  |
| 189  | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Tank       | SLC    | 1991 | Failure to Run   | Complete          | During the performance of a special test on Unit 1 to determine the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.  |
| 190  | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Suction | Tank       | ESW    | 1986 | Failure to Run   | Complete          | Loss of prime in the condenser circulating water siphon flow system caused loss of low pressure service water pumps. Pumps lost suction during a test due to poor design.  |
| 191  | Test             | Design          | Operational/ Human Error   | Driver  | Breaker    | AFW    | 1985 | Failure to Start | Complete          | Both AFW pumps failed to start when tested, due to the circuit breakers not being racked in properly.  |
| 192  | Test             | Design          | Other  | Driver  | I&C        | RHR-B  | 1982 | Failure to Start | Partial           | A functional test revealed a sliding link in control room panel open. Further investigation revealed a total of four links open. These links, left open, negated all autostart capability of 2 of 4 RHR pumps. It could not be determined why these four links were open.  |
| 193  | Test             | Design          | Other  | Driver  | I&C        | ESW    | 1992 | Failure to Start | Partial           | Valve position contacts prevented ESW pump circuit breakers from closing. Poor design resulted in water intrusion in the valve limit switch box.   |
| 194  | Test             | Design          | Other  | Driver  | Breaker    | SLC    | 1986 | Failure to Start | Complete          | During a test, both Squib Valve Detonators shorted after firing and the Control Power Transformer fuse blew causing the pump motor trip. This was caused by improper fuse coordination between the Control Power Transformer fuse and the Squib Valve Detonator fuses. The redundant system's Squib Valve was also fired during this test, without running the associated pump, and one of the Squib Valve Detonators shorted after firing. The same fuse coordination problem existed for both systems.   |
| 195  | Test             | Design          | Other  | Suction | I&C        | AFW    | 1985 | Failure to Run   | Almost Complete   | Testing of the turbine driven AFW pump resulted in a low suction trip of the motor driven pump. The turbine driven pump had a faulty governor. It was during the post maintenance test of turbine driven pump that speed oscillations occurred causing pressure oscillations in the suction of the motor driven pump that was in service. Foreign material in the suction gauge protectors resulted in the pressure sensors sensing only the low pressures and not the high pressures of the oscillations, so the motor driven pump tripped on low pressure. |
| 196  | Test             | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Pump    | Coupling   | ESW    | 1987 | Failure to Start | Partial           | Test showed two ESW pumps failed. Pump shafts were corroded and found to be made of incorrect material.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause       | Segment   | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|-----------------------|-----------|---------------------|--------|------|------------------|-------------------|---|
| 197  | Test             | Environmental   | External Environment  | Discharge | Recirc              | HPI    | 1992 | Failure to Run   | Almost Complete   | Safety Injection pumps were declared inoperable due to an observed declining trend in the pump's recirculation flow. The cause of the Safety Injection pump reduced recirculation flow is attributed to foreign material blockage within the associated minimum flow recirculation line flow orifice. |
| 198  | Test             | Environmental   | External Environment  | Driver    | Bearing             | RHR-B  | 1991 | Failure to Run   | Partial           | Two LCI pumps were declared inoperable due to high motor vibration.   |
| 199  | Test             | Environmental   | Internal to Component | Discharge | Recirc              | HPI    | 1991 | Failure to Run   | Partial           | Something in HPI pump recirculation line was restricting flow. The piece later dislodged and no identification was made. Both SI pumps had inadequate recirculation flow.   |
| 200  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1982 | Failure to Run   | Partial           | Low ESW pump head values were caused excessive wear of pump impeller due to foreign material in the service water.  |
| 201  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1991 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted.  |
| 202  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.   |
| 203  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1995 | Failure to Start | Partial           | Marine growth caused low flow and speed condition for two service water pumps   |
| 204  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1993 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head values. The low pump heads were caused by excessive wear of pump impeller due to sand in the service water.  |
| 205  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. A rag was found in one impeller and a plastic bottle in the other.  |
| 206  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1982 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.  |
| 207  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1995 | Failure to Start | Partial           | Pumps failed performance test. Sand in water eroded pump internals. Pump lift was adjusted.   |
| 208  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | HPI    | 1984 | Failure to Run   | Almost Complete   | One HPI pump seized, the second would have seized if operated.  |
| 209  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1994 | Failure to Start | Partial           | Degraded performance identified during testing. Sand in water was causing accelerated wear of the pump internals. Lift was adjusted for three pumps and one pump internals were replaced.   |
| 210  | Test             | Environmental   | Internal to Component | Pump      | Bearing             | ESW    | 1992 | Failure to Run   | Partial           | Abrasive particles present in ocean water produced accelerated wear of shaft bearing journals.  |
| 211  | Test             | Environmental   | Internal to Component | Pump      | Impeller/Wear Rings | ESW    | 1990 | Failure to Run   | Partial           | ESW pump impeller lift out of adjustment.   |
| 212  | Test             | Environmental   | Internal to Component | Suction   | Strainer            | ESW    | 1990 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by suction blockage due to foreign material in the service water.   |
| 213  | Test             | Environmental   | Internal to Component | Suction   | Piping              | ESW    | 1990 | Failure to Start | Partial           | ESW pumps failed flow testing. Foreign material blocked the suction.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment   | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--|-----------|---------------------|--------|------|------------------|-------------------|--|
| 214  | Test             | Environmental   | Internal to Component                                      | Suction   | Strainer            | ESW    | 1982 | Failure to Run   | Partial           | Failures occurred on residual heat removal service water pumps. The pumps failed to meet flow and pressure requirements. Failure was due to debris lodging in pump impellers. Source of debris was maintenance activities, broken traveling water screens, and the inadvertent opening of a RHR minimum flow line which washed materials into suction pit. |
| 215  | Test             | Maintenance     | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump      | Casing              | ESW    | 1997 | Failure to Run   | Almost Complete   | Both ESW pumps failed due to installation of wrong material for pump casing flanges by vendor during pump overhaul. The vendor overhauled the pumps without changing material. The plant returned the pumps to the warehouse also without verifying material.  |
| 216  | Test             | Maintenance     | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump      | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.   |
| 217  | Test             | Maintenance     | Internal to Component                                      | Discharge | Valve               | HPI    | 1984 | Failure to Start | Partial           | CCP pump low flow rates due to inaccuracies in positioning the throttle valves.  |
| 218  | Test             | Maintenance     | Internal to Component                                      | Driver    | Breaker             | RHR-B  | 1997 | Failure to Start | Partial           | Breaker latch check switch failed on both pumps. Lack of lubrication.  |
| 219  | Test             | Maintenance     | Internal to Component                                      | Driver    | Breaker             | AFW    | 1997 | Failure to Start | Almost Complete   | The circuit breakers associated with the AFW Pumps failed to close as required. The root cause of the failure was the binding in the operating mechanism. The plunger apparently did not always complete its upward movement to close and latch the breaker, due to accumulated dirt and lubricants.   |
| 220  | Test             | Maintenance     | Internal to Component                                      | Driver    | Breaker             | ESW    | 1998 | Failure to Start | Partial           | Service water pumps fail to start due to circuit breaker failures. Pump breakers failed to close due to failures of the charging spring/motor and closing spring motor.  |
| 221  | Test             | Maintenance     | Internal to Component                                      | Driver    | Breaker             | RHR-B  | 1986 | Failure to Start | Partial           | RHR pump circuit breakers failed during a start for testing. Bend switch and binding mechanism. Attributed to inadequate maintenance.  |
| 222  | Test             | Maintenance     | Internal to Component                                      | Driver    | Breaker             | ESW    | 1998 | Failure to Start | Partial           | Two RHR service water pump breakers would not close due to dirty contacts in breakers.   |
| 223  | Test             | Maintenance     | Internal to Component                                      | Driver    | Bearing             | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps exhibited vibration. Attributed to normal wear.  |
| 224  | Test             | Maintenance     | Internal to Component                                      | Pump      | Impeller/Wear Rings | ESW    | 1998 | Failure to Start | Partial           | Two ESW pumps failed to develop adequate flow/pressure - pumps degraded.   |
| 225  | Test             | Maintenance     | Internal to Component                                      | Pump      | Lubrication         | SLC    | 1992 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. The gasket between the crankcase frame cap and the gear housing cover was worn.   |
| 226  | Test             | Maintenance     | Internal to Component                                      | Pump      | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | Emergency service water pumps discharge pressure below allowable limits. Causes were loose impellers, dropped impeller, and worn internals.  |
| 227  | Test             | Maintenance     | Internal to Component                                      | Pump      | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. One pump also exhibited high vibration.   |
| 228  | Test             | Maintenance     | Internal to Component                                      | Pump      | Coupling            | ESW    | 1987 | Failure to Start | Almost Complete   | Two ESW pumps had failed couplings. Cause attributed to abnormal stress.   |
| 229  | Test             | Maintenance     | Internal to Component                                      | Pump      | Coupling            | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause       | Segment | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|-----------------------|---------|---------------------|--------|------|------------------|-------------------|--|
| 230  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1982 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 231  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | RHR-B  | 1985 | Failure to Start | Partial           | The first pump failed to meet required flow rate. The second was drawing excessive amperage. Both conditions were attributed to worn internals.  |
| 232  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1983 | Failure to Run   | Partial           | RHR Service Water pumps failed flow tests due to wearout and had to be rebuilt.  |
| 233  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.   |
| 234  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1994 | Failure to Start | Partial           | Two ESW pumps had low discharge pressure during testing. Each pump had worn internals and both pump internals were replaced.   |
| 235  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted. |
| 236  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 237  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed due to worn internals.  |
| 238  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 239  | Test             | Maintenance     | Internal to Component | Pump    | Bearing             | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 240  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1985 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 241  | Test             | Maintenance     | Internal to Component | Pump    | Shaft               | ESW    | 1993 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.   |
| 242  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.  |
| 243  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1984 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.  |
| 244  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1988 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 245  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1990 | Failure to Start | Partial           | ESW impeller gaps too wide. Gaps adjusted.   |
| 246  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1981 | Failure to Start | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 247  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | Wear caused high ESW pump bearing temperatures, vibration, and low amperage/flow.  |
| 248  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | ESW pump performance decreased 15% and 8% respectively since last test. Pumps were replaced.   |
| 249  | Test             | Maintenance     | Internal to Component | Pump    | Impeller/Wear Rings | ESW    | 1984 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |

| Item | Discovery Method | Coupling Factor | Proximate Cause          | Segment | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|------------------|-----------------|--------------------------|---------|---------------------|--------|------|------------------|-------------------|--|
| 250  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | HPI    | 1985 | Failure to Start | Partial           | The CCPs were tested and had low flow rates. The most probable cause is attributed to observed degradation of the pumps. The CCPs are subject to normal wear associated with their secondary duty of providing normal charging flow.   |
| 251  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | HPI    | 1983 | Failure to Start | Partial           | SI pump and both CCPs failed to meet the minimum head curve requirements. The cause of pump head capacity degradation has been attributed to normal pump operation. The inability to balance flows has been attributed to the lower head capacity of the pumps.  |
| 252  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1991 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 253  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1984 | Failure to Run   | Partial           | Containment spray raw water pumps failed flow tests. Aging and normal wear.  |
| 254  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to brackish water corrosion.   |
| 255  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1989 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 256  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1987 | Failure to Run   | Partial           | ESW pump low flow. Worn impellers.   |
| 257  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1990 | Failure to Run   | Partial           | ESW pumps had worn and cracked impellers. Aging and normal wear.   |
| 258  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1985 | Failure to Run   | Partial           | The charging pump service water pumps degraded. Caused by expected wear of pump due to erosion and corrosion properties of the process fluid involved  |
| 259  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1986 | Failure to Run   | Partial           | ESW pumps had worn impellers and one had a plugged strainer.   |
| 260  | Test             | Maintenance     | Internal to Component    | Pump    | Packing/Seals       | ESW    | 1981 | Failure to Start | Partial           | RHR service water pumps failed to meet flow requirements due to seal water leakage and pump wearout.   |
| 261  | Test             | Maintenance     | Internal to Component    | Pump    | Impeller/Wear Rings | ESW    | 1994 | Failure to Run   | Partial           | Two ESW pumps had internal deterioration, one of which was indicated by high vibration readings.   |
| 262  | Test             | Maintenance     | Operational/ Human Error | Driver  | I&C                 | ESW    | 1989 | Failure to Start | Partial           | Emergency equipment service water pump relays were not reset following a load shedding test 30 hours before.   |
| 263  | Test             | Maintenance     | Operational/ Human Error | Driver  | Motor               | ESW    | 1994 | Failure to Run   | Partial           | Leak test of the containment cooling service water pump vault watertight door revealed excessive leakage. Flooding and leakage past this door would make inoperable two of four containment cooling service water pumps. Procedural inadequacy was cited as the cause for the degraded door seals.   |
| 264  | Test             | Maintenance     | Operational/ Human Error | Pump    | Casing              | RHR-P  | 1989 | Failure to Start | Complete          | Both loops of the residual heat removal system were declared inoperable due to gas binding of both RHR pumps. The gas binding was caused by entry of nitrogen gas into the reactor coolant system from accumulator. The root cause of this event has been attributed to personnel error. Personnel did not comply with the specific requirements in the accumulator discharge check valve full flow test procedure due to inattention to detail. |

| Item | Discovery Method | Coupling Factor | Proximate Cause  | Segment   | Piece Part          | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|------------------|-----------------|--|-----------|---------------------|--------|------|------------------|-------------------|---|
| 265  | Test             | Maintenance     | Operational/ Human Error                                   | Pump      | Packing/Seals       | AFW    | 1996 | Failure to Run   | Partial           | During the performance of Steam-Driven Emergency Feedwater Pump testing, sparks were observed emanating from the outboard mechanical seal area. The sparks appeared to be due to a mechanical interference within the mechanical seal assembly. The pump mechanical seal was disassembled and determined to have been improperly installed during the last refueling outage. The evaluation identified a mechanical seal design deficiency and inadequate corrective action for a previously identified event as the primary causes for this event. A contributing cause for this event was found to be inadequate predictive maintenance techniques. The electric AFW pump exhibited the same problem. |
| 266  | Test             | Maintenance     | Other  | Driver    | Breaker             | ESW    | 1984 | Failure to Start | Partial           | ESW pump breakers tripped due to failed voltage control devices.  |
| 267  | Test             | Maintenance     | Other  | Driver    | Breaker             | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker overcurrent trip devices tripping too low.   |
| 268  | Test             | Operational     | Design/ Construction/ Manufacture/ Installation Inadequacy | Discharge | Check Valve         | ESW    | 1999 | Failure to Run   | Partial           | Two ESW pumps had low flow due to interaction with the two other pumps when all four pumps were running.  |
| 269  | Test             | Operational     | Operational/ Human Error                                   | Driver    | I&C                 | ESW    | 1990 | Failure to Start | Complete          | An emergency service water pump failed to start and was declared inoperable. Further investigation determined that the failure of the pump to start was due to a tripped emergency engine shutdown device. Operations personnel performing the testing did not recognize the need to reset it prior to starting the pump. Examination of the other two ESW pumps revealed that their emergency shutdown devices were also in the tripped condition.   |
| 270  | Test             | Operational     | Operational/ Human Error                                   | Suction   | Piping              | ESW    | 1989 | Failure to Run   | Partial           | Inadequate procedure led to air binding of operating ESW pumps.   |
| 271  | Test             | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver    | Breaker             | LCS    | 1980 | Failure to Start | Complete          | Relay extra contacts left connected during construction, prevented Core Spray pump start with emergency diesel generator breakers racked out.   |
| 272  | Test             | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Driver    | I&C                 | AFW    | 1980 | Failure to Start | Complete          | During surveillance testing, neither motor-driven AFW pump would start. The pump control circuit was found with autostart defeat switches labeled backwards, causing all autostarts except the low-low steam generator level to be defeated. The labels were corrected and the links were closed. The original installation error was the result of an inadequate design change process that did not require sufficient verification and testing of the modification.   |
| 273  | Test             | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | Pump      | Impeller/Wear Rings | ESW    | 1986 | Failure to Start | Partial           | Testing of the service water system disclosed that the performance of the three service water pumps was below requirements. The condition is the result of both an inadequate system design and the installation of replacement impellers, which were not modified by the vendor to improve performance, as were the original impellers.  |
| 274  | Test             | Quality         | Operational/ Human Error                                   | Driver    | I&C                 | ESW    | 1982 | Failure to Start | Partial           | Two ESW pumps failed to start. One ESW pump failed to function as a result of loose wires on relay terminals in both pump logic schemes, a loose states link and an instantaneous contact found out of adjustment on the other pump logic scheme.   |





**Appendix B**  
**Pump Data Summary by Segment**



## **Appendix B**

### **Pump Data Summary by Segment**

This appendix is a summary of the data evaluated in the common-cause failure (CCF) data collection effort for pumps. This appendix supports the charts in Chapter 4. The table is sorted alphabetically, by the first four columns.

## Appendix B

|  |   |
|--|---|
| Table B-1. Pump CCF event summary, sorted by segment. .... | 3 |
|--|---|

Table B-1. Pump CCF event summary, sorted by segment.

| Item | Segment   | Proximate Cause  | Discovery Method | Piece Part  | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------|--|------------------|-------------|-----------------|--------|------|------------------|-------------------|---|
| 1    | Discharge | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Valve       | Design          | AFW    | 1986 | Failure to Start | Partial           | Both the turbine driven and motor driven AFW pumps could not produce full flow because the cages in their discharge valve trapped debris and plugged.   |
| 2    | Discharge | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Valve       | Design          | AFW    | 1985 | Failure to Start | Partial           | Controller problems in the steam and diesel driven AFW pumps caused the pumps to trip on low suction pressure. The pump discharge flow controller valves were also not set properly after last maintenance. Low suction trips were due to design error.   |
| 3    | Discharge | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Check Valve | Operational     | ESW    | 1999 | Failure to Run   | Partial           | Two ESW pumps had low flow due to interaction with the two other pumps when all four pumps were running.  |
| 4    | Discharge | External Environment   | Demand           | Check Valve | Design          | AFW    | 1983 | Failure to Start | Almost Complete   | Hot water in the AFW pump casings caused the pumps to become vapor bound. The hot water was from leaking check valves upstream of the pumps. This event occurred once on the turbine driven pump and 5 times on the motor driven pump.  |
| 5    | Discharge | External Environment   | Inspection       | Piping      | Design          | HPI    | 1994 | Failure to Run   | Partial           | Due to a leaking socket weld in the common recirculation line, all three SI pumps were declared inoperable. The underlying cause of the leak was a crack in the socket weld in the common recirculation line, caused by pipe displacement from air entrainment and pump misalignment.   |
| 6    | Discharge | External Environment   | Test             | Recirc      | Environmental   | HPI    | 1992 | Failure to Run   | Almost Complete   | Safety Injection pumps were declared inoperable due to an observed declining trend in the pump's recirculation flow. The cause of the Safety Injection pump reduced recirculation flow is attributed to foreign material blockage within the associated minimum flow recirculation line flow orifice.   |
| 7    | Discharge | Internal to Component  | Demand           | Valve       | Environmental   | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitrol cages for these valves were clogged with shredded Asiatic clam shells. |
| 8    | Discharge | Internal to Component  | Demand           | Valve       | Environmental   | AFW    | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitrol cages for these valves were clogged with shredded Asiatic clam shells. |
| 9    | Discharge | Internal to Component  | Inspection       | Check Valve | Maintenance     | AFW    | 1990 | Failure to Start | Almost Complete   | Leakage past AFW check valves caused AFW pumps to become steam bound. Closed motor operated valve in line. Scheduled check valves for replacement next outage.  |
| 10   | Discharge | Internal to Component  | Test             | Recirc      | Environmental   | HPI    | 1991 | Failure to Run   | Partial           | Something in HPI pump recirculation line was restricting flow. The piece later dislodged and no identification was made. Both SI pumps had inadequate recirculation flow.   |

| Item | Segment   | Proximate Cause  | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|-----------|--|------------------|------------|-----------------|--------|------|------------------|-------------------|---|
| 11   | Discharge | Internal to Component                                      | Test             | Valve      | Maintenance     | HPI    | 1984 | Failure to Start | Partial           | CCP pump low flow rates due to inaccuracies in positioning the throttle valves.   |
| 12   | Discharge | Operational/ Human Error                                   | Inspection       | Valve      | Operational     | HPI    | 1987 | Failure to Start | Almost Complete   | While attempting to fill the safety injection accumulators, it was discovered that two of three SI pumps had been isolated from the high head injection flowpath.   |
| 13   | Discharge | Operational/ Human Error                                   | Inspection       | Valve      | Operational     | HPI    | 1993 | Failure to Run   | Partial           | One AFW pump failed due to incorrect procedure which allowed pump to be run without flow, other AFW pump was allowed to run past max flow rate. It is unclear whether these mistakes were due to inadequate procedures or staff errors, but it was assumed to be a failure to follow procedure.   |
| 14   | Discharge | Operational/ Human Error                                   | Inspection       | Valve      | Operational     | AFW    | 1994 | Failure to Start | Complete          | Following a trip, the AFW Pumps were secured and the discharge flow control valves for the Motor Driven Pumps were closed. Later, an operator discovered during a routine Control Board walkdown that the valves were closed. Subsequent investigation revealed the AFW system had not been placed in standby readiness per the operating procedure after the system was secured.   |
| 15   | Discharge | Other  | Demand           | Valve      | Maintenance     | ESW    | 1980 | Failure to Start | Partial           | RHR service water pumps were started to put torus cooling in service. When these pumps would not deliver required discharge pressure, they were declared inoperable. The seal in an air release valve was bad, allowing a vent on the discharge line.   |
| 16   | Driver    | Design/ Construction/ Manufacture/ Installation Inadequacy | Demand           | Breaker    | Quality         | ESW    | 1996 | Failure to Start | Partial           | Two RHRSW pumps fail to start due to breaker failures. Wrong contacts were installed. Design called for contacts to have a minimum current interrupt rating of 6 amps; contacts installed (that subsequently failed) had current interrupt rating of only 2.2 amps.   |
| 17   | Driver    | Design/ Construction/ Manufacture/ Installation Inadequacy | Demand           | I&C        | Design          | AFW    | 1997 | Failure to Run   | Partial           | One actual AFW pump failure due to spurious electronic overspeed trip. Determined that all three pumps were susceptible to spurious overspeed trips.  |
| 18   | Driver    | Design/ Construction/ Manufacture/ Installation Inadequacy | Demand           | I&C        | Design          | AFW    | 1981 | Failure to Start | Almost Complete   | A modification to the control instrumentation for two AFW pumps resulted in a backfeed situation such that when called upon to start, both pumps would not start.   |
| 19   | Driver    | Design/ Construction/ Manufacture/ Installation Inadequacy | Demand           | I&C        | Design          | AFW    | 1981 | Failure to Start | Almost Complete   | Two AFW pumps failed to automatically start due to low suction pressure trips. A modification was installed to prevent this. This effect was discovered previously, but apparently had not been corrected prior to an attempt to start the pumps three weeks later.   |
| 20   | Driver    | Design/ Construction/ Manufacture/ Installation Inadequacy | Demand           | I&C        | Quality         | AFW    | 1989 | Failure to Start | Complete          | Both motor driven auxiliary feedwater pumps failed to start when the operator tried to start them manually. While preparing a design change, the designer failed to review all the unit specific documentation associated with the motor-driven AFW pump wiring and made the erroneous assumption that both units switchgear compartment internal wiring was identical. In fact, the wiring for each unit was different. Consequently, when the design change was installed, it was installed in accordance with the erroneous design. The wiring discrepancy was corrected and the motor-driven AFW pumps were tested and returned to service. |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part  | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|--|------------------|-------------|-----------------|--------|------|------------------|-------------------|---|
| 21   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Lubrication | Design          | RHR-P  | 2000 | Failure to Run   | Complete          | Both RHR/LPI pumps fail to run due to improper oil in system. High bearing temperatures occurred when the pumps were operated. This was due to the wrong lube oil being used, which had too high a viscosity. Inadequate vender design information resulted in the higher viscosity oil being used and additional exacerbating problems such as insufficient bearing clearances.  |
| 22   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Motor       | Quality         | ESW    | 1987 | Failure to Start | Partial           | ESW pump motors tripped on overcurrent. The overcurrent trip was due to a ground and a short on the pump motor.   |
| 23   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | I&C         | Design          | AFW    | 1994 | Failure to Start | Partial           | Single failure would prevent auto initiation of AFW. Circuit design did not provide separation required by standards and code. The single failure identified was a short circuit across two conductors of the actuation relays associated with the initiation logic matrix.   |
| 24   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Lubrication | Design          | HPI    | 2000 | Failure to Run   | Partial           | CVC makeup oil pump motor too small for certain accidents.  |
| 25   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Piping      | Environmental   | HPI    | 2000 | Failure to Run   | Partial           | Microbiologically induced corrosion leak on service water lines to two charging/HPI pump lube oil coolers.  |
| 26   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Supports    | Design          | RHR-B  | 1986 | Failure to Start | Partial           | RHR motor internal supports were cracked due to stress and vibration. Design improvements were made.  |
| 27   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | I&C         | Design          | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |
| 28   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | I&C         | Design          | HPI    | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus. |



| Item | Segment | Proximate Cause  | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|--|------------------|------------|-----------------|--------|------|------------------|-------------------|--|
| 29   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Breaker    | Quality         | LCS    | 1980 | Failure to Start | Complete          | Relay extra contacts left connected during construction, prevented Core Spray pump start with emergency diesel generator breakers racked out.  |
| 30   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Breaker    | Design          | HPI    | 1980 | Failure to Start | Partial           | Upon testing the safety injection pumps it was found that the 6900-v breakers would lock-out preventing pump start if they were given a close signal for >0.32 seconds when a trip condition existed. There is no indication to operations when this locked-out condition exists. The breaker appears to be available for service when it actually is not. The only means of clearing the condition is to remove and reinstall the fuses at the breaker or manually change the state of the relays.  |
| 31   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | I&C        | Quality         | AFW    | 1980 | Failure to Start | Complete          | During surveillance testing, neither motor-driven AFW pump would start. The pump control circuit was found with autostart defeat switches labeled backwards, causing all autostarts except the low-low steam generator level to be defeated. The labels were corrected and the links were closed. The original installation error was the result of an inadequate design change process that did not require sufficient verification and testing of the modification.  |
| 32   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | I&C        | Design          | AFW    | 1992 | Failure to Start | Complete          | A modification design error (in 1983-1984) removed a start permissive interlock contact. At cold shutdown this de-energized the auxiliary lube oil pump, consequently, when one AFW pump was started it ran for 2.5 seconds and tripped on low oil pressure. Further investigation showed that both units AFW pumps would be affected in the same way. The design error combined with insufficient post modification testing led to this CCF event.  |
| 33   | Driver  | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | I&C        | Design          | AFW    | 1981 | Failure to Start | Almost Complete   | Two low suction pressure trips for the AFW pumps were mis-calibrated, which prevented the pumps from starting.   |
| 34   | Driver  | External Environment   | Demand           | Breaker    | Maintenance     | AFW    | 1990 | Failure to Run   | Partial           | AFW pumps circuit breakers degraded.   |
| 35   | Driver  | External Environment   | Demand           | I&C        | Environmental   | AFW    | 1984 | Failure to Start | Complete          | Both AFW pumps failed to start. The problem was traced to two relays (1 per pump). Examination of the relays revealed open circuiting and severe degradation of the insulation.  |
| 36   | Driver  | External Environment   | Demand           | Motor      | Environmental   | ESW    | 1985 | Failure to Run   | Partial           | Two service water motors failed on demand as a result of cement dust contamination.  |
| 37   | Driver  | External Environment   | Inspection       | I&C        | Operational     | HPI    | 1990 | Failure to Run   | Complete          | It was determined that the common minimum flow path return line for the safety injection pumps to the refueling water storage tank was frozen. Previous actions to investigate problems with the freeze protection system were unsuccessful in preventing development of this condition. The two HPI pumps were declared inoperable with this return line frozen. A faulty ambient temperature switch for the RWST heat trace system prevented the heat trace from activating and was subsequently replaced. In addition, administrative controls did not sufficiently recognize the safety significance of flow through this line and the need to ensure flow capability. |
| 38   | Driver  | External Environment   | Maintenance      | Motor      | Environmental   | ESW    | 1987 | Failure to Start | Partial           | During an extended service water bay flooding incident, one ESW pump was found grounded by testing, later two more pumps were found to be failed also.   |

| Item | Segment | Proximate Cause       | Discovery Method | Piece Part    | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|-----------------------|------------------|---------------|-----------------|--------|------|------------------|-------------------|---|
| 39   | Driver  | External Environment  | Test             | Bearing       | Environmental   | RHR-B  | 1991 | Failure to Run   | Partial           | Two LCI pumps were declared inoperable due to high motor vibration.   |
| 40   | Driver  | Internal to Component | Demand           | Breaker       | Design          | ESW    | 2000 | Failure to Start | Almost Complete   | Two ESW pumps failed to start due to their breakers failing to close. The breakers' prop spring bracket has slipped thus preventing proper interfacing between the prop and the prop pin.   |
| 41   | Driver  | Internal to Component | Demand           | Breaker       | Maintenance     | RHR-B  | 1987 | Failure to Start | Partial           | RHR pump breakers failed to close when operated remotely from the control room. It was found that the latch roller bearings and the cam follower bearing (internal piece parts of the breaker) were not operating correctly. This prevented the trip latch assembly from resetting and allowing the breaker to close.   |
| 42   | Driver  | Internal to Component | Demand           | I&C           | Maintenance     | ESW    | 1991 | Failure to Start | Partial           | Two ESW pumps failed to start due to failed breakers. Inadequate maintenance.   |
| 43   | Driver  | Internal to Component | Demand           | Lubrication   | Maintenance     | HPI    | 1984 | Failure to Run   | Partial           | Charging pump lube oil cooler fan motor trips on thermal overload. Probable cause: normal wear on motor resulting in increased friction replaced worn motor with spare. During routine inservice testing found that another charging pump lube oil cooler fan motor had a current imbalance. Probable cause: normal aging of motor insulation has resulted in a current imbalance.  |
| 44   | Driver  | Internal to Component | Inspection       | Bearing       | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | One service water pump motor upper bearing oil reservoir leaking from cover plate. Another service water pump motor upper oil cooler oil reservoir leaking.   |
| 45   | Driver  | Internal to Component | Inspection       | Bearing       | Maintenance     | ESW    | 1981 | Failure to Run   | Partial           | ESW motor to pump alignment problems. Bearings worn out.  |
| 46   | Driver  | Internal to Component | Inspection       | Breaker       | Maintenance     | ESW    | 1996 | Failure to Start | Partial           | ESW pump breakers fail due to misalignment of the breaker mechanism and internals developed over the years of operation.  |
| 47   | Driver  | Internal to Component | Inspection       | I&C           | Design          | ESW    | 1982 | Failure to Start | Partial           | Open circuit breaker resulted in loss of two RHR service water pumps.   |
| 48   | Driver  | Internal to Component | Inspection       | Packing/Seals | Maintenance     | HPI    | 1988 | Failure to Run   | Almost Complete   | Smoke was discovered coming from the speed increaser unit for a centrifugal charging pump. Investigation found the two gland seal retaining bolts inside the speed increaser lube oil pump backed out allowing the gland seal to loosen. The gland seal being loosened, caused reduced oil flow to the speed increaser internals and ultimate damage. Other CCPs were inspected, and the same gland seal bolts as on the first pump were found loosened. The cause of the bolts backing out was determined to be lack of a periodic adjustment of the gland seal bolts. |
| 49   | Driver  | Internal to Component | Maintenance      | Breaker       | Maintenance     | HPI    | 1991 | Failure to Start | Partial           | HPI pump breakers failed due to a broken pawl, and a broken closing coil.   |
| 50   | Driver  | Internal to Component | Maintenance      | Breaker       | Maintenance     | SLC    | 1999 | Failure to Start | Partial           | SLC Pump Breakers Fail to pickup on degraded voltage test   |
| 51   | Driver  | Internal to Component | Maintenance      | Breaker       | Maintenance     | ESW    | 1985 | Failure to Start | Partial           | Two raw water pump breaker main wipes were out of adjustment.   |
| 52   | Driver  | Internal to Component | Maintenance      | Breaker       | Maintenance     | AFW    | 1992 | Failure to Start | Partial           | With the unit in a refueling outage, following repairs to a motor driven auxiliary feedwater pump local/remote switch of the circuit breaker, personnel found that the switch contacts would not close. This failure rendered one of three auxiliary feedwater pumps inoperable. The cause of the failure appears to be due to dirty/corroded contacts on the switch.   |
| 53   | Driver  | Internal to Component | Test             | Bearing       | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps exhibited vibration. Attributed to normal wear.   |

| Item | Segment | Proximate Cause          | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|--------------------------|------------------|------------|-----------------|--------|------|------------------|-------------------|---|
| 54   | Driver  | Internal to Component    | Test             | Breaker    | Maintenance     | RHR-B  | 1986 | Failure to Start | Partial           | RHR pump circuit breakers failed during a start for testing. Bend switch and binding mechanism. Attributed to inadequate maintenance.   |
| 55   | Driver  | Internal to Component    | Test             | Breaker    | Maintenance     | ESW    | 1998 | Failure to Start | Partial           | Two RHR service water pump breakers would not close due to dirty contacts in breakers.  |
| 56   | Driver  | Internal to Component    | Test             | Breaker    | Maintenance     | RHR-B  | 1997 | Failure to Start | Partial           | Breaker latch check switch failed on both pumps. Lack of lubrication.   |
| 57   | Driver  | Internal to Component    | Test             | Breaker    | Maintenance     | AFW    | 1997 | Failure to Start | Almost Complete   | The circuit breakers associated with the AFW Pumps failed to close as required. The root cause of the failure was the binding in the operating mechanism. The plunger apparently did not always complete its upward movement to close and latch the breaker, due to accumulated dirt and lubricants.  |
| 58   | Driver  | Internal to Component    | Test             | Breaker    | Maintenance     | ESW    | 1998 | Failure to Start | Partial           | Service water pumps fail to start due to circuit breaker failures. Pump breakers failed to close due to failures of the charging spring/motor and closing spring motor.   |
| 59   | Driver  | Operational/ Human Error | Demand           | Breaker    | Maintenance     | ESW    | 1993 | Failure to Start | Partial           | Operations personnel were attempting to swap the running service water pump with the idle service water pump. Personnel placed the control switch to start and the service water pump did not start. Breaker malfunction. Later, another service water pump failed to start because of the breaker.   |
| 60   | Driver  | Operational/ Human Error | Demand           | Breaker    | Maintenance     | ESW    | 1988 | Failure to Run   | Partial           | Service water pump high dropout over current protection devices were less than running current conditions and trip setpoints did not account for changing load conditions due to modified impellers. Three pump trips had occurred.   |
| 61   | Driver  | Operational/ Human Error | Demand           | Breaker    | Maintenance     | ESW    | 1987 | Failure to Start | Partial           | One breaker failed to linkage alignment and second from loose relay connections. Inadequate maintenance.  |
| 62   | Driver  | Operational/ Human Error | Demand           | I&C        | Operational     | AFW    | 1983 | Failure to Start | Complete          | An operator incorrectly secured the diesel and steam driven AFW pumps, which prevented their restart on low SG level.   |
| 63   | Driver  | Operational/ Human Error | Demand           | I&C        | Operational     | ESW    | 1981 | Failure to Start | Partial           | Alarm circuit breaker was de-energized resulting in a loss of two RHR service water pumps.  |
| 64   | Driver  | Operational/ Human Error | Demand           | I&C        | Design          | ESW    | 1980 | Failure to Start | Partial           | Instrument isolation valve closed causing a low suction trip signal to two RHRSW pumps.   |
| 65   | Driver  | Operational/ Human Error | Inspection       | Bearing    | Maintenance     | RHR-P  | 1988 | Failure to Run   | Partial           | Residual heat removal pump motor upper bearing housings were observed to be leaking oil. The cause of the failure was attributed to a lack of sealant being applied and gasket installed after the last maintenance was performed on the motor bearing housing.   |
| 66   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Operational     | HPI    | 1982 | Failure to Start | Complete          | During the draining of the reactor coolant system, both centrifugal charging pumps were rendered inoperable. The initial conditions in the draining procedure contained a confusing statement, which led to an erroneous assumption that both CCP breakers had to be racked out and tagged.   |
| 67   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Design          | ESW    | 1984 | Failure to Start | Partial           | During an attempt to perform preventive maintenance for unit one's RHR service water pumps, plant personnel mistakenly disconnected the motor leads for unit two's RHR service water pump.  |
| 68   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Quality         | ESW    | 1992 | Failure to Start | Partial           | The fit between an ESW pump breaker primary disconnects and the associated breaker cubicle stabs was inadequate. The poor fit between the disconnects and the stabs led to arcing in the breaker cubicle when the pump was started, resulting in a fire. Shortly after identifying the cause of the fire, the remaining ESW breakers, which had recently been replaced along with the failed breaker, as part of a design modification package, were found to be inadequate also. |

| Item | Segment | Proximate Cause          | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|--------------------------|------------------|------------|-----------------|--------|------|------------------|-------------------|---|
| 69   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Operational     | CSS    | 1991 | Failure to Start | Complete          | CSR control power de-energized prior to mode change. Technical Specification violation. Inadequate procedure review.  |
| 70   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Operational     | ESW    | 1981 | Failure to Start | Almost Complete   | Control breakers for two ESW pumps were open due to inadvertent operator action.  |
| 71   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Maintenance     | RHR-P  | 1981 | Failure to Start | Complete          | All RHR pumps de-energized to replace RHR Relief valve. T.S. allows this condition for 1 hour. Operated in the mode in excess of 5 hours.   |
| 72   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Operational     | HPI    | 1988 | Failure to Start | Complete          | HPI pumps not restored before mode change due to procedural inadequacy.   |
| 73   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Operational     | HPI    | 1990 | Failure to Start | Complete          | By opening incorrect breaker, HPI pump tripped while others were unavailable.   |
| 74   | Driver  | Operational/ Human Error | Inspection       | Breaker    | Operational     | HPI    | 1989 | Failure to Start | Partial           | HPI Pump B not retested, then HPI Pump A removed from service.  |
| 75   | Driver  | Operational/ Human Error | Inspection       | I&C        | Maintenance     | RHR-P  | 1992 | Failure to Start | Complete          | Both trains of RHR were rendered inoperable for two minutes, while performing an operational readiness test surveillance procedure. The surveillance procedure required that the one RHR train pump be placed in pull to lock and the other train heat exchanger flow control valve throttled to 30-40% open. The procedure directed the operators to perform operations that resulted in both trains of RHR being inoperable |
| 76   | Driver  | Operational/ Human Error | Inspection       | I&C        | Operational     | HPI    | 1988 | Failure to Start | Complete          | With alternate CCP pump out-of-service, the remaining operable pump was erroneously placed in pull-to-lock.   |
| 77   | Driver  | Operational/ Human Error | Inspection       | I&C        | Operational     | HPI    | 1992 | Failure to Start | Almost Complete   | Two charging pumps and one charging pump service water pump were removed from service simultaneously which is a condition not allowed by technical specifications.  |
| 78   | Driver  | Operational/ Human Error | Inspection       | I&C        | Operational     | RHR-P  | 1995 | Failure to Start | Complete          | The switches for the containment spray and recirculation pumps were in a trip pullout when the Technical Specifications and plant procedures required the pumps to be operable.   |
| 79   | Driver  | Operational/ Human Error | Inspection       | I&C        | Operational     | HPI    | 1990 | Failure to Start | Partial           | Both safety injection pumps were in the pull-to-lock position. With the switches in pull-to-lock, the pumps would not have automatically started upon receipt of an initiating signal. This event was caused by cognitive personnel error by a utility licensed operator in failure to follow an approved procedure.  |
| 80   | Driver  | Operational/ Human Error | Inspection       | I&C        | Maintenance     | AFW    | 1990 | Failure to Start | Complete          | During testing one AFW pump was tested and other was tested without returning first to auto. Both pumps were unavailable at the same time. The procedure was the cause.   |
| 81   | Driver  | Operational/ Human Error | Maintenance      | Breaker    | Maintenance     | RHR-B  | 1991 | Failure to Start | Partial           | While performing preventive maintenance calibration check on the protective relays for a residual heat removal pump motor 4kv breaker, it was found that all overcurrent relays for two pumps were out of calibration   |
| 82   | Driver  | Operational/ Human Error | Maintenance      | Breaker    | Maintenance     | RHR-B  | 1990 | Failure to Start | Partial           | RHR pump breaker overcurrent trips out of calibration.  |
| 83   | Driver  | Operational/ Human Error | Test             | Breaker    | Design          | AFW    | 1985 | Failure to Start | Complete          | Both AFW pumps failed to start when tested, due to the circuit breakers not being racked in properly.   |
| 84   | Driver  | Operational/ Human Error | Test             | I&C        | Maintenance     | ESW    | 1989 | Failure to Start | Partial           | Emergency equipment service water pump relays were not reset following a load shedding test 30 hours before.  |
| 85   | Driver  | Operational/ Human Error | Test             | I&C        | Quality         | ESW    | 1982 | Failure to Start | Partial           | Two ESW pumps failed to start. One ESW pump failed to function as a result of loose wires on relay terminals in both pump logic schemes, a loose states link and an instantaneous contact found out of adjustment on the other pump logic scheme.   |

| Item | Segment | Proximate Cause          | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|--------------------------|------------------|------------|-----------------|--------|------|------------------|-------------------|--|
| 86   | Driver  | Operational/ Human Error | Test             | I&C        | Operational     | ESW    | 1990 | Failure to Start | Complete          | An emergency service water pump failed to start and was declared inoperable. Further investigation determined that the failure of the pump to start was due to a tripped emergency engine shutdown device. Operations personnel performing the testing did not recognize the need to reset it prior to starting the pump. Examination of the other two ESW pumps revealed that their emergency shutdown devices were also in the tripped condition.  |
| 87   | Driver  | Operational/ Human Error | Test             | Motor      | Maintenance     | ESW    | 1994 | Failure to Run   | Partial           | Leak test of the containment cooling service water pump vault watertight door revealed excessive leakage. Flooding and leakage past this door would make inoperable two of four containment cooling service water pumps. Procedural inadequacy was cited as the cause for the degraded door seals.   |
| 88   | Driver  | Other                    | Demand           | Breaker    | Maintenance     | RHR-P  | 1987 | Failure to Start | Complete          | Two LPI pumps, when given a start signal, would not start. An ongoing investigation revealed the probable root cause of the event to be poor electrical contact of the breaker auxiliary stabs for the pumps.  |
| 89   | Driver  | Other                    | Demand           | I&C        | Maintenance     | ESW    | 1982 | Failure to Start | Complete          | Following a reactor scram, an attempt to initiate suppression pool cooling revealed that both RHRSW loops were inoperable as neither loop's pumps could be started. Low suction header pressure lockout signals in each loop prevented starting each loop's pumps. Plugging of the sensing line to each loop's suction header pressure switch prevented both switches from sensing actual pressure, although a lack of operating fluid in one switch and an open power supply breaker to the other switch also would have prevented pumps from starting. |
| 90   | Driver  | Other                    | Demand           | I&C        | Design          | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure. This is a second event two months later.  |
| 91   | Driver  | Other                    | Demand           | I&C        | Design          | ESW    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure.   |
| 92   | Driver  | Other                    | Demand           | Piping     | Design          | HCI    | 1999 | Failure to Start | Complete          | Water entered the HCI and RCI steam supply lines, rendering both pumps inoperable. Failed reactor vessel instrumentation allowed water to overflow and fill the HCI/RCI steam lines. Pumps were unavailable.   |
| 93   | Driver  | Other                    | Inspection       | I&C        | Design          | AFW    | 1983 | Failure to Start | Partial           | Both AFW pumps had to be rendered inoperable to allow repairs to actuation circuitry.  |
| 94   | Driver  | Other                    | Inspection       | Motor      | Environmental   | ESW    | 1981 | Failure to Run   | Partial           | The float guide failed in a RHRSW pump air valve and caused the valve to fail open and flood pump room.  |
| 95   | Driver  | Other                    | Inspection       | Motor      | Environmental   | AFW    | 1990 | Failure to Start | Partial           | Both motor driven AFW pumps were sprayed when a service water pipe developed a through wall leak.  |
| 96   | Driver  | Other                    | Maintenance      | Breaker    | Maintenance     | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker failures, broken screw, no lubrication, and a bent track  |
| 97   | Driver  | Other                    | Maintenance      | Breaker    | Maintenance     | ESW    | 1982 | Failure to Start | Partial           | ESW pump circuit breakers found damaged. Defective arc chute and cracked secondary coupler.  |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part             | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|--|------------------|------------------------|-----------------|--------|------|------------------|-------------------|---|
| 98   | Driver  | Other  | Test             | Breaker                | Design          | SLC    | 1986 | Failure to Start | Complete          | During a test, both Squib Valve Detonators shorted after firing and the Control Power Transformer fuse blew causing the pump motor trip. This was caused by improper fuse coordination between the Control Power Transformer fuse and the Squib Valve Detonator fuses. The redundant system's Squib Valve was also fired during this test, without running the associated pump, and one of the Squib Valve Detonators shorted after firing. The same fuse coordination problem existed for both systems.  |
| 99   | Driver  | Other  | Test             | Breaker                | Maintenance     | ESW    | 1984 | Failure to Start | Partial           | ESW pump breaker overcurrent trip devices tripping too low.   |
| 100  | Driver  | Other  | Test             | Breaker                | Maintenance     | ESW    | 1984 | Failure to Start | Partial           | ESW pump breakers tripped due to failed voltage control devices.  |
| 101  | Driver  | Other  | Test             | I&C                    | Design          | RHR-B  | 1982 | Failure to Start | Partial           | A functional test revealed a sliding link in control room panel open. Further investigation revealed a total of four links open. These links, left open, negated all autostart capability of 2 of 4 RHR pumps. It could not be determined why these four links were open.   |
| 102  | Driver  | Other  | Test             | I&C                    | Design          | ESW    | 1992 | Failure to Start | Partial           | Valve position contacts prevented ESW pump circuit breakers from closing. Poor design resulted in water intrusion in the valve limit switch box.  |
| 103  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Impeller/Wear<br>Rings | Environmental   | ESW    | 2000 | Failure to Start | Almost Complete   | Two of the River Water pumps tripped on overcurrent when they were attempted to be started. The trips were a result of physical contact between the impeller and the lower casing liner of the pumps. This condition was due to differential thermal expansion between the pump shaft and the pump casing as a result of an elevated seal injection water temperature. The elevated temperature was due to an abnormal configuration of the Filtered Water System (the backup seal water supply).   |
| 104  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Impeller/Wear<br>Rings | Design          | ESW    | 1981 | Failure to Run   | Complete          | Both charging pump service water pumps failed. A carbon cap screw failed allowing the impeller of one pump to bind on the casing. The ensuing leakage shorted the motor windings of the other pump.   |
| 105  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Impeller/Wear<br>Rings | Quality         | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps drawing excessive current. Carbon steel snap rings corroded allowing impeller to come in contact with casing. The third pump, although not exhibiting abnormal current, had similar corrosion   |
| 106  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Impeller/Wear<br>Rings | Design          | ESW    | 1996 | Failure to Run   | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part             | Coupling Factor | System | Year | Failure Mode      | Degree of Failure  | Description   |
|------|---------|--|------------------|------------------------|-----------------|--------|------|-------------------|--------------------|---|
| 107  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Impeller/Wear<br>Rings | Quality         | ESW    | 1996 | Failure<br>to Run | Partial            | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 108  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Impeller/Wear<br>Rings | Design          | ESW    | 1986 | Failure<br>to Run | Partial            | All four emergency service water pumps showed cavitation damage. Two of the pumps had minor damage and were placed back in service. Recirculation cavitation occurs at flows significantly less than design.  |
| 109  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Casing                 | Quality         | HPI    | 1987 | Failure<br>to Run | Partial            | During inspection of a centrifugal charging pump, a portion of the stainless steel cladding on the inside surface of the pump casing exhibited corrosion. Corrosion of the pump casing was through the stainless steel cladding into the carbon steel base material. Inspection of the other CCP revealed similar corrosion. The cause of this event was a manufacturing deficiency. Corrosion observed at the pump casing discharge nozzle was attributed to a cladding breakthrough during final machining. Corrosion observed at the pump casing inlet end was attributed to either over-machining of the cladding or inadequate overlay of two adjacent weld beads.   |
| 110  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Casing                 | Quality         | AFW    | 1983 | Failure<br>to Run | Partial            | Two AFW pumps thrust tolerance was out of specification. These events were caused by improperly installed balancing drum parts. One turbine driven and one motor driven pump was involved.  |
| 111  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Lubrication            | Environmental   | HPI    | 1995 | Failure<br>to Run | Partial            | High lube oil temperatures were observed during HPI pump operation. Zinc particles from anode were discovered plugging the lube oil coolers. Accelerated corrosion was attributed to a corrosion inhibitor that was added to the system, which chemically interacted with the zinc.   |
| 112  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Packing/Seals          | Maintenance     | ESW    | 1997 | Failure<br>to Run | Partial            | Both ESW pumps leaking greater than 4 gpm because of inappropriate material for packing and sleeve (nitronic 60).   |
| 113  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Casing                 | Maintenance     | ESW    | 1997 | Failure<br>to Run | Almost<br>Complete | Both ESW pumps failed due to installation of wrong material for pump casing flanges by vendor during pump overhaul. The vendor overhauled the pumps without changing material. The plant returned the pumps to the warehouse also without verifying material.   |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part          | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|--|------------------|---------------------|-----------------|--------|------|------------------|-------------------|--|
| 114  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Coupling            | Environmental   | ESW    | 1987 | Failure to Start | Partial           | Test showed two ESW pumps failed. Pump shafts were corroded and found to be made of incorrect material.  |
| 115  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Coupling            | Design          | ESW    | 1994 | Failure to Start | Partial           | Pump produced no flow when started. A shaft coupling failed. Material was determined to be brittle and have low impact properties. The coupling was replaced on all pumps with a type of material more suitable for this application.  |
| 116  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.   |
| 117  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Impeller/Wear Rings | Quality         | ESW    | 1986 | Failure to Start | Partial           | Testing of the service water system disclosed that the performance of the three service water pumps was below requirements. The condition is the result of both an inadequate system design and the installation of replacement impellers, which were not modified by the vendor to improve performance, as were the original impellers.   |
| 118  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Shaft               | Design          | AFW    | 1988 | Failure to Run   | Almost Complete   | An auxiliary feedwater pump failed its performance test. Subsequent inspection of the pump internals revealed significant damage, including a split in the center shaft sleeve. The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.  |
| 119  | Pump    | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Shaft               | Design          | AFW    | 1988 | Failure to Run   | Partial           | The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.  |
| 120  | Pump    | External Environment   | Inspection       | Bearing             | Design          | HPI    | 1991 | Failure to Run   | Almost Complete   | Charging/safety pumps beyond operational limits. Damage was found to the thrust bearings. Air was introduced into this train of chilled water during modifications and testing being performed on the system. This air became trapped in high points of either, or both of, the supply and return chilled water lines to the charging pump. At the reduced flow rate, sufficient cooling was not available and oil temperature increased to the point where bearing damage occurred. |
| 121  | Pump    | External Environment   | Inspection       | Coupling            | Environmental   | ESW    | 1993 | Failure to Run   | Partial           | Entrained debris caused ESW pump shaft coupling to fail. Plant equipment did not prevent this debris from entering pump.   |
| 122  | Pump    | External Environment   | Inspection       | Packing/Seals       | Environmental   | RHR-P  | 1985 | Failure to Start | Complete          | Following a trip, water was found spraying from both low head safety injection pump wedge control rod seals. Both pumps were declared inoperable. Postulated failure on the seals was from a minor flow induced pressure transient.  |
| 123  | Pump    | Internal to Component  | Demand           | Bearing             | Maintenance     | AFW    | 1984 | Failure to Run   | Partial           | One ESW bearing failed and pump seized; second motor bearing failed.   |



| Item | Segment | Proximate Cause       | Discovery Method | Piece Part          | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|-----------------------|------------------|---------------------|-----------------|--------|------|------------------|-------------------|---|
| 124  | Pump    | Internal to Component | Demand           | Casing              | Maintenance     | ESW    | 1998 | Failure to Start | Partial           | Two ESW pump started and ran, but would not develop sufficient pressure or flow rate. Exact cause not known for either failure, however, one pump was noted to have microbiological induced corrosion fouling on internal surfaces. |
| 125  | Pump    | Internal to Component | Demand           | Impeller/Wear Rings | Environmental   | ESW    | 1994 | Failure to Run   | Partial           | Raw water pump currents stayed high after starting. The primary cause of these events was determined to be elevated sand content in the river, resulting in excessive sand accumulation around the suction area of the pumps.       |
| 126  | Pump    | Internal to Component | Demand           | Impeller/Wear Rings | Quality         | AFW    | 1988 | Failure to Run   | Partial           | Following a plant trip, it was discovered that the auxiliary feedwater pumps had internal damage. Some channel ring vanes had chips missing, and several parts were found in the SG auxiliary feedwater piping.                     |
| 127  | Pump    | Internal to Component | Demand           | Packing/Seals       | Maintenance     | AFW    | 1998 | Failure to Run   | Partial           | AFW MDP and TDPs failed due to incorrect packing installed.   |
| 128  | Pump    | Internal to Component | Inspection       | Bearing             | Maintenance     | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. Loose fittings and lack of thread sealant.   |
| 129  | Pump    | Internal to Component | Inspection       | Bearing             | Maintenance     | ESW    | 1987 | Failure to Run   | Partial           | Service water pumps had high shaft vibration. The excessive vibrations caused by worn bearings and shaft sleeves.   |
| 130  | Pump    | Internal to Component | Inspection       | Casing              | Maintenance     | ESW    | 1988 | Failure to Run   | Partial           | RHR service water pumps. Pump diffuser eroded on first pump and a through wall casing leak developed on the second.   |
| 131  | Pump    | Internal to Component | Inspection       | Casing              | Maintenance     | ESW    | 1986 | Failure to Run   | Partial           | Cracked seal water and vent lines.  |
| 132  | Pump    | Internal to Component | Inspection       | Impeller/Wear Rings | Environmental   | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Marine growth in suction.  |
| 133  | Pump    | Internal to Component | Inspection       | Impeller/Wear Rings | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. The cause of the failure is suspected to be binding.   |
| 134  | Pump    | Internal to Component | Inspection       | Lubrication         | Environmental   | HPI    | 1983 | Failure to Run   | Partial           | Oysters and miscellaneous mollusks plugged HPI oil coolers. Two pumps were required to be shutdown due to rising lubricating oil temperatures.  |
| 135  | Pump    | Internal to Component | Inspection       | Lubrication         | Design          | HPI    | 1981 | Failure to Run   | Partial           | Corrosion of HPI pump cooler heads. Improper material led to corrosion  |
| 136  | Pump    | Internal to Component | Inspection       | Lubrication         | Maintenance     | RHR-B  | 1990 | Failure to Run   | Partial           | Both pump motor oil coolers were leaking due to aging of components. The first case involved through wall corrosion and the pump was immediately removed from service. The second case was a packing leak.                          |
| 137  | Pump    | Internal to Component | Inspection       | Packing             | Maintenance     | AFW    | 1986 | Failure to Run   | Partial           | The packing was worn on both the motor-driven and one turbine-driven aux. feedwater pump, causing high temperature on one packing gland, and excessive leaking on the other pump.   |
| 138  | Pump    | Internal to Component | Inspection       | Packing/Seals       | Maintenance     | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking profusely at the packing. The failure of the packing was attributed to normal wear.  |
| 139  | Pump    | Internal to Component | Inspection       | Packing/Seals       | Maintenance     | SLC    | 1987 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing adjusted.  |
| 140  | Pump    | Internal to Component | Inspection       | Packing/Seals       | Maintenance     | AFW    | 1990 | Failure to Run   | Partial           | Both motor-driven aux. feedwater pumps had excessive packing leaks, due to worn packing.  |
| 141  | Pump    | Internal to Component | Inspection       | Packing/Seals       | Environmental   | ESW    | 1994 | Failure to Run   | Partial           | Backup seal water regulators did not provide required flow during testing on two pumps. The third pump lost seal flow while operating. The cause was attributed to plugged lines.   |

| Item | Segment | Proximate Cause       | Discovery Method | Piece Part          | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|-----------------------|------------------|---------------------|-----------------|--------|------|------------------|-------------------|--|
| 142  | Pump    | Internal to Component | Inspection       | Packing/Seals       | Maintenance     | ESW    | 1986 | Failure to Run   | Partial           | Excessive packing leakage. Both events occurred after previous maintenance had been performed for the same problems.   |
| 143  | Pump    | Internal to Component | Inspection       | Packing/Seals       | Maintenance     | ESW    | 1989 | Failure to Run   | Partial           | ESW pump excessive packing leakage.  |
| 144  | Pump    | Internal to Component | Inspection       | Packing/Seals       | Maintenance     | SLC    | 1988 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing replaced.                           |
| 145  | Pump    | Internal to Component | Inspection       | Plunger/Cylinder    | Maintenance     | SLC    | 1989 | Failure to Run   | Partial           | Standby Liquid Control pump seal was leaking excessively. The cause of this failure was normal wear of the plungers, packing, and head gaskets for the plungers (piece parts of the pump). |
| 146  | Pump    | Internal to Component | Maintenance      | Bearing             | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | High ESW pump vibration was caused by wearing of the upper bearings.   |
| 147  | Pump    | Internal to Component | Maintenance      | Lubrication         | Environmental   | HPI    | 1991 | Failure to Run   | Partial           | HPI pump lube oil cooler leaks. Degraded tubes.  |
| 148  | Pump    | Internal to Component | Maintenance      | Lubrication         | Environmental   | HPI    | 1986 | Failure to Run   | Almost Complete   | Clams/sludge fouling of lube oil cooler caused high temperature alarms on two HPI pumps.   |
| 149  | Pump    | Internal to Component | Maintenance      | Lubrication         | Environmental   | HPI    | 1980 | Failure to Run   | Partial           | HPI pump lube oil cooler with tube leak allowed water into oil reservoir.  |
| 150  | Pump    | Internal to Component | Maintenance      | Packing/Seals       | Environmental   | ESW    | 1985 | Failure to Run   | Partial           | First pump developed seal leak due to sand. Second pump had high bearing temperatures due to trash clogging cooling water lines.   |
| 151  | Pump    | Internal to Component | Test             | Bearing             | Environmental   | ESW    | 1992 | Failure to Run   | Partial           | Abrasive particles present in ocean water produced accelerated wear of shaft bearing journals.   |
| 152  | Pump    | Internal to Component | Test             | Bearing             | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 153  | Pump    | Internal to Component | Test             | Coupling            | Maintenance     | ESW    | 1987 | Failure to Start | Almost Complete   | Two ESW pumps had failed couplings. Cause attributed to abnormal stress.   |
| 154  | Pump    | Internal to Component | Test             | Coupling            | Maintenance     | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.                                  |
| 155  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 156  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1991 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 157  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1986 | Failure to Run   | Partial           | ESW pumps had worn impellers and one had a plugged strainer.   |
| 158  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | The charging pump service water pumps degraded. Caused by expected wear of pump due to erosion and corrosion properties of the process fluid involved                                      |
| 159  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1985 | Failure to Start | Partial           | Emergency service water pumps discharge pressure below allowable limits. Causes were loose impellers, dropped impeller, and worn internals.  |
| 160  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1984 | Failure to Run   | Partial           | Containment spray raw water pumps failed flow tests. Aging and normal wear.  |
| 161  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1990 | Failure to Run   | Partial           | ESW pumps had worn and cracked impellers. Aging and normal wear.   |

| Item | Segment | Proximate Cause       | Discovery Method | Piece Part          | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|-----------------------|------------------|---------------------|-----------------|--------|------|------------------|-------------------|--|
| 162  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1987 | Failure to Run   | Partial           | ESW pump low flow. Worn impellers.   |
| 163  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1988 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to brackish water corrosion.   |
| 164  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1990 | Failure to Start | Partial           | ESW impeller gaps too wide. Gaps adjusted.   |
| 165  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1994 | Failure to Run   | Partial           | Two ESW pumps had internal deterioration, one of which was indicated by high vibration readings.   |
| 166  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. One pump also exhibited high vibration.   |
| 167  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1991 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted. |
| 168  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1982 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.                     |
| 169  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1984 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 170  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1986 | Failure to Run   | Partial           | ESW pump performance decreased 15% and 8% respectively since last test. Pumps were replaced.   |
| 171  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | Wear caused high ESW pump bearing temperatures, vibration, and low amperage/flow.  |
| 172  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1981 | Failure to Start | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 173  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 174  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1990 | Failure to Run   | Partial           | ESW pump impeller lift out of adjustment.  |
| 175  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1985 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 176  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1998 | Failure to Start | Partial           | Two ESW pumps failed to develop adequate flow/pressure - pumps degraded.   |
| 177  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | HPI    | 1984 | Failure to Run   | Almost Complete   | One HPI pump seized, the second would have seized if operated.   |
| 178  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1993 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head values. The low pump heads were caused by excessive wear of pump impeller due to sand in the service water.                                 |
| 179  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1985 | Failure to Start | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted. |
| 180  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1982 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |

| Item | Segment | Proximate Cause       | Discovery Method | Piece Part          | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|-----------------------|------------------|---------------------|-----------------|--------|------|------------------|-------------------|---|
| 181  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1988 | Failure to Start | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.  |
| 182  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1988 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 183  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | HPI    | 1983 | Failure to Start | Partial           | SI pump and both CCPs failed to meet the minimum head curve requirements. The cause of pump head capacity degradation has been attributed to normal pump operation. The inability to balance flows has been attributed to the lower head capacity of the pumps. |
| 184  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | RHR-B  | 1985 | Failure to Start | Partial           | The first pump failed to meet required flow rate. The second was drawing excessive amperage. Both conditions were attributed to worn internals.   |
| 185  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1995 | Failure to Start | Partial           | Pumps failed performance test. Sand in water eroded pump internals. Pump lift was adjusted.   |
| 186  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1983 | Failure to Run   | Partial           | RHR Service Water pumps failed flow tests due to wearout and had to be rebuilt.   |
| 187  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1994 | Failure to Start | Partial           | Degraded performance identified during testing. Sand in water was causing accelerated wear of the pump internals. Lift was adjusted for three pumps and one pump internals were replaced.   |
| 188  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1989 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.  |
| 189  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1994 | Failure to Start | Partial           | Two ESW pumps had low discharge pressure during testing. Each pump had worn internals and both pump internals were replaced.  |
| 190  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | HPI    | 1985 | Failure to Start | Partial           | The CCPs were tested and had low flow rates. The most probable cause is attributed to observed degradation of the pumps. The CCPs are subject to normal wear associated with their secondary duty of providing normal charging flow.                            |
| 191  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. A rag was found in one impeller and a plastic bottle in the other.  |
| 192  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1985 | Failure to Run   | Partial           | ESW pumps failed due to worn internals.   |
| 193  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1982 | Failure to Run   | Partial           | Low ESW pump head values were caused excessive wear of pump impeller due to foreign material in the service water.  |
| 194  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.   |
| 195  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Maintenance     | ESW    | 1984 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.   |
| 196  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1995 | Failure to Start | Partial           | Marine growth caused low flow and speed condition for two service water pumps   |
| 197  | Pump    | Internal to Component | Test             | Impeller/Wear Rings | Environmental   | ESW    | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.   |
| 198  | Pump    | Internal to Component | Test             | Lubrication         | Maintenance     | SLC    | 1992 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. The gasket between the crankcase frame cap and the gear housing cover was worn.  |
| 199  | Pump    | Internal to Component | Test             | Packing/Seals       | Maintenance     | ESW    | 1981 | Failure to Start | Partial           | RHR service water pumps failed to meet flow requirements due to seal water leakage and pump wearout.  |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part          | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|--|------------------|---------------------|-----------------|--------|------|------------------|-------------------|---|
| 200  | Pump    | Internal to Component                                      | Test             | Shaft               | Maintenance     | ESW    | 1993 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.  |
| 201  | Pump    | Operational/ Human Error                                   | Demand           | Casing              | Maintenance     | AFW    | 1983 | Failure to Run   | Partial           | During testing, the outboard bearing temperature was high on the turbine-driven AFW pump, due to improper balance drum clearances, caused by improper maintenance. The procedure will be modified and the balance drum clearance reset. While the unit was starting up, the motor-driven AFW pump outboard bearing temperature was high. Excessive thrust bearing clearance caused the balance drum to unbalance, causing the thrust bearing to overheat.   |
| 202  | Pump    | Operational/ Human Error                                   | Demand           | Impeller/Wear Rings | Design          | AFW    | 1990 | Failure to Run   | Almost Complete   | Due to a combination of management error and procedural deficiency, the turbine driven auxiliary feedwater pump was run deadheaded. The operation damaged the pump. When the pump was manually tripped, steam vented back into the suction line, caused another AFW pump to also trip, on a low suction pressure signal.  |
| 203  | Pump    | Operational/ Human Error                                   | Inspection       | Lubrication         | Operational     | HPI    | 1983 | Failure to Start | Complete          | A routine preventive maintenance (oil change) was mistakenly performed on the north charging pump instead of the south as scheduled. Since the south pump was previously cleared for this oil change, and the test pump was valved out, none of these three pumps were in service as required by tech specs for the approximately 20 minutes it took to change the oil in the north pump.   |
| 204  | Pump    | Operational/ Human Error                                   | Maintenance      | Lubrication         | Maintenance     | HPI    | 1991 | Failure to Run   | Partial           | Following an overhaul of the HPI pumps. Too much oil flow led to excessive oil leakage, which would have failed HPI pumps before end of mission.  |
| 205  | Pump    | Operational/ Human Error                                   | Maintenance      | Lubrication         | Operational     | ESW    | 1993 | Failure to Run   | Partial           | Low pressure RHR bearing oil level not maintained high enough when new smaller sightglass installed. Second event the sightglass was broken when adding oil.  |
| 206  | Pump    | Operational/ Human Error                                   | Test             | Casing              | Maintenance     | RHR-P  | 1989 | Failure to Start | Complete          | Both loops of the residual heat removal system were declared inoperable due to gas binding of both RHR pumps. The gas binding was caused by entry of nitrogen gas into the reactor coolant system from accumulator. The root cause of this event has been attributed to personnel error. Personnel did not comply with the specific requirements in the accumulator discharge check valve full flow test procedure due to inattention to detail.  |
| 207  | Pump    | Operational/ Human Error                                   | Test             | Packing/Seals       | Maintenance     | AFW    | 1996 | Failure to Run   | Partial           | During the performance of Steam-Driven Emergency Feedwater Pump testing, sparks were observed emanating from the outboard mechanical seal area. The sparks appeared to be due to a mechanical interference within the mechanical seal assembly. The pump mechanical seal was disassembled and determined to have been improperly installed during the last refueling outage. The evaluation identified a mechanical seal design deficiency and inadequate corrective action for a previously identified event as the primary causes for this event. A contributing cause for this event was found to be inadequate predictive maintenance techniques. The electric AFW pump exhibited the same problem. |
| 208  | Pump    | Other  | Inspection       | Bearing             | Operational     | ESW    | 1991 | Failure to Run   | Partial           | Lube oil cooling water isolated during a test. Pumps continued to run with no cooling.  |
| 209  | Suction | Design/ Construction/ Manufacture/ Installation Inadequacy | Demand           | I&C                 | Maintenance     | HPI    | 1997 | Failure to Run   | Complete          | HPI pumps fail due to operation with inadequate suction head. Two pumps damaged due to operation with inadequate suction, but all three system pumps were unavailable due to the loss of the suction source. Suction source level instrumentation was the cause.  |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|--|------------------|------------|-----------------|--------|------|------------------|-------------------|--|
| 210  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1996 | Failure to Start | Partial           | Freezing of diesel generator service water piping in intake bay. Inadequate initial design.                              |
| 211  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1981 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.                               |
| 212  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1981 | Failure to Run   | Complete          | Increasing flow to chillers robs NPSH from charging service water pumps.   |
| 213  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1983 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.                               |
| 214  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the Charging Water Service Water pumps. |
| 215  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.  |
| 216  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1982 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.                               |
| 217  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1982 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.                               |
| 218  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.  |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description   |
|------|---------|--|------------------|------------|-----------------|--------|------|------------------|-------------------|---|
| 219  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Design          | ESW    | 1983 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Pump Service Water pumps.   |
| 220  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Demand           | Piping     | Quality         | ESW    | 1984 | Failure to Start | Partial           | Both RHR service water pumps tripped as a result of inadequate venting of suction header resulting from poor orientation of the vent line.  |
| 221  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Piping     | Design          | HPI    | 1991 | Failure to Start | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the alternate boration line and the gravity feed line from the boric acid storage tank.   |
| 222  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Piping     | Design          | HPI    | 1988 | Failure to Run   | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the suction piping.   |
| 223  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Piping     | Design          | HPI    | 1988 | Failure to Start | Partial           | It was determined that various pipes of the safety injection system and chemical volume and control system collected or trapped gas which might affect the functions of these systems. There was a concern that the gas pockets may adversely effect pump operation. Voids were detected in some of the high head SI pump piping.   |
| 224  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Piping     | Quality         | HPI    | 1988 | Failure to Run   | Partial           | Vortex breakers had not been installed in the containment emergency sumps. Vortex breakers are required to be installed in the containment emergency sumps to prevent the formation of vortices which could adversely affect performance of safety injection pumps during the safety injection and containment spray systems were declared inoperable.  |
| 225  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Piping     | Design          | HPI    | 1990 | Failure to Start | Partial           | A quantity of gas was found in the centrifugal charging pump suction header that exceeded the maximum allowed gas volume. It was subsequently determined that hydrogen gas had been coming out of solution on both units and accumulating in the suction piping as a probable result of gas stripping by the CCP miniflow orifices. In addition, entrainment of hydrogen bubbles from the volume control tank to the CCP suction pipe may be a contributor as well. |
| 226  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Inspection       | Strainer   | Environmental   | ESW    | 2000 | Failure to Run   | Partial           | RHRWS Pumps Failed to Develop flow/pressure. Debris in intake structure. Requires modifications to the traveling Water Screen.  |

| Item | Segment | Proximate Cause  | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode   | Degree of Failure | Description   |
|------|---------|--|------------------|------------|-----------------|--------|------|----------------|-------------------|---|
| 227  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | Tank       | Design          | ESW    | 1985 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 228  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | Tank       | Design          | ESW    | 1985 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 229  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | Tank       | Design          | ESW    | 1990 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 230  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | Tank       | Design          | ESW    | 1990 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 231  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | Tank       | Design          | ESW    | 1985 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 232  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Maintenance      | Tank       | Design          | ESW    | 1990 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.   |
| 233  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Piping     | Design          | AFW    | 1999 | Failure to Run | Partial           | All AFW trains declared inoperable due to inadequate suction flow capability from the nuclear service water alternate source. Inadequate flow caused by corroded piping. Piping is undersized so there is little margin for piping degradation. Since this is 1 of 4 suction sources, the safety significance is limited. |
| 234  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Tank       | Design          | ESW    | 1986 | Failure to Run | Complete          | Loss of prime in the condenser circulating water siphon flow system caused loss of low pressure service water pumps. Pumps lost suction during a test due to poor design.   |
| 235  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Tank       | Design          | SLC    | 1991 | Failure to Run | Complete          | During the performance of a special test on Unit 1 to determine the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.     |



| Item | Segment | Proximate Cause  | Discovery Method | Piece Part   | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|--|------------------|--------------|-----------------|--------|------|------------------|-------------------|--|
| 236  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Tank         | Design          | SLC    | 1991 | Failure to Run   | Complete          | During the performance of a special test on the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.  |
| 237  | Suction | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | Test             | Valve        | Design          | ESW    | 1983 | Failure to Start | Partial           | Low discharge pressure was caused by insufficient suction pressure. Service water flow to parallel components was adjusted.  |
| 238  | Suction | External Environment   | Demand           | Piping       | Environmental   | HPI    | 1984 | Failure to Start | Complete          | Boron solidification in the suction and gas binding of pumps led to the failure of all three safety injection pumps. Flushing procedures inadequate.   |
| 239  | Suction | Internal to Component  | Demand           | Piping       | Environmental   | ESW    | 1986 | Failure to Start | Partial           | RHR service water pumps failed flow testing due to blocked suctions and abnormal wear of impellers.  |
| 240  | Suction | Internal to Component  | Demand           | Strainer     | Environmental   | ESW    | 1980 | Failure to Run   | Partial           | Foreign material was allowed to enter the suction of the charging pump service water pumps resulting in low flow conditions.   |
| 241  | Suction | Internal to Component  | Inspection       | Strainer     | Environmental   | ESW    | 1984 | Failure to Run   | Partial           | Two RHR service water pumps had blown seals and sparks and smoke between the bearing housing and shaft. A piece of hard rubber valve liner was found in the pumps.   |
| 242  | Suction | Internal to Component  | Test             | Piping       | Environmental   | ESW    | 1990 | Failure to Start | Partial           | ESW pumps failed flow testing. Foreign material blocked the suction.   |
| 243  | Suction | Internal to Component  | Test             | Strainer     | Environmental   | ESW    | 1990 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by suction blockage due to foreign material in the service water.  |
| 244  | Suction | Internal to Component  | Test             | Strainer     | Environmental   | ESW    | 1982 | Failure to Run   | Partial           | Failures occurred on residual heat removal service water pumps. The pumps failed to meet flow and pressure requirements. Failure was due to debris lodging in pump impellers. Source of debris was maintenance activities, broken traveling water screens, and the inadvertent opening of a RHR minimum flow line which washed materials into suction pit.   |
| 245  | Suction | Operational/ Human Error   | Demand           | Booster Pump | Operational     | ESW    | 1980 | Failure to Start | Partial           | The service water RHR booster pump was de-energized during maintenance. The attempt to start service water pumps failed due to low suction pressure.   |
| 246  | Suction | Operational/ Human Error   | Demand           | Piping       | Design          | RHR-P  | 1984 | Failure to Run   | Almost Complete   | On two occasions, RHR pumps cavitated due to low RCS level while draining the RCS.   |
| 247  | Suction | Operational/ Human Error   | Demand           | Piping       | Design          | RHR-P  | 1985 | Failure to Run   | Complete          | Swap over of RHR pumps resulted in both trains becoming inoperable due to air injection into the suction of the pumps. This required both pumps to be vented and required RCS level to be raised to prevent a possible recurrence of the vortex problem.   |
| 248  | Suction | Operational/ Human Error   | Demand           | Piping       | Operational     | RHR-P  | 1980 | Failure to Run   | Complete          | While attempting to increase RHR flow, the plant experienced a total loss of flow due to the pumps being air-bound. The pump was not vented when starting to increase flow. Operating procedures have been changed to have an operator present while changing flow in the RHR system. There have been losses of RHR flow in the past because the pumps were air-bound and methods are being investigated to improve the system design. |

| Item | Segment | Proximate Cause          | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|--------------------------|------------------|------------|-----------------|--------|------|------------------|-------------------|--|
| 249  | Suction | Operational/ Human Error | Demand           | Piping     | Operational     | RHR-P  | 1984 | Failure to Run   | Complete          | The control room operators started a second residual heat removal pump in preparation for removing the operating RHR pump from service. With both pumps running, flow became excessive for the half-loop condition causing cavitation and air binding of both pumps. To prevent recurrence the procedure which controls the operation of the RHR pumps has been changed to include specific instructions to stop the operating pump prior to starting the second pump while at half-loop.              |
| 250  | Suction | Operational/ Human Error | Demand           | Piping     | Maintenance     | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.  |
| 251  | Suction | Operational/ Human Error | Demand           | Piping     | Operational     | ESW    | 1988 | Failure to Run   | Complete          | The procedure failed to adequately caution the operator to slowly fill a drained line. Rapid filling resulted in a loss of NPSH to the charging service water pumps.   |
| 252  | Suction | Operational/ Human Error | Demand           | Piping     | Maintenance     | ESW    | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test.  |
| 253  | Suction | Operational/ Human Error | Demand           | Piping     | Operational     | ESW    | 1986 | Failure to Run   | Complete          | Failure to properly vent and fill a newly installed pipe introduced air into the charging pump service water system.   |
| 254  | Suction | Operational/ Human Error | Demand           | Piping     | Design          | RHR-P  | 1980 | Failure to Run   | Complete          | The reactor vessel vent eductor was in service in preparation for refueling with RHR operating. A low flow alarm was received and low flow and low motor current were indicated. A second pump was started and became air-bound. Putting the vessel vent eductor system into service was the root cause of the incident.   |
| 255  | Suction | Operational/ Human Error | Demand           | Piping     | Design          | RHR-P  | 1982 | Failure to Run   | Complete          | Suction was lost to both RHR pumps. RHR flow was less than 3000 gpm and pump amps were fluctuating prior to taking corrective action. Each of these events appear to have been caused by a slow decrease in RCS level in conjunction with the vortex action at the pump suction.   |
| 256  | Suction | Operational/ Human Error | Demand           | Tank       | Design          | AFW    | 1980 | Failure to Run   | Complete          | Both emergency feedwater pumps lost feed pump suction. The emergency feedwater pump suction flashed to steam due to the feedwater train flashing and forcing hot water back through the startup and blowdown tanks and into the feedwater pump suction. To prevent this recurrence, the operating procedures have been changed to require isolating the startup and blowdown effluent as a source of emergency feedwater suction prior to increasing power.  |
| 257  | Suction | Operational/ Human Error | Inspection       | Valve      | Maintenance     | SLC    | 1991 | Failure to Start | Partial           | SLC pumps were potentially inoperable during part of test due to valve lineup.   |
| 258  | Suction | Operational/ Human Error | Maintenance      | Piping     | Maintenance     | RHR-P  | 1982 | Failure to Run   | Complete          | Shutdown cooling was lost due to nitrogen intrusion because of backflushing a filter in the purification system.   |
| 259  | Suction | Operational/ Human Error | Maintenance      | Strainer   | Operational     | ESW    | 1986 | Failure to Run   | Complete          | A service water strainer was placed in service without being vented resulting in air binding system and loss of charging pump service water pumps.   |
| 260  | Suction | Operational/ Human Error | Maintenance      | Strainer   | Maintenance     | HPI    | 1985 | Failure to Run   | Partial           | Strainers found still installed in the suction piping of the high-pressure injection pumps was a condition not considered in the operating design. The strainers were found during maintenance to repair a slight flange leak. The strainers had been placed in the suction piping during construction and were to be in place during system flushing to prevent any debris from reaching the pumps. However, the strainers should have been removed after system flushing prior to functional testing |

| Item | Segment | Proximate Cause          | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode     | Degree of Failure | Description  |
|------|---------|--------------------------|------------------|------------|-----------------|--------|------|------------------|-------------------|--|
| 261  | Suction | Operational/ Human Error | Test             | Piping     | Operational     | ESW    | 1989 | Failure to Run   | Partial           | Inadequate procedure led to air binding of operating ESW pumps.  |
| 262  | Suction | Other                    | Demand           | I&C        | Design          | HPI    | 1997 | Failure to Run   | Partial           | Letdown storage tank reference leg not full, which gave erroneous indication of sufficient tank level. One HPI pump severely damaged, other pump not as damaged, and could have run. The root cause was a combination of a design weakness of a common reference leg for the Letdown storage tank level instruments and a leaking instrument fitting due to an inadequate work practice.   |
| 263  | Suction | Other                    | Demand           | Piping     | Design          | HPI    | 1982 | Failure to Start | Complete          | Hydrogen from the suction dampener got into suction piping and failed both CCPs.   |
| 264  | Suction | Other                    | Demand           | Piping     | Maintenance     | RHR-P  | 1986 | Failure to Run   | Complete          | SDC pumps cavitated due to lowering RCS level. Level indication was in error.  |
| 265  | Suction | Other                    | Demand           | Piping     | Design          | ESW    | 1980 | Failure to Run   | Almost Complete   | Air ingress exceeded the air removal capability of the constant vent valves. A design change was implemented to remove the air compressor cooling from the service water system.   |
| 266  | Suction | Other                    | Demand           | Piping     | Maintenance     | RHR-P  | 1983 | Failure to Run   | Complete          | The RHR pumps began to cavitate and eventually both pumps were stopped. The reactor vessel level gauge being used to provide an indication that the level was approaching the vessel flange level had been isolated (reactor coolant drain tank isolation valve had been closed during an attempt to reduce leakage). Additionally, procedures did not require visual monitoring of cavity level.  |
| 267  | Suction | Other                    | Demand           | Piping     | Design          | RHR-P  | 1982 | Failure to Run   | Complete          | RHR Suction lost due to erroneous RCS level while draining the RCS.  |
| 268  | Suction | Other                    | Demand           | Piping     | Design          | RHR-P  | 1982 | Failure to Run   | Complete          | With unit drained to centerline of the nozzles, suction to both RHR pumps was lost for 36 minutes. Suction to the RHR pumps was lost because of ambiguous reactor coolant system level indication while drained to centerline of the nozzles. The actual RCS level was lower than observed.  |
| 269  | Suction | Other                    | Demand           | Piping     | Maintenance     | RHR-P  | 1981 | Failure to Run   | Complete          | Temporary coolant loop level indicator showed level slowly increasing over a period of days. The system was periodically drained to maintain 65 percent indicated level. A RHR pump lost suction on reduction of actual level. The second pump was started, and lost suction. Indication drift was due to evaporation of reference leg.  |
| 270  | Suction | Other                    | Demand           | Piping     | Maintenance     | RHR-P  | 1980 | Failure to Run   | Complete          | A complete loss of RHR flow occurred while plant operators were increasing RHR heat exchanger flow by closing down on the heat exchanger bypass valve.   |
| 271  | Suction | Other                    | Demand           | Piping     | Design          | RHR-P  | 1987 | Failure to Run   | Complete          | RHR flow was interrupted when both RHR trains became inoperable due to air bound RHR pumps. The loss of RCS inventory to the reactor coolant drain tank due to a leaking valve caused a decrease in RCS water level, vortexing in the pumps' suction line, and air entrainment in the RHR pumps.   |
| 272  | Suction | Other                    | Demand           | Valve      | Design          | RHR-P  | 1984 | Failure to Run   | Complete          | Both RHR pumps were unable to operate due to the introduction of air into the RHR system. The incident occurred during the drain down of the RCS, when the level of the RCS was being monitored via a standpipe off the centerline of one of the RCS loops. The isolation valve to which the standpipe was attached became clogged sometime during the drain down and falsely indicated above centerline when in fact the level was below the RHR suction line (below centerline). |

| Item | Segment | Proximate Cause | Discovery Method | Piece Part | Coupling Factor | System | Year | Failure Mode   | Degree of Failure | Description  |
|------|---------|-----------------|------------------|------------|-----------------|--------|------|----------------|-------------------|--|
| 273  | Suction | Other           | Test             | I&C        | Design          | AFW    | 1985 | Failure to Run | Almost Complete   | Testing of the turbine driven AFW pump resulted in a low suction trip of the motor driven pump. The turbine driven pump had a faulty governor. It was during the post maintenance test of turbine driven pump that speed oscillations occurred causing pressure oscillations in the suction of the motor driven pump that was in service. Foreign material in the suction gauge protectors resulted in the pressure sensors sensing only the low pressures and not the high pressures of the oscillations, so the motor driven pump tripped on low pressure. |
| 274  | Suction | Unknown         | Demand           | Piping     | Design          | RHR-P  | 1983 | Failure to Run | Complete          | RHR pumps cavitated. Unable to repeat. Unknown cause.  |



**Appendix C**  
**Pump CCF Data Summary by System**



## **Appendix C**

### **Pump CCF Data Summary by System**

This appendix is a summary of the data evaluated in the common-cause failure (CCF) data collection effort for pumps. This appendix supports the charts in Chapter 5. The table is sorted alphabetically, by the first four columns.



## Appendix C

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| Table C-1. Pump CCF event summary, by system. .... | 3 |
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Table C-1. Pump CCF event summary, by system.

| Item | System | Segment   | Piece Part  | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|-----------|-------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 1    | AFW    | Discharge | Check Valve | Demand           | Design          | External Environment                                       | 1983 | Failure to Start | Almost Complete   | Hot water in the AFW pump casings caused the pumps to become vapor bound. The hot water was from leaking check valves upstream of the pumps. This event occurred once on the turbine driven pump and 5 times on the motor driven pump.  |
| 2    | AFW    | Discharge | Check Valve | Inspection       | Maintenance     | Internal to Component                                      | 1990 | Failure to Start | Almost Complete   | Leakage past AFW check valves caused AFW pumps to become steam bound. Closed motor operated valve in line. Scheduled check valves for replacement next outage.  |
| 3    | AFW    | Discharge | Valve       | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1985 | Failure to Start | Partial           | Controller problems in the steam and diesel driven AFW pumps caused the pumps to trip on low suction pressure. The pump discharge flow controller valves were also not set properly after last maintenance. Low suction trips were due to design error.   |
| 4    | AFW    | Discharge | Valve       | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1986 | Failure to Start | Partial           | Both the turbine driven and motor driven AFW pumps could not produce full flow because the cages in their discharge valve trapped debris and plugged.   |
| 5    | AFW    | Discharge | Valve       | Demand           | Environmental   | Internal to Component                                      | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitrol cages for these valves were clogged with shredded Asiatic clam shells. |
| 6    | AFW    | Discharge | Valve       | Demand           | Environmental   | Internal to Component                                      | 1988 | Failure to Run   | Partial           | After automatic start, motor driven AFW pump swapped suction automatically to the nuclear service water system when a sustained low suction pressure condition was sensed, and raw water entered two steam generators. After the initial trip recovery, it was noted that AFW flow to steam generators had degraded following the suction swap. Inspections revealed that the cavitrol cages for these valves were clogged with shredded Asiatic clam shells. |
| 7    | AFW    | Discharge | Valve       | Inspection       | Operational     | Operational/ Human Error                                   | 1994 | Failure to Start | Complete          | Following a trip, the AFW Pumps were secured and the discharge flow control valves for the Motor Driven Pumps were closed. Later, an operator discovered during a routine Control Board walkdown that the valves were closed. Subsequent investigation revealed the AFW system had not been placed in standby readiness per the operating procedure after the system was secured.   |
| 8    | AFW    | Driver    | Breaker     | Demand           | Maintenance     | External Environment                                       | 1990 | Failure to Run   | Partial           | AFW pumps circuit breakers degraded.  |
| 9    | AFW    | Driver    | Breaker     | Maintenance      | Maintenance     | Internal to Component                                      | 1992 | Failure to Start | Partial           | With the unit in a refueling outage, following repairs to a motor driven auxiliary feedwater pump local/remote switch of the circuit breaker, personnel found that the switch contacts would not close. This failure rendered one of three auxiliary feedwater pumps inoperable. The cause of the failure appears to be due to dirty/corroded contacts on the switch.   |
| 10   | AFW    | Driver    | Breaker     | Test             | Maintenance     | Internal to Component                                      | 1997 | Failure to Start | Almost Complete   | The circuit breakers associated with the AFW Pumps failed to close as required. The root cause of the failure was the binding in the operating mechanism. The plunger apparently did not always complete its upward movement to close and latch the breaker, due to accumulated dirt and lubricants.  |

| Item | System | Segment | Piece Part | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 11   | AFW    | Driver  | Breaker    | Test             | Design          | Operational/ Human Error                                   | 1985 | Failure to Start | Complete          | Both AFW pumps failed to start when tested, due to the circuit breakers not being racked in properly.   |
| 12   | AFW    | Driver  | I&C        | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1981 | Failure to Start | Almost Complete   | Two AFW pumps failed to automatically start due to low suction pressure trips. A modification was installed to prevent this. This effect was discovered previously, but apparently had not been corrected prior to an attempt to start the pumps three weeks later.   |
| 13   | AFW    | Driver  | I&C        | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1997 | Failure to Run   | Partial           | One actual AFW pump failure due to spurious electronic overspeed trip. Determined that all three pumps were susceptible to spurious overspeed trips.  |
| 14   | AFW    | Driver  | I&C        | Demand           | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | 1989 | Failure to Start | Complete          | Both motor driven auxiliary feedwater pumps failed to start when the operator tried to start them manually. While preparing a design change, the designer failed to review all the unit specific documentation associated with the motor-driven AFW pump wiring and made the erroneous assumption that both units switchgear compartment internal wiring was identical. In fact, the wiring for each unit was different. Consequently, when the design change was installed, it was installed in accordance with the erroneous design. The wiring discrepancy was corrected and the motor-driven AFW pumps were tested and returned to service. |
| 15   | AFW    | Driver  | I&C        | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1981 | Failure to Start | Almost Complete   | A modification to the control instrumentation for two AFW pumps resulted in a backfeed situation such that when called upon to start, both pumps would not start.   |
| 16   | AFW    | Driver  | I&C        | Demand           | Operational     | Operational/ Human Error                                   | 1983 | Failure to Start | Complete          | An operator incorrectly secured the diesel and steam driven AFW pumps, which prevented their restart on low SG level.   |
| 17   | AFW    | Driver  | I&C        | Demand           | Environmental   | External Environment                                       | 1984 | Failure to Start | Complete          | Both AFW pumps failed to start. The problem was traced to two relays (1 per pump). Examination of the relays revealed open circuiting and severe degradation of the insulation.   |
| 18   | AFW    | Driver  | I&C        | Inspection       | Design          | Other  | 1983 | Failure to Start | Partial           | Both AFW pumps had to be rendered inoperable to allow repairs to actuation circuitry.   |
| 19   | AFW    | Driver  | I&C        | Inspection       | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1994 | Failure to Start | Partial           | Single failure would prevent auto initiation of AFW. Circuit design did not provide separation required by standards and code. The single failure identified was a short circuit across two conductors of the actuation relays associated with the initiation logic matrix.   |
| 20   | AFW    | Driver  | I&C        | Inspection       | Maintenance     | Operational/ Human Error                                   | 1990 | Failure to Start | Complete          | During testing one AFW pump was tested and other was tested without returning first to auto. Both pumps were unavailable at the same time. The procedure was the cause.   |
| 21   | AFW    | Driver  | I&C        | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1981 | Failure to Start | Almost Complete   | Two low suction pressure trips for the AFW pumps were mis-calibrated, which prevented the pumps from starting.  |

| Item | System | Segment | Piece Part          | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|---------------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 22   | AFW    | Driver  | I&C                 | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1992 | Failure to Start | Complete          | A modification design error (in 1983-1984) removed a start permissive interlock contact. At cold shutdown this de-energized the auxiliary lube oil pump, consequently, when one AFW pump was started it ran for 2.5 seconds and tripped on low oil pressure. Further investigation showed that both units AFW pumps would be affected in the same way. The design error combined with insufficient post modification testing led to this CCF event.                   |
| 23   | AFW    | Driver  | I&C                 | Test             | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1980 | Failure to Start | Complete          | During surveillance testing, neither motor-driven AFW pump would start. The pump control circuit was found with autostart defeat switches labeled backwards, causing all autostarts except the low-low steam generator level to be defeated. The labels were corrected and the links were closed. The original installation error was the result of an inadequate design change process that did not require sufficient verification and testing of the modification. |
| 24   | AFW    | Driver  | Motor               | Inspection       | Environmental   | Other  | 1990 | Failure to Start | Partial           | Both motor driven AFW pumps were sprayed when a service water pipe developed a through wall leak.   |
| 25   | AFW    | Pump    | Bearing             | Demand           | Maintenance     | Internal to Component  | 1984 | Failure to Run   | Partial           | One ESW bearing failed and pump seized; second motor bearing failed.  |
| 26   | AFW    | Pump    | Casing              | Demand           | Maintenance     | Operational/ Human Error   | 1983 | Failure to Run   | Partial           | During testing, the outboard bearing temperature was high on the turbine-driven AFW pump, due to improper balance drum clearances, caused by improper maintenance. The procedure will be modified and the balance drum clearance reset. While the unit was starting up, the motor-driven AFW pump outboard bearing temperature was high. Excessive thrust bearing clearance caused the balance drum to unbalance, causing the thrust bearing to overheat.             |
| 27   | AFW    | Pump    | Casing              | Inspection       | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1983 | Failure to Run   | Partial           | Two AFW pumps thrust tolerance was out of specification. These events were caused by improperly installed balancing drum parts. One turbine driven and one motor driven pump was involved.  |
| 28   | AFW    | Pump    | Impeller/Wear Rings | Demand           | Quality         | Internal to Component  | 1988 | Failure to Run   | Partial           | Following a plant trip, it was discovered that the auxiliary feedwater pumps had internal damage. Some channel ring vanes had chips missing, and several parts were found in the SG auxiliary feedwater piping.   |
| 29   | AFW    | Pump    | Impeller/Wear Rings | Demand           | Design          | Operational/ Human Error   | 1990 | Failure to Run   | Almost Complete   | Due to a combination of management error and procedural deficiency, the turbine driven auxiliary feedwater pump was run deadheaded. The operation damaged the pump. When the pump was manually tripped, steam vented back into the suction line, caused another AFW pump to also trip, on a low suction pressure signal.  |
| 30   | AFW    | Pump    | Packing             | Inspection       | Maintenance     | Internal to Component  | 1986 | Failure to Run   | Partial           | The packing was worn on both the motor-driven and one turbine-driven aux. feedwater pump, causing high temperature on one packing gland, and excessive leaking on the other pump.   |
| 31   | AFW    | Pump    | Packing/Seals       | Demand           | Maintenance     | Internal to Component  | 1998 | Failure to Run   | Partial           | AFW MDP and TDPs failed due to incorrect packing installed.   |
| 32   | AFW    | Pump    | Packing/Seals       | Inspection       | Maintenance     | Internal to Component  | 1990 | Failure to Run   | Partial           | Both motor-driven aux. feedwater pumps had excessive packing leaks, due to worn packing.  |

| Item | System | Segment   | Piece Part    | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|-----------|---------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 33   | AFW    | Pump      | Packing/Seals | Test             | Maintenance     | Operational/ Human Error                                   | 1996 | Failure to Run   | Partial           | During the performance of Steam-Driven Emergency Feedwater Pump testing, sparks were observed emanating from the outboard mechanical seal area. The sparks appeared to be due to a mechanical interference within the mechanical seal assembly. The pump mechanical seal was disassembled and determined to have been improperly installed during the last refueling outage. The evaluation identified a mechanical seal design deficiency and inadequate corrective action for a previously identified event as the primary causes for this event. A contributing cause for this event was found to be inadequate predictive maintenance techniques. The electric AFW pump exhibited the same problem. |
| 34   | AFW    | Pump      | Shaft         | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1988 | Failure to Run   | Almost Complete   | An auxiliary feedwater pump failed its performance test. Subsequent inspection of the pump internals revealed significant damage, including a split in the center shaft sleeve. The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 35   | AFW    | Pump      | Shaft         | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1988 | Failure to Run   | Partial           | The AFW pumps were susceptible to corrosion cracking of their bushings. A different material was needed for the bushings.   |
| 36   | AFW    | Suction   | I&C           | Test             | Design          | Other  | 1985 | Failure to Run   | Almost Complete   | Testing of the turbine driven AFW pump resulted in a low suction trip of the motor driven pump. The turbine driven pump had a faulty governor. It was during the post maintenance test of turbine driven pump that speed oscillations occurred causing pressure oscillations in the suction of the motor driven pump that was in service. Foreign material in the suction gauge protectors resulted in the pressure sensors sensing only the low pressures and not the high pressures of the oscillations, so the motor driven pump tripped on low pressure.  |
| 37   | AFW    | Suction   | Piping        | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1999 | Failure to Run   | Partial           | All AFW trains declared inoperable due to inadequate suction flow capability from the nuclear service water alternate source. Inadequate flow caused by corroded piping. Piping is undersized so there is little margin for piping degradation. Since this is 1 of 4 suction sources, the safety significance is limited.   |
| 38   | AFW    | Suction   | Tank          | Demand           | Design          | Operational/ Human Error                                   | 1980 | Failure to Run   | Complete          | Both emergency feedwater pumps lost feed pump suction. The emergency feedwater pump suction flashed to steam due to the feedwater train flashing and forcing hot water back through the startup and blowdown tanks and into the feedwater pump suction. To prevent this recurrence, the operating procedures have been changed to require isolating the startup and blowdown effluent as a source of emergency feedwater suction prior to increasing power.   |
| 39   | CSS    | Driver    | Breaker       | Inspection       | Operational     | Operational/ Human Error                                   | 1991 | Failure to Start | Complete          | CSR control power de-energized prior to mode change. Technical Specification violation. Inadequate procedure review.  |
| 40   | ESW    | Discharge | Check Valve   | Test             | Operational     | Design/ Construction/ Manufacture/ Installation Inadequacy | 1999 | Failure to Run   | Partial           | Two ESW pumps had low flow due to interaction with the two other pumps when all four pumps were running.  |
| 41   | ESW    | Discharge | Valve         | Demand           | Maintenance     | Other  | 1980 | Failure to Start | Partial           | RHR service water pumps were started to put torus cooling in service. When these pumps would not deliver required discharge pressure, they were declared inoperable. The seal in an air release valve was bad, allowing a vent on the discharge line.   |

| Item | System | Segment | Piece Part | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 42   | ESW    | Driver  | Bearing    | Inspection       | Maintenance     | Internal to Component                                      | 1985 | Failure to Run   | Partial           | One service water pump motor upper bearing oil reservoir leaking from cover plate. Another service water pump motor upper oil cooler oil reservoir leaking.   |
| 43   | ESW    | Driver  | Bearing    | Inspection       | Maintenance     | Internal to Component                                      | 1981 | Failure to Run   | Partial           | ESW motor to pump alignment problems. Bearings worn out.  |
| 44   | ESW    | Driver  | Bearing    | Test             | Maintenance     | Internal to Component                                      | 1985 | Failure to Run   | Partial           | Service water pumps exhibited vibration. Attributed to normal wear.   |
| 45   | ESW    | Driver  | Breaker    | Demand           | Maintenance     | Operational/ Human Error                                   | 1993 | Failure to Start | Partial           | Operations personnel were attempting to swap the running service water pump with the idle service water pump. Personnel placed the control switch to start and the service water pump did not start. Breaker malfunction. Later, another service water pump failed to start because of the breaker.   |
| 46   | ESW    | Driver  | Breaker    | Demand           | Design          | Internal to Component                                      | 2000 | Failure to Start | Almost Complete   | Two ESW pumps failed to start due to their breakers failing to close. The breakers' prop spring bracket has slipped thus preventing proper interfacing between the prop and the prop pin.   |
| 47   | ESW    | Driver  | Breaker    | Demand           | Maintenance     | Operational/ Human Error                                   | 1987 | Failure to Start | Partial           | One breaker failed to linkage alignment and second from loose relay connections. Inadequate maintenance.  |
| 48   | ESW    | Driver  | Breaker    | Demand           | Maintenance     | Operational/ Human Error                                   | 1988 | Failure to Run   | Partial           | Service water pump high dropout over current protection devices were less than running current conditions and trip setpoints did not account for changing load conditions due to modified impellers. Three pump trips had occurred.   |
| 49   | ESW    | Driver  | Breaker    | Demand           | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | 1996 | Failure to Start | Partial           | Two RHRSW pumps fail to start due to breaker failures. Wrong contacts were installed. Design called for contacts to have a minimum current interrupt rating of 6 amps; contacts installed (that subsequently failed) had current interrupt rating of only 2.2 amps.   |
| 50   | ESW    | Driver  | Breaker    | Inspection       | Quality         | Operational/ Human Error                                   | 1992 | Failure to Start | Partial           | The fit between an ESW pump breaker primary disconnects and the associated breaker cubicle stabs was inadequate. The poor fit between the disconnects and the stabs led to arcing in the breaker cubicle when the pump was started, resulting in a fire. Shortly after identifying the cause of the fire, the remaining ESW breakers, which had recently been replaced along with the failed breaker, as part of a design modification package, were found to be inadequate also. |
| 51   | ESW    | Driver  | Breaker    | Inspection       | Operational     | Operational/ Human Error                                   | 1981 | Failure to Start | Almost Complete   | Control breakers for two ESW pumps were open due to inadvertent operator action.  |
| 52   | ESW    | Driver  | Breaker    | Inspection       | Maintenance     | Internal to Component                                      | 1996 | Failure to Start | Partial           | ESW pump breakers fail due to misalignment of the breaker mechanism and internals developed over the years of operation.  |
| 53   | ESW    | Driver  | Breaker    | Inspection       | Design          | Operational/ Human Error                                   | 1984 | Failure to Start | Partial           | During an attempt to perform preventive maintenance for unit one's RHR service water pumps, plant personnel mistakenly disconnected the motor leads for unit two's RHR service water pump.  |
| 54   | ESW    | Driver  | Breaker    | Maintenance      | Maintenance     | Other  | 1984 | Failure to Start | Partial           | ESW pump breaker failures, broken screw, no lubrication, and a bent track   |
| 55   | ESW    | Driver  | Breaker    | Maintenance      | Maintenance     | Internal to Component                                      | 1985 | Failure to Start | Partial           | Two raw water pump breaker main wipes were out of adjustment.   |
| 56   | ESW    | Driver  | Breaker    | Maintenance      | Maintenance     | Other  | 1982 | Failure to Start | Partial           | ESW pump circuit breakers found damaged. Defective arc chute and cracked secondary coupler.   |

| Item | System | Segment | Piece Part | Discovery Method | Coupling Factor | Proximate Cause          | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|------------|------------------|-----------------|--------------------------|------|------------------|-------------------|--|
| 57   | ESW    | Driver  | Breaker    | Test             | Maintenance     | Internal to Component    | 1998 | Failure to Start | Partial           | Two RHR service water pump breakers would not close due to dirty contacts in breakers.   |
| 58   | ESW    | Driver  | Breaker    | Test             | Maintenance     | Internal to Component    | 1998 | Failure to Start | Partial           | Service water pumps fail to start due to circuit breaker failures. Pump breakers failed to close due to failures of the charging spring/motor and closing spring motor.  |
| 59   | ESW    | Driver  | Breaker    | Test             | Maintenance     | Other                    | 1984 | Failure to Start | Partial           | ESW pump breaker overcurrent trip devices tripping too low.  |
| 60   | ESW    | Driver  | Breaker    | Test             | Maintenance     | Other                    | 1984 | Failure to Start | Partial           | ESW pump breakers tripped due to failed voltage control devices.   |
| 61   | ESW    | Driver  | I&C        | Demand           | Design          | Other                    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure. This is a second event two months later.  |
| 62   | ESW    | Driver  | I&C        | Demand           | Maintenance     | Other                    | 1982 | Failure to Start | Complete          | Following a reactor scram, an attempt to initiate suppression pool cooling revealed that both RHRSW loops were inoperable as neither loop's pumps could be started. Low suction header pressure lockout signals in each loop prevented starting each loop's pumps. Plugging of the sensing line to each loop's suction header pressure switch prevented both switches from sensing actual pressure, although a lack of operating fluid in one switch and an open power supply breaker to the other switch also would have prevented pumps from starting. |
| 63   | ESW    | Driver  | I&C        | Demand           | Operational     | Operational/ Human Error | 1981 | Failure to Start | Partial           | Alarm circuit breaker was de-energized resulting in a loss of two RHR service water pumps.   |
| 64   | ESW    | Driver  | I&C        | Demand           | Design          | Other                    | 1981 | Failure to Start | Partial           | Attempt was made to place the a RHRSW subsystem into service for use in suppression pool cooling, the subsystems' pumps could not be started due to a pump suction header low pressure lockout signal from the header pressure switch. The threaded plug in the switch diaphragm housing became loose and allowed the diaphragm fluid to leak out and caused the switch to sense a low pressure.   |
| 65   | ESW    | Driver  | I&C        | Demand           | Design          | Operational/ Human Error | 1980 | Failure to Start | Partial           | Instrument isolation valve closed causing a low suction trip signal to two RHRSW pumps.  |
| 66   | ESW    | Driver  | I&C        | Demand           | Maintenance     | Internal to Component    | 1991 | Failure to Start | Partial           | Two ESW pumps failed to start due to failed breakers. Inadequate maintenance.  |
| 67   | ESW    | Driver  | I&C        | Inspection       | Design          | Internal to Component    | 1982 | Failure to Start | Partial           | Open circuit breaker resulted in loss of two RHR service water pumps.  |
| 68   | ESW    | Driver  | I&C        | Test             | Design          | Other                    | 1992 | Failure to Start | Partial           | Valve position contacts prevented ESW pump circuit breakers from closing. Poor design resulted in water intrusion in the valve limit switch box.   |
| 69   | ESW    | Driver  | I&C        | Test             | Operational     | Operational/ Human Error | 1990 | Failure to Start | Complete          | An emergency service water pump failed to start and was declared inoperable. Further investigation determined that the failure of the pump to start was due to a tripped emergency engine shutdown device. Operations personnel performing the testing did not recognize the need to reset it prior to starting the pump. Examination of the other two ESW pumps revealed that their emergency shutdown devices were also in the tripped condition.  |
| 70   | ESW    | Driver  | I&C        | Test             | Quality         | Operational/ Human Error | 1982 | Failure to Start | Partial           | Two ESW pumps failed to start. One ESW pump failed to function as a result of loose wires on relay terminals in both pump logic schemes, a loose states link and an instantaneous contact found out of adjustment on the other pump logic scheme.  |

| Item | System | Segment | Piece Part | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|------------|------------------|-----------------|--|------|------------------|-------------------|--|
| 71   | ESW    | Driver  | I&C        | Test             | Maintenance     | Operational/ Human Error                                   | 1989 | Failure to Start | Partial           | Emergency equipment service water pump relays were not reset following a load shedding test 30 hours before.   |
| 72   | ESW    | Driver  | Motor      | Demand           | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | 1987 | Failure to Start | Partial           | ESW pump motors tripped on overcurrent. The overcurrent trip was due to a ground and a short on the pump motor.  |
| 73   | ESW    | Driver  | Motor      | Demand           | Environmental   | External Environment                                       | 1985 | Failure to Run   | Partial           | Two service water motors failed on demand as a result of cement dust contamination.  |
| 74   | ESW    | Driver  | Motor      | Inspection       | Environmental   | Other  | 1981 | Failure to Run   | Partial           | The float guide failed in a RHRSW pump air valve and caused the valve to fail open and flood pump room.  |
| 75   | ESW    | Driver  | Motor      | Maintenance      | Environmental   | External Environment                                       | 1987 | Failure to Start | Partial           | During an extended service water bay flooding incident, one ESW pump was found grounded by testing, later two more pumps were found to be failed also.   |
| 76   | ESW    | Driver  | Motor      | Test             | Maintenance     | Operational/ Human Error                                   | 1994 | Failure to Run   | Partial           | Leak test of the containment cooling service water pump vault watertight door revealed excessive leakage. Flooding and leakage past this door would make inoperable two of four containment cooling service water pumps. Procedural inadequacy was cited as the cause for the degraded door seals. |
| 77   | ESW    | Pump    | Bearing    | Inspection       | Operational     | Other  | 1991 | Failure to Run   | Partial           | Lube oil cooling water isolated during a test. Pumps continued to run with no cooling.   |
| 78   | ESW    | Pump    | Bearing    | Inspection       | Maintenance     | Internal to Component                                      | 1987 | Failure to Run   | Partial           | Service water pumps had high shaft vibration. The excessive vibrations caused by worn bearings and shaft sleeves.  |
| 79   | ESW    | Pump    | Bearing    | Maintenance      | Maintenance     | Internal to Component                                      | 1985 | Failure to Run   | Partial           | High ESW pump vibration was caused by wearing of the upper bearings.   |
| 80   | ESW    | Pump    | Bearing    | Test             | Maintenance     | Internal to Component                                      | 1985 | Failure to Run   | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 81   | ESW    | Pump    | Bearing    | Test             | Environmental   | Internal to Component                                      | 1992 | Failure to Run   | Partial           | Abrasive particles present in ocean water produced accelerated wear of shaft bearing journals.   |
| 82   | ESW    | Pump    | Casing     | Demand           | Maintenance     | Internal to Component                                      | 1998 | Failure to Start | Partial           | Two ESW pump started and ran, but would not develop sufficient pressure or flow rate. Exact cause not known for either failure, however, one pump was noted to have microbiological induced corrosion fouling on internal surfaces.  |
| 83   | ESW    | Pump    | Casing     | Inspection       | Maintenance     | Internal to Component                                      | 1988 | Failure to Run   | Partial           | RHR service water pumps. Pump diffuser eroded on first pump and a through wall casing leak developed on the second.  |
| 84   | ESW    | Pump    | Casing     | Inspection       | Maintenance     | Internal to Component                                      | 1986 | Failure to Run   | Partial           | Cracked seal water and vent lines.   |
| 85   | ESW    | Pump    | Casing     | Test             | Maintenance     | Design/ Construction/ Manufacture/ Installation Inadequacy | 1997 | Failure to Run   | Almost Complete   | Both ESW pumps failed due to installation of wrong material for pump casing flanges by vendor during pump overhaul. The vendor overhauled the pumps without changing material. The plant returned the pumps to the warehouse also without verifying material.                                      |
| 86   | ESW    | Pump    | Coupling   | Inspection       | Environmental   | External Environment                                       | 1993 | Failure to Run   | Partial           | Entrained debris caused ESW pump shaft coupling to fail. Plant equipment did not prevent this debris from entering pump.   |



| Item | System | Segment | Piece Part          | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|---------------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 87   | ESW    | Pump    | Coupling            | Test             | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1987 | Failure to Start | Partial           | Test showed two ESW pumps failed. Pump shafts were corroded and found to be made of incorrect material.   |
| 88   | ESW    | Pump    | Coupling            | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1994 | Failure to Start | Partial           | Pump produced no flow when started. A shaft coupling failed. Material was determined to be brittle and have low impact properties. The coupling was replaced on all pumps with a type of material more suitable for this application.   |
| 89   | ESW    | Pump    | Coupling            | Test             | Maintenance     | Internal to Component  | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.   |
| 90   | ESW    | Pump    | Coupling            | Test             | Maintenance     | Internal to Component  | 1987 | Failure to Start | Almost Complete   | Two ESW pumps had failed couplings. Cause attributed to abnormal stress.  |
| 91   | ESW    | Pump    | Impeller/Wear Rings | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1996 | Failure to Run   | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 92   | ESW    | Pump    | Impeller/Wear Rings | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1988 | Failure to Run   | Partial           | ESW pumps drawing excessive current. Carbon steel snap rings corroded allowing impeller to come in contact with casing. The third pump, although not exhibiting abnormal current, had similar corrosion   |
| 93   | ESW    | Pump    | Impeller/Wear Rings | Demand           | Environmental   | Internal to Component  | 1994 | Failure to Run   | Partial           | Raw water pump currents stayed high after starting. The primary cause of these events was determined to be elevated sand content in the river, resulting in excessive sand accumulation around the suction area of the pumps.   |
| 94   | ESW    | Pump    | Impeller/Wear Rings | Demand           | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1996 | Failure to Run   | Partial           | A Nuclear Service Water pump tripped on overcurrent after operating for approximately 20 minutes. Initial troubleshooting indicated that the pump was binding and disassembly was required to determine the cause. It was determined that the pump impeller thrust ring had become loose due to thrust ring retainer bolt failure, which allowed the impeller to slip on the shaft and resulted in pump binding and the overcurrent condition. The bolts failed due to corrosion. Similar bolt degradation was discovered on other service water pumps. The investigation results indicate the primary cause of the bolt failures was corrosion induced by galvanic coupling of the retainer bolting and other pump components. |
| 95   | ESW    | Pump    | Impeller/Wear Rings | Demand           | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 2000 | Failure to Start | Almost Complete   | Two of the River Water pumps tripped on overcurrent when they were attempted to be started. The trips were a result of physical contact between the impeller and the lower casing liner of the pumps. This condition was due to differential thermal expansion between the pump shaft and the pump casing as a result of an elevated seal injection water temperature. The elevated temperature was due to an abnormal configuration of the Filtered Water System (the backup seal water supply).   |

| Item | System | Segment | Piece Part          | Discovery Method | Coupling Factor | Proximate Cause   | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|---------------------|------------------|-----------------|---|------|------------------|-------------------|--|
| 96   | ESW    | Pump    | Impeller/Wear Rings | Demand           | Design          | Design/Construction/Manufacture/Installation Inadequacy | 1981 | Failure to Run   | Complete          | Both charging pump service water pumps failed. A carbon cap screw failed allowing the impeller of one pump to bind on the casing. The ensuing leakage shorted the motor windings of the other pump.                      |
| 97   | ESW    | Pump    | Impeller/Wear Rings | Demand           | Design          | Design/Construction/Manufacture/Installation Inadequacy | 1986 | Failure to Run   | Partial           | All four emergency service water pumps showed cavitation damage. Two of the pumps had minor damage and were placed back in service. Recirculation cavitation occurs at flows significantly less than design.             |
| 98   | ESW    | Pump    | Impeller/Wear Rings | Inspection       | Environmental   | Internal to Component                                   | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Marine growth in suction.   |
| 99   | ESW    | Pump    | Impeller/Wear Rings | Inspection       | Maintenance     | Internal to Component                                   | 1985 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. The cause of the failure is suspected to be binding.  |
| 100  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1994 | Failure to Start | Partial           | Two ESW pumps had low discharge pressure during testing. Each pump had worn internals and both pump internals were replaced.   |
| 101  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1987 | Failure to Run   | Partial           | ESW pump low flow. Worn impellers.   |
| 102  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1990 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 103  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1989 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 104  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1988 | Failure to Start | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.   |
| 105  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1988 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to brackish water corrosion.   |
| 106  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component                                   | 1991 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted. |
| 107  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by wear and aging of internals.  |
| 108  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1990 | Failure to Run   | Partial           | ESW pumps had worn and cracked impellers. Aging and normal wear.   |
| 109  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1991 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 110  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1990 | Failure to Start | Partial           | ESW impeller gaps too wide. Gaps adjusted.   |
| 111  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                   | 1985 | Failure to Run   | Partial           | ESW pumps failed due to worn internals.  |
| 112  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component                                   | 1982 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water.                     |

| Item | System | Segment | Piece Part          | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|---------------------|------------------|-----------------|--|------|------------------|-------------------|--|
| 113  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1988 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 114  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component  | 1985 | Failure to Run   | Partial           | ESW pumps failed to meet the minimum flow requirements of test. A rag was found in one impeller and a plastic bottle in the other.   |
| 115  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1985 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 116  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. One pump also exhibited high vibration.   |
| 117  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component  | 1990 | Failure to Run   | Partial           | ESW pump impeller lift out of adjustment.  |
| 118  | ESW    | Pump    | Impeller/Wear Rings | Test             | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1986 | Failure to Start | Partial           | Testing of the service water system disclosed that the performance of the three service water pumps was below requirements. The condition is the result of both an inadequate system design and the installation of replacement impellers, which were not modified by the vendor to improve performance, as were the original impellers. |
| 119  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1985 | Failure to Run   | Partial           | The charging pump service water pumps degraded. Caused by expected wear of pump due to erosion and corrosion properties of the process fluid involved  |
| 120  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component  | 1995 | Failure to Start | Partial           | Pumps failed performance test. Sand in water eroded pump internals. Pump lift was adjusted.  |
| 121  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1998 | Failure to Start | Partial           | Two ESW pumps failed to develop adequate flow/pressure - pumps degraded.   |
| 122  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component  | 1994 | Failure to Start | Partial           | Degraded performance identified during testing. Sand in water was causing accelerated wear of the pump internals. Lift was adjusted for three pumps and one pump internals were replaced.  |
| 123  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component  | 1993 | Failure to Run   | Partial           | Essential service water pumps were declared inoperable, due to low pump head values. The low pump heads were caused by excessive wear of pump impeller due to sand in the service water.   |
| 124  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component  | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings. Cause determined to be normal wear and high sand content of river water.  |
| 125  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1985 | Failure to Run   | Partial           | Wear caused high ESW pump bearing temperatures, vibration, and low amperage/flow.  |
| 126  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1984 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 127  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1985 | Failure to Start | Partial           | Emergency service water pumps discharge pressure below allowable limits. Causes were loose impellers, dropped impeller, and worn internals.  |
| 128  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1981 | Failure to Start | Partial           | Loss of Service Water pump due to wearout at end of life.  |
| 129  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1985 | Failure to Start | Partial           | ESW pumps failed to meet the minimum flow requirements of test. The cause of the failure is normal wearout of the pump impeller due to the high sand content of the water being pumped. Pump impeller lift was adjusted.   |
| 130  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1982 | Failure to Run   | Partial           | Loss of Service Water pump due to wearout at end of life.  |

| Item | System | Segment | Piece Part          | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|---------------------|------------------|-----------------|--|------|------------------|-------------------|--|
| 131  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component                                      | 1982 | Failure to Run   | Partial           | Low ESW pump head values were caused excessive wear of pump impeller due to foreign material in the service water.   |
| 132  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1984 | Failure to Run   | Partial           | Containment spray raw water pumps failed flow tests. Aging and normal wear.  |
| 133  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Design/ Construction/ Manufacture/ Installation Inadequacy | 1988 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by excessive wear of pump impeller due to foreign material in the service water. |
| 134  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1986 | Failure to Run   | Partial           | ESW pump performance decreased 15% and 8% respectively since last test. Pumps were replaced.   |
| 135  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1986 | Failure to Run   | Partial           | ESW pumps had worn impellers and one had a plugged strainer.   |
| 136  | ESW    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component                                      | 1995 | Failure to Start | Partial           | Marine growth caused low flow and speed condition for two service water pumps  |
| 137  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1992 | Failure to Start | Partial           | ESW pumps had reduced flow and discharge pressure. Worn impellers/wearing rings.   |
| 138  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1994 | Failure to Run   | Partial           | Two ESW pumps had internal deterioration, one of which was indicated by high vibration readings.   |
| 139  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1984 | Failure to Start | Partial           | Essential service water pumps were declared inoperable, due to low pump head values. The low pump heads were caused by wear and aging of internals.  |
| 140  | ESW    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1983 | Failure to Run   | Partial           | RHR Service Water pumps failed flow tests due to wearout and had to be rebuilt.  |
| 141  | ESW    | Pump    | Lubrication         | Maintenance      | Operational     | Operational/ Human Error                                   | 1993 | Failure to Run   | Partial           | Low pressure RHR bearing oil level not maintained high enough when new smaller sightglass installed. Second event the sightglass was broken when adding oil.   |
| 142  | ESW    | Pump    | Packing/Seals       | Inspection       | Maintenance     | Internal to Component                                      | 1986 | Failure to Run   | Partial           | Excessive packing leakage. Both events occurred after previous maintenance had been performed for the same problems.   |
| 143  | ESW    | Pump    | Packing/Seals       | Inspection       | Maintenance     | Design/ Construction/ Manufacture/ Installation Inadequacy | 1997 | Failure to Run   | Partial           | Both ESW pumps leaking greater than 4 gpm because of inappropriate material for packing and sleeve (nitronic 60).  |
| 144  | ESW    | Pump    | Packing/Seals       | Inspection       | Environmental   | Internal to Component                                      | 1994 | Failure to Run   | Partial           | Backup seal water regulators did not provide required flow during testing on two pumps. The third pump lost seal flow while operating. The cause was attributed to plugged lines.                    |
| 145  | ESW    | Pump    | Packing/Seals       | Inspection       | Maintenance     | Internal to Component                                      | 1989 | Failure to Run   | Partial           | ESW pump excessive packing leakage.  |
| 146  | ESW    | Pump    | Packing/Seals       | Maintenance      | Environmental   | Internal to Component                                      | 1985 | Failure to Run   | Partial           | First pump developed seal leak due to sand. Second pump had high bearing temperatures due to trash clogging cooling water lines.   |
| 147  | ESW    | Pump    | Packing/Seals       | Test             | Maintenance     | Internal to Component                                      | 1981 | Failure to Start | Partial           | RHR service water pumps failed to meet flow requirements due to seal water leakage and pump wearout.   |
| 148  | ESW    | Pump    | Shaft               | Test             | Maintenance     | Internal to Component                                      | 1993 | Failure to Run   | Partial           | Service water pumps were noted to have high vibrations and low discharge pressure. Uneven wear caused pump to be out of balance.   |

| Item | System | Segment | Piece Part   | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|--------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 149  | ESW    | Suction | Booster Pump | Demand           | Operational     | Operational/ Human Error                                   | 1980 | Failure to Start | Partial           | The service water RHR booster pump was de-energized during maintenance. The attempt to start service water pumps failed due to low suction pressure.  |
| 150  | ESW    | Suction | Piping       | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1996 | Failure to Start | Partial           | Freezing of diesel generator service water piping in intake bay. Inadequate initial design.   |
| 151  | ESW    | Suction | Piping       | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1982 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 152  | ESW    | Suction | Piping       | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1981 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 153  | ESW    | Suction | Piping       | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1982 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.  |
| 154  | ESW    | Suction | Piping       | Demand           | Maintenance     | Operational/ Human Error                                   | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test. |
| 155  | ESW    | Suction | Piping       | Demand           | Design          | Other  | 1980 | Failure to Run   | Almost Complete   | Air ingress exceeded the air removal capability of the constant vent valves. A design change was implemented to remove the air compressor cooling from the service water system.  |
| 156  | ESW    | Suction | Piping       | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1983 | Failure to Run   | Almost Complete   | Increased flow to chillers resulted in loss of NPSH to Charging Pump Service Water pumps.   |
| 157  | ESW    | Suction | Piping       | Demand           | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | 1984 | Failure to Start | Partial           | Both RHR service water pumps tripped as a result of inadequate venting of suction header resulting from poor orientation of the vent line.  |
| 158  | ESW    | Suction | Piping       | Demand           | Maintenance     | Operational/ Human Error                                   | 1996 | Failure to Run   | Complete          | Both trains of both units charging pump service water pumps became air bound. Underwater diving maintenance activities on one units circulating water and service water lines was identified as the source of the air. The air entered the service water supply lines when a valve was opened in preparation for a Safety Injection logic test. |

| Item | System | Segment | Piece Part | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|------------|------------------|-----------------|--|------|------------------|-------------------|--|
| 159  | ESW    | Suction | Piping     | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1983 | Failure to Run   | Complete          | Increased flow to chillers resulted in loss of NPSH to Charging Water Service Water pumps.   |
| 160  | ESW    | Suction | Piping     | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the Charging Water Service Water pumps.   |
| 161  | ESW    | Suction | Piping     | Demand           | Operational     | Operational/ Human Error   | 1988 | Failure to Run   | Complete          | The procedure failed to adequately caution the operator to slowly fill a drained line. Rapid filling resulted in a loss of NPSH to the charging service water pumps. |
| 162  | ESW    | Suction | Piping     | Demand           | Environmental   | Internal to Component  | 1986 | Failure to Start | Partial           | RHR service water pumps failed flow testing due to blocked suction and abnormal wear of impellers.   |
| 163  | ESW    | Suction | Piping     | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1981 | Failure to Run   | Complete          | Increasing flow to chillers robs NPSH from charging service water pumps.   |
| 164  | ESW    | Suction | Piping     | Demand           | Operational     | Operational/ Human Error   | 1986 | Failure to Run   | Complete          | Failure to properly vent and fill a newly installed pipe introduced air into the charging pump service water system.   |
| 165  | ESW    | Suction | Piping     | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.  |
| 166  | ESW    | Suction | Piping     | Demand           | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1982 | Failure to Run   | Complete          | The use of service water by the chillers can cause a loss of suction pressure to the charging pump service water pumps.  |
| 167  | ESW    | Suction | Piping     | Test             | Operational     | Operational/ Human Error   | 1989 | Failure to Run   | Partial           | Inadequate procedure led to air binding of operating ESW pumps.  |
| 168  | ESW    | Suction | Piping     | Test             | Environmental   | Internal to Component  | 1990 | Failure to Start | Partial           | ESW pumps failed flow testing. Foreign material blocked the suction.   |
| 169  | ESW    | Suction | Strainer   | Demand           | Environmental   | Internal to Component  | 1980 | Failure to Run   | Partial           | Foreign material was allowed to enter the suction of the charging pump service water pumps resulting in low flow conditions.   |
| 170  | ESW    | Suction | Strainer   | Inspection       | Environmental   | Internal to Component  | 1984 | Failure to Run   | Partial           | Two RHR service water pumps had blown seals and sparks and smoke between the bearing housing and shaft. A piece of hard rubber valve liner was found in the pumps.   |
| 171  | ESW    | Suction | Strainer   | Inspection       | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 2000 | Failure to Run   | Partial           | RHRSW Pumps Failed to Develop flow/pressure. Debris in intake structure. Requires modifications to the traveling Water Screen.                                       |

| Item | System | Segment | Piece Part | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode   | Degree of Failure | Description  |
|------|--------|---------|------------|------------------|-----------------|--|------|----------------|-------------------|--|
| 172  | ESW    | Suction | Strainer   | Maintenance      | Operational     | Operational/ Human Error                                   | 1986 | Failure to Run | Complete          | A service water strainer was placed in service without being vented resulting in air binding system and loss of charging pump service water pumps.   |
| 173  | ESW    | Suction | Strainer   | Test             | Environmental   | Internal to Component                                      | 1982 | Failure to Run | Partial           | Failures occurred on residual heat removal service water pumps. The pumps failed to meet flow and pressure requirements. Failure was due to debris lodging in pump impellers. Source of debris was maintenance activities, broken traveling water screens, and the inadvertent opening of a RHR minimum flow line which washed materials into suction pit. |
| 174  | ESW    | Suction | Strainer   | Test             | Environmental   | Internal to Component                                      | 1990 | Failure to Run | Partial           | Essential service water pumps were declared inoperable, due to low pump head valves. The low pump heads were caused by suction blockage due to foreign material in the service water.  |
| 175  | ESW    | Suction | Tank       | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1985 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.  |
| 176  | ESW    | Suction | Tank       | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1990 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.  |
| 177  | ESW    | Suction | Tank       | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1985 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.  |
| 178  | ESW    | Suction | Tank       | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1990 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.  |
| 179  | ESW    | Suction | Tank       | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1990 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.  |
| 180  | ESW    | Suction | Tank       | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1985 | Failure to Run | Partial           | An engineering evaluation revealed that ESW had been inoperable several times due to low NPSH. All three units were affected.  |
| 181  | ESW    | Suction | Tank       | Test             | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1986 | Failure to Run | Complete          | Loss of prime in the condenser circulating water siphon flow system caused loss of low pressure service water pumps. Pumps lost suction during a test due to poor design.  |

| Item | System | Segment   | Piece Part | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|-----------|------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 182  | ESW    | Suction   | Valve      | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1983 | Failure to Start | Partial           | Low discharge pressure was caused by insufficient suction pressure. Service water flow to parallel components was adjusted.   |
| 183  | HCI    | Driver    | Piping     | Demand           | Design          | Other  | 1999 | Failure to Start | Complete          | Water entered the HCI and RCI steam supply lines, rendering both pumps inoperable. Failed reactor vessel instrumentation allowed water to overflow and fill the HCI/RCI steam lines. Pumps were unavailable.  |
| 184  | HPI    | Discharge | Piping     | Inspection       | Design          | External Environment   | 1994 | Failure to Run   | Partial           | Due to a leaking socket weld in the common recirculation line, all three SI pumps were declared inoperable. The underlying cause of the leak was a crack in the socket weld in the common recirculation line, caused by pipe displacement from air entrainment and pump misalignment.   |
| 185  | HPI    | Discharge | Recirc     | Test             | Environmental   | Internal to Component  | 1991 | Failure to Run   | Partial           | Something in HPI pump recirculation line was restricting flow. The piece later dislodged and no identification was made. Both SI pumps had inadequate recirculation flow.   |
| 186  | HPI    | Discharge | Recirc     | Test             | Environmental   | External Environment   | 1992 | Failure to Run   | Almost Complete   | Safety Injection pumps were declared inoperable due to an observed declining trend in the pump's recirculation flow. The cause of the Safety Injection pump reduced recirculation flow is attributed to foreign material blockage within the associated minimum flow recirculation line flow orifice.   |
| 187  | HPI    | Discharge | Valve      | Inspection       | Operational     | Operational/ Human Error   | 1987 | Failure to Start | Almost Complete   | While attempting to fill the safety injection accumulators, it was discovered that two of three SI pumps had been isolated from the high head injection flowpath.   |
| 188  | HPI    | Discharge | Valve      | Inspection       | Operational     | Operational/ Human Error   | 1993 | Failure to Run   | Partial           | One AFW pump failed due to incorrect procedure which allowed pump to be run without flow, other AFW pump was allowed to run past max flow rate. It is unclear whether these mistakes were due to inadequate procedures or staff errors, but it was assumed to be a failure to follow procedure.   |
| 189  | HPI    | Discharge | Valve      | Test             | Maintenance     | Internal to Component  | 1984 | Failure to Start | Partial           | CCP pump low flow rates due to inaccuracies in positioning the throttle valves.   |
| 190  | HPI    | Driver    | Breaker    | Inspection       | Operational     | Operational/ Human Error   | 1982 | Failure to Start | Complete          | During the draining of the reactor coolant system, both centrifugal charging pumps were rendered inoperable. The initial conditions in the draining procedure contained a confusing statement, which led to an erroneous assumption that both CCP breakers had to be racked out and tagged.   |
| 191  | HPI    | Driver    | Breaker    | Inspection       | Operational     | Operational/ Human Error   | 1989 | Failure to Start | Partial           | HPI Pump B not retested, then HPI Pump A removed from service.  |
| 192  | HPI    | Driver    | Breaker    | Inspection       | Operational     | Operational/ Human Error   | 1988 | Failure to Start | Complete          | HPI pumps not restored before mode change due to procedural inadequacy.   |
| 193  | HPI    | Driver    | Breaker    | Inspection       | Operational     | Operational/ Human Error   | 1990 | Failure to Start | Complete          | By opening incorrect breaker, HPI pump tripped while others were unavailable.   |
| 194  | HPI    | Driver    | Breaker    | Maintenance      | Maintenance     | Internal to Component  | 1991 | Failure to Start | Partial           | HPI pump breakers failed due to a broken pawl, and a broken closing coil.   |
| 195  | HPI    | Driver    | Breaker    | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1980 | Failure to Start | Partial           | Upon testing the safety injection pumps it was found that the 6900-v breakers would lock-out preventing pump start if they were given a close signal for >0.32 seconds when a trip condition existed. There is no indication to operations when this locked-out condition exists. The breaker appears to be available for service when it actually is not. The only means of clearing the condition is to remove and reinstall the fuses at the breaker or manually change the state of the relays. |



| Item | System | Segment | Piece Part    | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|---------------|------------------|-----------------|--|------|------------------|-------------------|--|
| 196  | HPI    | Driver  | I&C           | Inspection       | Operational     | External Environment                                       | 1990 | Failure to Run   | Complete          | It was determined that the common minimum flow path return line for the safety injection pumps to the refueling water storage tank was frozen. Previous actions to investigate problems with the freeze protection system were unsuccessful in preventing development of this condition. The two HPI pumps were declared inoperable with this return line frozen. A faulty ambient temperature switch for the RWST heat trace system prevented the heat trace from activating and was subsequently replaced. In addition, administrative controls did not sufficiently recognize the safety significance of flow through this line and the need to ensure flow capability. |
| 197  | HPI    | Driver  | I&C           | Inspection       | Operational     | Operational/ Human Error                                   | 1990 | Failure to Start | Partial           | Both safety injection pumps were in the pull-to-lock position. With the switches in pull-to-lock, the pumps would not have automatically started upon receipt of an initiating signal. This event was caused by cognitive personnel error by a utility licensed operator in failure to follow an approved procedure.   |
| 198  | HPI    | Driver  | I&C           | Inspection       | Operational     | Operational/ Human Error                                   | 1992 | Failure to Start | Almost Complete   | Two charging pumps and one charging pump service water pump were removed from service simultaneously which is a condition not allowed by technical specifications.   |
| 199  | HPI    | Driver  | I&C           | Inspection       | Operational     | Operational/ Human Error                                   | 1988 | Failure to Start | Complete          | With alternate CCP pump out-of-service, the remaining operable pump was erroneously placed in pull-to-lock.  |
| 200  | HPI    | Driver  | I&C           | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus.  |
| 201  | HPI    | Driver  | I&C           | Maintenance      | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1996 | Failure to Run   | Partial           | A lead was lifted in an emergency bus DC control circuit resulting in one charging pump tripping while running on the alternate power supply. Further investigation into this event revealed an anomaly, which could result in having no operating charging pumps. The cause of the event has been determined to be an error in the original design of the charging pump interlock logic. The anomaly would occur upon a loss of the DC control power to one emergency bus if 'C' charging pump was powered from the other bus.  |
| 202  | HPI    | Driver  | Lubrication   | Demand           | Maintenance     | Internal to Component                                      | 1984 | Failure to Run   | Partial           | Charging pump lube oil cooler fan motor trips on thermal overload. Probable cause: normal wear on motor resulting in increased friction replaced worn motor with spare. During routine inservice testing found that another charging pump lube oil cooler fan motor had a current imbalance. Probable cause: normal aging of motor insulation has resulted in a current imbalance.   |
| 203  | HPI    | Driver  | Lubrication   | Inspection       | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 2000 | Failure to Run   | Partial           | CVC makeup oil pump motor too small for certain accidents.   |
| 204  | HPI    | Driver  | Packing/Seals | Inspection       | Maintenance     | Internal to Component                                      | 1988 | Failure to Run   | Almost Complete   | Smoke was discovered coming from the speed increaser unit for a centrifugal charging pump. Investigation found the two gland seal retaining bolts inside the speed increaser lube oil pump backed out allowing the gland seal to loosen. The gland seal being loosened, caused reduced oil flow to the speed increaser internals and ultimate damage. Other CCPs were inspected, and the same gland seal bolts as on the first pump were found loosened. The cause of the bolts backing out was determined to be lack of a periodic adjustment of the gland seal bolts.  |

| Item | System | Segment | Piece Part          | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|---------------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 205  | HPI    | Driver  | Piping              | Inspection       | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 2000 | Failure to Run   | Partial           | Microbiologically induced corrosion leak on service water lines to two charging/HPI pump lube oil coolers.  |
| 206  | HPI    | Pump    | Bearing             | Inspection       | Design          | External Environment   | 1991 | Failure to Run   | Almost Complete   | Charging/safety pumps beyond operational limits. Damage was found to the thrust bearings. Air was introduced into this train of chilled water during modifications and testing being performed on the system. This air became trapped in high points of either, or both of, the supply and return chilled water lines to the charging pump. At the reduced flow rate, sufficient cooling was not available and oil temperature increased to the point where bearing damage occurred.  |
| 207  | HPI    | Pump    | Casing              | Inspection       | Quality         | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1987 | Failure to Run   | Partial           | During inspection of a centrifugal charging pump, a portion of the stainless steel cladding on the inside surface of the pump casing exhibited corrosion. Corrosion of the pump casing was through the stainless steel cladding into the carbon steel base material. Inspection of the other CCP revealed similar corrosion. The cause of this event was a manufacturing deficiency. Corrosion observed at the pump casing discharge nozzle was attributed to a cladding breakthrough during final machining. Corrosion observed at the pump casing inlet end was attributed to either over-machining of the cladding or inadequate overlay of two adjacent weld beads. |
| 208  | HPI    | Pump    | Impeller/Wear Rings | Test             | Environmental   | Internal to Component  | 1984 | Failure to Run   | Almost Complete   | One HPI pump seized, the second would have seized if operated.  |
| 209  | HPI    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1983 | Failure to Start | Partial           | SI pump and both CCPs failed to meet the minimum head curve requirements. The cause of pump head capacity degradation has been attributed to normal pump operation. The inability to balance flows has been attributed to the lower head capacity of the pumps.   |
| 210  | HPI    | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component  | 1985 | Failure to Start | Partial           | The CCPs were tested and had low flow rates. The most probable cause is attributed to observed degradation of the pumps. The CCPs are subject to normal wear associated with their secondary duty of providing normal charging flow.  |
| 211  | HPI    | Pump    | Lubrication         | Inspection       | Design          | Internal to Component  | 1981 | Failure to Run   | Partial           | Corrosion of HPI pump cooler heads. Improper material led to corrosion  |
| 212  | HPI    | Pump    | Lubrication         | Inspection       | Operational     | Operational/ Human Error   | 1983 | Failure to Start | Complete          | A routine preventive maintenance (oil change) was mistakenly performed on the north charging pump instead of the south as scheduled. Since the south pump was previously cleared for this oil change, and the test pump was valved out, none of these three pumps were in service as required by tech specs for the approximately 20 minutes it took to change the oil in the north pump.   |
| 213  | HPI    | Pump    | Lubrication         | Inspection       | Environmental   | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1995 | Failure to Run   | Partial           | High lube oil temperatures were observed during HPI pump operation. Zinc particles from anode were discovered plugging the lube oil coolers. Accelerated corrosion was attributed to a corrosion inhibitor that was added to the system, which chemically interacted with the zinc.   |
| 214  | HPI    | Pump    | Lubrication         | Inspection       | Environmental   | Internal to Component  | 1983 | Failure to Run   | Partial           | Oysters and miscellaneous mollusks plugged HPI oil coolers. Two pumps were required to be shutdown due to rising lubricating oil temperatures.  |
| 215  | HPI    | Pump    | Lubrication         | Maintenance      | Environmental   | Internal to Component  | 1991 | Failure to Run   | Partial           | HPI pump lube oil cooler leaks. Degraded tubes.   |

| Item | System | Segment | Piece Part  | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|-------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 216  | HPI    | Pump    | Lubrication | Maintenance      | Maintenance     | Operational/ Human Error                                   | 1991 | Failure to Run   | Partial           | Following an overhaul of the HPI pumps. Too much oil flow led to excessive oil leakage, which would have failed HPI pumps before end of mission.  |
| 217  | HPI    | Pump    | Lubrication | Maintenance      | Environmental   | Internal to Component                                      | 1980 | Failure to Run   | Partial           | HPI pump lube oil cooler with tube leak allowed water into oil reservoir.   |
| 218  | HPI    | Pump    | Lubrication | Maintenance      | Environmental   | Internal to Component                                      | 1986 | Failure to Run   | Almost Complete   | Clams/sludge fouling of lube oil cooler caused high temperature alarms on two HPI pumps.  |
| 219  | HPI    | Suction | I&C         | Demand           | Maintenance     | Design/ Construction/ Manufacture/ Installation Inadequacy | 1997 | Failure to Run   | Complete          | HPI pumps fail due to operation with inadequate suction head. Two pumps damaged due to operation with inadequate suction, but all three system pumps were unavailable due to the loss of the suction source. Suction source level instrumentation was the cause.  |
| 220  | HPI    | Suction | I&C         | Demand           | Design          | Other  | 1997 | Failure to Run   | Partial           | Letdown storage tank reference leg not full, which gave erroneous indication of sufficient tank level. One HPI pump severely damaged, other pump not as damaged, and could have run. The root cause was a combination of a design weakness of a common reference leg for the Letdown storage tank level instruments and a leaking instrument fitting due to an inadequate work practice.  |
| 221  | HPI    | Suction | Piping      | Demand           | Environmental   | External Environment                                       | 1984 | Failure to Start | Complete          | Boron solidification in the suction and gas binding of pumps led to the failure of all three safety injection pumps. Flushing procedures inadequate.  |
| 222  | HPI    | Suction | Piping      | Demand           | Design          | Other  | 1982 | Failure to Start | Complete          | Hydrogen from the suction dampener got into suction piping and failed both CCPs.  |
| 223  | HPI    | Suction | Piping      | Inspection       | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1991 | Failure to Start | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the alternate boration line and the gravity feed line from the boric acid storage tank.   |
| 224  | HPI    | Suction | Piping      | Inspection       | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | 1988 | Failure to Run   | Partial           | Vortex breakers had not been installed in the containment emergency sumps. Vortex breakers are required to be installed in the containment emergency sumps to prevent the formation of vortices which could adversely affect performance of safety injection pumps during the safety injection and containment spray systems were declared inoperable.  |
| 225  | HPI    | Suction | Piping      | Inspection       | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1988 | Failure to Start | Partial           | It was determined that various pipes of the safety injection system and chemical volume and control system collected or trapped gas which might affect the functions of these systems. There was a concern that the gas pockets may adversely effect pump operation. Voids were detected in some of the high head SI pump piping.   |
| 226  | HPI    | Suction | Piping      | Inspection       | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1990 | Failure to Start | Partial           | A quantity of gas was found in the centrifugal charging pump suction header that exceeded the maximum allowed gas volume. It was subsequently determined that hydrogen gas had been coming out of solution on both units and accumulating in the suction piping as a probable result of gas stripping by the CCP miniflow orifices. In addition, entrainment of hydrogen bubbles from the volume control tank to the CCP suction pipe may be a contributor as well. |

| Item | System | Segment | Piece Part          | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|---------------------|------------------|-----------------|--|------|------------------|-------------------|--|
| 227  | HPI    | Suction | Piping              | Inspection       | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1988 | Failure to Run   | Partial           | Ultrasonic examination of the chemical and volume control system suction piping was performed. These examinations revealed voids in the suction piping.  |
| 228  | HPI    | Suction | Strainer            | Maintenance      | Maintenance     | Operational/ Human Error                                   | 1985 | Failure to Run   | Partial           | Strainers found still installed in the suction piping of the high-pressure injection pumps was a condition not considered in the operating design. The strainers were found during maintenance to repair a slight flange leak. The strainers had been placed in the suction piping during construction and were to be in place during system flushing to prevent any debris from reaching the pumps. However, the strainers should have been removed after system flushing prior to functional testing |
| 229  | LCS    | Driver  | Breaker             | Test             | Quality         | Design/ Construction/ Manufacture/ Installation Inadequacy | 1980 | Failure to Start | Complete          | Relay extra contacts left connected during construction, prevented Core Spray pump start with emergency diesel generator breakers racked out.  |
| 230  | RHR-B  | Driver  | Bearing             | Test             | Environmental   | External Environment                                       | 1991 | Failure to Run   | Partial           | Two LCI pumps were declared inoperable due to high motor vibration.  |
| 231  | RHR-B  | Driver  | Breaker             | Demand           | Maintenance     | Internal to Component                                      | 1987 | Failure to Start | Partial           | RHR pump breakers failed to close when operated remotely from the control room. It was found that the latch roller bearings and the cam follower bearing (internal piece parts of the breaker) were not operating correctly. This prevented the trip latch assembly from resetting and allowing the breaker to close.  |
| 232  | RHR-B  | Driver  | Breaker             | Maintenance      | Maintenance     | Operational/ Human Error                                   | 1990 | Failure to Start | Partial           | RHR pump breaker overcurrent trips out of calibration.   |
| 233  | RHR-B  | Driver  | Breaker             | Maintenance      | Maintenance     | Operational/ Human Error                                   | 1991 | Failure to Start | Partial           | While performing preventive maintenance calibration check on the protective relays for a residual heat removal pump motor 4kv breaker, it was found that all overcurrent relays for two pumps were out of calibration  |
| 234  | RHR-B  | Driver  | Breaker             | Test             | Maintenance     | Internal to Component                                      | 1997 | Failure to Start | Partial           | Breaker latch check switch failed on both pumps. Lack of lubrication.  |
| 235  | RHR-B  | Driver  | Breaker             | Test             | Maintenance     | Internal to Component                                      | 1986 | Failure to Start | Partial           | RHR pump circuit breakers failed during a start for testing. Bend switch and binding mechanism. Attributed to inadequate maintenance.  |
| 236  | RHR-B  | Driver  | I&C                 | Test             | Design          | Other  | 1982 | Failure to Start | Partial           | A functional test revealed a sliding link in control room panel open. Further investigation revealed a total of four links open. These links, left open, negated all autostart capability of 2 of 4 RHR pumps. It could not be determined why these four links were open.  |
| 237  | RHR-B  | Driver  | Supports            | Inspection       | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 1986 | Failure to Start | Partial           | RHR motor internal supports were cracked due to stress and vibration. Design improvements were made.   |
| 238  | RHR-B  | Pump    | Impeller/Wear Rings | Test             | Maintenance     | Internal to Component                                      | 1985 | Failure to Start | Partial           | The first pump failed to meet required flow rate. The second was drawing excessive amperage. Both conditions were attributed to worn internals.  |
| 239  | RHR-B  | Pump    | Lubrication         | Inspection       | Maintenance     | Internal to Component                                      | 1990 | Failure to Run   | Partial           | Both pump motor oil coolers were leaking due to aging of components. The first case involved through wall corrosion and the pump was immediately removed from service. The second case was a packing leak.   |

| Item | System | Segment | Piece Part    | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description   |
|------|--------|---------|---------------|------------------|-----------------|--|------|------------------|-------------------|---|
| 240  | RHR-P  | Driver  | Bearing       | Inspection       | Maintenance     | Operational/ Human Error                                   | 1988 | Failure to Run   | Partial           | Residual heat removal pump motor upper bearing housings were observed to be leaking oil. The cause of the failure was attributed to a lack of sealant being applied and gasket installed after the last maintenance was performed on the motor bearing housing.   |
| 241  | RHR-P  | Driver  | Breaker       | Demand           | Maintenance     | Other  | 1987 | Failure to Start | Complete          | Two LPI pumps, when given a start signal, would not start. An ongoing investigation revealed the probable root cause of the event to be poor electrical contact of the breaker auxiliary stabs for the pumps.   |
| 242  | RHR-P  | Driver  | Breaker       | Inspection       | Maintenance     | Operational/ Human Error                                   | 1981 | Failure to Start | Complete          | All RHR pumps de-energized to replace RHR Relief valve. T.S. allows this condition for 1 hour. Operated in the mode in excess of 5 hours.   |
| 243  | RHR-P  | Driver  | I&C           | Inspection       | Maintenance     | Operational/ Human Error                                   | 1992 | Failure to Start | Complete          | Both trains of RHR were rendered inoperable for two minutes, while performing an operational readiness test surveillance procedure. The surveillance procedure required that the one RHR train pump be placed in pull to lock and the other train heat exchanger flow control valve throttled to 30-40% open. The procedure directed the operators to perform operations that resulted in both trains of RHR being inoperable   |
| 244  | RHR-P  | Driver  | I&C           | Inspection       | Operational     | Operational/ Human Error                                   | 1995 | Failure to Start | Complete          | The switches for the containment spray and recirculation pumps were in a trip pullout when the Technical Specifications and plant procedures required the pumps to be operable.   |
| 245  | RHR-P  | Driver  | Lubrication   | Demand           | Design          | Design/ Construction/ Manufacture/ Installation Inadequacy | 2000 | Failure to Run   | Complete          | Both RHR/LPI pumps fail to run due to improper oil in system. High bearing temperatures occurred when the pumps were operated. This was due to the wrong lube oil being used, which had too high a viscosity. Inadequate vender design information resulted in the higher viscosity oil being used and additional exacerbating problems such as insufficient bearing clearances.  |
| 246  | RHR-P  | Pump    | Casing        | Test             | Maintenance     | Operational/ Human Error                                   | 1989 | Failure to Start | Complete          | Both loops of the residual heat removal system were declared inoperable due to gas binding of both RHR pumps. The gas binding was caused by entry of nitrogen gas into the reactor coolant system from accumulator. The root cause of this event has been attributed to personnel error. Personnel did not comply with the specific requirements in the accumulator discharge check valve full flow test procedure due to inattention to detail.  |
| 247  | RHR-P  | Pump    | Packing/Seals | Inspection       | Environmental   | External Environment                                       | 1985 | Failure to Start | Complete          | Following a trip, water was found spraying from both low head safety injection pump wedge control rod seals. Both pumps were declared inoperable. Postulated failure on the seals was from a minor flow induced pressure transient.   |
| 248  | RHR-P  | Suction | Piping        | Demand           | Design          | Other  | 1982 | Failure to Run   | Complete          | RHR Suction lost due to erroneous RCS level while draining the RCS.   |
| 249  | RHR-P  | Suction | Piping        | Demand           | Operational     | Operational/ Human Error                                   | 1984 | Failure to Run   | Complete          | The control room operators started a second residual heat removal pump in preparation for removing the operating RHR pump from service. With both pumps running, flow became excessive for the half-loop condition causing cavitation and air binding of both pumps. To prevent recurrence the procedure which controls the operation of the RHR pumps has been changed to include specific instructions to stop the operating pump prior to starting the second pump while at half-loop. |
| 250  | RHR-P  | Suction | Piping        | Demand           | Design          | Unknown  | 1983 | Failure to Run   | Complete          | RHR pumps cavitated. Unable to repeat. Unknown cause.   |
| 251  | RHR-P  | Suction | Piping        | Demand           | Maintenance     | Other  | 1986 | Failure to Run   | Complete          | SDC pumps cavitated due to lowering RCS level. Level indication was in error.   |
| 252  | RHR-P  | Suction | Piping        | Demand           | Design          | Other  | 1987 | Failure to Run   | Complete          | RHR flow was interrupted when both RHR trains became inoperable due to air bound RHR pumps. The loss of RCS inventory to the reactor coolant drain tank due to a leaking valve caused a decrease in RCS water level, vortexing in the pumps' suction line, and air entrainment in the RHR pumps.  |

| Item | System | Segment | Piece Part | Discovery Method | Coupling Factor | Proximate Cause          | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|------------|------------------|-----------------|--------------------------|------|------------------|-------------------|--|
| 253  | RHR-P  | Suction | Piping     | Demand           | Design          | Operational/ Human Error | 1984 | Failure to Run   | Almost Complete   | On two occasions, RHR pumps cavitated due to low RCS level while draining the RCS.   |
| 254  | RHR-P  | Suction | Piping     | Demand           | Design          | Operational/ Human Error | 1985 | Failure to Run   | Complete          | Swap over of RHR pumps resulted in both trains becoming inoperable due to air injection into the suction of the pumps. This required both pumps to be vented and required RCS level to be raised to prevent a possible recurrence of the vortex problem.   |
| 255  | RHR-P  | Suction | Piping     | Demand           | Maintenance     | Other                    | 1983 | Failure to Run   | Complete          | The RHR pumps began to cavitate and eventually both pumps were stopped. The reactor vessel level gauge being used to provide an indication that the level was approaching the vessel flange level had been isolated (reactor coolant drain tank isolation valve had been closed during an attempt to reduce leakage). Additionally, procedures did not require visual monitoring of cavity level.  |
| 256  | RHR-P  | Suction | Piping     | Demand           | Design          | Operational/ Human Error | 1982 | Failure to Run   | Complete          | Suction was lost to both RHR pumps. RHR flow was less than 3000 gpm and pump amps were fluctuating prior to taking corrective action. Each of these events appear to have been caused by a slow decrease in RCS level in conjunction with the vortex action at the pump suction.   |
| 257  | RHR-P  | Suction | Piping     | Demand           | Maintenance     | Other                    | 1981 | Failure to Run   | Complete          | Temporary coolant loop level indicator showed level slowly increasing over a period of days. The system was periodically drained to maintain 65 percent indicated level. A RHR pump lost suction on reduction of actual level. The second pump was started, and lost suction. Indication drift was due to evaporation of reference leg.  |
| 258  | RHR-P  | Suction | Piping     | Demand           | Maintenance     | Other                    | 1980 | Failure to Run   | Complete          | A complete loss of RHR flow occurred while plant operators were increasing RHR heat exchanger flow by closing down on the heat exchanger bypass valve.   |
| 259  | RHR-P  | Suction | Piping     | Demand           | Design          | Operational/ Human Error | 1980 | Failure to Run   | Complete          | The reactor vessel vent eductor was in service in preparation for refueling with RHR operating. A low flow alarm was received and low flow and low motor current were indicated. A second pump was started and became air-bound. Putting the vessel vent eductor system into service was the root cause of the incident.   |
| 260  | RHR-P  | Suction | Piping     | Demand           | Operational     | Operational/ Human Error | 1980 | Failure to Run   | Complete          | While attempting to increase RHR flow, the plant experienced a total loss of flow due to the pumps being air-bound. The pump was not vented when starting to increase flow. Operating procedures have been changed to have an operator present while changing flow in the RHR system. There have been losses of RHR flow in the past because the pumps were air-bound and methods are being investigated to improve the system design.   |
| 261  | RHR-P  | Suction | Piping     | Demand           | Design          | Other                    | 1982 | Failure to Run   | Complete          | With unit drained to centerline of the nozzles, suction to both RHR pumps was lost for 36 minutes. Suction to the RHR pumps was lost because of ambiguous reactor coolant system level indication while drained to centerline of the nozzles. The actual RCS level was lower than observed.  |
| 262  | RHR-P  | Suction | Piping     | Maintenance      | Maintenance     | Operational/ Human Error | 1982 | Failure to Run   | Complete          | Shutdown cooling was lost due to nitrogen intrusion because of backflushing a filter in the purification system.   |
| 263  | RHR-P  | Suction | Valve      | Demand           | Design          | Other                    | 1984 | Failure to Run   | Complete          | Both RHR pumps were unable to operate due to the introduction of air into the RHR system. The incident occurred during the drain down of the RCS, when the level of the RCS was being monitored via a standpipe off the centerline of one of the RCS loops. The isolation valve to which the standpipe was attached became clogged sometime during the drain down and falsely indicated above centerline when in fact the level was below the RHR suction line (below centerline). |
| 264  | SLC    | Driver  | Breaker    | Maintenance      | Maintenance     | Internal to Component    | 1999 | Failure to Start | Partial           | SLC Pump Breakers Fail to pickup on degraded voltage test  |

| Item | System | Segment | Piece Part       | Discovery Method | Coupling Factor | Proximate Cause  | Year | Failure Mode     | Degree of Failure | Description  |
|------|--------|---------|------------------|------------------|-----------------|--|------|------------------|-------------------|--|
| 265  | SLC    | Driver  | Breaker          | Test             | Design          | Other  | 1986 | Failure to Start | Complete          | During a test, both Squib Valve Detonators shorted after firing and the Control Power Transformer fuse blew causing the pump motor trip. This was caused by improper fuse coordination between the Control Power Transformer fuse and the Squib Valve Detonator fuses. The redundant system's Squib Valve was also fired during this test, without running the associated pump, and one of the Squib Valve Detonators shorted after firing. The same fuse coordination problem existed for both systems. |
| 266  | SLC    | Pump    | Bearing          | Inspection       | Maintenance     | Internal to Component  | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. Loose fittings and lack of thread sealant.  |
| 267  | SLC    | Pump    | Lubrication      | Test             | Maintenance     | Internal to Component  | 1992 | Failure to Run   | Partial           | Standby Liquid Control pumps lost oil while running. The gasket between the crankcase frame cap and the gear housing cover was worn.   |
| 268  | SLC    | Pump    | Packing/Seals    | Inspection       | Maintenance     | Internal to Component  | 1988 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing replaced.   |
| 269  | SLC    | Pump    | Packing/Seals    | Inspection       | Maintenance     | Internal to Component  | 1987 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking excessively at the packing. The failure of the packing was attributed to normal wear. Packing adjusted.   |
| 270  | SLC    | Pump    | Packing/Seals    | Inspection       | Maintenance     | Internal to Component  | 1989 | Failure to Run   | Partial           | Standby Liquid Control pumps were observed to be leaking profusely at the packing. The failure of the packing was attributed to normal wear.   |
| 271  | SLC    | Pump    | Plunger/Cylinder | Inspection       | Maintenance     | Internal to Component  | 1989 | Failure to Run   | Partial           | Standby Liquid Control pump seal was leaking excessively. The cause of this failure was normal wear of the plungers, packing, and head gaskets for the plungers (piece parts of the pump).   |
| 272  | SLC    | Suction | Tank             | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1991 | Failure to Run   | Complete          | During the performance of a special test on Unit 1 to determine the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.  |
| 273  | SLC    | Suction | Tank             | Test             | Design          | Design/<br>Construction/<br>Manufacture/<br>Installation<br>Inadequacy | 1991 | Failure to Run   | Complete          | During the performance of a special test on the available NPSH of the SLC pumps, the pumps began to cavitate unexpectedly. The SLC systems of both units were declared inoperable. The causes of this event are inadequate modification testing and an error in the original design calculations.  |
| 274  | SLC    | Suction | Valve            | Inspection       | Maintenance     | Operational/ Human Error   | 1991 | Failure to Start | Partial           | SLC pumps were potentially inoperable during part of test due to valve lineup.   |

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| 11. ABSTRACT (200 words or less)<br>This report documents a study performed on the set of common-cause failures (CCF) of pumps from 1980 to 2000. The data studied here were derived from the NRC CCF database, which is based on US commercial nuclear power plant event data. This report is the result of an in-depth review of the pump CCF data and presents several insights about the pump CCF data. The objective of this document is to look beyond the CCF parameter estimates that can be obtained from the CCF data to gain further understanding of why CCF events occur and what measures may be taken to prevent, or at least mitigate the effect of, pump CCF events. This report presents quantitative presentation of the pump CCF data and discussion of some engineering aspects of the pump events. |  |  |  |  |              |
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